

A time-variant/invariant equivalence for the transient response of rotor blades in resonance crossing

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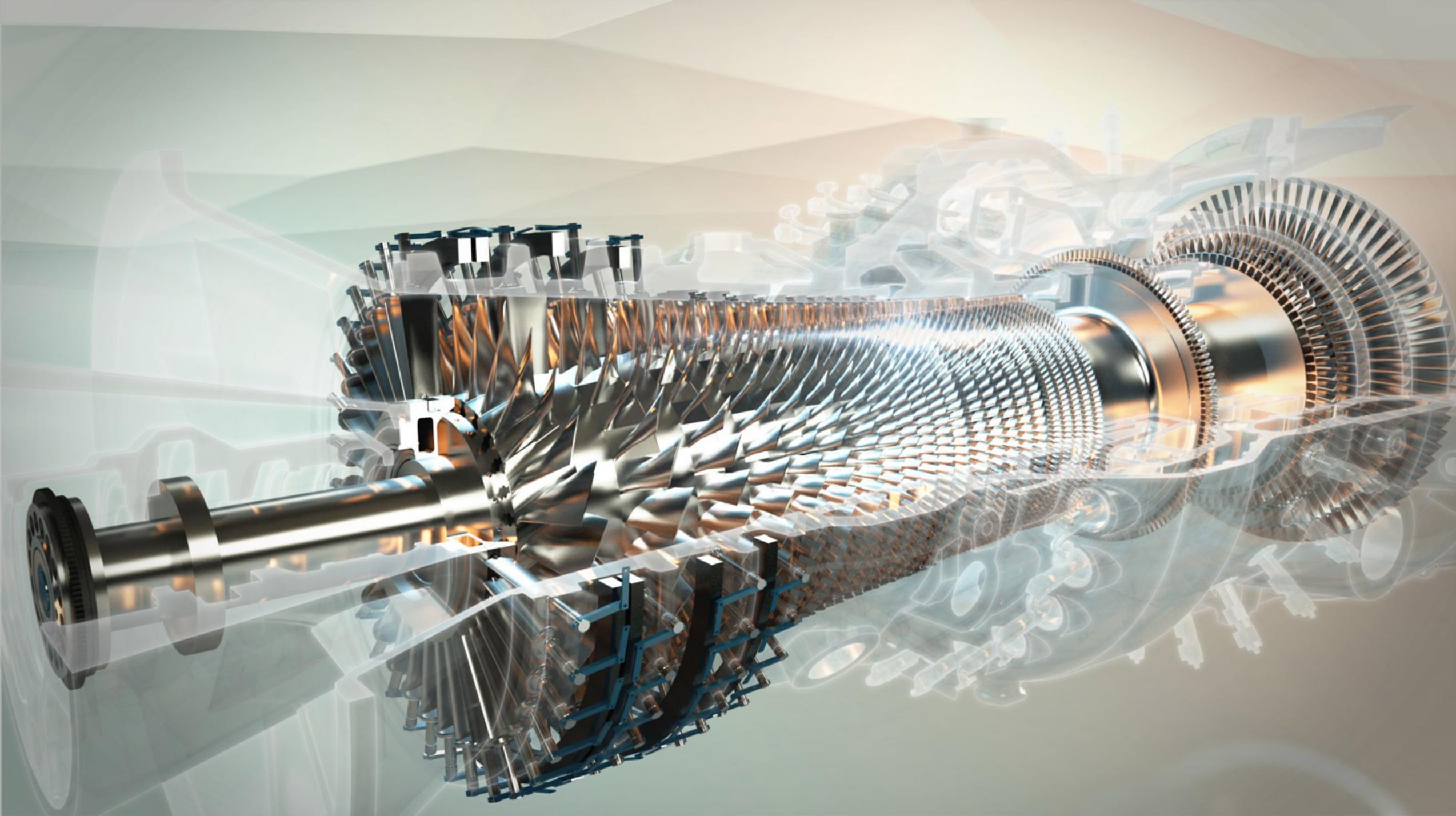


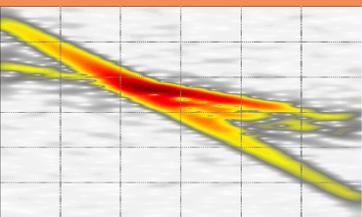
Summary

1. **Motivation.** *Modal identification of rotor components*
2. **Background.** *Why we like the Fourier transform*
3. Analytic signals and **demodulation.** *The most brilliant ideas... never work*
4. **Fourier transform, short-time Fourier transform, wavelet transform.** *On the origin of species (by means of natural selection ...)*
5. **Implementation and interpretation.** *How to make an elegant framework useful*
6. **Time-frequency spectral analysis.** *A numeric magnifying glass*
7. **System identification** by TFA. *Leveraging a 2D representation*

TO BE UPDATED

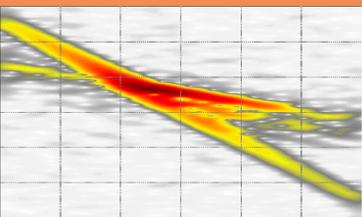
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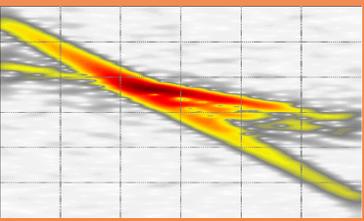


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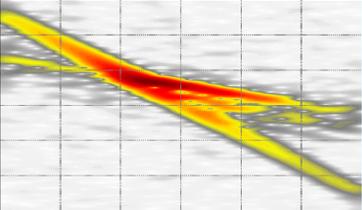


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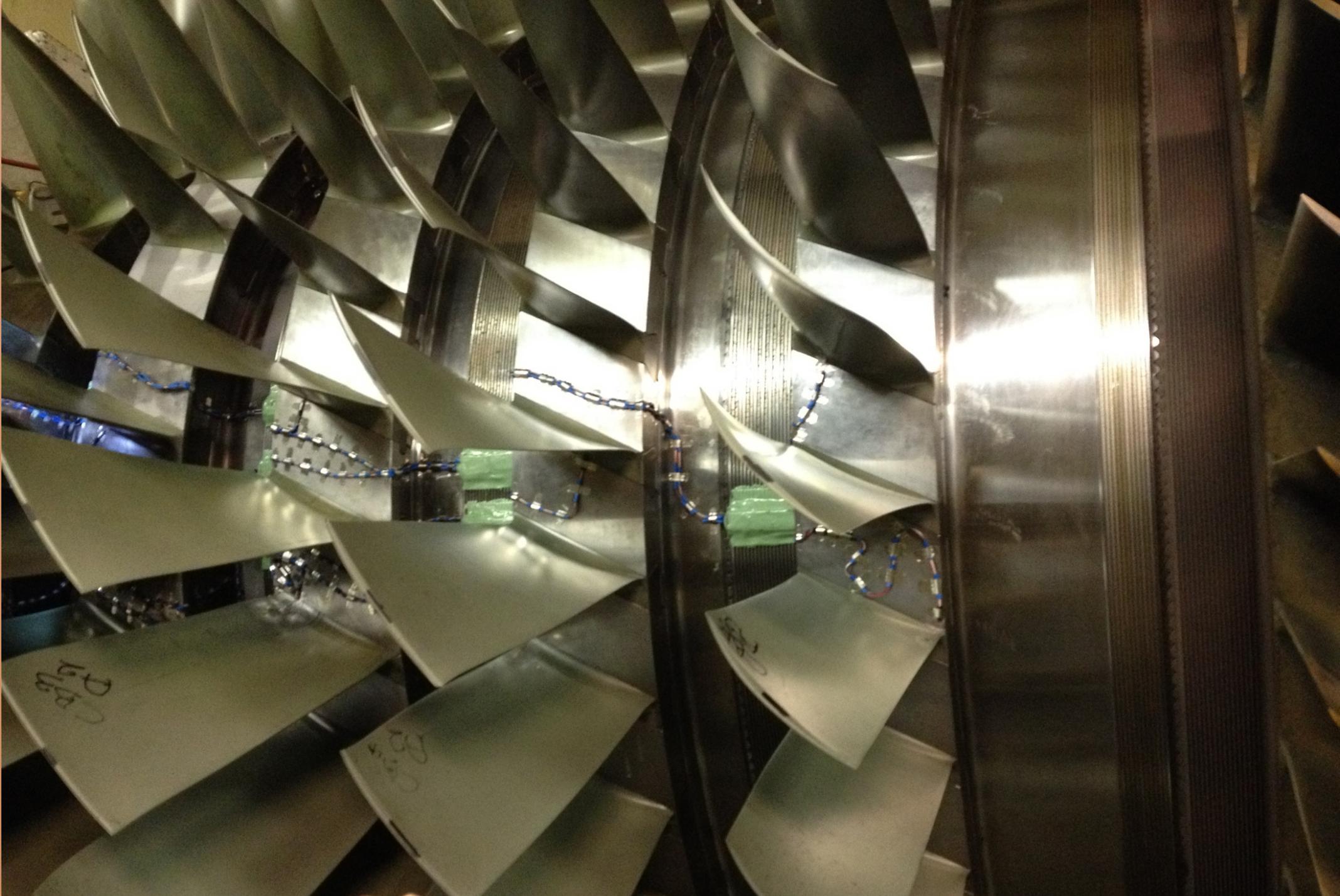


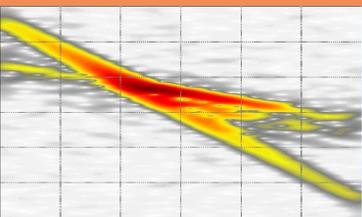
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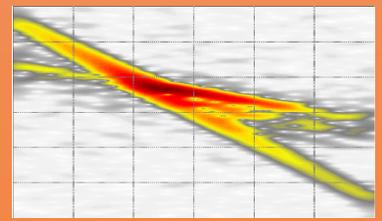




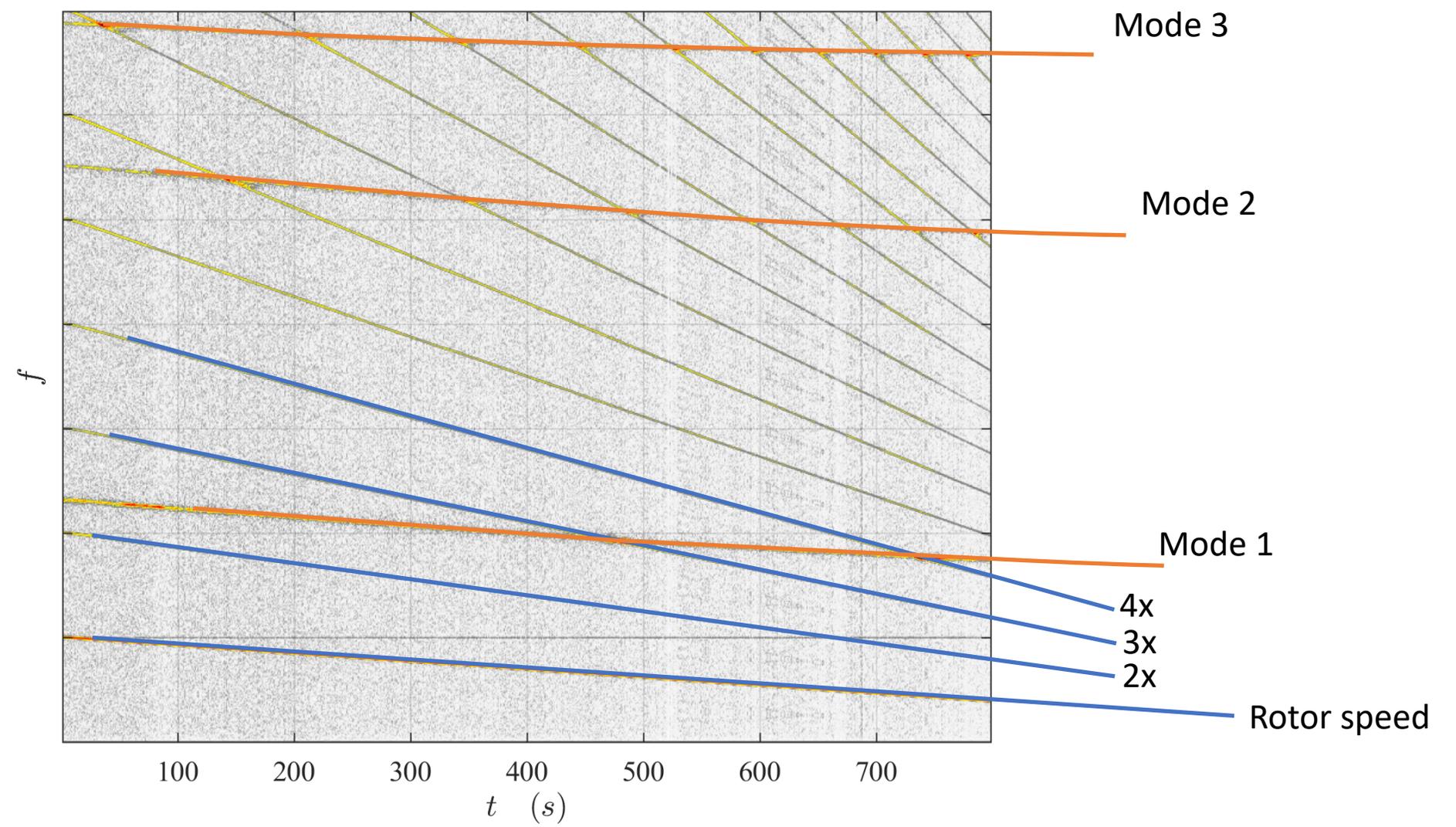
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Modal identification of rotor components

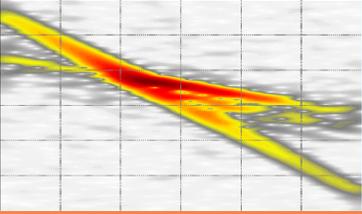
Campbell diagram



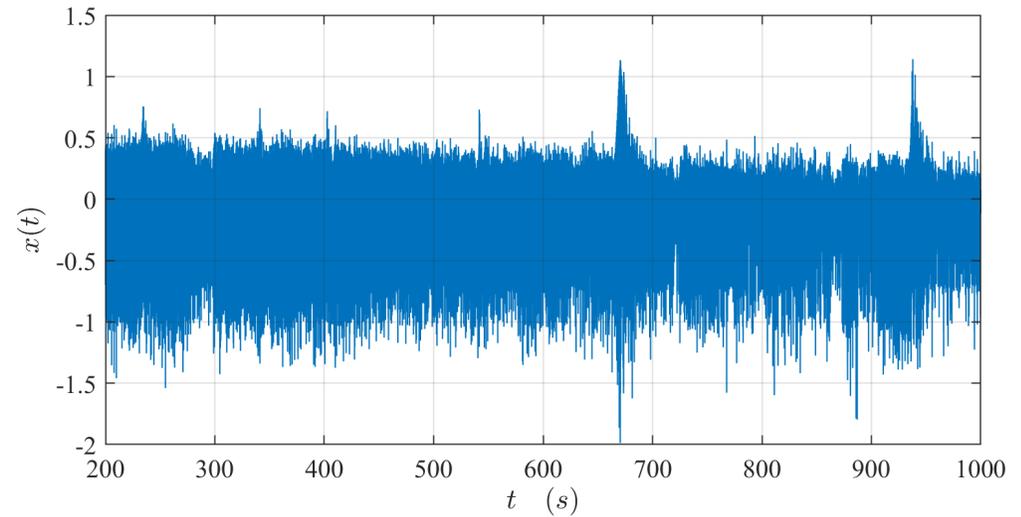
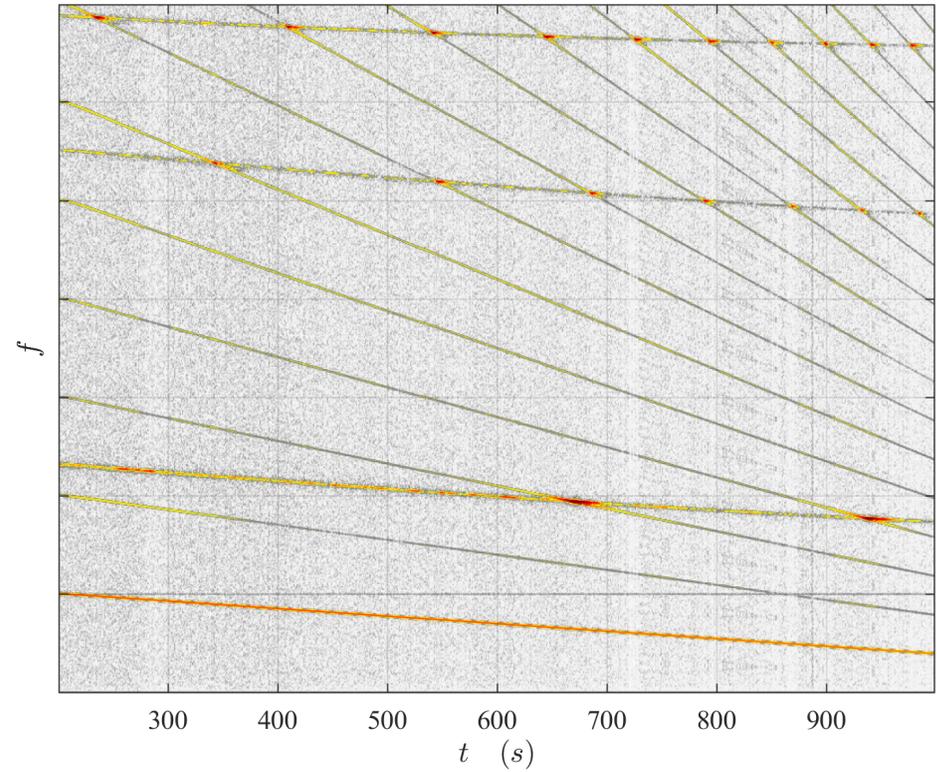
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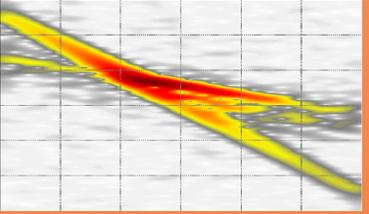
Resonance crossing



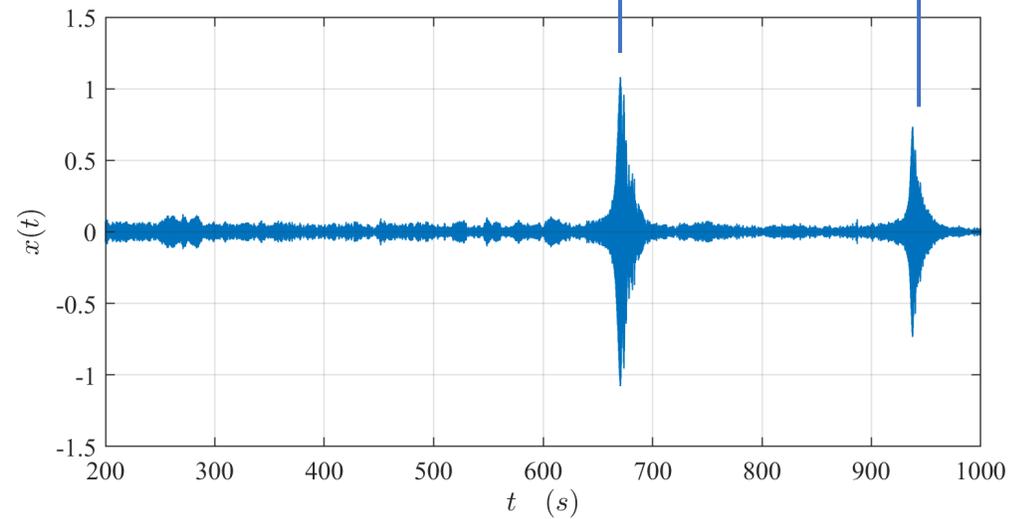
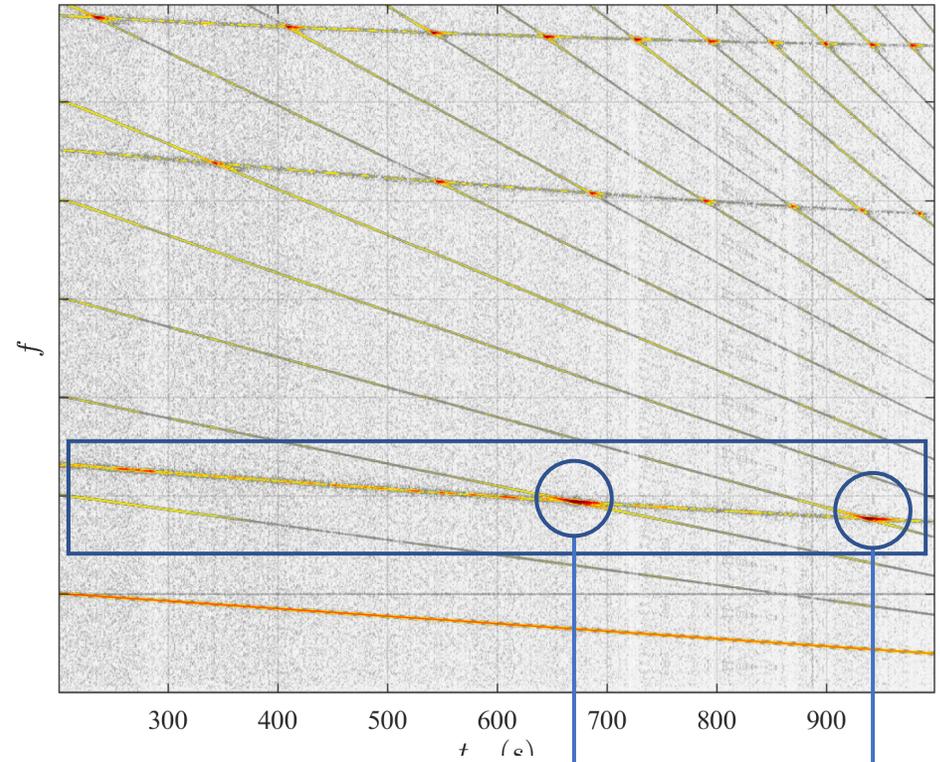
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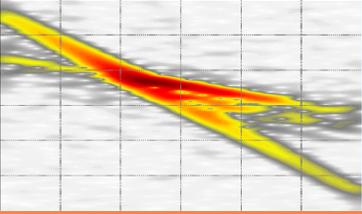
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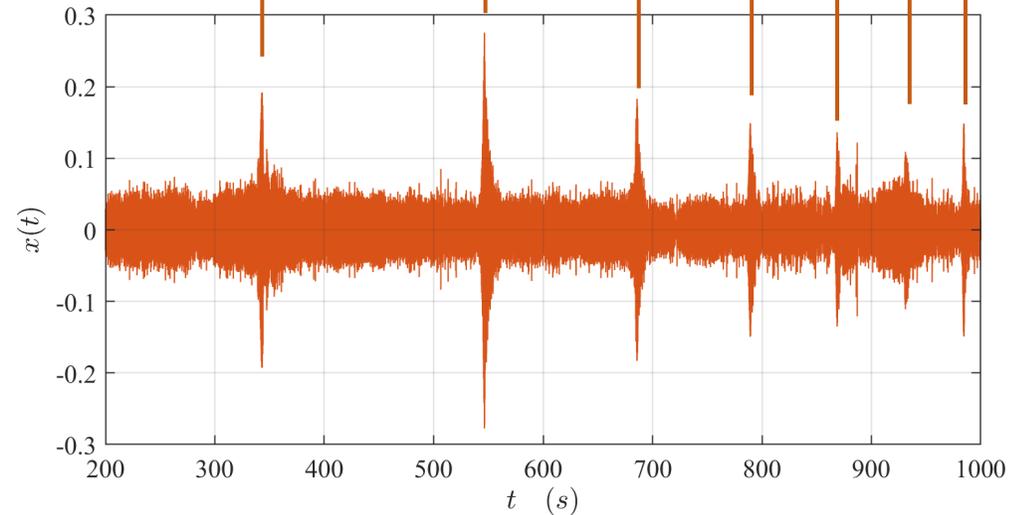
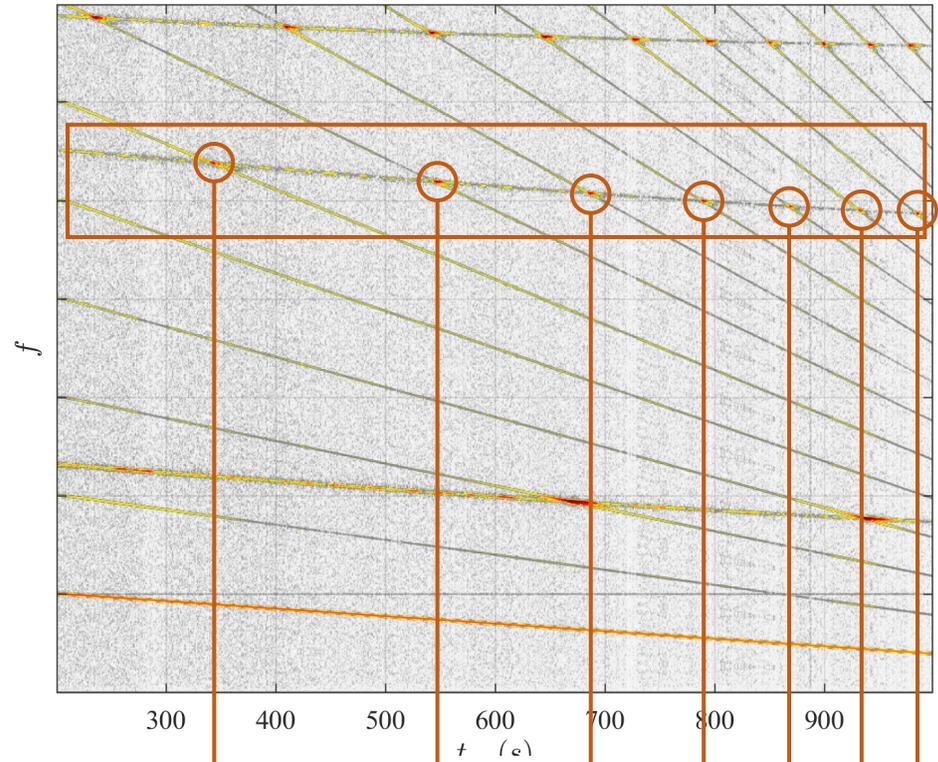
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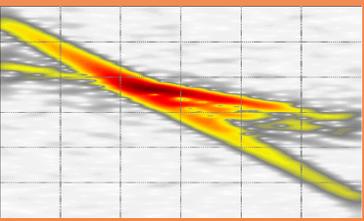
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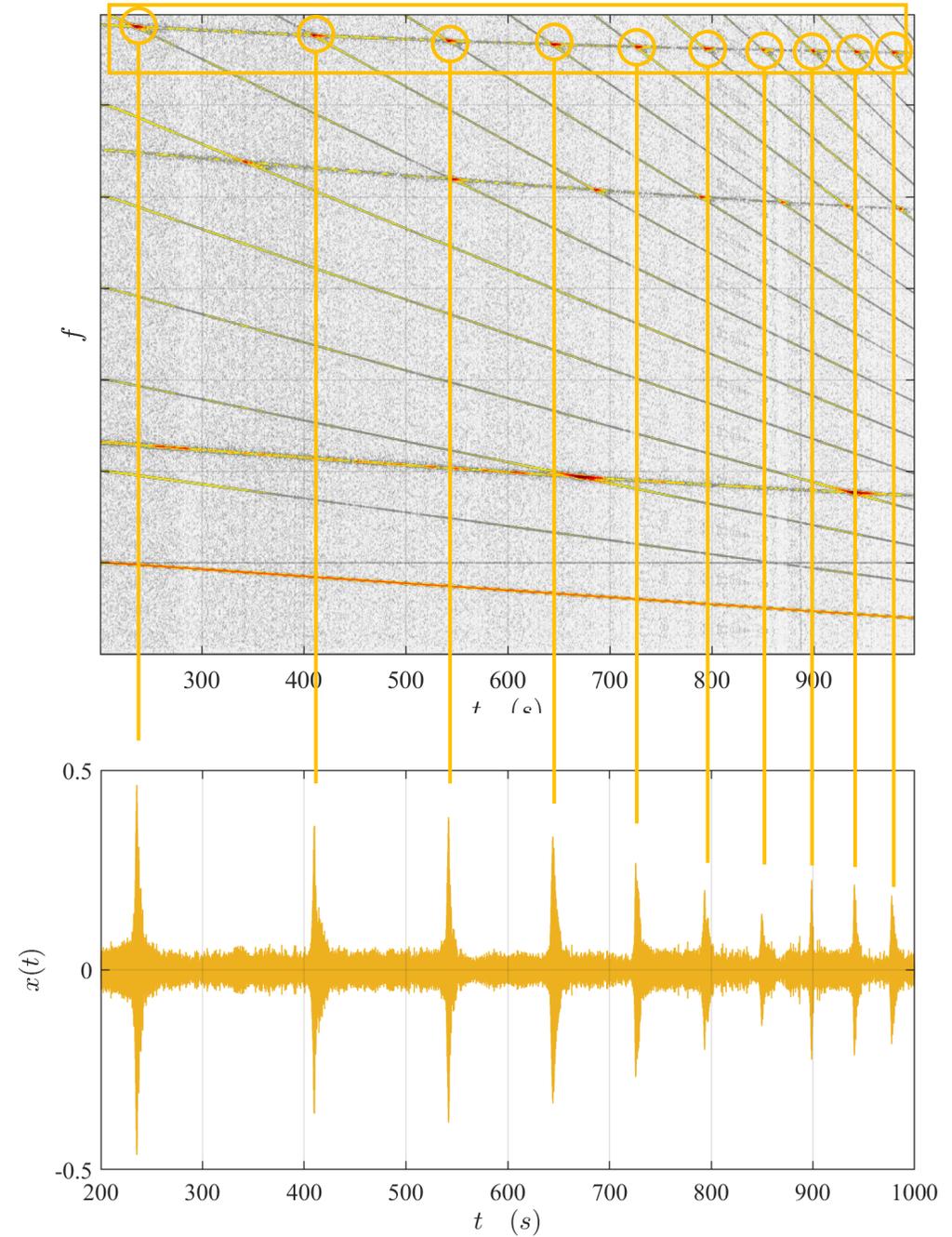
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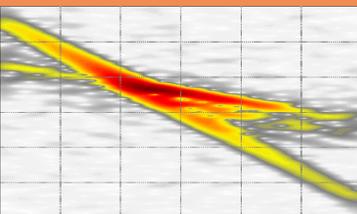
Resonance crossing



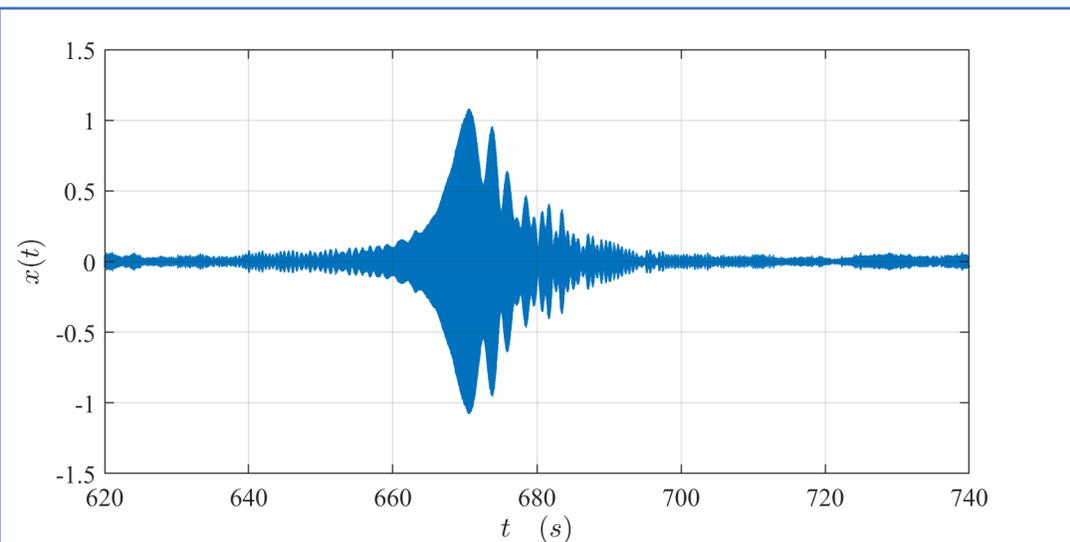
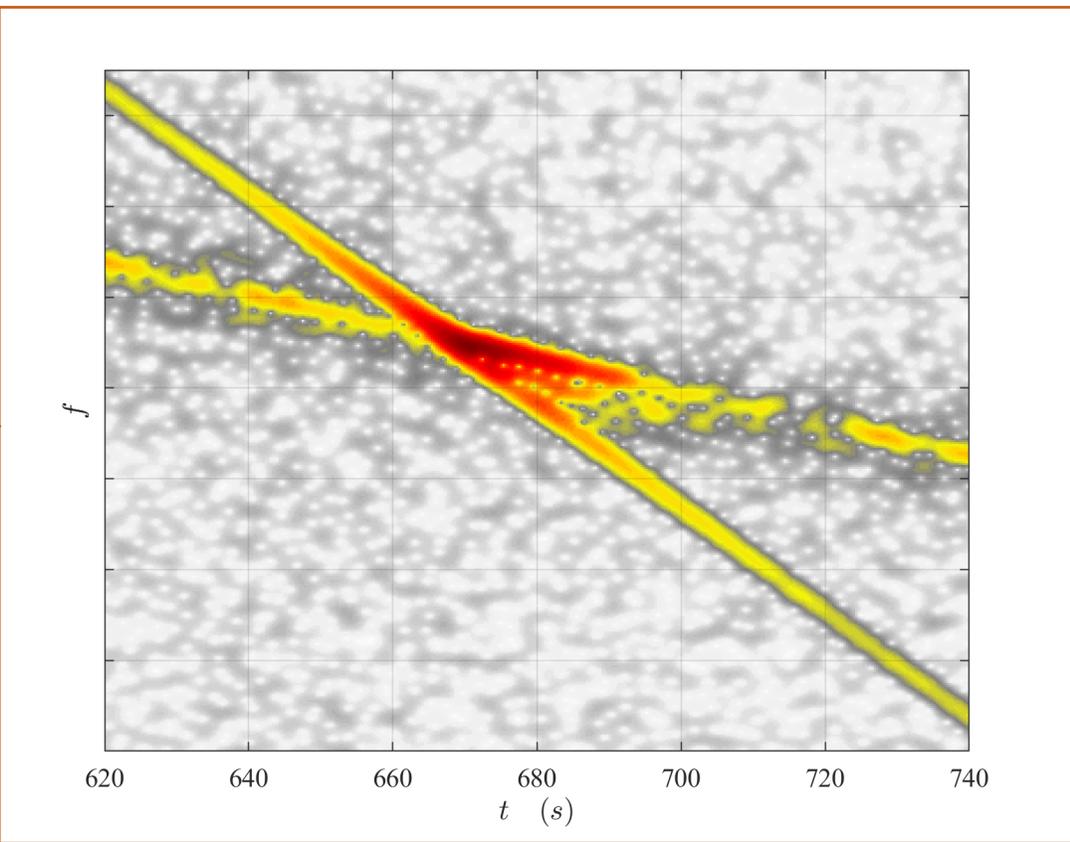
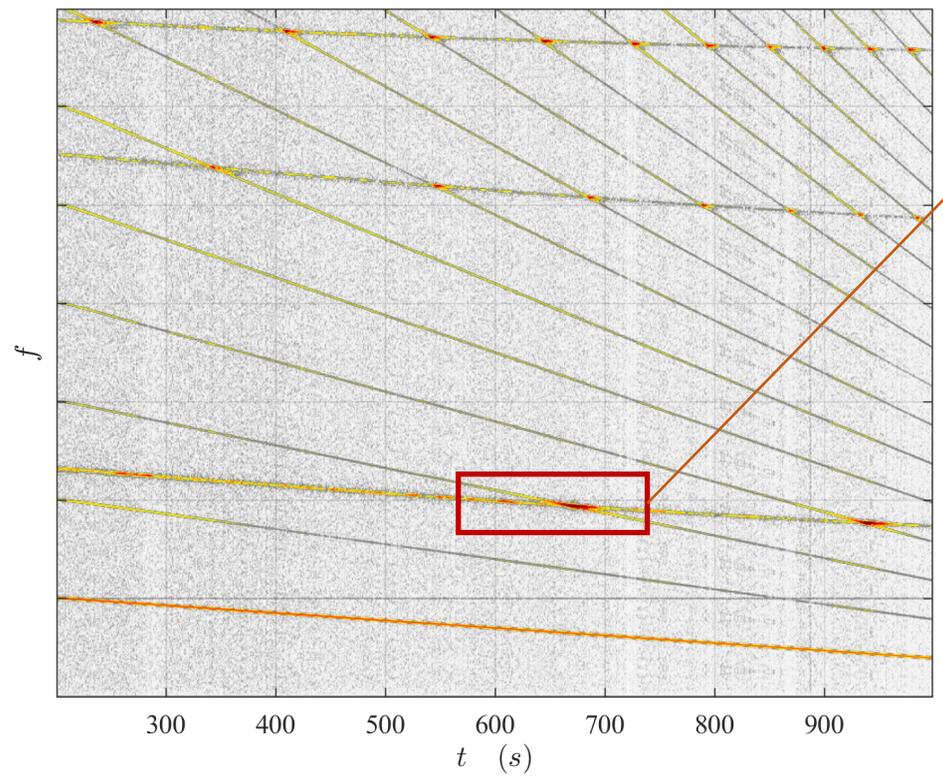
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Resonance crossing



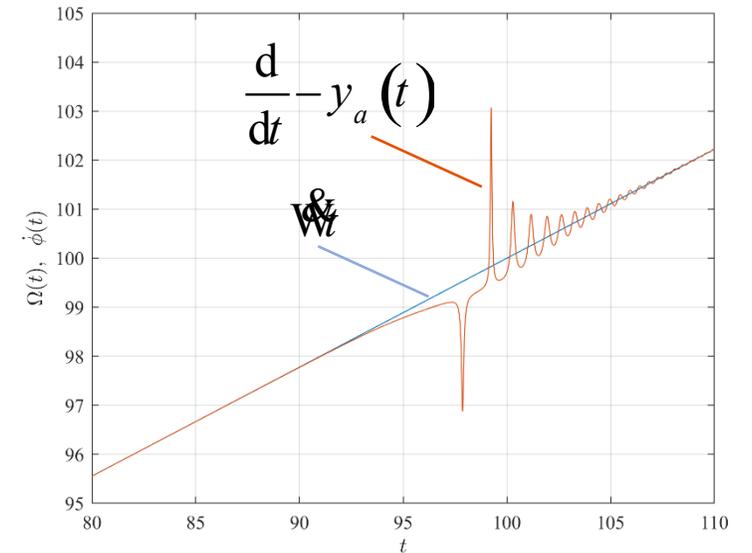
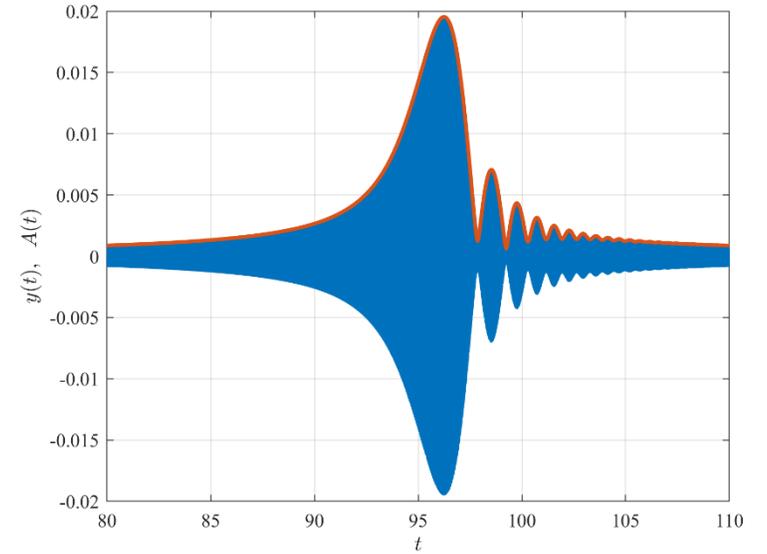
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Single-dof model

$$\ddot{x} + 2\xi\omega_n\dot{x} + \omega_n^2x = \frac{1}{m}p(t)$$
$$= \frac{P}{m}\cos\left(\frac{1}{2}\dot{\Omega}t^2\right)$$

$$\Omega(t) = \dot{\Omega}t$$



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Minimalistic modeling : single-DOF model

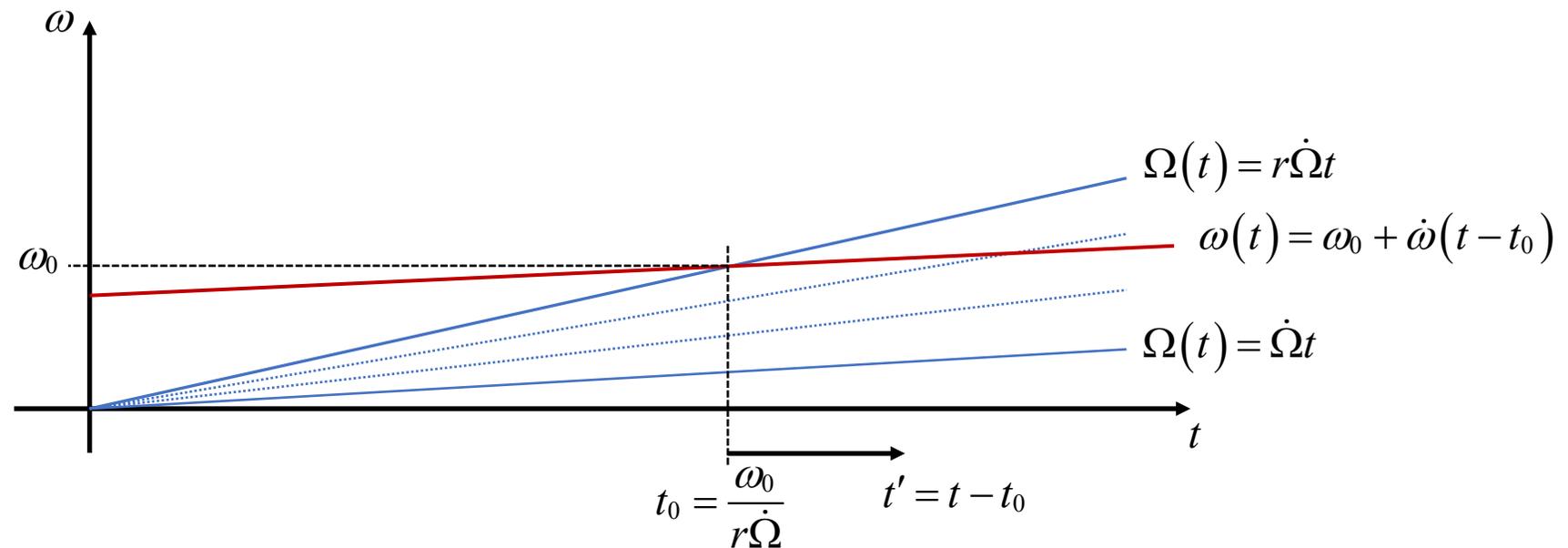
Governing equation

$$\ddot{q} + 2\xi\omega(t)\dot{q} + \omega^2(t)q = \frac{p_0}{m} e^{\frac{i}{2}r\dot{\Omega}t^2}$$

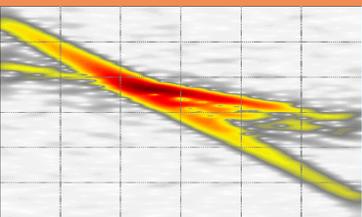
Time-varying natural frequency

$$\omega(t) = \omega_0 + \dot{\omega}(t - t_0) \quad t_0 = \omega_0 / r\dot{\Omega}$$

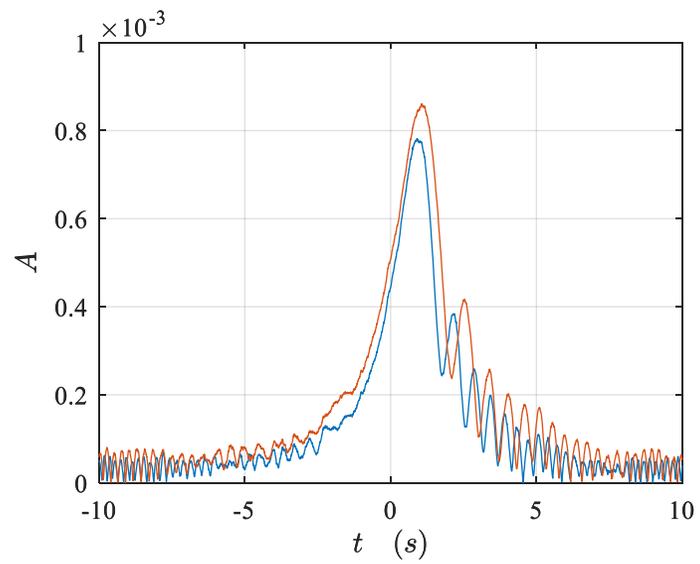
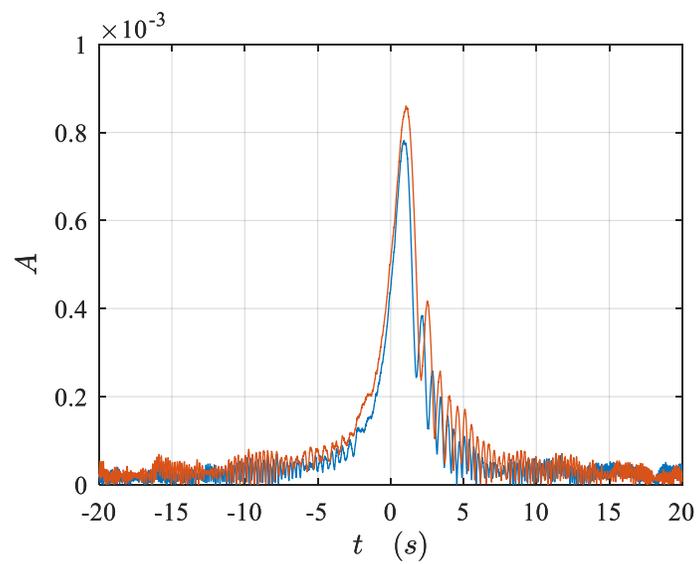
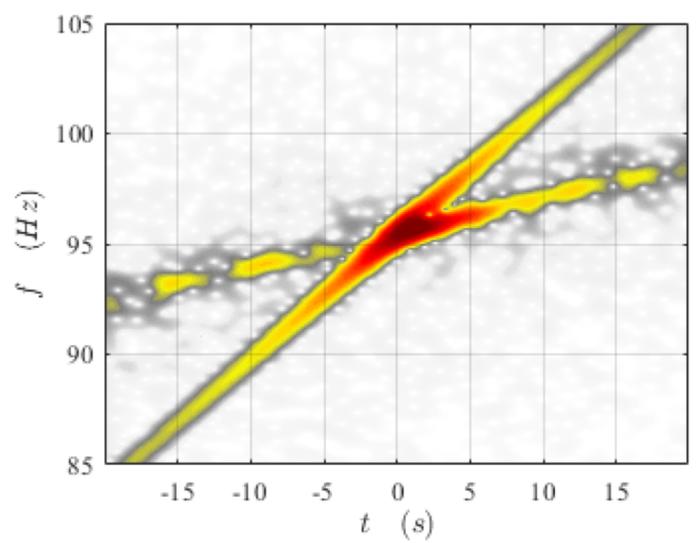
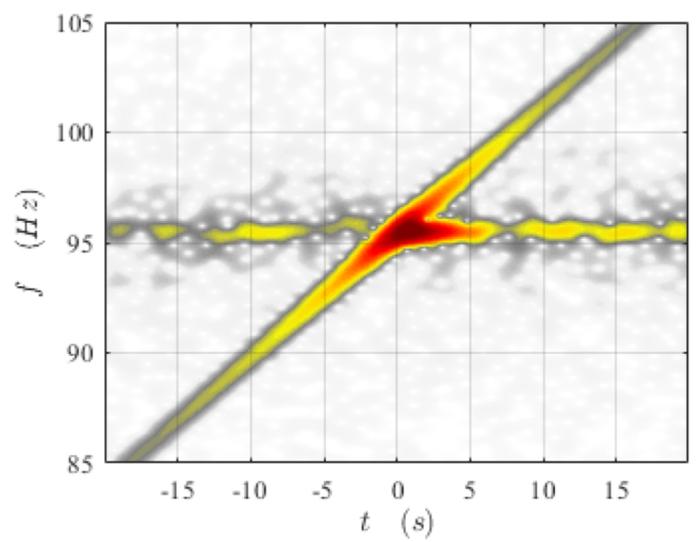
$$\ddot{q} + 2\xi(\omega_0 + \dot{\omega}t')\dot{q} + (\omega_0 + \dot{\omega}t')^2 q = \frac{p_0}{m} e^{\frac{i}{2}r\dot{\Omega}t'^2} e^{ir\dot{\Omega}t_0t'} e^{\frac{i}{2}r\dot{\Omega}t_0^2} = \frac{p_0}{m} e^{\frac{i}{2}r\dot{\Omega}t'^2} e^{i\omega_0t'} e^{i\Phi_0}$$

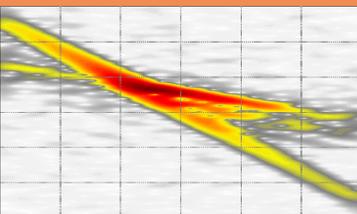


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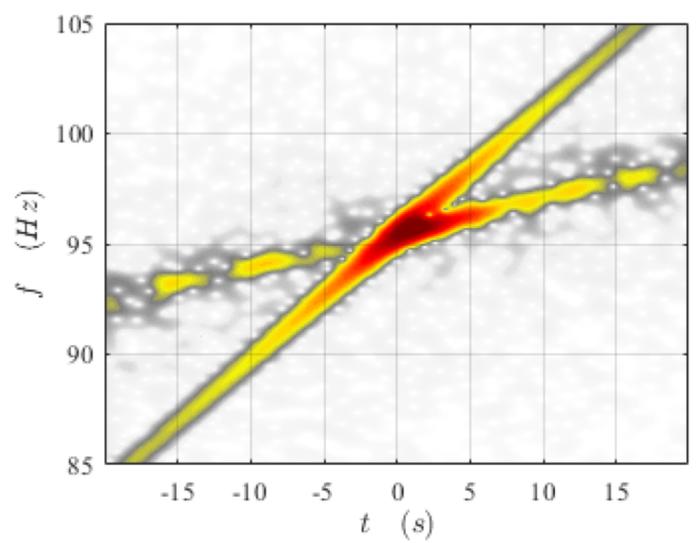
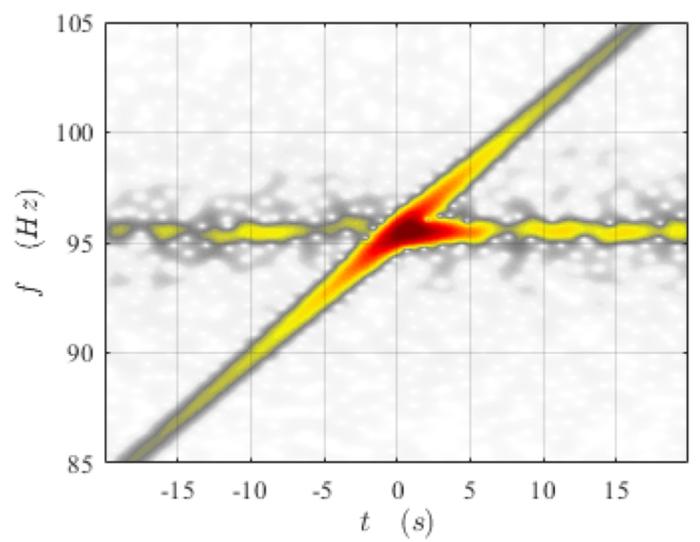


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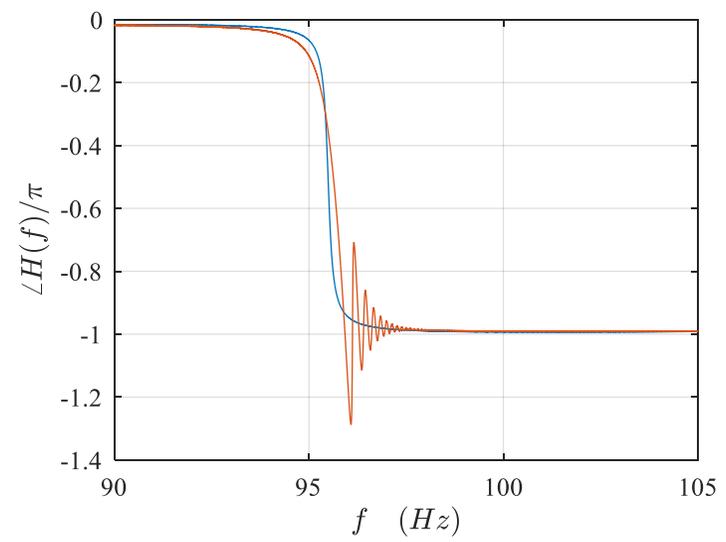
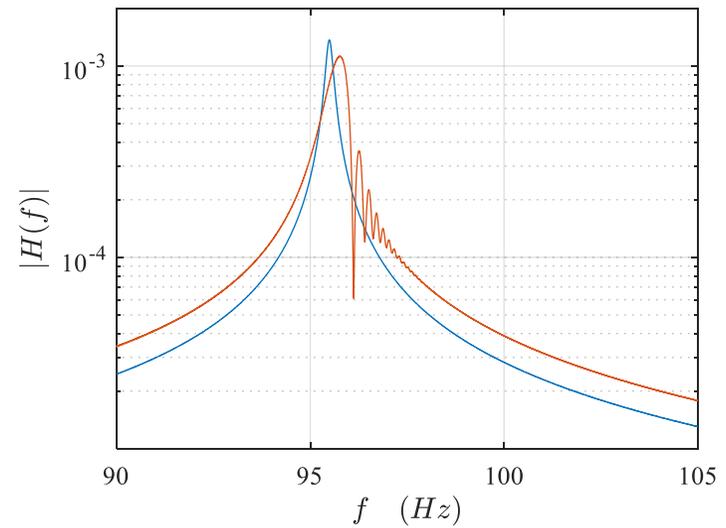




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$$H(f) = \frac{F_{\omega}[q]}{F_{\omega}[p]}$$



Single-DOF model

Governing equation

$$\ddot{q} + 2\xi \underbrace{(\omega_0 + \dot{\omega}t')}_{\text{Time-varying natural frequency}} \dot{q} + \underbrace{(\omega_0 + \dot{\omega}t')^2}_{\text{Time-varying natural frequency}} q = \frac{p_0}{m} e^{\frac{i}{2} \underbrace{\dot{\Omega}t'^2}_{\text{Sweeping rate}}} e^{i\omega_0 t'}$$

Define a dimensionless time and a dimensionless response : $\tau = \omega_0 t'$ $x = \frac{p_0}{2\xi i m \omega_0^2} q$

$$x'' + 2\xi \left(1 + \frac{\dot{\omega}}{\omega_0^2} \tau\right) x' + \left(1 + \frac{\dot{\omega}}{\omega_0^2} \tau\right)^2 x = 2\xi i e^{i\tau} e^{\frac{i}{2} \frac{\dot{\Omega}}{\omega_0^2} \tau^2}$$

Small dimensionless parameters :

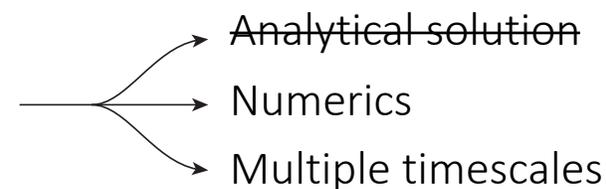
$$\xi = \varepsilon$$

$$\frac{\dot{\Omega}}{\omega_0^2} = \kappa \varepsilon^2$$

$$\frac{\dot{\omega}}{\omega_0^2} = \alpha \frac{\dot{\omega}}{\dot{\Omega}} = \alpha \kappa \varepsilon^2$$

$$x'' + 2\varepsilon \left(1 + \kappa \alpha \varepsilon^2 \tau\right) x' + \left(1 + \kappa \alpha \varepsilon^2 \tau\right)^2 x = 2\varepsilon i e^{i\tau} e^{\frac{i}{2} \kappa \varepsilon^2 \tau^2}$$

Slowly time-varying oscillator subject to slow sine sweep



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Single-DOF model

Multiple timescale solution

$$x'' + 2\varepsilon (1 + \kappa\alpha\varepsilon^2\tau) x' + (1 + \kappa\alpha\varepsilon^2\tau)^2 x = 2\varepsilon i e^{i\tau} e^{\frac{i}{2}\kappa\varepsilon^2\tau^2}$$

Fast time : τ Slow time : $T = \varepsilon\tau$ Ansatz : $x(\tau, T) = x_0(\tau, T) + \varepsilon x_1(\tau, T) + \dots$

Leading order solution : $x_0 = A(T) e^{i\tau}$ (slowly modulated harmonic response)

Secularity condition : $A'(T) + (1 - i\kappa\alpha T) A(T) = e^{\frac{i}{2}\kappa T^2}$ (linear time varying 1st order ODE)

General solution

$$A(T) = e^{-T} e^{\frac{i}{2}\kappa\alpha T^2} \mathcal{D}(T; \kappa(1 - \alpha))$$

Particular solution $\alpha = 0$

$$A(T) = e^{-T} \mathcal{D}(T; \kappa)$$

$$\mathcal{D}(T; \kappa) = \sqrt{\frac{i\pi}{2\kappa}} e^{\frac{i}{2\kappa}T} \operatorname{erfc} \left[e^{-i\frac{\pi}{4}} \frac{i - \kappa T}{\sqrt{2\kappa}} \right]$$

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Single-DOF model

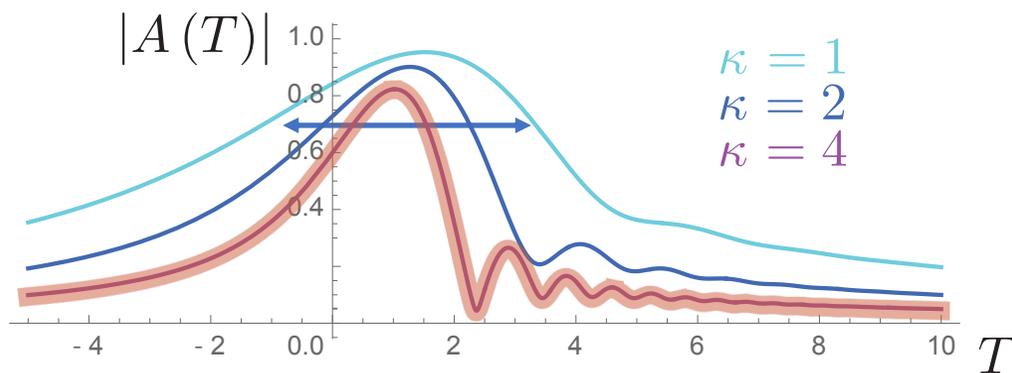
Multiple timescale solution - illustration

General solution

$$A(T) = e^{-T} e^{\frac{i}{2} \kappa \alpha T^2} \mathcal{D}(T; \kappa(1 - \alpha))$$

$$\alpha = 0.5$$

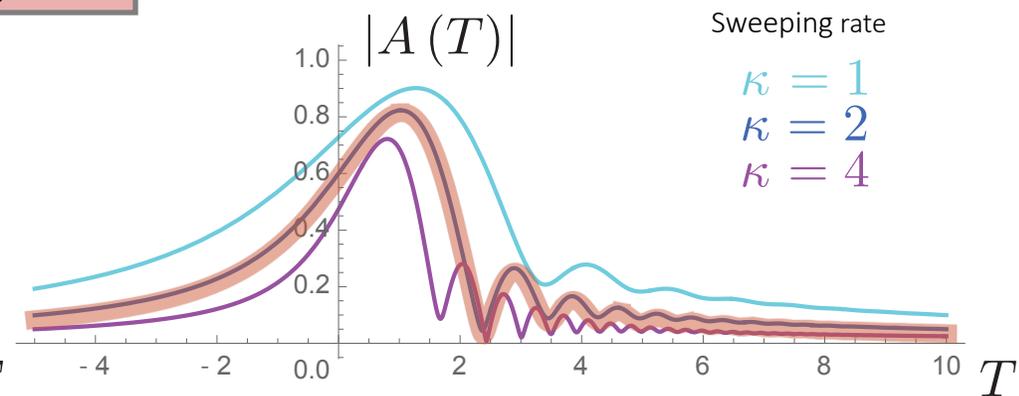
$$\kappa(1 - \alpha) = 2$$



Particular solution

$$A(T) = e^{-T} \mathcal{D}(T; \kappa)$$

$$\alpha = 0$$



The width of the peak increases. Translates an apparently larger damping. More time spent near resonance

Norm of envelope only depends on $\kappa(1 - \alpha)$

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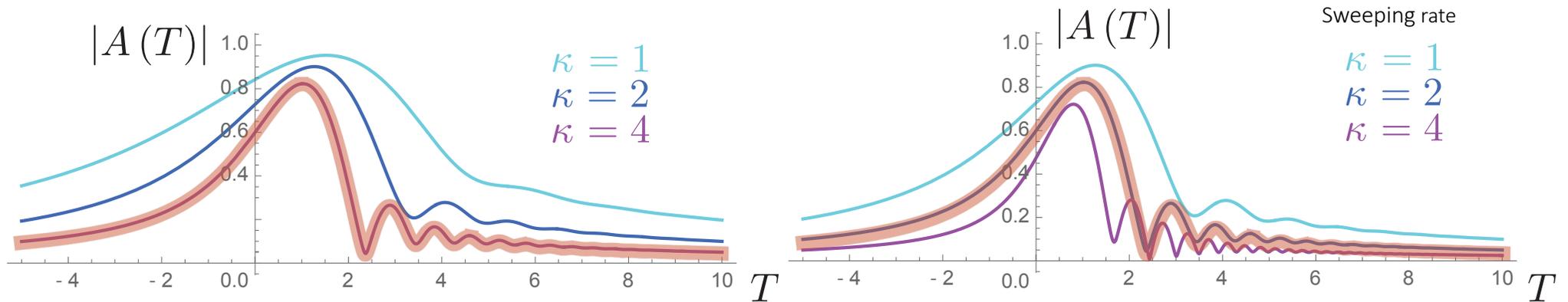
Equivalence

General solution

$$A(T) = e^{-T} \cancel{e^{\frac{i}{2}\kappa\alpha T^2}} \mathcal{D}(T; \kappa(1-\alpha))$$

Particular solution

$$A(T) = e^{-T} \mathcal{D}(T; \kappa)$$



As long as we focus on the **envelope of the response** :

An LTV system can be replaced by an equivalent LTI system subjected to a (smaller) sweeping rate

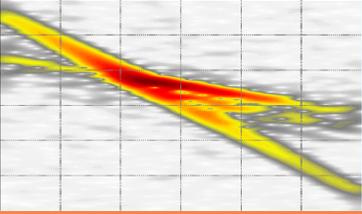
$$\kappa_{LTV} (1 - \alpha) = \kappa_{LTI}$$

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Conclusions

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