

A Local Frame Approach for Line-to-Line Contact between thin Beams with circular Cross-Sections

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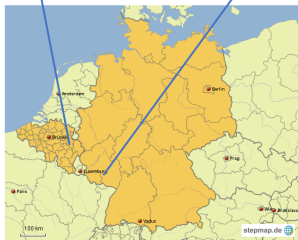
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Outline

- 1 Context
- 2 The Local Frame Approach
- 3 The Mortar Method
- 4 Problem Formulation
- 5 First Results
- 6 Conclusions and Perspectives

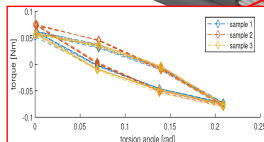
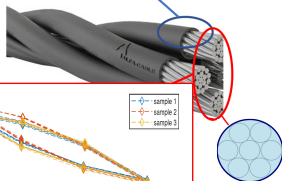
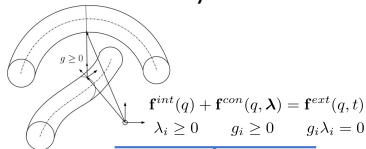
Mesoscopic Cable Modeling

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THREAD
EUROPEAN TRAINING NETWORK

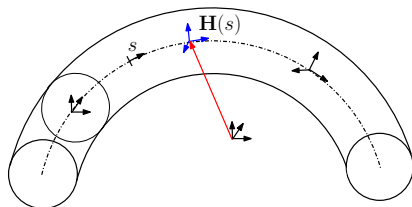
Flexible Multibody Contact



Constitutive modeling

Geometrically exact beam in $SE(3)$

Simo beam revisited (large rotations, shearing): [Sonneville 2013]



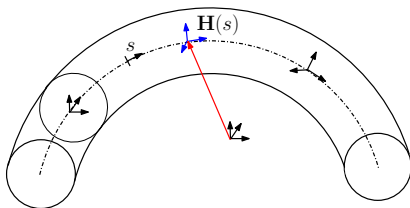
- Frames as unknowns:
$$\begin{bmatrix} \mathbf{R} & \mathbf{x} \\ \mathbf{0} & 1 \end{bmatrix} = \mathbf{H}(s) \in SE(3)$$
- Simple access to strains through the **Lie group derivative**
- Solve the dynamics in the **local** frame (reduced non-linearity)

Kinematic equation

$$\mathbf{H}'(s) = \mathbf{H}(s)\tilde{\mathbf{f}} \quad \implies \quad \boldsymbol{\epsilon} = \mathbf{f} - \mathbf{f}_0$$

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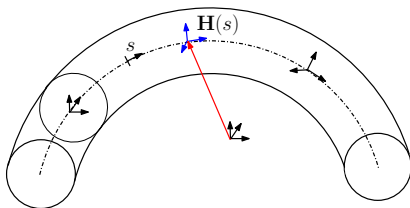
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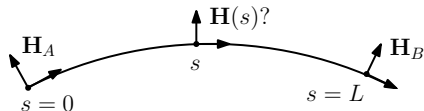
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Beam finite element on $SE(3)$

Verify the kinematic equation by construction



- Interpolation formula

$$\mathbf{H}(s) = \mathbf{H}_A \exp(s\tilde{\epsilon}^*)$$

- Discrete deformation

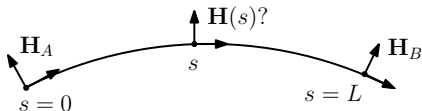
$$\tilde{\epsilon}^* = \frac{\log(\mathbf{H}_A^{-1}\mathbf{H}_B)}{L}$$

Discretization

- First order interpolation generalized to a non-linear space
- Frame invariant interpolation
- Natural coupling of translation and rotation
- Automatically locking free
- Geometric non-linearities reduce with mesh refinement

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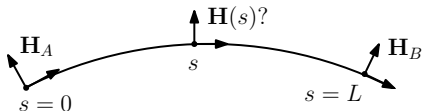
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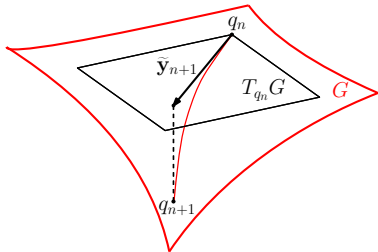
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- **First order** interpolation generalized to a non-linear space
- **Frame invariant** interpolation
- Natural **coupling** of translation and rotation
- Automatically **locking free**
- Geometric non-linearities reduce with mesh **refinement**

Lie group time integrator

The equations of motion are directly formulated on the Lie group. The numerical solution at every step **remains** on the manifold. [Bruls 2012]



$$q_{n+1} = q_n \circ \exp(\tilde{y}_{n+1})$$

$$\mathbf{f}^{int}(q_{n+1}) = \mathbf{f}^{ext}(q_{n+1}, t_{n+1})$$

Integration scheme

- No global **parametrization** of rotation
- **Local** parametrization of the manifold at each time step using the **exponential map**
- Only admissible **group operations**
- **Simple** implementation

No tricks! Just the formalism.

State of the art for beam contact

- Point contact
- **Node** to segment approaches

Correct representation of **contact forces**?

Integration of contact contributions

- Line to line contact with **penalty** approach. [Meier 2016]

→ **Smooth** contact forces

→ Dependence of the solution on the penalty **parameter**

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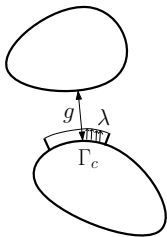
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Mortar methods for unilateral contact

Weak enforcement of the non-penetration condition using Lagrange multipliers. [Popp 2012]



$$g \geq 0 \quad \lambda \geq 0 \quad \lambda g = 0$$

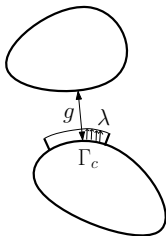
\downarrow

$$\int_{\Gamma_c} (\delta\lambda - \lambda)g \, d\gamma_c \geq 0 \quad \forall \delta\lambda \in \mathcal{M}$$

- Numerical **stability** and **optimal convergence** rates if correct discretized spaces are used (**inf-sup** condition)
- Usually combined with **active set strategies** to deal with the inherent **non-smoothness** of unilateral contact problems
- Difficulty: Finding the **contact region**

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Contact kinematics and constraint gradient

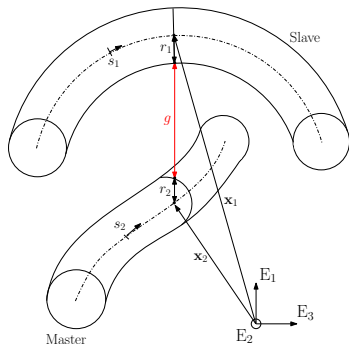
"1" is the slave or non-mortar beam and "2" the master or mortar beam

$$q = \{\mathbf{H}_1, \mathbf{H}_2\}, \quad \delta \mathbf{h} = \begin{bmatrix} \delta \mathbf{h}_1 \\ \delta \mathbf{h}_2 \end{bmatrix}$$

Circular cross section:

$$g(q) = \|\mathbf{x}_2 - \mathbf{x}_1\| - r_1 - r_2$$

$$\delta g(q) = \mathbf{G}(q) \delta \mathbf{h}, \quad \mathbf{G}^T = \begin{bmatrix} -\mathbf{M}\mathbf{N} \\ \mathbf{M}\mathbf{R}_2^T \mathbf{R}_1 \mathbf{N} \end{bmatrix}$$



Simplified kinematics

- Kinematics entirely written in terms of the **centerline**
- The constraint gradient only depends on **local relative** quantities

Continuous formulation

Find $q \in \mathcal{U}$ and $\lambda \in \mathcal{M}$ such that

$$\begin{aligned}\delta \mathcal{W}_{int}(q, \delta \mathbf{h}) + \delta \mathcal{W}_{con}(q, \lambda, \delta \mathbf{h}) &= \delta \mathcal{W}_{ext}(q, \delta \mathbf{h}, t) \quad \forall \delta \mathbf{h} \in \mathcal{V} \\ \delta \mathcal{W}_{lag}(q, \delta \lambda) &\geq 0 \quad \forall \delta \lambda \in \mathcal{M}\end{aligned}$$

where we define

$$\begin{aligned}\delta \mathcal{W}_{con}(q, \lambda, \delta \mathbf{h}) &= \int_{s_1^a}^{s_1^b} (\delta \mathbf{h})^T \mathbf{G}^T \lambda \, ds_1 \\ \delta \mathcal{W}_{lag}(q, \delta \lambda) &= \int_{s_1^a}^{s_1^b} (\delta \lambda - \lambda) g \, ds_1\end{aligned}$$

Invariance property

- Equilibrium expressed in the **local frame**
- All contributions only depend on the **relative** configuration

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Semi discrete formulation

Idea: We use a **first order interpolation** for the configuration, thus we do the same for the Lagrange multipliers [Belgacem 1999]: $\lambda^h = \Phi(s_1)\lambda$

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Solution

- Solved using Lie group versions of semi-smooth Newton or **Augmented Lagrangian** methods. [Cavaliere 2012]
- The **weak** constraints are enforced **exactly**

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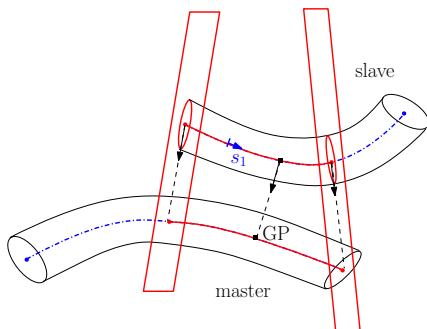
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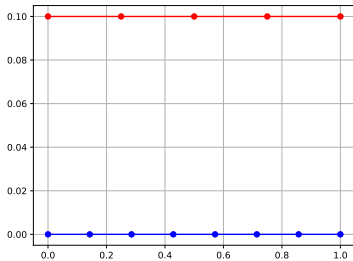
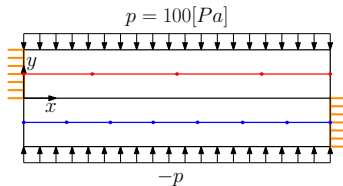
Simple computation of contact patch

Intersection between the slave cross section plane and the master centerline

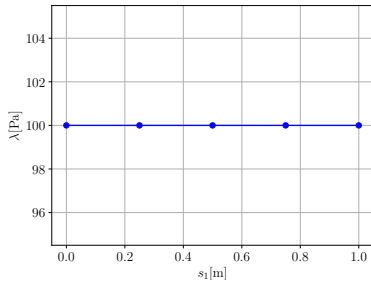
- Appropriate for **thin** beams with **circular** cross sections



Patch test: Two clamped beams



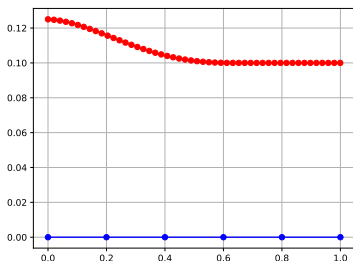
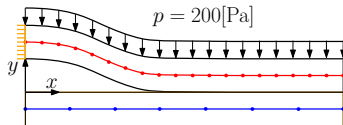
centerlines



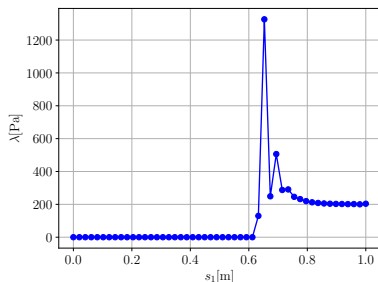
Lagrange multiplier

- Correct representation of **constant distributed pressure** field.

Cantilever: Non-smooth contact pressures



centerlines



Lagrange multiplier

- Imposed **kink in the curvature** produces impulse-like contact force at the contact interface. [Civilek 1975]

Flying helix spaghetti

Conclusion, perspectives and open questions

Conclusion

- In $SE(3)$ the contact formulation for beams has **interesting properties**
- The Mortar method allows one to correctly represent line contact geometries and an accurate computation of contact pressures.
- Both are **combined** with the aim to construct a robust method

Perspectives

- Use it to simulate realistic cable assemblies
- Extend the formulation to **friction**

Open Question

- Is it possible to derive a **unified model** that is efficient for both point and line contact?

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-  Sonnevile, V. et al. Geometrically exact beam finite element formulated on the special Euclidean group $SE(3)$. *Comput. Methods Appl. Mech. Engrg.* (2014) **268**:451-474.
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-  F. B. Belgacem and Y. Renard. Hybrid finite element methods for the Signorini problem. *Mathematics of Computation* (2003) **72**:1117-1145.

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