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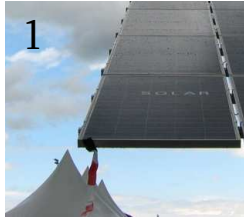
## Numerical simulation and performance assessment of an absorption solar air-conditioning system coupled with an office building



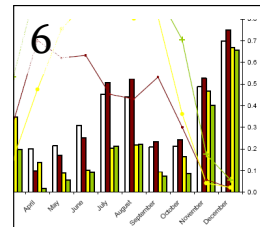
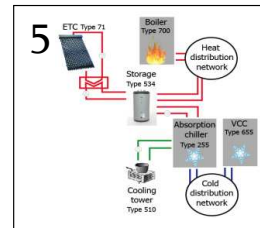
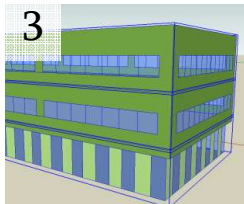
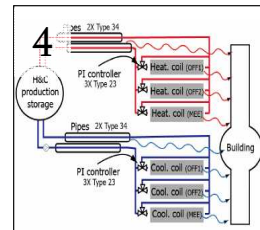
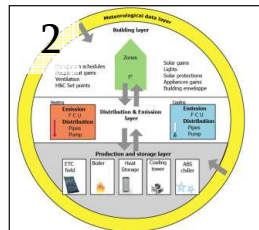
Sébastien Thomas and Philippe André

Building Energy Monitoring and Simulation, University of Liège

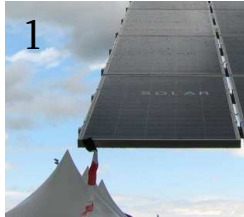
185 Avenue de Longwy, 6700 ARLON, Belgium



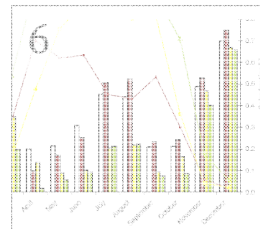
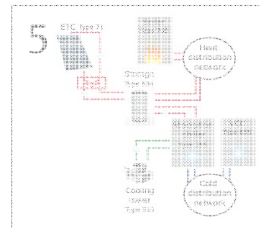
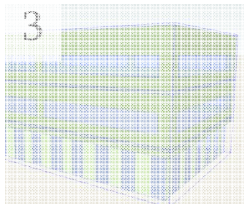
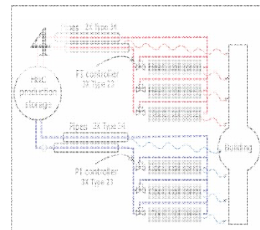
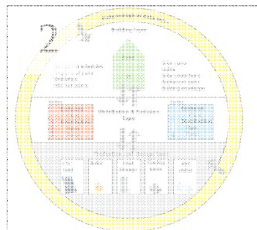
## Presentation overview



1. Introduction
2. Simulation environment overview
3. Building
4. H&C emission and distribution
5. H&C production and storage
6. Results & Conclusion



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## Introduction : Solar air-conditioning context

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*Building cooling has an important impact on energy consumption, therefore on CO<sub>2</sub> emissions.*

*Moreover, strong increase in cooling installed capacity has been encountered last years.*

### Assets for solar energy :

- It is one of the largest renewable energy resource
- Various technologies are available for cold production
- Sunny locations have more cooling needs

### Market available solar cooling technologies

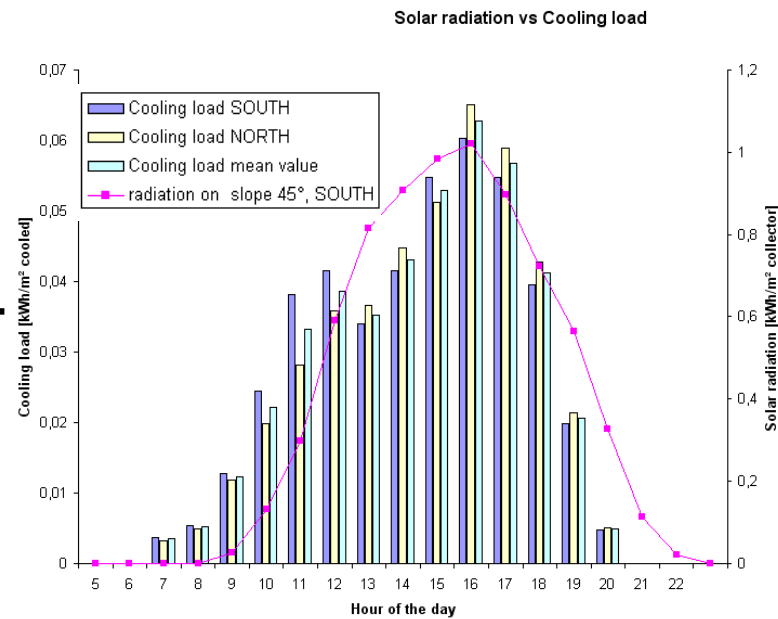
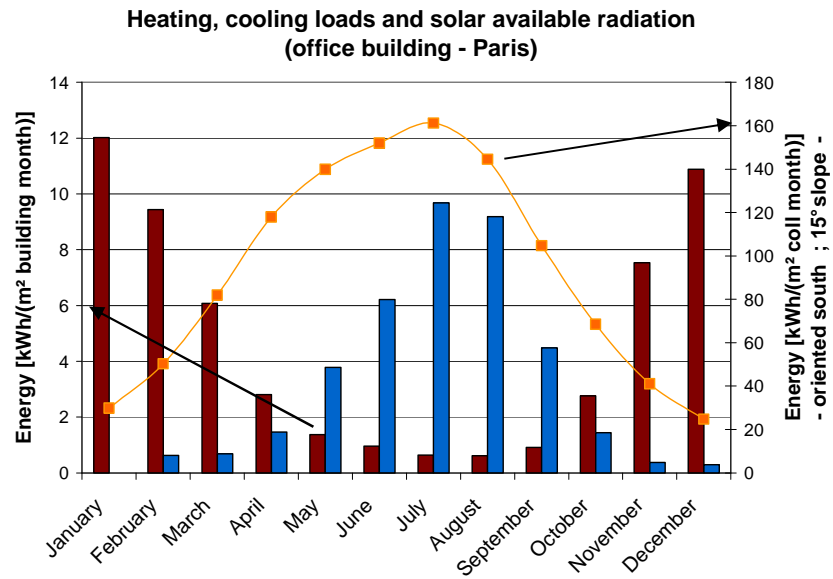
- Absorption chiller
- Adsorption chiller
- Desiccant cooling system
- PV panels with classical vapour compression chiller

## Introduction : key asset of solar energy

- The cooling load of buildings is generally high when solar radiation is high

→ yearly basis

→ daily basis



For an office building in August in Paris

## Introduction : Market status of Solar air-conditioning

World wide : 288 installations total of 16 MW

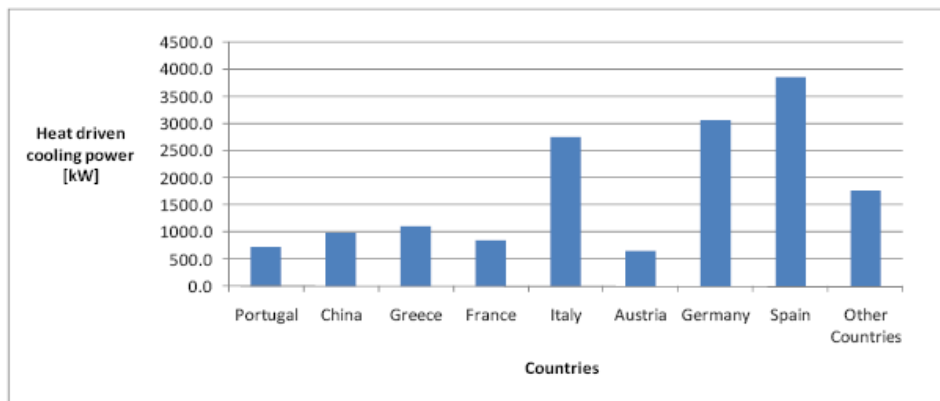
Small scale : 1.6 MW

≈ 200 syst < 15kW

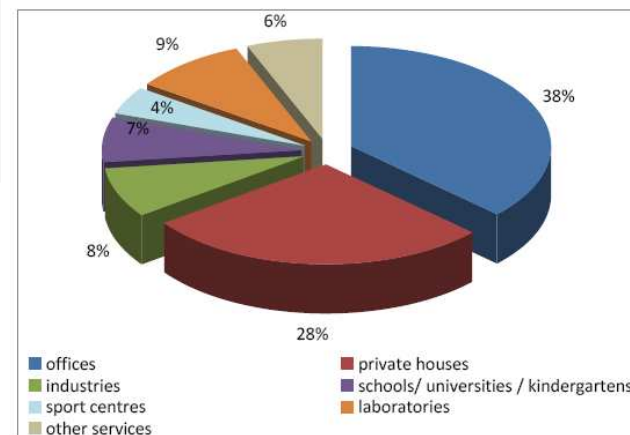
Large scale : 14 MW

≈ 100 Syst > 15kW

### Country distribution



### Type of building



### Technology distribution

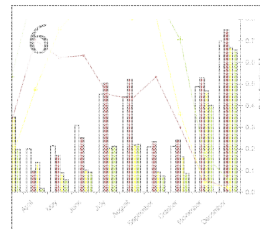
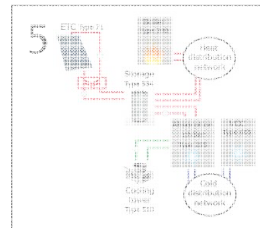
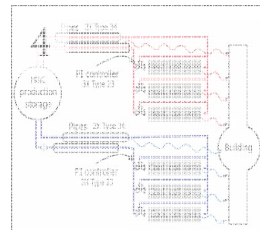
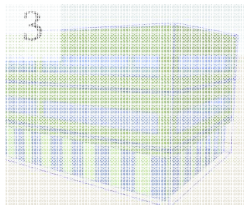
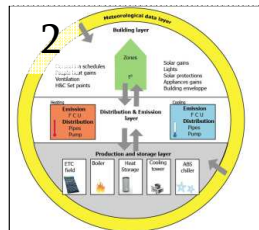
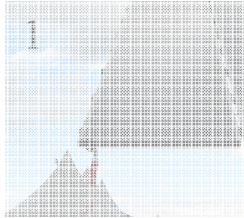
71% Absorption

16 % Desiccant (14% solid – 2% liquid)

13% Adsorption

Src : IEA-SHC Task 38 Subtask B Reports 2009-11-12

## Presentation overview



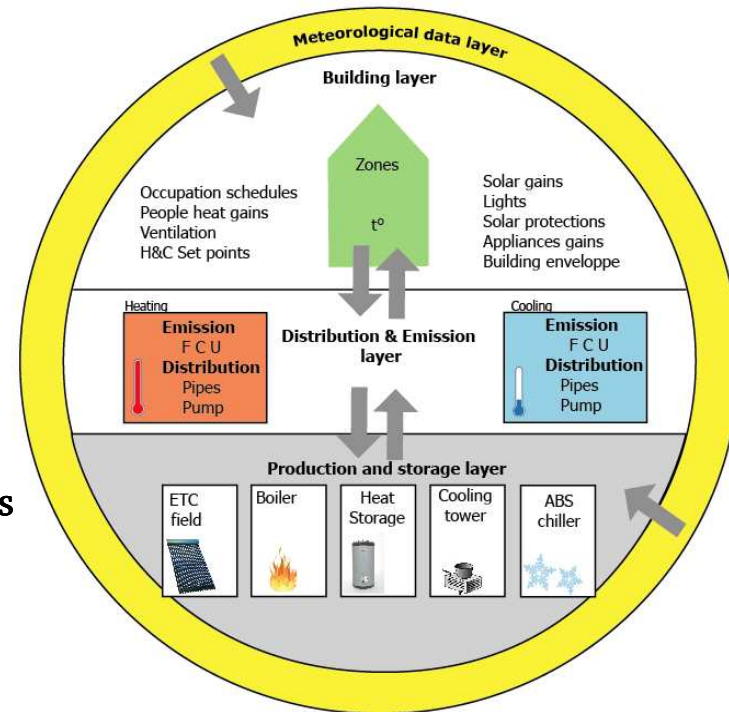
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## Simulation environment overview

*Integral approach to evaluate energy savings : Complete simulation environment is presented (...using TRNSYS)*

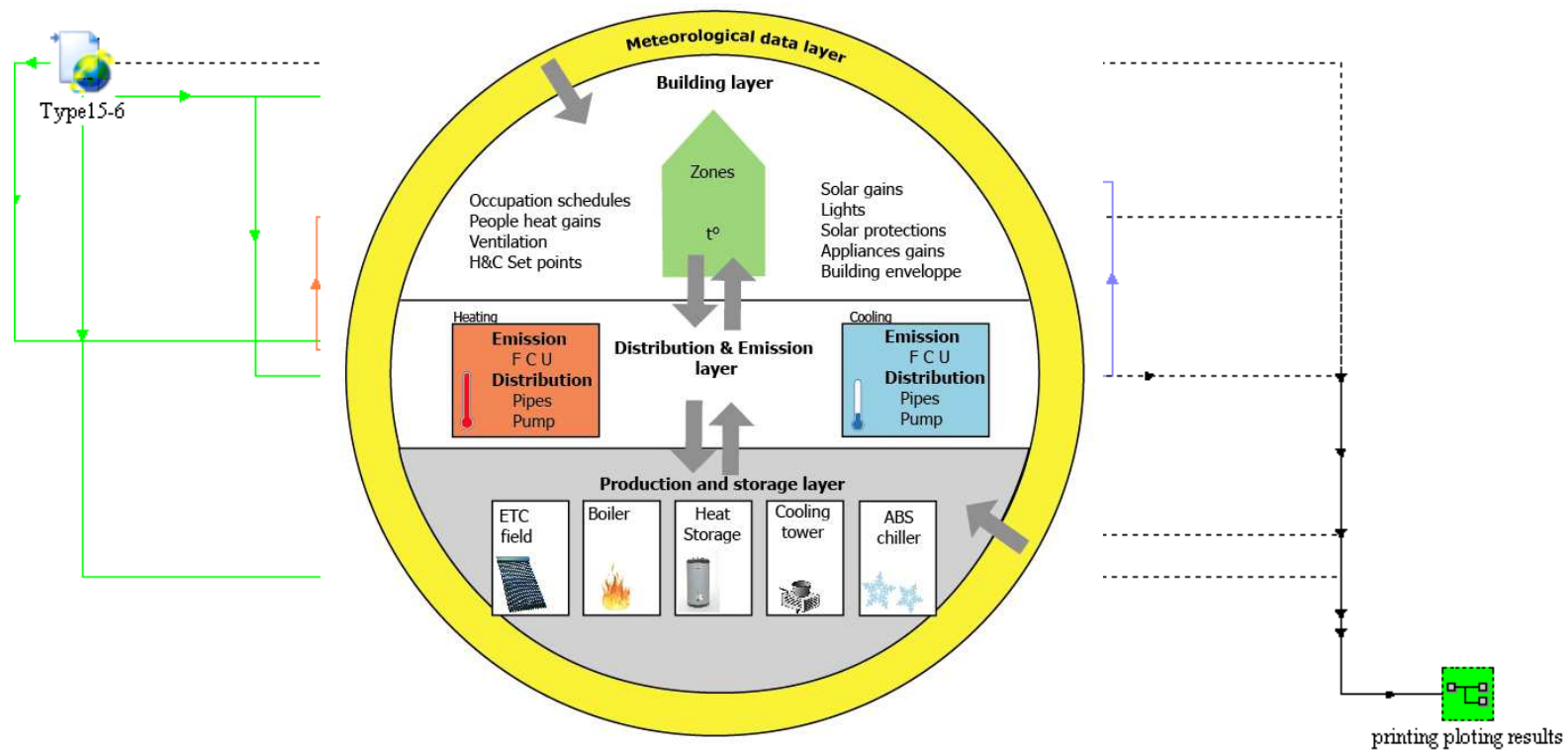
Sub-systems implementation :

- Building
  - Heat and Cold distribution and emission
  - Heat and Cold production and storage
  - Climate
- Combined simulation of these sub-systems
- Possibility to substitute components



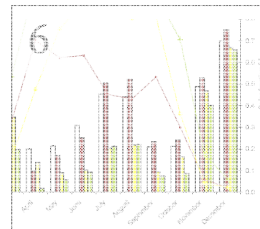
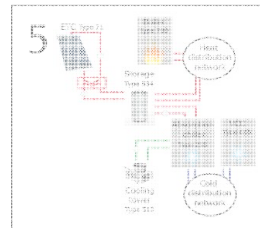
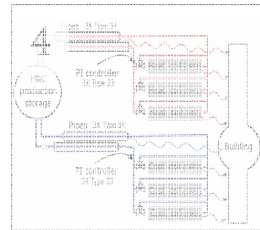
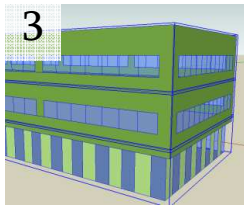
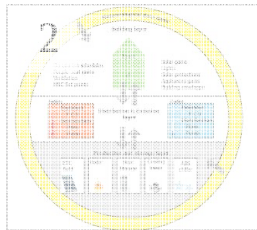
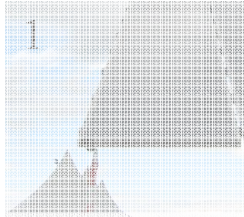
## Simulation environment overview

### *TRNSYS implementation of Sub-systems*



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## Building modelling

IEA-ECBCS 48 European typical office building 1-c

→ Paris Climate



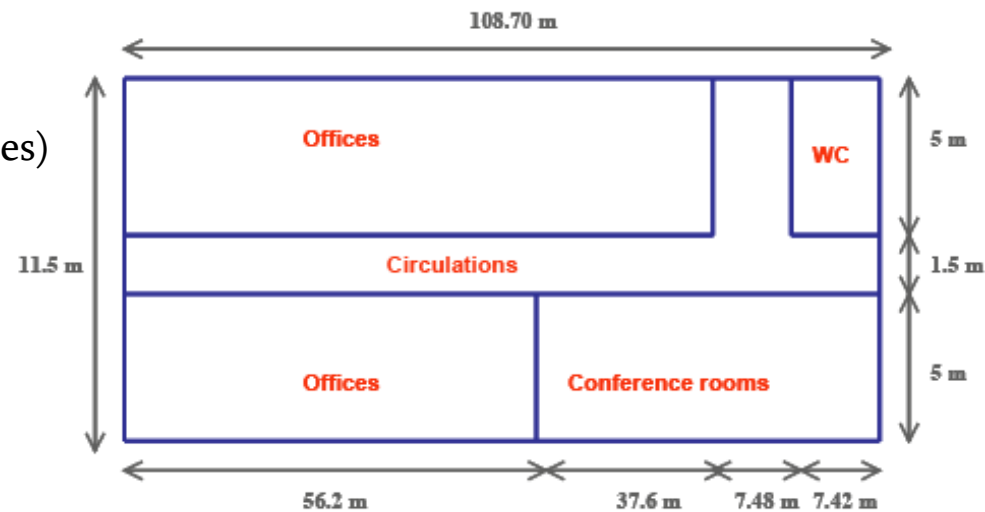
### Building modeling (one floor)

including :

- 5 thermal zones (3 for H&C)  
(glazing on South and North facades)
- Ventilation
- External shading modulation
- Light intensity modulation
- Occupancy profile for each zone
- Internal gains profiles
  - > People
  - > Appliances
  - > Light

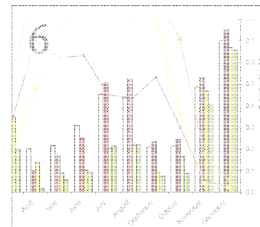
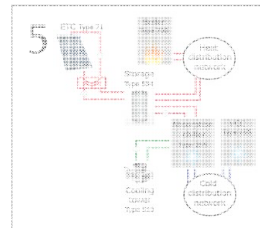
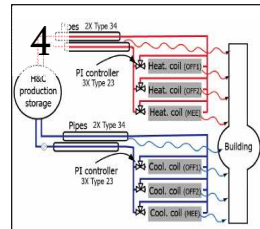
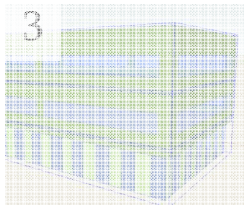
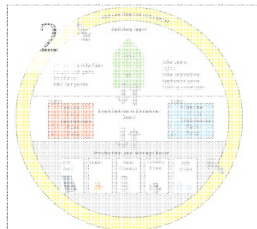
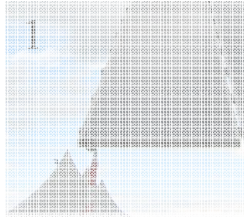
**Total area : 3750 m<sup>2</sup> (1250 m<sup>2</sup>/floor)**

**Around 250 people**



Stabat P. 2007. IEA48 – Description of Type 1c air-conditioned office buildings for simulation  
IEA-ECBCS Annex 48 working document.

## Presentation overview

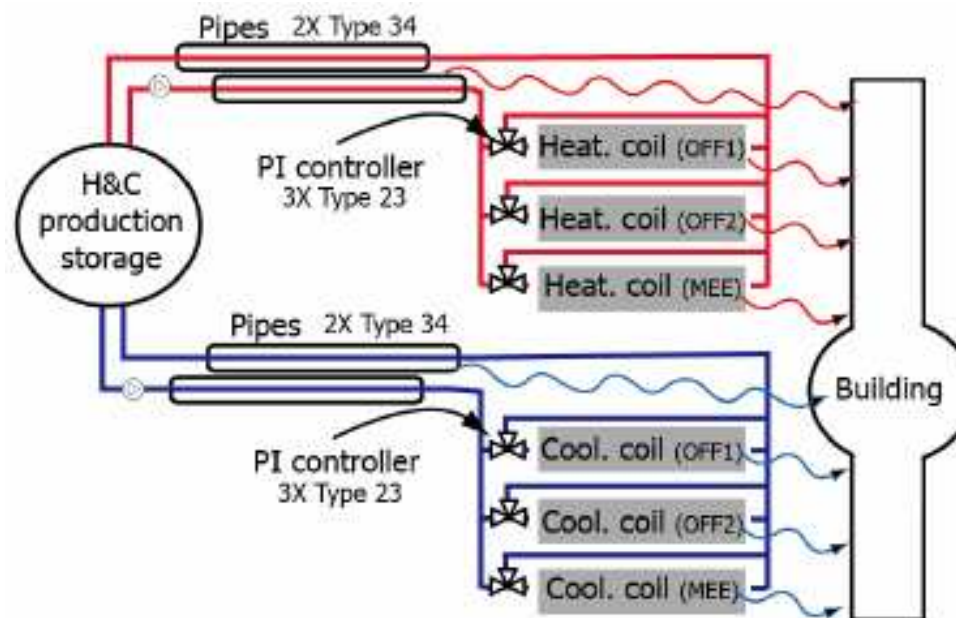


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## Heat and Cold emission and distribution

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*This sub-system is linked with H&C production and building sub-systems*



Pipes, pumps, control and fan coil units are modelled

## Fan Coil Units modelling

Objective : A real FCU where manufacturer data is available should be implemented.

Existing heating/cooling coil classical TRNSYS models are not suited to this objective



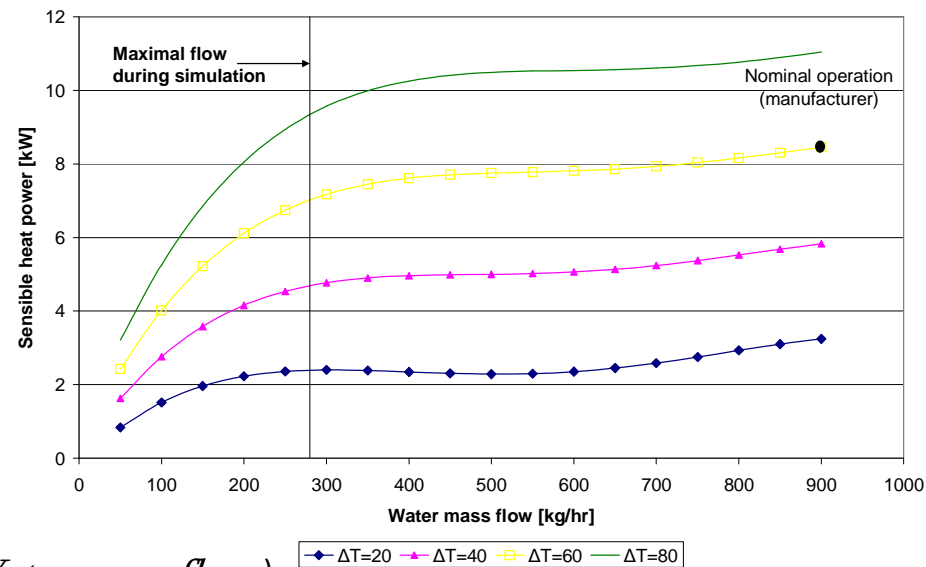
Polynomial approximation of the coil is implemented (based on manufacturer data)

Head/Cold emission is controlled by a thermostatic valve (PI controller) modifying the water mass flow.

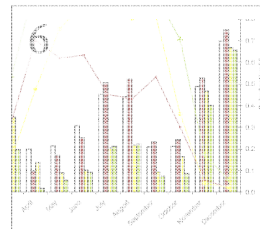
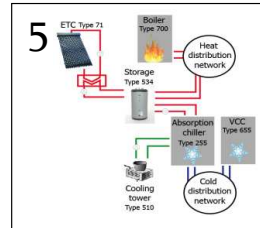
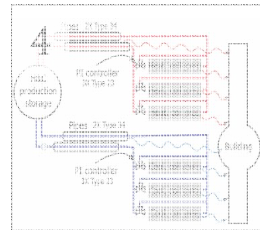
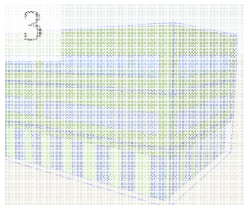
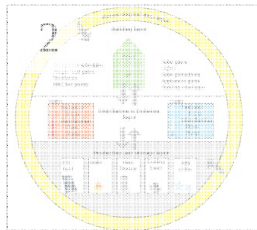
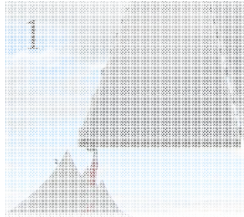
*Heating coil sensible*

*heating energy =*

*$f((T_{\text{water supply}} - T_{\text{room}}), \text{Water mass flow})$*



## Presentation overview



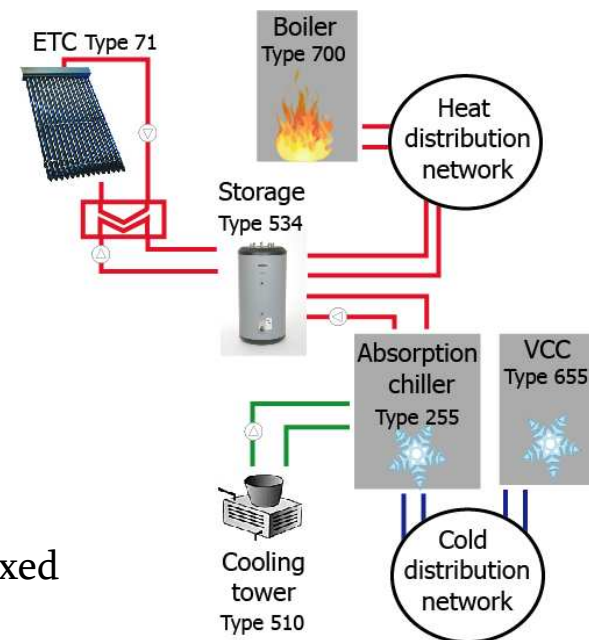
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## Heat and Cold production and storage

Sub-system main components (for one floor) :

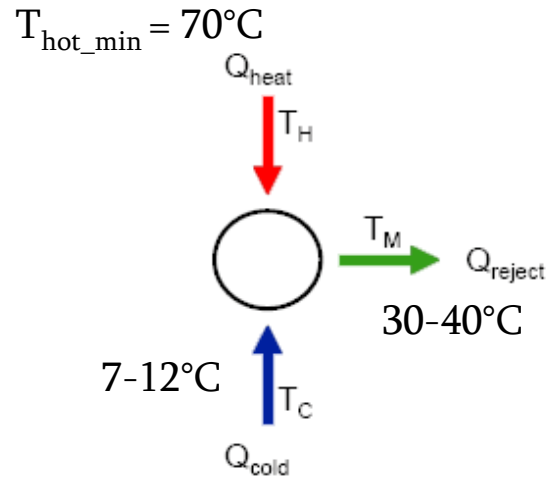
- Evacuated tube collector 142 m<sup>2</sup>
- Storage tank 7 m<sup>3</sup>
- Absorption chiller (ABS) 105 kW<sub>C</sub>  
COP<sub>nom</sub> 0.695
- Back up gas boiler 150 kW
- Cooling tower 263 kW
- Backup chiller (VCC) 105 kW<sub>C</sub>  
COP<sub>nom</sub> 3.5

For each component, parameters have been fixed based on market available equipment.



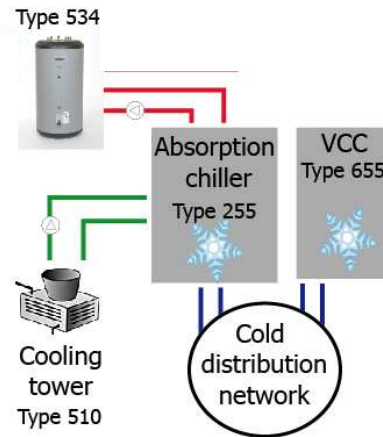
## Cold production : Absorption chiller

### Thermodynamic scheme



$$0.55 < COP = \frac{Q_{\text{cold}}}{Q_{\text{hot}}} < 0.73$$

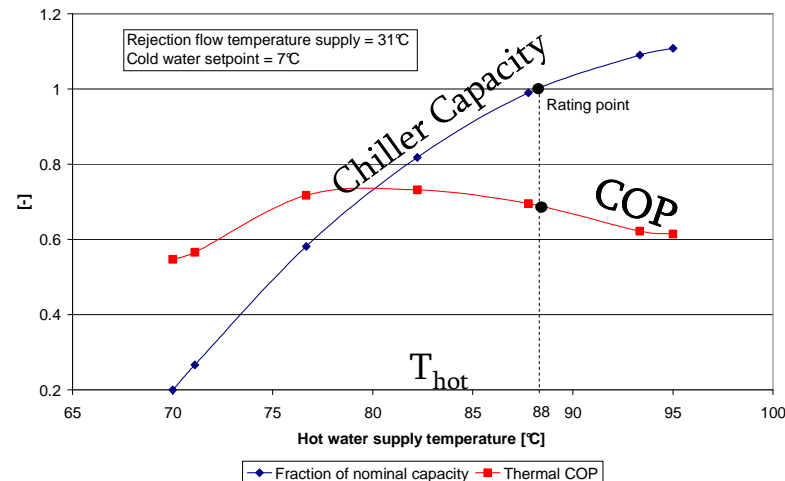
ABS chiller model in TRNSYS :  
 Energy balance  
*No transient effects !*



### Cooling control:

Absorption chiller switched on if storage temperature high enough.

Vapour comp. chiller in every other case



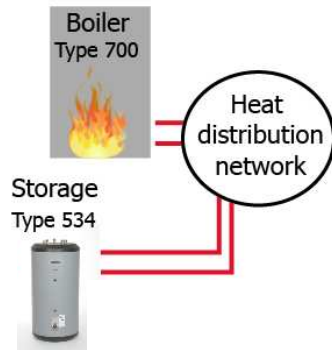
## Heat production : Gas boiler and storage tank

### Gas boiler

Efficiency depends on  
part load ratio : 88.2-89.2 %

Losses at 0% load : 1.3 kW

Boiler set point : heating curve



### Heating control:

Hot water comes from  
Storage if temperature  
higher than heating curve

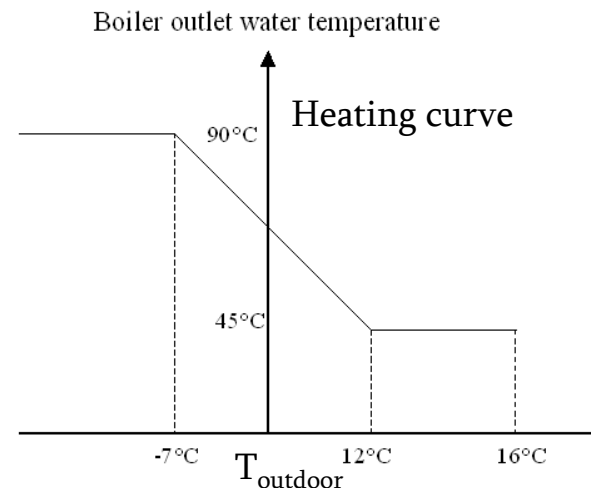
Gas boiler  
in every other case

### Storage tank

Heated by solar collectors only

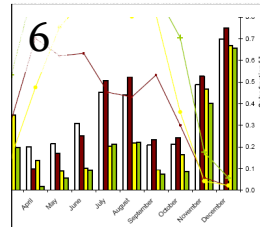
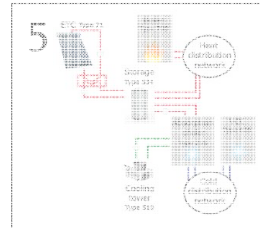
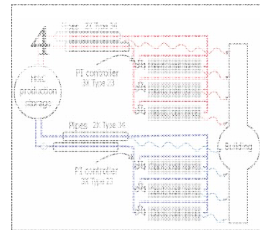
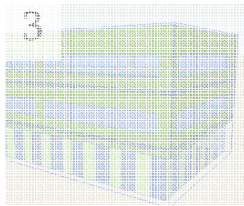
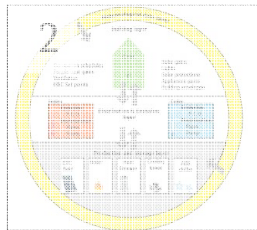
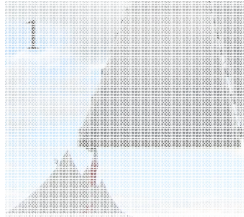
Two heat sinks :

- building heating network
- absorption chiller



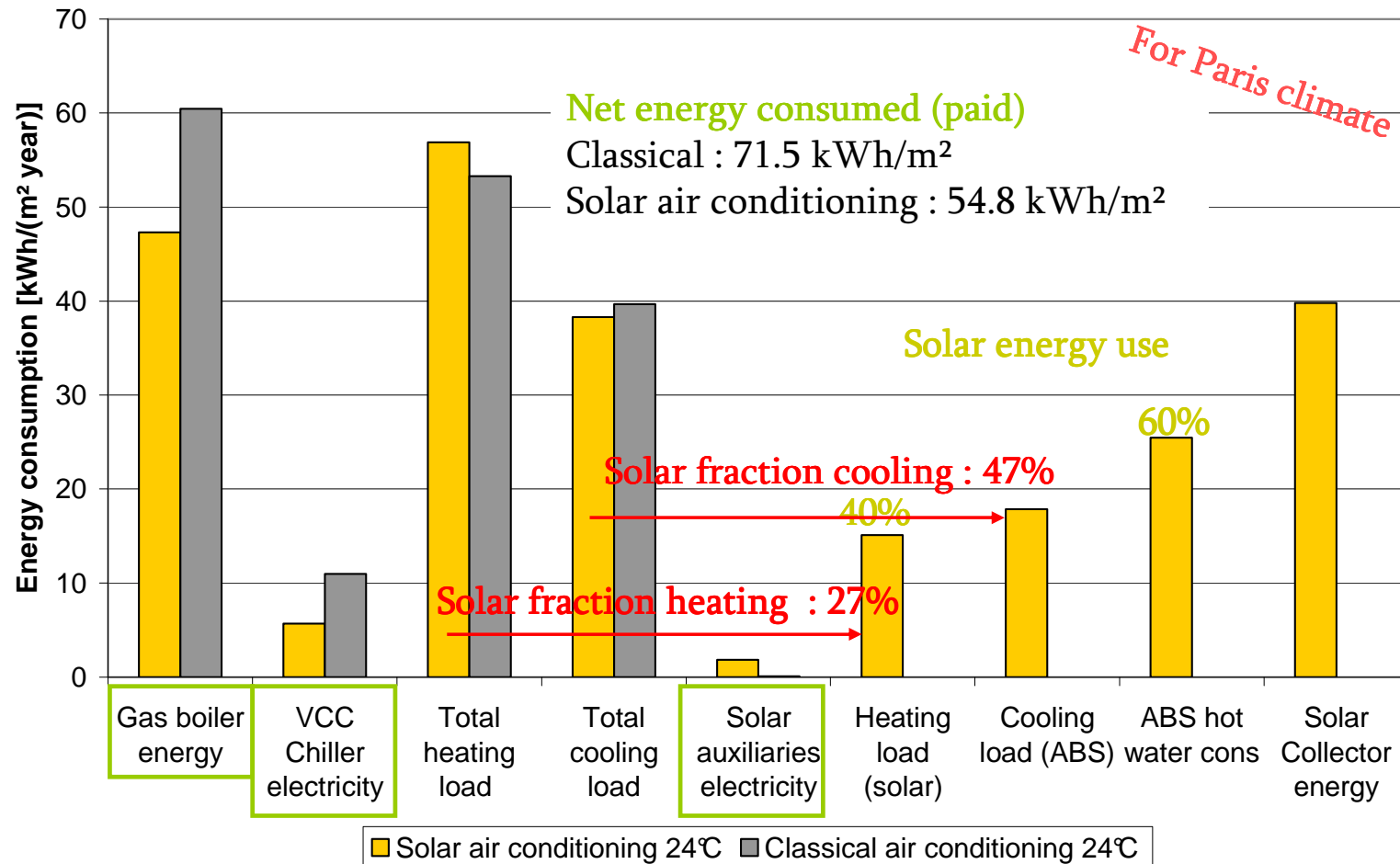
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## Yearly results : net energy for heating and cooling



## Yearly results : performance indicators and auxiliaries

### Performance indicators

Case	Solar fraction	Solar fraction Cooling	Collector energy	Primary energy savings per collector area	Yearly ABS COP	Elec. COP
Units	[-]	[-]	[kWh/m <sup>2</sup> coll]	[kWh/m <sup>2</sup> coll]	[-]	[-]
24°C Set point	0.43	0.47	350.19	192.12	0.70	9.85

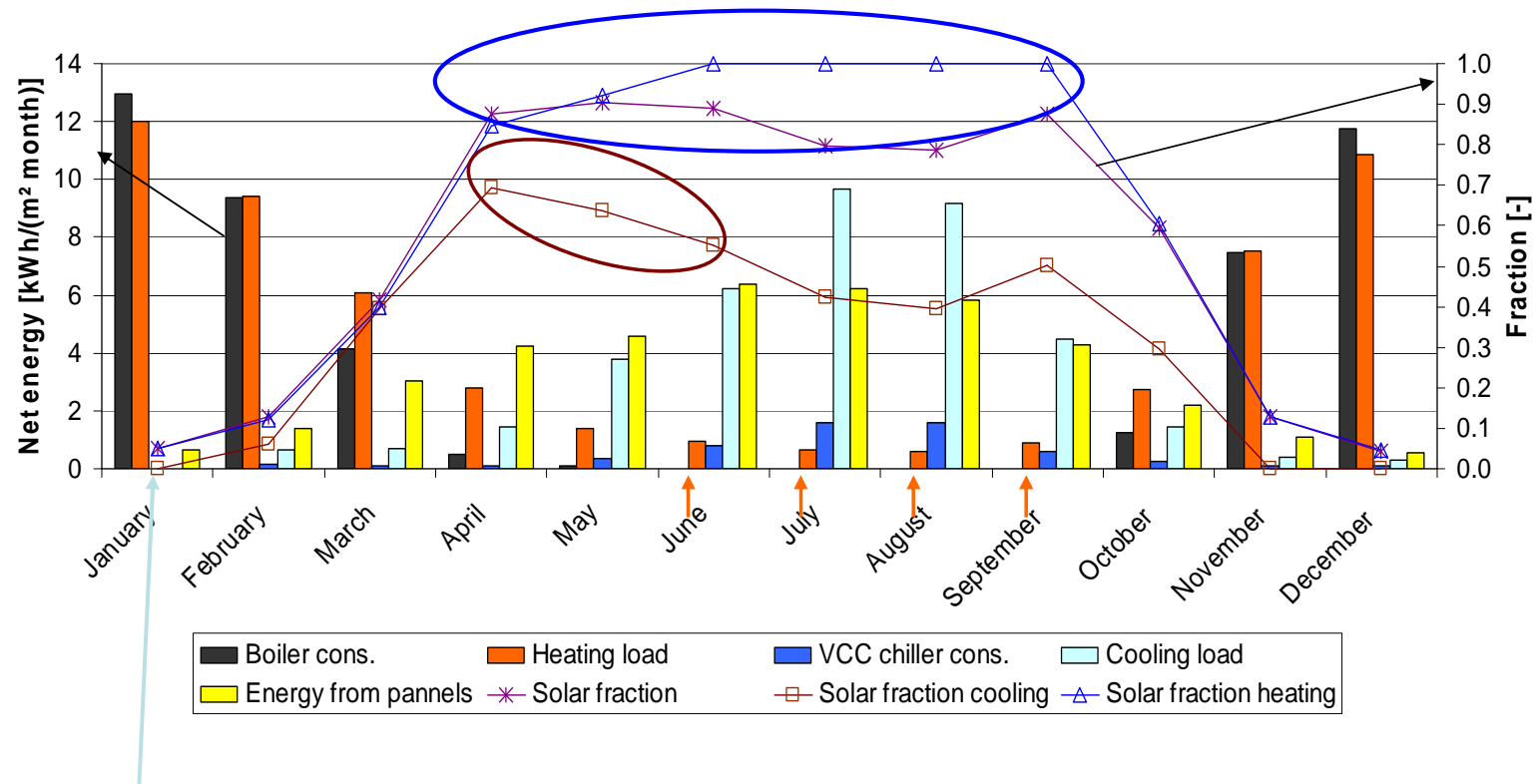
$$SF = \frac{\text{Collector energy}}{\text{Collector energy} + \text{Boiler cons.} + \text{VCC cons.}}$$

$$SF_{\text{Cooling}} = \frac{\text{Load met by ABS chiller}}{\text{Load met by ABS chiller} + \text{Load met by VCC chiller.}}$$

### Building auxiliaries consumption

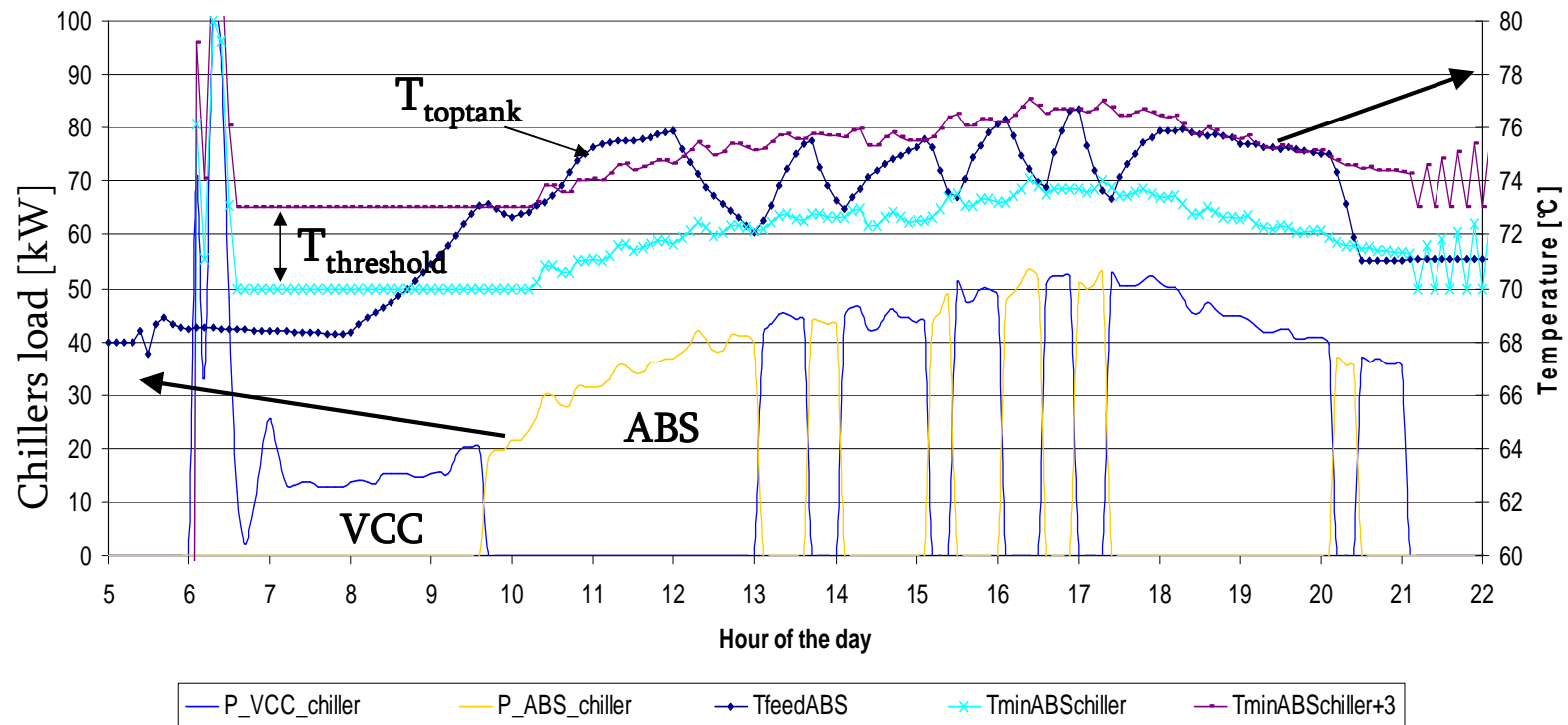
	Auxiliaries sum (primary)	Auxiliaries sum (elec.cons.)	Hot pump	Cold pump	FCU Fans	Ventilation	Appliances	Light
$\frac{kWh}{m^2 \text{ year}}$	180.03	72.01	1.09	10.40	8.08	3.72	25.56	20.36

## Monthly results : Energy for heating and cooling



No cooling load

## Daily results : one typical hot day



## Conclusions and next developments

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A comprehensive coupling between an office building and a solar air-conditioning application and provided results about the whole energy consumption.

Solar collector field and absorption chiller achieve considerable energy savings 22-23 % compared to classical air-conditioning using electricity driven vapour compression chiller. Moreover some performance indicators are presented that gives the possibility to assess energy efficiency of such systems.

Solar air-conditioning is efficient when used in efficient buildings. Work can be done to decrease cooling load. In actual office building simulations, auxiliaries have a great impact on the whole building energy consumption.

Limitations:

Steady state absorption chiller model, handling of the latent load not accurate

Next development :

Parametric analysis : internal load variation, collector field size,...

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