Preprint article

# Experimental assessment of pollutant emissions from residential fuel cells and comparative benchmark analysis

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#### **Abstract:**

Energy transition currently brings focus on fuel cell micro-combined heat and power (mCHP) systems for residential uses. The two main technologies already commercialized are the Proton Exchange Membrane Fuel Cells (PEMFCs) and Solid Oxide Fuel Cells (SOFCs). The pollutant emissions of one system of each technology have been tested with a portable probe both in laboratory and field-test configurations. In this paper, the nitrogen oxides ( $NO_x$ ), sulphur dioxide ( $SO_2$ ), and carbon monoxide (CO) emission levels are compared to other combustion technologies such as a recent Euro 6 diesel automotive vehicle, a classical gas condensing boiler, and a gas absorption heat pump. At last, a method of converting the concentration of pollutants (in ppm) measured by the sensors into pollutant intensity per unit of energy (in mg/kWh) is documented and reported. This allows for comparing the pollutant emissions levels with relevant literature, especially other studies conducted with other measuring sensors.

Both tested residential fuel cell technologies fed by natural gas can be considered clean regarding  $SO_2$  and  $NO_x$  emissions. The CO emissions can be considered quite low for the tested SOFC and even nil for the tested PEMFC. The biggest issue of natural gas fuel cell technologies still lies in the carbon dioxide (CO<sub>2</sub>) emissions associated with the fossil fuel they consume. The gas absorption heat pump however shows worse  $NO_x$  and CO levels than the classical gas condensing boiler.

At last, this study illustrates that the high level of hybridization between a fuel cell and a gas boiler may be responsible for unexpected ON/OFF cycling behaviours and therefore prevent both sub-systems from operating as optimally and reliably as they would have as standalone units.

#### **Keywords:**

SO<sub>2</sub>, Fuel cell, emission factor, NO<sub>x</sub>, pollutant emissions, CO.

## 1. Introduction

In its latest Sixth Assessment Report in April 2022, the Intergovernmental Panel on Climate Change has reported a maximum carbon budget of 890 GtCO<sub>2</sub> that humanity can emit from January 1<sup>st</sup> 2020 in order for global warming to likely remain under the +2 °C widely acknowledged limit compared to preindustrial temperature levels (Paulus, 2023). Even at residential scales, this much-needed GreenHouse Gases (GHG) mitigation brings focus to cleaner power sources and combined heat and power (CHP) systems, such as fuel cells (Dávila et al., 2022a). The two primary technologies that have already been commercialized are the Proton Exchange Membrane Fuel Cells (PEMFCs) and the Solid Oxide Fuel Cells (SOFCs), which are later on compared in Table 1. GHG emissions (in terms of CO<sub>2</sub> or CO<sub>2eq</sub>) of such systems have already been addressed in previous studies (Paulus et al., 2022a;

\*Corresponding author. Quai Gloesener No. 6, 4000 Liège, Belgium Email address: <u>nicolas.paulus@hepl.be</u> Paulus and Lemort, 2022a) but another key element in assessing the environmental impacts of those technologies lies in the other common air pollutants: the emissions of nitrogen oxides  $(NO_x)$ , sulphur dioxide  $(SO_2)$ , and carbon monoxide (CO).

The novelty of this study lies within the evaluation of  $SO_2$ ,  $NO_x$ , and CO emissions of fuel cell technologies commercialized for residential applications in both laboratory and field-test configurations (in real dwellings in Belgium). This has been performed on several machines of different ages, for one PEMFC-based and one SOFCbased micro-combined heat and power (mCHP) technology, thanks to a combustion analyser portable meter. This study compares the emission levels of those pollutants, measured for the studied fuel cell systems, with other combustion technologies, such as a recent Euro 6 diesel automotive vehicle and classical gas condensing boilers. To facilitate comparison with relevant literature, a method to convert sensor-detected pollutant concentrations (measured in ppm) into pollutant intensity per unit of energy (in g/kWh) has been documented and reported in this study. This represents another unprecedented contribution to academic literature, as far as the authors are aware. This approach indeed enables the assessment of pollutant emissions levels across different studies, including those conducted using alternative measuring sensors.

It is noteworthy that the initial study upon which this research builds was presented at a conference (Paulus and Lemort, 2023a) and subsequently selected for publication in this journal. However, this paper still delivers new results and novel analyses distinct from the earlier work.

# 2. Background

## 2.1 SOFC and PEMFC current technologies

In residential systems currently commercialized, SOFC technology is characterized by its oxygen ion-conducting electrolyte, usually a (solid) ceramic that includes oxides (Nicolas Paulus, 2024a, 2024b). The charge carrier therefore consists of oxygen ions, leading to the common 'O-SOFC' more explicit acronym (Nicolas Paulus, 2024a) and its working principle is depicted in Figure 1(a), reproduced and adapted from reference (Guney and Tepe, 2017). Oppositely, currently commercialized PEMFC systems only consist of LT-PEMFCs (Low-Temperature Proton Exchange Membrane Fuel Cells), characterized by a (solid) polymer electrolyte that is permeable to protons, i.e. hydrogen cations (Nicolas Paulus, 2024a, 2024b). Its working principle is presented in Figure 1(b), reproduced and adapted from reference (Alshorman, 2016).



**Figure 1.** Working principle of the two existing residential fuel cell technologies. It is noteworthy that the respective charge carriers have opposite charges in both technologies and therefore flow through the respective electrolytes in opposite directions. (a) O-SOFC. Reproduced and adapted from reference (Guney and Tepe, 2017); (b) PEMFC (valid for Low-Temperature and High-Temperature PEMFCs). Reproduced and adapted from reference (Alshorman, 2016).

A detailed comparison of these two technologies has been provided in Table 1. Besides their distinctive charge carriers and electrolytes, Solid Oxide Fuel Cells (SOFCs) operate at a markedly higher temperature, approximately

1000°C, whereas Proton Exchange Membrane Fuel Cells (PEMFCs) function at temperatures below 100°C. This substantially higher stack operating temperature of SOFCs involves drawbacks, such as low startup time, higher thermal stresses, higher costs, *etc*, but also leads to significantly higher fuel flexibility, tolerance to contaminants, and electrical efficiency (when operated on hydrocarbons).

#### Table 1.

Comparison between PEMFCs and SOFCs. Reproduced and adapted from references (Nicolas Paulus, 2024a; Sharaf and Orhan, 2014).

Fuel cell type & Charge carrier	Typical electrolyte	Major contaminants	Stack operating temperature (°C)	Specific advantages	Specific disadvantages	LHV Electrical efficiency (%)
PEMFC & H+	Solid Nafion®, a polymer	Carbon monoxide (CO) <sup>a</sup> Hydrogen sulfide $(H_2S)^a$	60–80 Only low- temperature PEMFCs are currently commercialized (Element Energy, 2021)	Highly modular for most applications High power density Compact structure Rapid startup due to low-temperature operation Excellent dynamic response	Complex water and thermal management <sup>a</sup> Low-grade heat High sensitivity to contaminants <sup>d</sup> Expensive catalyst Expensive Nafion® membrane (Park and Hong, 2016) Low fuel flexibility	40-60 (with H <sub>2</sub> ) Currently limited to 38.5 with CH <sub>4</sub> as some fuel needs to be burned to provide heat to a methane reformer (Perna and Minutillo, 2020)
SOFC & O <sup>2-</sup>	Solid yttria- stabilized zirconia, i.e. YSZ, a ceramic	Sulfides	800-1000	High electrical efficiencies High-grade heat High tolerance to contaminants Possibility of internal reforming Fuel flexibility Inexpensive catalyst Simpler water management - SOFC can work in a perfect drying state (Wen, 2002)	Slow startup Low power density Strict material requirements High thermal stresses Sealing issues Durability issues High manufacturing costs	55-65 (with $H_2$ ) Currently limited to 60%-65% with CH <sub>4</sub> (Bloom Energy, 2023; Element Energy, 2021), i.e. still high thanks to the SOFC fuel flexibility

<sup>a</sup> Contaminants, thermal, and water management of PEMFC stacks have been discussed more deeply in another work (Paulus et al., 2024).

### 2.2 Harmful effects of NO<sub>x</sub>, SO<sub>2</sub>, and CO

#### 2.2.1 NO<sub>x</sub>

Nitrogen oxides, i.e.  $NO_x$ , in the ambient air consist primarily of nitric oxide, i.e. NO, and the much more harmful nitrogen dioxide, i.e.  $NO_2$  (Cheremisinoff, 2002), that NO readily turns into in the atmosphere in the presence of volatile organic compounds (Cheremisinoff, 2002), i.e. VOCs (Cheremisinoff and Young, 1977).  $NO_2$  indeed irritates the lungs and promotes respiratory infections (Gupta, 2018). The two forms of gaseous nitrogen oxides are not only considered as pollutants of the lower atmosphere, but they also can have a significant impact on the upper atmosphere. Indeed,  $NO_2$  has been reported to contribute to acid rain (Cheremisinoff, 2002), which subsequently, in addition to endangering vegetation, ecosystems and freshwater (Cheremisinoff, 2002) or increase human exposure to nitrate/nitrite consumption (Galloway et al., 2013), enhance the soil nitrous oxide ( $N_2O$ ) emissions (Cao et al., 2021; Thomson et al., 2012). In fact, this strong GHG pollutant, with a global warming potential over 100 years that is evaluated about 300 times greater than  $CO_2$  (Muret et al., 2019), is also responsible for most of the stratospheric ozone depletion (Portmann et al., 2012). It is worth mentioning that, through the subsequent formation of  $N_2O$ , the global warming potential over 100 years of  $NO_x$  is estimated between 7 and 10 (Lammel and Graßl, 1995).

In the lower atmosphere, in addition to being again a precursor of acid precipitation, it is also a precursor of fine particulate matter (Kuo et al., 2017), which can penetrate deep into vital systems, causing cardiovascular and respiratory diseases (Peng, 2008) as bad as lung cancers (Hamra et al., 2014), increasing morbidity and mortality (Brook et al., 2010). Nitrogen dioxide is also a precursor of tropospheric ozone ( $O_3$ ) formation (Cheremisinoff, 2002). And  $O_3$  is a poison even more harmful than NO<sub>2</sub> (Gupta, 2018). Indeed, it damages vegetation, irritates lung tissues, and can lead to smog, even more harmful to respiratory functions than ozone (Cheremisinoff, 2002). Smog is known to cause deaths as it can coarse many other pollutants of different toxicities, such as particulate matter (Mishra, 2017). Indeed, smog primarily consists of ozone and 'secondary' pollutants that are produced through photochemical reactions of directly emitted species, mainly consisting once again of VOCs (Sher, 1998),

in processes that are driven by sunlight and accelerated by warm temperatures (Sillman, 2003). The chemical composition of smog can vary according to mereological conditions, primarily temperature, humidity, and radiation (Raza et al., 2021). It also specifically varies according to the concentration of other pollutants (not only VOCs) and other atmospheric species (Raza et al., 2021). For example, the effect of smog is even aggravated when it contains fine particulate matter (Jiang et al., 2016). Smog can also contain and induce the health issues of the following species: lead (Grzywa-Celińska et al., 2020), peroxylacyl nitrates (PAN), aldehydes, CO, SO<sub>2</sub>, NO<sub>x</sub> (Raza et al., 2021) or their sulphate/nitrate derivatives (Zhou et al., 2015), etc.

## 2.2.2 SO<sub>2</sub>

Similarly to NO<sub>2</sub>, SO<sub>2</sub> is a major precursor of acid rains and fine particulate matter (Geddes and Murphy, 2012) and can thus be associated with their environmental and health issues. Another similarity is that it is an irritating gas that leads to respiratory illnesses (Cheremisinoff, 2002). It also has been reported that it can aggravate existing heart (and pulmonary) diseases (Hanrahan, 2012). Oppositely to NO<sub>x</sub>, SO<sub>2</sub> has a negative contribution to global warming with a GWP100 estimated between -18 and -25 (Rypdal et al., 2009). This is because SO<sub>2</sub> is a precursor of sulphate or sulfuric acid aerosols (Gupta, 2018). It is noteworthy that decreasing the emissions of those negative forcing aerosol pollutants will unfortunately have the unwanted consequence of increasing net radiative forcing, which is commonly called as an 'unmasking' effect (N. Paulus, 2024).

While SO<sub>2</sub> is not a direct precursor of ozone, it has been reported that sulfur compounds initiated by SO<sub>2</sub> emissions can function similarly to VOCs in the ozone formation cycle, in a mechanism of re-oxidation of NO into NO<sub>2</sub>, following the reaction of NO<sub>2</sub> with O<sub>2</sub> to form O<sub>3</sub> and NO (Graedel, 1976). At last, as stated, it can aggravate smog toxicity. This can occur either directly (Grzywa-Celińska et al., 2020), through its sulphate derivatives (Zhou et al., 2015), or through the fine particulate matter it can coarse into (Geddes and Murphy, 2012; Raza et al., 2021).

## 2.2.3 CO

CO is a well-known major air pollutant as it is also known as the 'silent killer' (invisible and odourless): at a concentration of 12000 ppm, it kills in two to three breaths as it blocks the ability of haemoglobin to transport oxygen to the cells of the body (Varma et al., 2015). However, at lower concentrations, it also has many impacts on human health: impaired vision, reduced brain function (Hanrahan, 2012), coma, seizures, heart and respiratory diseases, physical weakness (Gupta, 2018), tissue damage (Eichhorn et al., 2018). It is also considered as a minor ozone precursor in urban areas as it can play the role of VOCs in the ozone formation cycle (Chameides et al., 1992). At last, even though the lifetime of CO in the atmosphere is quite short, i.e. a few months (Rotmans and Den Elzen, 1992), its GWP100 is still estimated at 5 because of its interaction with methane (Rotmans and Den Elzen, 1992). Indeed, this is mainly because the main removal process of both CO and CH<sub>4</sub> includes the reaction with hydroxyl radicals (Johnson and Derwent, 1996). CO emissions reduce the hydroxyl radicals concentration for methane removal, which has a quite high GWP100 of about 28 (Paulus et al., 2022a), thus leading to a significant indirect impact of CO emissions on global warming.

## 2.3 SO<sub>2</sub> and NO<sub>x</sub> emission factors of space heating appliances

 $SO_2$  and  $NO_x$  emission factors of typical space heating appliances have respectively been reported in Table 2 and Table 3. The data required to feed such a comparative table relative to CO emission factors of typical space heating appliances (and Belgian grid electricity in Belgium) could not (yet) be computed with the literature review conducted through this study.

#### Table 2.

Combustion only and Life Cycle Assessment (LCA)  $SO_2$  emission level reported from Energie+ (Energie+, 2007), which is a website developed by the University of Louvain-la-Neuve and the Energy Department of the Walloon Region, in Belgium. It is noteworthy that the electrical radiator emission factor has been established based on (and is therefore relevant for) the Belgian electrical mix.

Space-heating appliance	$SO_2$ (source from 2007: Fondation Rurale de Wallonie - combustion only) mg/kWh <sub>th</sub>	SO <sub>2</sub> (source accessed in 2007: Gemis 4.5 - complete LCA cycle) mg/kWh <sub>th</sub>
Oil-fired boiler	504	600
Gas condensing boiler	$0^{a}$	111
Electrical radiators (Joule heating)	Unavailable	392 <sup>b</sup>
Old log wood boiler	36	Unavailable
Modern log wood boiler	36	320
Wood chip boiler (wood chips)	36	Unavailable
Condensing wood boiler (pellets)	Unavailable	472

<sup>a</sup> Combustion-only SO<sub>2</sub> emissions from natural gas combustion are definitely low as most natural gas markets require less than 4 ppm of (all) sulfur-containing compounds in the gas (Speight, 2019). Indeed, decentralized desulfurization is implemented in natural gas processing (Xu et al., 2020). However, the SO<sub>2</sub> combustion-only emissions of gas condensing boilers are not completely nil as other studies have reported about 2 mg/khW<sub>th</sub> (Papadopoulo et al., 2011) or between 3.5 and 4 mg/khW<sub>th</sub> (Proszak-Miasik and Rabczak, 2018). This latter study considers a residential domestic hot water demand of 300 L a day at 45°C (Proszak-Miasik and Rabczak, 2018) and this has been considered to correspond to 12.3 kWh<sub>th</sub> a day (Fuentes et al., 2018).

<sup>b</sup> The power plant type has a strong influence on the SO<sub>2</sub> emissions. As a comparison, in 2012 in the US, coal-fired and Combined Cycle Gas Turbine (CCGT) power plants were reported to have average combustion-only SO<sub>2</sub> emission intensities respectively of about 1200 mg/kWhel and about 2.4 mg/kWhei (de Gouw et al., 2014). Another study reported (in 2013) the following SO<sub>2</sub> LCA emission intensities: 10-320 mg/kWhel range for natural gas power plants (of all kinds, not only the most efficient ones, i.e. the CCGTs), 30-6700 mg/kWhel range for coalfired power plants, 3-38 mg/kWhel range for nuclear power plants (Turconi et al., 2013). At the time when Table 2 was originally computed (in 2007), coal-fired power plants (of high SO<sub>2</sub> emission intensity) were still in use for electricity generation. In fact, for example, Belgium closed its last coal-fired power plant in 2016 and was the seventh EU country to completely remove coal from its electrical mix (Asiaban et al., 2021). Therefore,  $SO_2$  emissions from electrical radiators can be considered much lower than reported in this table. For example, considering the 2022 Belgian electrical mix (ELIA, 2023) and the maximum SO<sub>2</sub> emission factors for electricity generation reported by the European Topic Centre on Air and Climate Change (Fritsche and Rausch, 2009), the LCA SO<sub>2</sub> intensity of the 2022 Belgian electrical mix can be considered to only 77 mg/kWhel, i.e. five times lower than the value reported in Table 2. In this calculation, the assumption that the 3.2 TWh electrical production referred to as 'Other' in the Belgian electrical mix (ELIA, 2023) entirely corresponds to the incineration of municipal waste with a  $SO_2$  emission factor of 1220 mg/k $Wh_{el}$ , coming from the most  $SO_2$  emitting waste incineration technology reported in the literature for China power plants (Chen and Christensen, 2010). This is a safe assumption (providing the worst  $SO_2$  intensity for the Belgian electrical mix) as municipal waste incineration in Belgium corresponded only to 1 TWh per year in 2014 (Psomopoulos et al., 2017) or to 1.5 TWh in more recent years (Belgian Waste-To-Energy, n.d.) while other power plant technologies (hydropower, geothermal, solar-thermal, and solid biomass) are much cleaner, or are at worst similar (liquid biofuels) in terms of SO<sub>2</sub> emissions (Fritsche and Rausch, 2009). It is noteworthy that the Belgian nuclear electricity SO<sub>2</sub> emission factor has not been reported by the considered reference, i.e. the European Topic Centre on Air and Climate Change (Fritsche and Rausch, 2009). However, this reference still reported the nuclear electricity SO<sub>2</sub> emission factor in France (Belgian's neighbouring country), which was considered in this study to compute the LCA SO2 intensity of the Belgian electrical mix.

#### Table 3.

Combustion only and Life Cycle Assessment (LCA)  $NO_x$  emission level reported from Energie+ (Energie+, 2007), which is a website developed by the University of Louvain-la-Neuve and the Energy Department of the Walloon Region, in Belgium. It is noteworthy that the electrical radiator emission factor has been established based on (and is therefore relevant for) the Belgian electrical mix.

Space-heating appliance	NO <sub>x</sub> range (source from 1998: Electrabel-SPE – combustion only) mg/kWh <sub>LHV</sub>	NO <sub>x</sub> (source from 2007: Fondation Rurale de Wallonie - combustion only) mg/kWh <sub>th</sub>	NO <sub>x</sub> (source accessed in 2007: Gemis 4.5 - complete LCA cycle) mg/kWh <sub>th</sub>
Old oil-fired boiler	up to 200	Unavailable	Unavailable
Non-Low NOx oil-fired boiler	150 - 180	144	244
Low NO <sub>x</sub> oil-fired boiler	90 - 120	Unavailable	Unavailable
Old gas boiler	150 - 200	Unavailable	Unavailable
Atmospheric gas boiler	100 - 180	Unavailable	Unavailable
Modulating gas condensing boiler	20 - 90	144	140
Electrical radiators (Joule heating)	420 <sup>a</sup>	Unavailable	459
Old log wood boiler	Unavailable	180	Unavailable
Modern log wood boiler	Unavailable	151	235
Wood chip boiler (wood chips)	Unavailable	162	Unavailable
Condensing wood boiler (pellets)	Unavailable	Unavailable	344

<sup>a</sup> Considering the average Belgian electricity generation efficiency of 38% in 1998. Following the same methodology (and references) as conducted in Table 2 to establish the SO<sub>2</sub> intensity of the current Belgian electrical mix, the LCA NO<sub>x</sub> intensity of the current Belgian electrical mix can be considered to be only 90 mg/kWh<sub>el</sub> or less, i.e. also about five times lower than reported in Table 3.

## 2.4 Pollutant emissions of residential fuel cells

A specific background study has been performed regarding the  $SO_2$ ,  $NO_x$ , and/or CO emissions of residential fuel cells, and its results have been presented in Table 4. As it can be observed, there is only a very small number of works that provide information about the  $SO_2$ ,  $NO_x$ , and/or CO emissions of residential fuel cell systems. In addition, in the consulted literature, the experimental emission measurement campaigns are seldom detailed (in terms of sensors or procedure). Pollutant emissions figures are even often reported without any references or are reproduced from the manufacturer's datasheets. It is noteworthy that pollutant emissions are never given by the same reference both in ppm (concentration) and in mg/kWh (intensity per unit of energy), which impedes the comparison between sources. These limitations therefore strengthen the interest of this experimental research.

## Table 4.

NO<sub>x</sub> and SO<sub>x</sub> (or SO<sub>2</sub>) emissions of fuel cell systems reported in the literature.

Technology	NO <sub>x</sub>	SO <sub>x</sub> (or SO <sub>2</sub> )	References	Comments
PEMFC	<1 ppm	<1 ppm	(de Bruijn, 2005)	Figures seem to have originated from a
			Same fuel cell and figures reported in another study: (Srinivasan and Miller, 2006)	document written by the manufacturer (original sources not available). Figures therefore not verified in an available documented experimental study.
Residential fuel cells in general.	<5 ppm	Not disclosed	(Krist, 1999)	Figure undocumented in the reference.
Evel cells in	< 9.07	Not disclosed	(Found et al. 2007)	Figure undocumented in the reference
general (residential market included). Unspecified.	mg/kWh <sup>a</sup>	Not disclosed	(i oudd ei all, 2007)	The micro-cogeneration market is mentioned by the reference so the reported figure can be assumed to apply to residential fuel cells.
PEMFC	27.22 mg/kWh <sup>a</sup>	Not disclosed	(Shipley and Elliott, 2004)	Figure documented in the reference as: 'Source: Personal Communication with Joel Bluestein 2002'.
				The residential or micro-cogeneration markets have not been mentioned regarding this figure.
SOFC	4.54 mg/kWh <sup>a</sup>	2.27 mg/kWh <sup>a</sup>	(Shipley and Elliott, 2004)	Figure documented in the reference as: 'Source: Personal Communication with Joel Bluestein 2002'.
				The residential or micro-cogeneration markets have not been mentioned regarding this figure.
PAFC (Phosphoric Acid Fuel Cell)	13.61 mg/kWh <sup>a</sup>	2.72 mg/kWh <sup>a</sup>	(Shipley and Elliott, 2004)	Figure documented in the reference as: 'Source: Personal Communication with Joel Bluestein 2002'.
				PAFCs are not considered for the residential market (Nicolas Paulus, 2024b). 200 $kW_{el}$ applications are indeed reported in the reference.
MCFC (Molten Carbonate Fuel Cell)	22.68 mg/kWh <sup>a</sup>	Not disclosed	(Shipley and Elliott, 2004)	Figure documented in the reference as: 'Source: Personal Communication with Joel Bluestein 2002'.
				MCFCs are not considered for the residential market (Nicolas Paulus, 2024b). 250-2000 $kW_{el}$ applications are indeed reported in the reference.

<sup>a</sup> Supposedly expressed in  $mg/kWh_{th}$  or  $mg/kWh_{LHV}$  (rather than in  $mg/kWh_{el}$ ) as it is usually considered in literature (and as it is the case in Table 2 and Table 3).

CO emissions are even more rarely reported. The only available figure is 10 ppm for a 20 MW<sub>el</sub> Molten Carbonate Fuel Cell (MCFC), i.e. a figure reported to be similar to the emissions of gas turbines (Karvountzi and Duby, 2008). However, MCFC is not a residential fuel cell technology (see Table 4). It is noteworthy that the same MCFC fuel cell is reported in the consulted study to feature 0.03 ppm of NO<sub>x</sub>, i.e. an order of magnitude lower than gas turbines and two orders of magnitude lower than reciprocating engines (Karvountzi and Duby, 2008). Regarding NO<sub>x</sub>, those order of magnitudes has been confirmed in another study (Shipley and Elliott, 2004), which however explicitly reports that the considered reciprocating engines do not feature any NO<sub>x</sub>-reducing catalyst in the exhaust.

## 2.5 Methane slip in natural gas-fed fuel cells

It should be mentioned that methane slip (or slippage), i.e. 'unburnt' methane emissions are sometimes mentioned in fuel cell literature. However, this term is usually rather used not for the methane content in the flue gases but

for the methane slipping from the reformer to the stack (to the anode of the fuel cell stack) if the fuel cell is not directly fed by hydrogen but by natural gas or other hydrocarbons (Aguiar et al., 2004; Chartrand, 2011; Kee et al., 2008).

Regarding Solid Oxide Fuel Cells (SOFCs), their high operating temperature fuel flexibility allows them to directly use methane (and other hydrocarbons) as fuel onto the stack anode, as it has been reported in Table 1. In addition, in the case of commercialized mCHP SOFC systems, existing studies of exhaust gases reported in the literature have shown that there is no or negligible methane slip (Payne et al., 2009). It has in fact been reported that the 'operation of the stack' is at a temperature between 700 and 800°C, which 'enables the internal reforming to proceed with negligible methane slip', which maximizes the amount of fuel available for the electrochemical process (Payne et al., 2009). Increased internal SOFC temperatures have indeed been reported to significantly reduce methane slip/emissions in exhaust gases (Wagner et al., 2002). Furthermore, specific SOFC literature confirms that, when operated on natural gas, SOFC do not have problems with methane slip in exhaust gases (Baldi et al., 2020; Kistner et al., 2021).

Regarding Proton Exchange Membrane Fuel Cells (PEMFCs), their stack relies on high-purity  $H_2$  fuel (Urdampilleta et al., 2007), which implies reforming processes if the fuel cell is fed by natural gas or other hydrocarbons (Bang et al., 2022; Rabiu et al., 2012), as it has been inferred in Table 1. In fact, CH<sub>4</sub> concentrations up to 20 ppm downstream of the fuel processor and upstream of the fuel cell stack (anode) have been reported not to affect the power generation performance (Minei et al., 2020). However, an afterburner is usually implemented with PEMFC systems fed by natural gas in order to ensure the (complete) utilization of the unreacted fuel, which mainly consists of CH<sub>4</sub> and H<sub>2</sub> (Rabiu et al., 2012; Schumann et al., 2008), simultaneously providing the necessary heat to the reforming processes (Rabiu et al., 2012).

For those reasons and also because of the unavailability of dedicated  $CH_4$  sensors to place in the exhausts of the systems in the experimental studies performed in this study, potential unlikely methane emissions of fuel cell mCHP systems have not been considered from here onwards.

# 3. Material and methods

## 3.1 Tested systems

#### 3.1.1 PEMFC hybridized to a gas-condensing boiler

The PEMFC is not a standalone unit. It is hybridized to a gas-condensing boiler and to a Domestic Hot Water (DHW) tank. It is fed by natural gas and is designed to cover all the heat demands, including DHW, of residential houses and to participate locally in the electrical production. This particular system exists in several versions all based upon the same PEMFC module of nominal constant power of  $0.75 kW_{el}$  (and  $1.1 kW_{th}$ ) and the same 220L DHW tank. The only module that may vary is the gas boiler which is supposed to ensure peak heat demands. Indeed, it exists in four rated power versions from 11.4 to  $30.8 kW_{th}$ , depending on thermal needs (Paulus and Lemort, 2023a). The heat rate output of the field-test system considered in this study is rated at 24.5 kW<sub>th</sub> and is located in Huy, in Belgium. The system's architecture is presented in Figure 2, which does not show the double-walled chimney used for both the air inlet and flue gas exhaust (Lichtenegger et al., 2015). The PEMFC gas boiler hybrid system (from the laboratory facilities) is presented in Figure 3(a) without its front panel cover. Main datasheet characteristics are presented in Table 5.

#### Table 5.

PEMFC gas boiler hybrid expected targets. Publicly available data from the datasheet (Paulus and Lemort, 2023a).

Datasheet figures	Values
Maximum electrical production a year	6200kWh <sub>el</sub>
Fuel cell rated electrical and thermal power as defined by EN 50465 (Dávila et al., 2022a)	$0.75 kW_{el} \& 1.1 kW_{th}$
Electrical fuel cell efficiency	37% (LHV)
Max global Fuel cell efficiency	92% (LHV)
Max boiler efficiency (at rated power) <sup>a</sup>	108.7% (LHV)
NO <sub>x</sub> class 6 (European Commission, 2021)	7.2 mg/kWh
Size without chimney (Height x Width x Depth)	1800 mm x 595 mm x 600 mm

<sup>a</sup> Considering High Heating Value (HHV) to Low Heating Value (LHV) ratio of 1.1094 (Jedlikowski et al., 2020).



**Figure 2.** PEMFC system's architecture, including two heat exchangers, several 3-way valves, several circulators, the gas condensing boiler, and the DHW tank. Reproduced from reference (Paulus et al., 2022a).

The complete system's behaviour is heat driven. Its PEMFC has not been designed to be driven by the electrical demand and it is preferable that it runs as long as possible. It includes a methane reforming apparatus to feed the fuel cell stack with clean hydrogen and requires an automated fuel cell shutdown recovery procedure of 2.5 hours at least every two days to handle some reversible aging processes (Paulus et al., 2024). For further information, this system has been quite exhaustively studied in other publications (Dávila et al., 2022a; Paulus et al., 2024, 2022a, 2022b; Paulus and Lemort, 2023b, 2022b).



Figure 3. Tested fuel cell mCHP systems in the laboratory facilities. (a) PEMFC; (b) SOFC.

## 3.1.2 SOFC

The studied SOFC is also fed by natural gas. It is designed to provide  $1.5 \text{ kW}_{el}$  of nominal output power with a high announced LHV electrical efficiency of 60%, along with a heat recovery of 0.6 kW<sub>th</sub> representing a LHV thermal efficiency of up to 25% (Paulus and Lemort, 2022a). The output power can be modulated down remotely (by the manufacturer, upon the owner's request) as wanted in the 0.5 - 1.5 W<sub>el</sub> range, affecting those announced efficiencies. It is not advised to completely shut it down because thermal cycles affect its durability and because

startup operations are long and have been reported in the user manual to last up to 30 hours (Paulus and Lemort, 2023c).

Discarding its chimney, the system is approximately the same size as a dishwasher, as it can be seen in Figure 3(b). Its internal schematics have not been disclosed but have been discussed in a previous publication (Paulus and Lemort, 2022a), based on observations of the system and cogeneration SOFC literature. Amongst other particularities, the reforming process of the inlet natural gas (into hydrogen) is not only internal (Paulus and Lemort, 2022a). Indeed, it does not only occur directly at the anode, onto the stack, with the high operating temperatures occurring with that fuel cell technology (Aguiar et al., 2004), but it also uses an external steam reformer upstream of the stack, called 'pre-former' (Paulus and Lemort, 2022a). For information, the newer version of this system is stated by the manufacturer as belonging to class 6 in terms of NO<sub>x</sub> (D2SERVICE, 2019), according to EN 15502-1 (European Commission, 2021). This is the most stringent NO<sub>x</sub> emission class, which corresponds to a limit of 56 mg/kWh based on natural gas high heating value (Bălănescu and Homutescu, 2018), i.e 62 mg/kWh based on natural gas low heating value (Jedlikowski et al., 2020), as it has been reported in Table 6. The exact emission levels have however not been reported, to the knowledge of the authors.

#### Table 6.

 $NO_x$  emissions classes for gas condensing boilers according to EN 15502 (European Commission, 2021) and for gas absorption heat pumps according to EN 12309 (European Commission, 2014). The  $NO_x$  emissions limits are identical. Reproduced from references (Venfield and Brown, 2018; VHK for the European Commission, 2019).

NO <sub>x</sub> class	EN 15502 NO <sub>x</sub> concentration limit Low heating value-based	EN 15502 NO <sub>x</sub> concentration limit High heating value-based	EN 12309 NO <sub>x</sub> concentration limit Low heating value-based
1	260	234 <sup>a</sup>	260
2	200	180 <sup>a</sup>	200
3	150	135 <sup>a</sup>	150
4	100	90 <sup>a</sup>	100
5	70	63 <sup>a</sup>	70
6	62 <sup>a</sup>	56	Unavailable

<sup>a</sup>Considering High Heating Value (HHV) to Low Heating Value (LHV) ratio of 1.1094 (Jedlikowski et al., 2020).

#### 3.1.3 Gas condensing boiler

The tested mural gas condensing boiler dates from 2005 and is quite classical. Its identification name is 'Buderus Logamax plus GB142-45' and it can provide up to 45 kW<sub>th</sub> (that can be modulated down to 30%). It can provide heat to an optional DHW tank but cannot provide instantaneous DHW directly as it has only one hydraulic inlet and one hydraulic outlet (used in close circuit configurations). The emissions of CO and NO<sub>x</sub> are reported by the manufacturer respectively to 15 mg/kWh and 20 mg/kWh (Buderus, 2011).

### 3.1.4 Euro 6 diesel vehicle

The tested vehicle is a 4-year BMW X1 sDrive18d that is proper maintenance and had 111210 kilometres on the odometer at the moment of the test. Its four-stroke engine has four cylinders and represents a displacement of 1995cm<sup>3</sup>. Net power is 100 kW at 4000 rpm. The certificate of conformity presents average emissions on the New European Driving Cycle (NEDC) for CO and NO<sub>x</sub> respectively of 86.8 mg/km and 19.2 mg/km. Maximum Real Driving Emissions (RDE) NO<sub>x</sub> emissions are reported to be equal to 168 mg/km. Considering an effective consumption of 6L per 100 km (according to the dashboard of the vehicle), considering a diesel LHV of 43.51 MJ/kg and a density of 827 kg/m<sup>3</sup> (Parravicini et al., 2020), those emissions correspond respectively to 145 mg/kWh (average CO emissions on the NEDC), 32 mg/kWh (average NO<sub>x</sub> emissions on the NEDC) and 280 mg/kWh (maximum Real Driving Emissions NO<sub>x</sub>). They are relative to the diesel LHV input to the engine.

#### 3.1.5 Gas absorption heat pump

It is noteworthy that another uncommon commercial natural gas space heating appliance was tested with the same sensors as the previously described systems. Indeed, similar pollutant tests were conducted in a laboratory environment over a gas absorption heat pump named 'Robur K18 Simplygas' from 2019. The system is based on the Water-Ammonia absorption cycle using outdoor air as renewable energy source (low-temperature heat source) and natural gas combustion as high-temperature heat source; the delivered hot water is the medium-temperature heat sink. This heat pump shows a nominal heating output of 18,9 kW with an outdoor temperature of 7°C and delivery temperature of 35°C, and its datasheet announces a 169% low heating value efficiency in those conditions (Robur, 2022). It is designed for domestic hot water and space heating production. Its working principle and performance analysis have been reported in a previous study (Dávila et al., 2022b).

For information, the newer version of this system is stated by the manufacturer as belonging to class 5 in terms of  $NO_x$  (Robur, 2020), according to EN 12309-1 (European Commission, 2014). The exact emission levels, expected

to be below the limit of 70 mg/kWh based on natural gas low heating value according to Table 6, have however not been reported to the knowledge of the authors.

## 3.2 Measurements device

To perform the pollutants emissions analyses of the tested systems, the same portable combustion analyser meter was used. It is called 'Multilyzer STx' and is shown in Figure 4 whereas its specifications are shown in Table 7.

It measures CO, NO, NO<sub>2</sub>, and SO<sub>2</sub> in ppm, whereas  $O_2$  and CO<sub>2</sub> concentration levels are expressed in percentage (by volume). Carbon monoxide sensors generally have a significant cross-sensitivity to hydrogen, meaning that the real carbon monoxide concentration can be overestimated if hydrogen is present as well in the tested gas sample (Hall et al., 2021). Therefore, as presented in Table 7, the 'Multilyzer STx' combustion analyser portable meter has implemented a hydrogen compensation for its carbon monoxide measurements. However, it can already be stated that, for the tests conducted in this study, there were not any significant differences between the signals provided by the sensor for the CO concentration measurement and for the hydrogen compensated CO measurements, indicating that hydrogen levels in the exhaust appliances were always negligible.



Figure 4. 'Multilyzer STx' combustion analyser portable meter.

### Table 7.

Specifications of the 'Multilyzer STx' combustion analyser portable meter. Publicly available data published by the manufacturer (AFRISO-EURO-INDEX Group, 2019).

Pollutant	Range	Accuracy	Resolution
NO	0 - 5000 ppm	± 5 ppm (< 50 ppm) ± 5% reading (> 50 ppm)	1 ppm
NO <sub>2</sub>	0 - 500 ppm	± 10 ppm (< 50 ppm) ± 10% reading (> 50 ppm)	1 ppm
SO <sub>2</sub>	0 - 5000 ppm	± 10 ppm (< 200 ppm) ± 5% reading (> 200 ppm)	1 ppm
CO (hydrogen compensated)	0 - 10000 ppm	± 5 ppm (< 50 ppm) ± 5% reading (> 50 ppm)	1 ppm
$O_2$	0 - 21 % vol.	± 0.2% vol.	0.1% vol.
CO <sub>2</sub> (calculated from O <sub>2</sub> level)	0 % vol. up to $(CO_2)_N$ which depends on fuel type, see Equation (1)	$\pm \ 0.2\%$ vol.	0.1% vol.
Gas temperature	0 - 1150 °C	± 1 °C (0 - 300°C) ± 1% reading (> 300°C)	0.1 °C

It can be interesting to compare fixed continuous pollutant analysers to portable analysers, such as the 'ENERAC Model 500' (Edelblute et al., 2015), or such as the 'Multilyzer STx' used in this study and presented in Table 7 (which present both similar performances). For example, as many others, a study conducted on a combustion

engine test bench used the very common 'AVL Digas 444N' gas analyser and established its performance, as reported in Table 8 (Balamurugan and Nalini, 2014). Comparing Table 7 and Table 8, it can be established that the measuring range of both sensors is quite similar, except for CO, which is an order of magnitude higher for the fixed sensor, considering ppm vol. and ppm values similar by the assumption of ideal gases behaviors (through the ideal gas law). This increased range has however as consequence that its CO measuring resolution is an order of magnitude worse than for the portable sensor used in this study. With the comparison of both the fixed and portable multi-gas analysers of Table 7 and Table 8, it can be established that there are no benefits of using a fixed measuring device instead of a portable one in terms of resolution, measuring range, and especially in terms of accuracy (which is even often worse with the fixed multi-gas sensor of Table 8 than for the one used in this study).

#### Table 8.

Specifications of the 'AVL Digas 444N' combustion analyser fixed meter. Reproduced and adapted from references (AVL, 2021; Balamurugan and Nalini, 2014). It is noteworthy in the datasheet of the 'AVL Digas 444N' sensor currently available, some of the measuring ranges have slightly been increased (AVL, 2021). Similarly, in another study, the accuracy regarding the  $O_2$  measurements has been established to ±1% vol. (Patnaik et al., 2016), but this is still not better than for the 'Multilyzer STx' used in this work (as reported in Table 7).

Pollutant	Range	Accuracy	Resolution
NO <sub>x</sub>	0 – 5000 ppm vol.	$\pm 10\%$ reading	1 ppm vol.
СО	0 - 10 % vol. (0 - 100 000 ppm vol.)	±0.03% vol.	10 ppm vol.
O <sub>2</sub>	0 - 22 % vol.	±5% vol.	0.01% vol.
$CO_2$	0 – 20 % vol.	±0.5% vol.	0.1% vol.

However, when considering only the NO<sub>x</sub> measurements (and not muti-gas analysers), some fixed continuous analysers feature truly enhanced resolution (and, to a lesser extent, accuracy). For example, the 'Serinus 40', the 'Thermo Scientific Model 42i', the 'API T200' and the 'API T500u' all exhibit resolutions of 0.4 ppb (for the '42i' in its selectable 500 ppb range) or 0.5% of reading (for the other sensors), which, depending on their range, can either reach the maximum values of 5 ppb (for the 'API T500u') or 100 ppb (for the 'Serinus 40' and the 'API T200'). Based on their reported linearity of  $\pm 1\%$  full scale, it can be assumed that the relative accuracy of those sensors is also enhanced compared to the portable sensor used in this study. The range of those sensors is however much more limited (as they are primarily designed for ambient air analyses rather than combustion analyses), i.e. up to 100 ppm for the 'API T500u'. These sensor performances have been reproduced from consulted studies and/or from the respective datasheets (acoem, 2022; Landis and Edgerton, 2024; Rana et al., 2019; Teledyne API, 2021a, 2021b; Thermo Scientific, 2021).

A commonly used NO<sub>x</sub> combustion fixed analyser with a similar range as the one used in this study (yet still five times lower, i.e. 1000 ppm) is the '4000VML' (Signal Instruments, 2006). It is indeed an interesting sensor because it also presents a linearity of  $\pm 1\%$  full scale, i.e. a figure that can be closely linked to the accuracy and which is therefore significantly increased compared to the portable sensor used in these experiments (Signal Instruments, 2006). Noticeably, in combustion analyses (fixed) test bench, it has been reported to have been combined with the 'AVL Digas 444N' sensor already presented in Table 8, to compensate for this latter's poor NO<sub>x</sub> measurements performance (Solomon et al., 2020).

Regarding SO<sub>2</sub> measurements, a commonly used fixed sensor worth citing is the 'Serinus 50', exhibiting a measuring range of up to 20 ppm, a resolution of 0.5% of reading, and a linearity of  $\pm$ 1% full scale (acoem, 2023; Park et al., 2021; Vrekoussis et al., 2022). Compared to the portable sensor used in this study, the range is significantly reduced while the resolution and accuracy can be assumed significantly increased.

To summarize this fixed-to-portable sensor comparison, for low pollutant levels (which, according to Table 4 can be expected with fuel cells), it could indeed be beneficial in terms of accuracy and resolution to use fixed continuous acquisition bays. However, in the case of this study, a lot of the tests had to be conducted in field-test applications (as it will be detailed in the *Testing procedure* section) and the choice of a portable sensor was mandatory.

### 3.3 Conversion of ppm to mg/kWh

As demonstrated in the previous section, literature on space heating appliances pollution levels is quite rare and pollutant emissions are regularly reported in terms of concentration, i.e. in ppm (McDonald, 2009), or in terms of intensity, i.e. in mg/kWh (Energie+, 2007). However, it is quite rare for both pieces of information to be provided. In this case, the pollutant emissions measurements are provided by the metering device in ppm (see Table 7) whereas, for comparison purposes, it would be more meaningful to express them in terms of mg/kWh. Indeed, Table 3 for example reports from literature the NO<sub>x</sub> emission levels of several space heating appliances in

mg/kWh. In addition, as reported in the *Tested systems* section, the datasheets of the tested space heating appliances only express the emissions in terms of mg/kWh. Therefore, to use those figures as references for this study, the emission measures performed in this work must be converted from ppm to mg/kWh.

For natural gas appliances, this can be performed thanks to Equation (1) for carbon monoxide emissions (European Commission, 2021):

$$CO_{(mg/kWh)} = 1.074 \times CO_{(ppm)} \times \frac{(CO_2)_N}{(CO_2)_M}$$
 (1)

Where  $CO_{(mg/kWh)}$  is the carbon monoxide emissions level per unit of energy (kWh) that must be established for the studied combustion test;  $CO_{(ppm)}$  is the measured carbon monoxide concentration at the exhaust of the system during the combustion test (in ppm);  $(CO_2)_M$  is the measured carbon dioxide concentration at the exhaust of the system during the combustion test (in %) and  $(CO_2)_N$  is the maximum carbon dioxide concentration of the dry, air-free, combustion products (in %), which depends only on the natural gas type that is fed to the studied system during the combustion.  $(CO_2)_N$  is equal to 11.7% for G20 natural gas and 11.5% for G25 natural gas (European Commission, 2021).

Indeed, in Belgium (Paulus and Lemort, 2023d), as in France or Germany (Bruijstens et al., 2008), natural gas comes from different sources, which implies different gas compositions and different HHV and leads to the appellations 'lean' and 'rich' gas, respectively for the natural gas source providing the lower and the higher HHV (Haeseldonckx and D'haeseleer, 2007). Lean gas is also called 'L-gas' (CREG, 2018), 'type L' gas (Paulus and Lemort, 2023d) or G25 (Bruijstens et al., 2008) whereas rich gas is also called 'H-gas' (CREG, 2018), 'type H' gas (Paulus and Lemort, 2023d) or G20 (Bruijstens et al., 2008). The type of gas provided on the grid only depends on the localization of the delivery point. All lean gas deliveries are supposed to be progressively replaced (in Belgium) by 2030 by rich gas deliveries (CREG, 2018).

As reported in the previous section,  $(CO_2)_M$  and  $CO_{(ppm)}$  are provided by the meter used in this work. Also, in Equation (1), the 1.074 constant is the unit conversion coefficient related to CO emissions from natural gas appliances (Zlateva et al., 2020).

Similarly, ppm to mg/kWh conversion for  $NO_x$  emissions is obtained thanks to Equation (2) for natural gas appliances (European Commission, 2021) :

$$NOx_{(mg/kWh)} = \frac{\left(C_g \times NOx_{(ppm)} \times \frac{(CO_2)_N}{(CO_2)_M}\right) - 0.85(20 - T_m) + \frac{0.34(h_m - 10)}{1 - 0.02(h_m - 10)}}{\left(1 + \frac{0.02(h_m - 10)}{1 - 0.02(h_m - 10)}\right)}$$
(2)

Where  $NOx_{(mg/kWh)}$  is the nitrogen oxide emissions level per unit of energy (kWh) that must be established for the studied combustion test;  $C_g$  is the unit conversion coefficient related to NO<sub>x</sub> emissions from natural gas appliances (Bălănescu and Homutescu, 2017) and is equal to 1.764 for G20, i.e. rich gas, or 1.767 for G25, i.e. lean gas (European Commission, 2021);  $NOx_{(ppm)}$  is the sum of the measured nitrogen dioxide and nitric oxide concentrations at the exhaust of the system during the combustion test (in ppm);  $(CO_2)_N$  and  $(CO_2)_M$  have already been described for Equation (1);  $T_m$  is the temperature of the outdoor air used for the combustion (in °C) and  $h_m$ is the absolute humidity of the outdoor air used for the combustion (in g of water per kg of dry air).  $h_m$  is the only variable of Equation (2) that is not provided by the combustion analyser meter (see Table 7). By assimilating inlet air to humid air of relative humidity between 40 and 80%, at atmospheric pressure and at the  $T_m$  temperature,  $h_m$ can be approximated with the Engineering Equation Solver (EES) software. It is worth mentioning regarding Equation (2) that the allowable ranges for  $T_m$ ,  $h_m NOx_{(mg/kWh)}$  and are respectively 15 - 25 °C, 5 - 15 g of water per kg of dry air, and 50-300 mg/kWh. However, industrial partners in this work advise using Equation (2) anyway even if some parameters are out of those ranges.

The European standard from which Equation (1) and Equation (2) are deduced (European Commission, 2021) unfortunately does not provide any information about  $SO_2$  emissions conversion. Fortunately, another reference (TSI Incorporated, 2004) provided Equation (3), which has been reported to be relevant not only for  $SO_2$  but also for CO and NO<sub>x</sub> emissions, giving similar conversion results, at least in its documented allowable range, respectively to Equation (1) and Equation (2).

$$PEI_{(mg/kWh)} = F \times PEC_{(ppm)} \times \frac{20.9}{20.9 - (O_2)_M}$$
(3)

Where  $PEI_{(mg/kWh)}$  is the pollutant emissions intensity, i.e. the emission level per unit of energy (kWh) that must be established for the studied combustion test; *F* is an emission rate conversion factor that depends on the pollutant (and the type of fuel) and that is given in Table 9,  $PEC_{(ppm)}$  is the measured pollutant emissions concentration at the exhaust of the system during the combustion test (in ppm) and  $(O_2)_M$  is the measured oxygen concentration at the exhaust of the system during the combustion test (in %). Equation (3) has the particularity to consider  $O_2$ concentration (in %) in the exhaust whereas Equation (1) and Equation (2) rely on CO<sub>2</sub> concentration (in %). Similar conversion equations for diesel engines have not been reported in this paper but can also be found in the literature (Pilusa et al., 2012).

#### Table 9.

Natural gas F coefficients for Equation (3) depending on the pollutant type (TSI Incorporated, 2004).

Pollutant	F
Tonutant	mg/(kWh-ppm)
СО	0.974313
NO <sub>x</sub>	1.608389
$SO_2$	2.242466

### **3.4** Testing procedure

The end of the probe of the 'Multilyzer STx' must be placed at the centre of the exhaust gas chimney (or tailpipe) and the probe axis can either be oriented in the perpendicular plane of this chimney (or tailpipe) or parallel to it (if the measurements are conducted at the exit of the chimney/tailpipe). The probe disposes of a conical adjustable mechanical stop to ensure the correct probe depth to the centre of the chimney (see Figure 4).

The studied PEMFC system, which is composed of a PEMFC stack hybridized to a gas condensing boiler (as seen in the *Tested systems* section), has the advantage of being equipped by design with a small, sealable access hole, fitted with a cap, directly at the exhaust of the system (in the first 5 cm of the chimney). There is thus no need for the PEMFC system to place the combustion analyser meter at the exit of the chimney, which access is very often difficult and potentially risky if it figures on the roof of the building. However, some measurements have still been taken at the exit of the chimney for comparison purposes (with the probe fully inserted in the chimney). Indeed, temperature (which varies all along the double-walled chimney that cools down the flue gases and heats the inlet air from outdoors) is not only known to influence the NO<sub>x</sub> formation but also the NO-NO<sub>2</sub> equilibrium. This is especially the case in the near-post-flame zone (Bowman, 1975), i.e. close to the outlet of the system, but also in the atmosphere, i.e. close to the exit of the chimney, in the presence of Volatile Organic Compounds (Cheremisinoff and Young, 1977), which can be co-emitted in hydrocarbons combustion (Bowman, 1975).

The PEMFC hybrid system was tested in two separate modes: with only the PEMFC turned on and with only the gas condensing boiler turned on. This system, installed in 2019, was tested in a field-test application (in a real house) in Huy (in Belgium). At the moment of the tests, the whole machine has been functioning for about 15000 hours, but its integrated fuel cell has only been producing electricity for about 5500 hours. It is worth mentioning that another machine of this system, which was perfectly new, was tested in a laboratory environment (as it will be seen in the results reported in Figure 6).

The studied mCHP SOFC system does not involve any hole in its chimney by design. However, since one machine of this system was tested in laboratory facilities, a hole was manufactured at a chimney height of 50 cm (above the system flue gases outlet). This was also the case for the tested gas absorption heat pump (which was tested in steady state, at its rated power). The SOFC tested in the laboratory was used for two pollutant test campaigns (conducted at minimum and intermediate electrical power output, i.e. 500  $W_{el}$  and 1000  $W_{el}$ ). This machine, installed in 2021, had already been functioning for about 6000 hours before being tested. In the laboratory facilities, the return temperature of the heat recovery circuit, which affects the exhaust gas temperature, could be controlled (Paulus and Lemort, 2023c). For the other pollutant test campaign (at full-rated electrical power output, i.e. 1500  $W_{el}$ ), the combustion analyser meter was placed in another configuration. It was indeed positioned at the exit of the chimney (and fully inserted in it) since this campaign was performed on another SOFC machine (with the same reference) in a field-test application in Riemst (Belgium). At the moment of the pollutant measurements, this second machine, installed in 2017, has already been functioning for about 45000 hours.

As mentioned in the *Tested systems* section, another classical gas condensing boiler has been tested (only at the exit of its chimney, with the probe fully inserted in it). This system was tested in a field-test application in Riemst (Belgium).

At last, the Euro 6 diesel vehicle was tested at the exit of both of its tailpipes. The probe of the sensor could be oriented parallel to the tailpipe, so it has either been fully inserted in the tailpipe (about 35 cm before its exit) or only inserted over about 15 cm. The purpose was to see the changes in the exhaust gas temperature and their impact on the pollutant measurements. It is worth mentioning that the car engine was tested at idle ( $\pm$ 850 rpm) and at 1500 rpm, but the clutch was always disengaged.

All the tests include a purge with clean air before starting the measurements. It is indeed a mandatory step requested by the 'Multilyzer STx' combustion analyser meter. At last, the sample time was always one second.

## 4. Results

All the tests may also have included other specificities in the way they have been conducted and those are reported accordingly in Table 10 along with the pollutant emissions results.

Firstly, for the tests conducted in this study, there were not any significant differences between the signals for the CO concentration measurement and for the hydrogen-compensated CO measurements provided by the sensor (as mentioned in the *Measurements device* section), indicating that hydrogen levels in the exhaust appliances were always negligible. Thus, only the CO measurements have been reported in Table 10 and in the rest of this work.

The results reported in the first row of Table 10, i.e. conducted for the studied PEMFC gas boiler hybrid system with only the fuel cell running, have been reproduced in Figure 5 for illustration purposes. It is interesting to point out that an inexplicable short CO peak of about 60 ppm is noticeable after about 25 min of measurements, with the fuel cell running constantly (without any change operated to its control by the user or technicians). It will later on be seen in Figure 6 that a similar CO peak also occurs at the fuel cell startup of the same PEMFC gas boiler hybrid system, but in this case, the fuel cell was already running. However, it is likely to be related to an online (yet unidentified) recovery operation. Indeed, such PEMFC fuel cells are known to be very sensitive to impurities (as reported in Table 1) and many recovery mechanisms (conducted online or offline) can be associated with this fuel cell technology (Paulus et al., 2024).

Later in the acquisition signals (after about 47 min of measurements), Figure 5 also shows combined short NO, NO<sub>2</sub>, and CO peaks that coincide with an elevation of temperature of the flue gases. Those peaks also coincided with a sudden increase in gas consumption (that was monitored for this field-test site) and it was concluded that it corresponded to an inexplicable startup (and shutdown) of the gas boiler of the system. In fact, over a quite longer pollutant test conducted on the PEMFC-gas boiler hybrid of the field-test site of Oostmalle reproduced on Figure 7, similar pollutant peaks have clearly been identified to startups and shutdowns of the gas boiler. Figure 7 indeed shows that the gas boiler embodied in the PEMFC hybrid system is exhibiting unwanted ON/OFF cycling behaviour. As inferred by the literature, this highly transient behaviour of the gas boiler is expected to decrease the overall efficiency of the system (Bennett and Elwell, 2020). Additionally, it is interesting to point out from Figure 5 and Figure 7 that, although NO<sub>x</sub> peaks only occur at boiler startups, CO peaks occur both at boiler startups and shutdowns.

Also, it is strongly believed that the high level of hybridization between the fuel cell and the gas boiler (seen in Figure 2) is responsible for those many unexpected boiler startups and shutdowns and therefore prevents both sub-systems from operating as optimally and reliably as they would have as standalone units.



At last, as it is the case in all the tests conducted through this study, no SO<sub>2</sub> emissions could be measured.

**Figure 5.** Pollutant measurements of the fuel cell (only) steady state phase of the PEMFC-gas condensing boiler hybrid system (performed at the field-test site in Huy). The gas temperature elevation at the start of the test corresponds to the moment when the probe of the pollutant sensor is moved from ambient (external) air to the exhaust chimney.

In addition, for information, a graphical example of another pollutant test (performed with the same sensor in a laboratory environment) is given in Figure 6 for the studied PEMFC system in its startup phase only (with the boiler turned off). In that test, no  $NO_x$  nor  $SO_2$  could be measured. The startup phase (duration between the machine's startup initiated thanks to a thermal demand and the moment when the fuel cell starts producing electricity) takes about 7 min whereas the total duration to reach a steady state is about 15-20 min (gradually from 0 W<sub>el</sub> to its nominal output power of 750 W<sub>el</sub>). A CO peak of about 2 minutes, with a maximum at 55 ppm, can again be noticed at the beginning of the power and heat generation phase, probably due to transient behaviours of the internal reformer required for this PEMFC fed by natural gas (Paulus et al., 2024). The stepped behaviour of the CO<sub>2</sub> percentage measurement is explained by the resolution of the sensor and the fact that it is not directly measured but established by the combustion analyser from O<sub>2</sub> measurements (Table 7). However, the sudden CO<sub>2</sub> peak is probably an outlier as it could not be explained.



**Figure 6.** Pollutant measurements of the fuel cell (only) startup phase of the PEMFC-gas condensing boiler hybrid system (performed in a laboratory environment).



**Figure 7.** Pollutant measurements of the PEMFC-gas condensing boiler hybrid system (performed at the field-test site in Oostmalle). The fuel cell is always running and the sudden changes in the signals are related to the gas boiler being turned on and then turned off by the system itself (without any change operated to its control by the user or technicians). The gas boiler embodied in the system is therefore exhibiting unwanted ON/OFF cycling behaviour. The gas temperature elevation at the start of the test corresponds to the moment when the probe of the pollutant sensor is moved from ambient (external) air to the exhaust chimney. (a) CO and NO; (b) NO<sub>2</sub>; (c) Associated  $O_2$  concentration and gas temperature.

Test and conditions	NO <sup>a</sup>	NO <sub>2</sub> <sup>a</sup>	CO <sup>a</sup>	Remarks
PEMFC hybrid system ( <b>PEMFC only mode</b> ) Measured on the field-test site in Huy without control on the return temperature (or on the exhaust gas temperature)	0	0	<ul> <li>Startup : short peak up to 55-60 ppm for 2 min (in total). Also measured in the laboratory (Figure 6).</li> <li>Steady state : 0 but an unexplainable short peak similar to FC startup has been measured while the PEMFC was running (Figure 5).</li> </ul>	<ul> <li>Boiler turned down by closing the radiator valves in the house</li> <li>No difference in the pollutant measurement between the exit of the chimney (on the roof) and the exit of the system</li> </ul>
PEMFC hybrid system ( <b>condensing gas</b> <b>boiler only mode</b> ) Measured on the field-test site in Huy without control on the return temperature (or on the exhaust gas temperature)	<ul><li>Startup : peak up to 7 ppm for</li><li>5 min (in total)</li><li>Steady state : 0</li></ul>	- Startup : 3 ppm - Steady state : 3 ppm, i.e. 6.7 mg/kWh	<ul> <li>Startup : short peak up to 80 ppm for 30 sec (in total)</li> <li>Steady state : 30 ppm, i.e. 40.7 mg/kWh</li> </ul>	- Boiler turned on by opening the radiator valves in the house and setting a high-temperature setpoint on the thermostat; the PEMFC happened to be turned off, probably conducting a regeneration procedure (Paulus et al., 2024)
$SOFC - 500$ and $1000 W_{el}$ output Tested in laboratory with different heat recovery temperatures, i.e. different exhaust gases temperature (from 45°C to 25°C)	0	0	5 ppm (at 500 W <sub>el</sub> ), i.e. 28.3 mg/kWh 11 ppm (at 1000 W <sub>el</sub> ), i.e. 41.5 mg/kWh	<ul> <li>Return temperature of the heat recovery circuit has no influence on the pollutant measurements</li> <li>Only steady state data (the system in supposed to be turned on continuously and the startup test was not conducted)</li> </ul>
$SOFC - 1500 W_{el}$ output Measured on the field-test site in Riemst with only one heat recovery temperature corresponding to 60°C of exhaust gases temperature	0	0	8 ppm, i.e. 17.0 mg/kWh	<ul> <li>Same SOFC reference as in the laboratory tests but a different machine)</li> <li>Only steady state data (the system in supposed to be turned on continuously and the startup test was not conducted)</li> </ul>
Classical gas condensing boiler - high DHW load (exhaust gases temperature of about 65°C at the exit of the chimney) Measured on the field-test site in Riemst	<ul> <li>Startup : peak up to 8 ppm for 2 min (in total)</li> <li>Steady state : 5 ppm, i.e. 10.1 mg/kWh</li> </ul>	- Startup : peak up to 4 ppm for 2,5 min (in total) - Steady state : 2 ppm, i.e. 4.1 mg/kWh	<ul> <li>Startup : peak up to 50 ppm for 2 min (in total)</li> <li>Steady state : 10 ppm, i.e. 12.3 mg/kWh</li> </ul>	- No remark
Classical gas condensing boiler – low- temperature space heating load (exhaust gases temperature of about 30°C at the exit of the chimney) Measured on the field-test site in Riemst	- Startup : Untested - Steady state : 0 ppm	- Startup : Untested - Steady state : 0 ppm	- Startup : Untested - Steady state : 8 ppm, i.e. 10.7 mg/kWh	- No remark
Gas absorption heat pump at rated power. Tested in laboratory.	- Steady state : 8 ppm, i.e. 16.9 mg/kWh	- Steady state : 12 ppm, i.e. 25.4 mg/kWh	- Steady state : 120 ppm, i.e. 153.7 mg/kWh	- No remark
Euro 6 Diesel Engine at idle, i.e. ±850 rpm (car in neutral)	<ul> <li>Startup : continuous increase for about 20 min up to 60 ppm</li> <li>Steady state : 55 ppm, i.e. 238 mg/kWh (Pilusa et al., 2012)</li> </ul>	0	<ul> <li>Startup : rapid increase for about 3 min to the 200-300 ppm range</li> <li>Steady state : 200-300 ppm, i.e. 800-1200 mg/kWh (Pilusa et al., 2012)</li> </ul>	<ul><li>The probe must be fully inserted in the tailpipe to record pollutant emissions</li><li>There is no difference between the left and right tailpipes</li></ul>
Euro 6 Diesel Engine at 1500 rpm	- Startup : unavailable (engine already warmed up) Steady state : 40 ppm, i.e. 173 mg/kWh (Pilusa et al., 2012)	0	- Startup: unavailable (engine already warmed up) Steady state : 850 ppm, i.e. 3430 mg/kWh (Pilusa et al., 2012)	<ul> <li>The probe must be fully inserted in the tailpipe to record pollutant emissions</li> <li>There is no difference between the left and right tailpipes</li> </ul>

#### Table 10. Pollutant emissions measurements results (in all tests, the sensor indicated 0 ppm of SO<sub>2</sub> emissions).

<sup>a</sup> Equation (3) has been used to convert ppm measurement into mg/kWh for steady state measurements only (of natural gas appliances). The similar conversion law for diesel engines comes from literature (Pilusa et al., 2012). Peaks and startups have highly transient dynamic behaviours both on the pollutant concentration and the O<sub>2</sub> percentage signal, making the ppm to mg/kWh conversion hazardous.

# 5. Discussion and limitations

None of the tested systems (PEMFC, SOFC, gas condensing boilers and Euro 6 diesel engine) showed any  $SO_2$  emissions. This is either an indication of an issue with the  $SO_2$  sensor or it proves the efficiency of the desulphurization treatment implemented in the natural gas process before it enters the grid (Paulus et al., 2024). In addition, both fuel cell systems include a desulphurizer in their respective fuel processors according to the consulted manufacturer's documentation. Regarding the diesel vehicle, the lack of  $SO_2$  emissions could be explained by low sulphur content of diesel in the EU, limited to 10 ppm according to the EN 590:2009 regulation (European Commission, 2009). It could also be explained by the oxidation of  $SO_2$  into  $SO_3$  (not measured by the sensor) in the selective catalytic converter used in the exhaust of the engine to reduce  $NO_x$  emissions (Wade and Farrauto, 2012).

Both fuel cell systems (PEMFC and SOFC) do not show any NO<sub>x</sub> emissions even if they involve high-temperature reforming processes (Paulus et al., 2024; Paulus and Lemort, 2022a). Oppositely NO<sub>x</sub> emissions of gas boilers (in steady state) were measured between 3 and 7 ppm, which is rather low. Using Equation (3) and thus considering the O<sub>2</sub> percentage measurement (not shown in Table 10), those figures can be converted in the 6.7-14.2 mg/kWh range, which is slightly better than provided by the literature (see combustion only figures reported in Table 3). The lower part of that range, i.e. 6.7 mg/kWh, corresponds to the gas boiler of the PEMFC hybrid system, and it is indeed under the 7.2 mg/kWh figure announced by the manufacturer. For the other classical gas condensing boiler, it is also under the announced value of 20 mg/kWh (as seen in the *Tested systems* section). In comparison, the diesel Euro 6 engine showed NO<sub>x</sub> concentrations of 55 ppm in neutral and 40 ppm at 1500 rpm (without any load since the clutch was not engaged). Equation (3) and the coefficients of Table 9 being only relevant for natural gas appliances, another conversion equation from ppm to mg/kWh provided by literature for diesel engines has been implemented (Pilusa et al., 2012). Considering a molar mass of NO of 30, these NO<sub>x</sub> emission concentrations correspond to the 173-238 mg/kWh range, i.e. under but close to the maximum NO<sub>x</sub> Real Driving Emissions announced at about 280 mg/kWh, assuming an average consumption of 6L per 100 km (as seen in Table 3).

For all tested systems except the gas absorption heat pump, the NO<sub>2</sub> emissions are either nil or quite low compared to NO emissions, which was expected as NO has been reported to be the predominant nitrogen oxide emitted by combustion devices (Bowman, 1975).

For the gas absorption heat pump, NO<sub>2</sub> and NO emission levels are similar. The total NO<sub>x</sub> emission level was measured at 42.3 mg/kWh. Those values are about three times greater than the tested classical gas condensing boiler given the worse NO<sub>x</sub> results. Although the system is reported to belong to the NO<sub>x</sub> class 5 (Robur, 2020), the system still exhibits NO<sub>x</sub> emissions levels below the limit of class 6, i.e. the most stringent emission class reported in Table 6 according to the applicable standards.

There were no CO emissions regarding the steady state operating conditions of the PEMFC system (other than an explicable peak that is similar to the transient CO peak that occurs at the PEMFC startup, as seen in Figure 6). This was expected since CO is a major pollutant of PEMFC stacks and since it has been reported that the system is equipped with a CO-removing apparatus in the fuel (natural gas) processing system before the stack (Paulus et al., 2024). Transient CO peaks are surely not caused by the fuel cell stack but by the fuel processor of the PEMFC system. For example, it could happen when the reforming processes start and are not yet at their steady state temperature levels, impeding the CO remover from operating efficiently. During these transients for which CO can occur, the PEMFC stack must indeed surely be bypassed (Paulus et al., 2024). Also, because methane reforming requires temperatures much higher than the one occurring in the PEMFC stack, it has been reported that the PEMFC system involves an afterburner for reforming purposes (Paulus et al., 2024). In addition to burning the stack exhaust gases when the PEMFC is running (the anode exhaust still contains unused hydrogen and the cathode contains an excess of air, which is at a higher temperature than the ambient air), this afterburner also requires a direct feed from the natural gas supply to ensure enough heat for the reforming processes (Paulus et al., 2024). The inexplicable CO peak while the PEMFC was running is likely to be related to this afterburner (after the stack) and it can once again be assumed that no CO has gone through the PEMFC stack.

Oppositely, the SOFC system (two different machines of the same reference tested) showed slight CO emissions (5 ppm, 11 ppm, and 8 ppm respectively at 500  $W_{el}$ , 1000  $W_{el}$ , and 1500  $W_{el}$  of power output) with no dependence on the thermal output or on the exhaust gases temperature (driven by the return temperature of the heat recovery system). Through Equation (3), these CO concentrations respectively correspond to 28.3 mg/kWh, 41.5 mg/kWh, and 17.0 mg/kWh. It is worth mentioning that the PEMFC system was mainly tested in field-test real applications so the return temperature (and the exhaust gas temperature) could not be controlled, although it is not believed to affect the pollutant emissions in steady state (which were nil).

CO emissions peak (between 50 and 60 ppm) at gas condensing boilers startup is probably due to the momentary incomplete combustion in this highly transient starting process. In steady state, the tested machines showed 8 to

30 ppm of CO emissions, corresponding to 10.7 mg/kWh to 40.7 mg/kWh using Equation (3), i.e. very similar to the CO emissions range of the SOFC). The gas condensing boilers were tested in field-test applications so the return temperature (and therefore the exhaust gases temperature) could not be controlled. Regarding the diesel engine, the steady state CO emissions were much higher, between 200 and 300 ppm at idle and up to 850 ppm at 1500 rpm. Equation (3) and the coefficients of Table 9 can only be used with natural gas. Therefore, another conversion law has been found in the literature (Pilusa et al., 2012), which leads to the 800-1200 mg/kWh range at idle and to about 3430 mg/kWh for the 1500 rpm test. Those levels of CO emissions are far greater than the one announced on the certificate of conformity for the average NEDC (calculated in this work to about 145 mg/kWh, as seen in the *Tested systems* section). This is another proof of the inadequacy of the NEDC to account for pollutant emissions (Pavlovic et al., 2017) but it also should be reminded that maintaining the engine at 1500 rpm while keeping the vehicle stationary is also not exactly representative of real driving conditions (although it provides interesting results for comparisons).

CO emissions of the tested gas absorption heat pump (120 ppm, corresponding to 153.7 mg/kWh related to the fuel input) are about four times greater than the tested gas condensing boilers giving the worse CO emissions results.

Regarding NO<sub>x</sub>, SO<sub>2</sub>, and CO emissions of residential fuel cells, the results of this research are in fact aligned with the insufficiently documented and unverified previous papers (as detailed in the *Pollutant emissions of residential fuel cells* section).

As implied in the *Measurements device* section and especially since its pollutant levels have been measured especially low, the laboratory tests conducted on the fuel cells could also have been performed with fixed continuous  $NO_x/SO_2$  analysers for optimal resolution and accuracy. For example, the  $NO_x$  analyser '4000VML' (Signal Instruments, 2006) and/or the SO<sub>2</sub> analyser 'Serinus 50' (acoem, 2023) could have been used, at least if the actual SO<sub>2</sub> emission levels are not completely nil as opposed to what this study tends to indicate (in which case, investing in an additional SO<sub>2</sub> sensor would have been quite pointless). However, more accurate fixed sensors would not have changed the statements inferred by this study regarding the low pollutant levels exhibited by the fuel cells. In addition, for all the tests conducted in field-test applications, it was mandatory to use a portable sensor (such as the one used in this study).

The main limitation of this research comes from the fact that Volatile Organic Compounds (VOCs) but mostly Particulate Matter (PM) emissions were not considered in this study for the tested systems. Actually, an infructuous attempt to measure fine particulates has been made using the 'AFRISO STM 225' dust sensor (AFRISO-EURO-INDEX Group, 2018) in the exhaust chimney of the studied fuel cell systems. Indeed, as it can be seen in Figure 8 for the tested SOFC system, it is not possible to distinguish in the resulting signal any particulate matter emissions when the probe is placed in the exhaust chimney. This can mean three very different things: the fuel cells did not emit any fine particulates; the PM emissions could have been beneath the sensitivity range of the sensor, or the sensor might not have been optimally suited for natural gas flue gas analysis. This latter assumption is the most probable as the sensor is originally designed to measure dust concentration from pellets or wood-fired systems (AFRISO-EURO-INDEX Group, 2018). Unfortunately, no other PM sensor was available for this study at the time.



**Figure 8.** Example of signal from the fine particulates sensor 'AFRISO STM 225' for the studied SOFC system. There is absolutely no change in the signal when the probe is placed in ambient (external) air and when it is placed inside the fuel cell exhaust chimney.

Another limitation of this work lies in the validity of the comparison of emissions intensity per unit of energy (in g/kWh) between the Euro 6 vehicle and the tested space heating (or CHP) appliances. Indeed, in the Euro 6 standard (for heavy-duty vehicles), the pollutant intensity limits are defined in g/kWh based on the energy available in the engine (Martyr and Rogers, 2021). The conversion law used in this study for the tested light-duty vehicle uses the same definition of pollutant intensity (Pilusa et al., 2012), i.e. based on the engine's mechanical

energy. Therefore, it is not comparable as-is to the pollutant intensity (per unit of energy) also in g/kWh reported or measured for space heating appliances, i.e. referring to its low heating value fuel energy input (Fumey et al., 2018). This limitation is also applicable to the gas absorption heat pump. Indeed, its pollutant emissions measurements are related to the fuel input and not the heat generated at its output. Although it is commonly accepted that a classical gas condensing boiler has an average low heating value efficiency of 90% (Baldi et al., 2017; CWaPE, 2005), the tested gas absorption heat pump is announced by the manufacturer to achieve 169% of low heating value efficiency in some specific conditions (Robur, 2020). Therefore, even if it has been demonstrated that such gas absorption heat pumps are not likely to be that efficient in field-test applications (Dávila et al., 2022b), the pollutant emission levels should always be considered taking into account the thermal efficiency of the systems.

Similarly, when considering the pollutant intensity (per unit of energy) of the fuel cells tested in this study, it is crucial to recognize that these units function as cogeneration systems. They provide both electrical and thermal energy and yet, the pollutant measurement reported in g/kWh is only anchored to their fuel input. This distinct basis for measuring emissions intensity necessitates careful consideration when making direct comparisons across these different categories of energy-producing appliances (and vehicles).

Yet, another way of comparison could be thought of to provide perspectives about the impact of the different appliances. The proposed approach is based on the computation of their average total pollutant emissions yearly use. For example, considering an average Belgian household with 17500 kWh<sub>th</sub> of space heating thermal demand per year (CREG, 2022) equipped with a classical gas condensing boiler of 90% of low heating value efficiency (Baldi et al., 2017; CWaPE, 2005), the related NO<sub>x</sub> emissions of the space heating would correspond to the ones of only between 775 and 2660 km driven with the 6L/100km Euro 6 car (depending on the different results obtained for the tested gas condensing boilers). In other words, with that Euro 6 car, a 16000 km distance traveled over a year, as it is a common assumption in recent studies in developed countries (Amatuni et al., 2020), corresponds in terms of NO<sub>x</sub> emissions to between 6 and 20 Belgian average households that use gas condensing boilers as space heating appliance.

Considering the tested gas absorption heat pump with an assumed (theoretical) seasonal efficiency of 150% (Dehghan B. et al., 2020) and again the average Belgian household with 17500 kWh<sub>th</sub> of space heating thermal demand per year (CREG, 2022), the associated NO<sub>x</sub> emissions levels would correspond to the ones about 4750 km driven with the 6L/100km Euro 6 car. The chosen example of an annual distance of 16000 km traveled by the tested car would thus correspond to a little more than three Belgian average households that use a gas absorption heat pump as their space heating appliance.

Similarly, based on its datasheet efficiencies, the yearly heat generation of the tested SOFC at its nominal power (1500  $W_{el}$ ) all year long can be estimated to about 5250 kWh<sub>th</sub>, which would correspond to about 372 gCO produced over the year (based on the results of this study given in Table 10). Considering the CO emissions intensity respectively at idle and at 1500 rpm in neutral given in Table 10, this corresponds to only 180 to 775 km driven with the Euro 6 car. In other words, between 20 and 89 machines of the tested SOFC would be needed to match the CO emissions that account for an annual average driving distance of 16000 km (Amatuni et al., 2020) with the considered Euro 6 vehicle.

Although NO<sub>x</sub>, SO<sub>2</sub>, and CO levels of residential fuel cells have been demonstrated in this study to be very low, their market penetration also (and mainly) lies in their profitability and in the CO<sub>2</sub> savings they are allowing. Fortunately, the exact PEMFC and SOFC systems presented in this work have already been investigated in field-test applications regarding these matters in previous studies (Paulus et al., 2022a; Paulus and Lemort, 2022a). Two machines of each technology have been monitored in field-test applications for the whole year 2021. Their results have been summarized in Table 11 and Table 12. Those cost and CO<sub>2</sub> indicators are based on a comparison with (current) reference energy production systems with their own emission factors and costs. Those reference systems consist of electricity from the (Belgian) grid and heat from a gas-condensing boiler of 90% of LHV efficiency. The considered electrical price is  $0.333 \text{€/kWh}_{el}$  whereas the considered gas price is  $0.093 \text{€/kWh}_{HHV}$ , and they were approximately valid for residential bills in Belgium (and close to the Germany-France-Nederland average) in 2021 (CREG, 2024). Interestingly, according to the Belgian energy regulator, those energy costs are quite close to the 2023 ones in the same countries/regions (CREG, 2024). The gas consumption emission factor considered was the one computed hourly and provided by Electricity Maps (https://www.electricitymaps.com). For information, the statistical average of the Belgian grid electricity consumption emission factors in 2021 was 167 gCO<sub>2eq</sub>/kWh<sub>el</sub>.

It is noteworthy that the results reported in Table 11 and Table 12 are extremely case dependent and not only of energy costs. They are indeed dependent on the dwelling (and notably on its hydraulic installation), the way the system has been installed, the way the owner is using it, the owner's ability to consume the electrical production from the system, *etc.* Those indicators shall thus be considered with caution before being replicated in similar fuel

cell systems. Regarding costs and  $CO_2$  balances, positive figures in Table 11 and Table 12 correspond to savings, so negative values indicate poorer performances than the reference systems.

#### Table 11.

2021 field-test cost and  $CO_2$  indicators for two of the PEMFC-gas boiler hybrid system studied in this work. That year accounted for 2286 degree-days (Gas.be, 2021) according to the base 16.5°C (The Chartered Institution of Building Services Engineers, 2006)). Reproduced and adapted from reference (Paulus et al., 2022a).

Monitored data	PEMFC #1	PEMFC #2	Monitored data	PEMFC #1	PEMFC #2
HHV equivalent energy consumed (kWh)	20083	38243	LHV Electrical efficiency (%)	11.1	9.3
Electrical production (kWh)	2011	3222	LHV Thermal efficiency (%)	69.4	84.5
Electrical consumption (kWh)	298	258	LHV Total efficiency (%)	80.5	93.8
DHW (kWh)	1627	2095	Space heating (kWh)	10941	27061
Utilization cost savings (€)	≈1430	≈1300	Utilization CO <sub>2eq</sub> savings (kgCO <sub>2eq</sub> )	-469	-45

#### Table 12.

2021 field-test cost and  $CO_2$  indicators for two of the SOFC system studied in this work. Climate hardiness is the same as reported in Table 11 but it is not as relevant as those SOFC systems are electrically driven and do not provide space heating (at least in the studied field-test sites). Reproduced and adapted from reference (Paulus and Lemort, 2022a).

Monitored data	SOFC #1	SOFC #2	Monitored data	SOFC #1	SOFC #2
HHV equivalent energy consumed (kWh)	25031	24273	LHV Electrical efficiency (%)	52.4	59.0
Electrical production (kWh)	11843	12922	LHV Thermal efficiency (%)	15.8	11.6
Electrical consumption (kWh)	11	2	LHV Total efficiency (%)	68.2	70.6
Heat recovered (kWh)	3569	2549	Utilization CO <sub>2eq</sub> savings (kgCO <sub>2eq</sub> )	-3013	-2969
Utilization cost savings (€)	≈-45	≈450			

From a global warming perspective, those systems are using fossil natural gas and Table 11 and Table 12 show that they cannot compete with the electrical grid (that involves renewables and should even involve more of those in the future). It must however be considered that in Belgium, in 2020 and 2021 (the author of this paper has not the information for 2022 and 2023), there was always some electrical production that came from natural gas power plants. Therefore, through the System Marginal Price (SMP) principle (Chae et al., 2012), the Marginal Emission Factor (MEF), which 'reflects the emissions intensities of the marginal generators in the system, i.e. the last generators needed to meet demand at a given time' (Siler-Evans et al., 2012), can always be considered equal to the emission factor for the electricity production from natural gas power plants. This would change the grid electricity emission factor to about three times more, i.e. to 456 gCO<sub>2eq</sub>/kWh<sub>el</sub> (Paulus et al., 2022a) and this would reverse the CO<sub>2</sub> balance of the SOFC systems to 700-1100 kgCO<sub>2eq</sub> savings (Paulus and Lemort, 2022a). This analysis is strengthened by the fact that SOFC systems are flexible in terms of electrical power output (within the 33-100% range for the system presented in this work) and could be controlled remotely to replace electrical production from natural gas power plants. In addition, the demonstrated electrical efficiencies of the studied SOFCs (Table 12) lie in the order of magnitude (and it is even slightly better for the SOFC #2) than typical natural gas power plants (Paulus and Lemort, 2022a). It is worth mentioning that decentralized local electrical production also avoids transportation and distribution losses, which can reach about 6-7% in the EU (Psomopoulos et al., 2010). This could have been considered in these ecological balances and they would actually be slightly improved.

The studied PEMFC system being not flexible in terms of power output and presenting at best a LHV electrical efficiency of 37% (Table 5), this Marginal Emission Factor approach is not as relevant and considered not applicable. Even so, it has been demonstrated in the mentioned previous study that the  $CO_2$  balance of the studied PEMFC-gas boiler system could even remain negative (with the MEF assumption), at least for the PEMFC #1 (Paulus et al., 2022a).

Regarding energy utilization costs, the analyses are similar as the PEMFC-gas boilers systems can exhibit losses compared to reference systems (as indicated for the PEMFC #1 in Table 11) and as the SOFCs indicate significant yearly savings (Table 12). For instance, with the results of Table 12, if an investor wanted to ensure reaching a return on investment before 10 years, the maximum capital costs of the system (including installation) should not reach more than about 13-14 k $\in$ , considering that the owner already has a main space heating appliance (Paulus and Lemort, 2022a). Although this SOFC's capital costs are unknown to the authors, it is interesting to point out

that back in 2015, residential SOFC technology cost (with auxiliaries) was estimated at the time to be about 10  $k \in /kW$  (Napoli et al., 2015), i.e. quite close to that cost threshold.

These are the probable reasons why it has been reported that historical PEMFC manufacturers concluded that the future of fuel cells in domestic built environment applications lies with SOFCs, and have therefore stopped PEMFC development (Elmer et al., 2015).

# 6. Conclusion

The  $SO_2$  and  $NO_x$  emission factors of (Belgian) grid electricity and common space heating appliances reproduced in this work represent an interesting contribution of this paper. In addition, this study has documented and reported equations that allow for converting pollutant concentration measurements, i.e. in ppm, into pollutant intensity per unit of energy, i.e. in mg/kWh, useful for comparative benchmarking analysis (especially considering the  $SO_2$  and  $NO_x$  emissions factors reported in this paper).

As a final conclusion, both tested residential fuel cell technologies fed by natural gas can be considered clean regarding  $SO_2$  and  $NO_x$  emissions. In addition, the CO emissions can be considered quite low for the tested SOFC and even nil for the tested PEMFC. Those statements apply even with machines that have already been running for up to 45000 hours. Therefore, the biggest issue of natural gas fuel cell technologies still lies in the  $CO_2$  emissions associated with the fossil fuel they consume.

However, this study still demonstrated that the gas condensing boiler embodied within the tested PEMFC-gas boiler hybrid system exhibited many undesired ON/OFF cycling (with associated small NO<sub>x</sub> and CO peaks). It is strongly believed that the high level of hybridization between the fuel cell and the gas boiler (seen in Figure 2) is responsible for those many unexpected boiler startups and shutdowns and therefore prevents both sub-systems from operating as optimally and reliably as they would have as standalone units.

At last, this research established that the tested gas absorption heat pump gives worse  $NO_x$  and CO emission levels than classical gas condensing boilers, even when considering that the gas absorption heat pump may exhibit an increased thermal efficiency of about 150% on a low heating value basis (Dehghan B. et al., 2020).

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# **CRediT** authorship contribution statement

**Nicolas Paulus:** Conceptualization, Formal analysis, Investigation, Resources, Data curation, Methodology, Software, Writing – original draft, Writing – review & editing, Visualization. **Vincent Lemort:** Funding acquisition, Writing – review & editing, Supervision, Project administration, Validation.

# **Declaration of competing interest**

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

# Data availability

Data will be made available on request.

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