

Performance analysis of a mini exhaust air heat pump integrated into a low energy detached house: experimental on-site performance

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Introduction

- ❑ In new buildings, the ventilation can account for 30 to 60 % of the total building heating demand.
- ❑ Consequently, highly efficient heat recovery systems on the ventilation must be developed.
- ❑ In that context, the **simple exhaust ventilation system + exhaust air heat pump** is an interesting solution.
- ❑ The on-site performance of this system were monitored in 2017 for a residential building situated in Belgium to assess the potential of the system.



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Building characteristics

- New residential building built in 2016
- Wooden two-story freestanding house
- Floor area: 155 m²
- Total exposed area: 389 m²
- $n_{50}=0.6 \text{ Vol/h}$
- U-values:

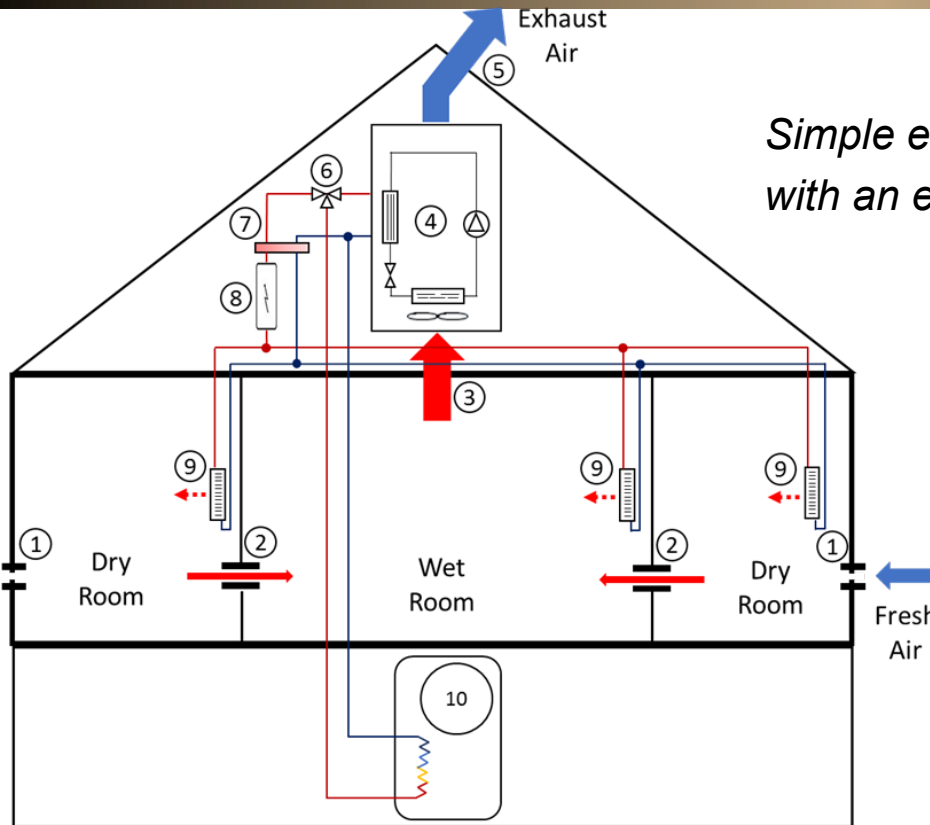
| Wall type | Composition | Area [m ²] | U-value [W/m ² -K] |
|------------|--|------------------------|-------------------------------|
| Outer wall | Wood structure + 40 cm cellulose | 139 | 0.11 |
| Roof | Wood structure + 40 cm cellulose | 127.2 | 0.15 |
| Floor | Concrete slab floor + 65 cm cellular glass | 82.7 | 0.07 |
| Window | Triple-glazed + aluminum frame | 18.5 | 0.84 |

- Global heat transfer coefficient: 0.2 W/m² - K



HVAC systems characteristics

Simple exhaust ventilation system combined with an exhaust air heat pump



Legend:

- ① : Fresh air inlet
- ② : Transfer orifice
- ③ : Exhaust air extraction
- ④ : Exhaust air heat pump
- ⑤ : Exhaust air evacuation
- ⑥ : Three-way valve
- ⑦ : Hydraulic separator
- ⑧ : Back-up electrical resistance
- ⑨ : Fan-coil unit
- ⑩ : Domestic hot water storage tank

- Ventilation: active 24 hours, 200 m³/h
- Heat pump water exhaust temperatures:
 - DHW: 25-55°C
 - Space heating: 40-45°C

- Nominal Heat pump capacity: 1400 W (A20W35)
- 3000 W electric resistance

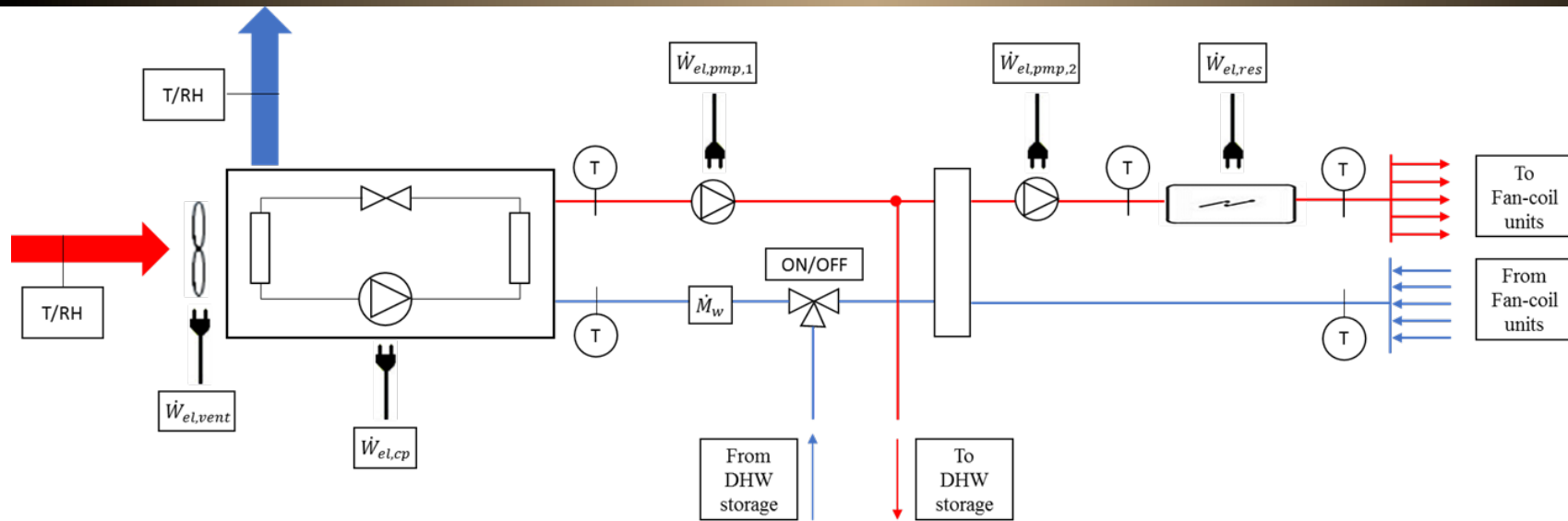


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System instrumentation



To measure the performance:

- 5 water temperature sensors
- 1 water flow meter
- 3 electric meters: compressor, resistance, total

Additional measurements:

- Temperature and humidity of air: supply and exhaust
- Weather station (less than 8 km)



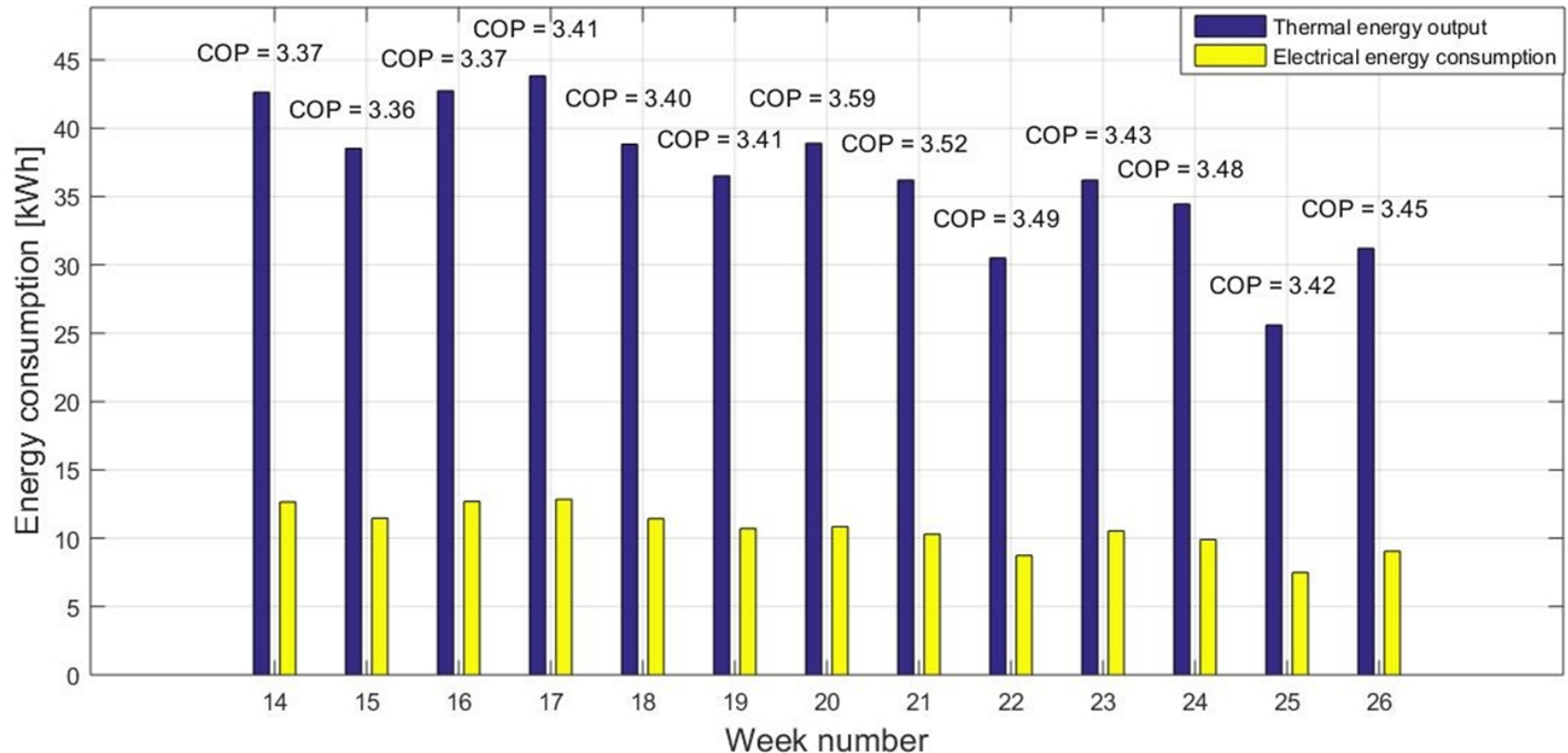
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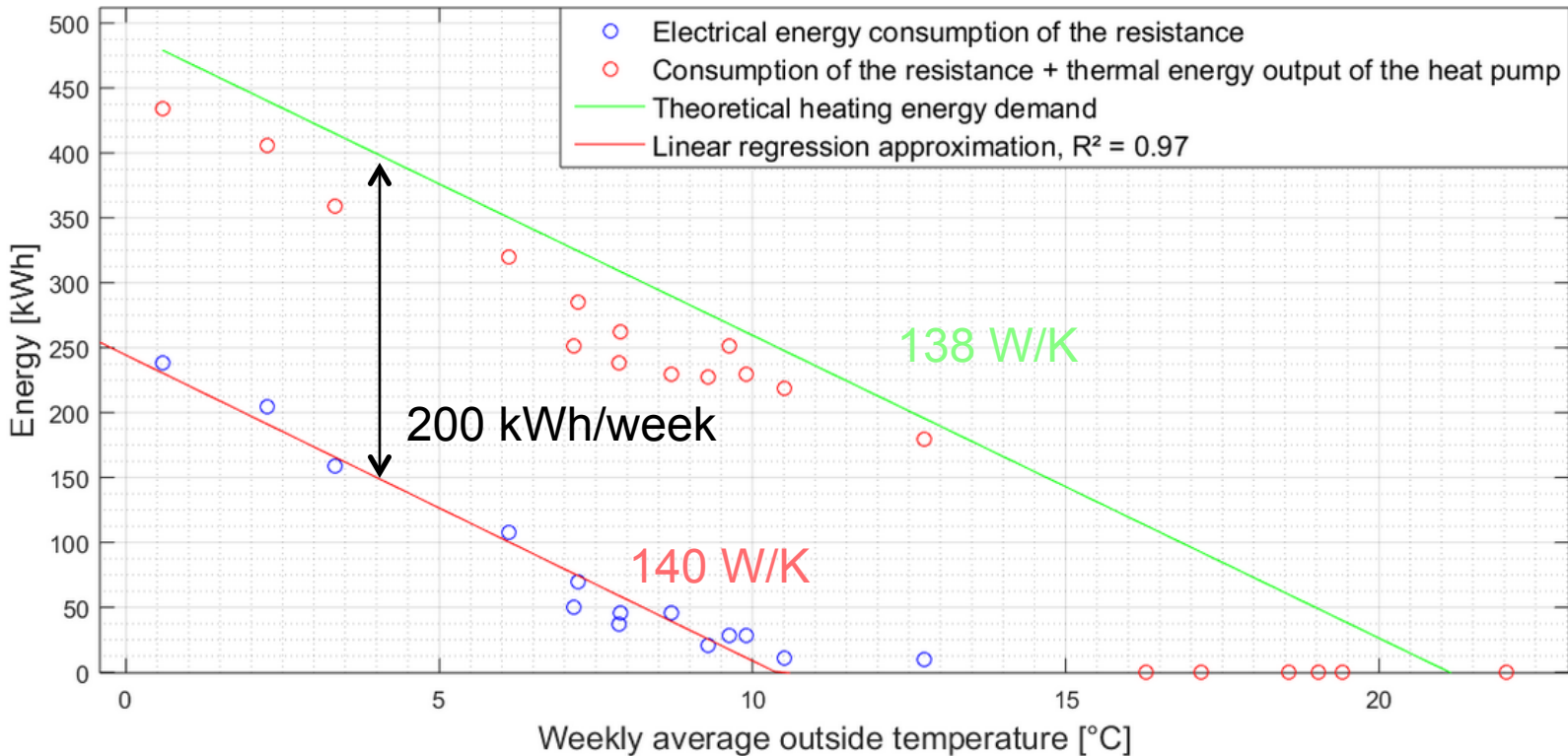
DHW production mode

- The COP of the machine is relatively constant over the weeks (from 3.36 to 3.59 for the weeks 14 to 26).
- The thermal energy output varies from 44 to 25 kWh/week





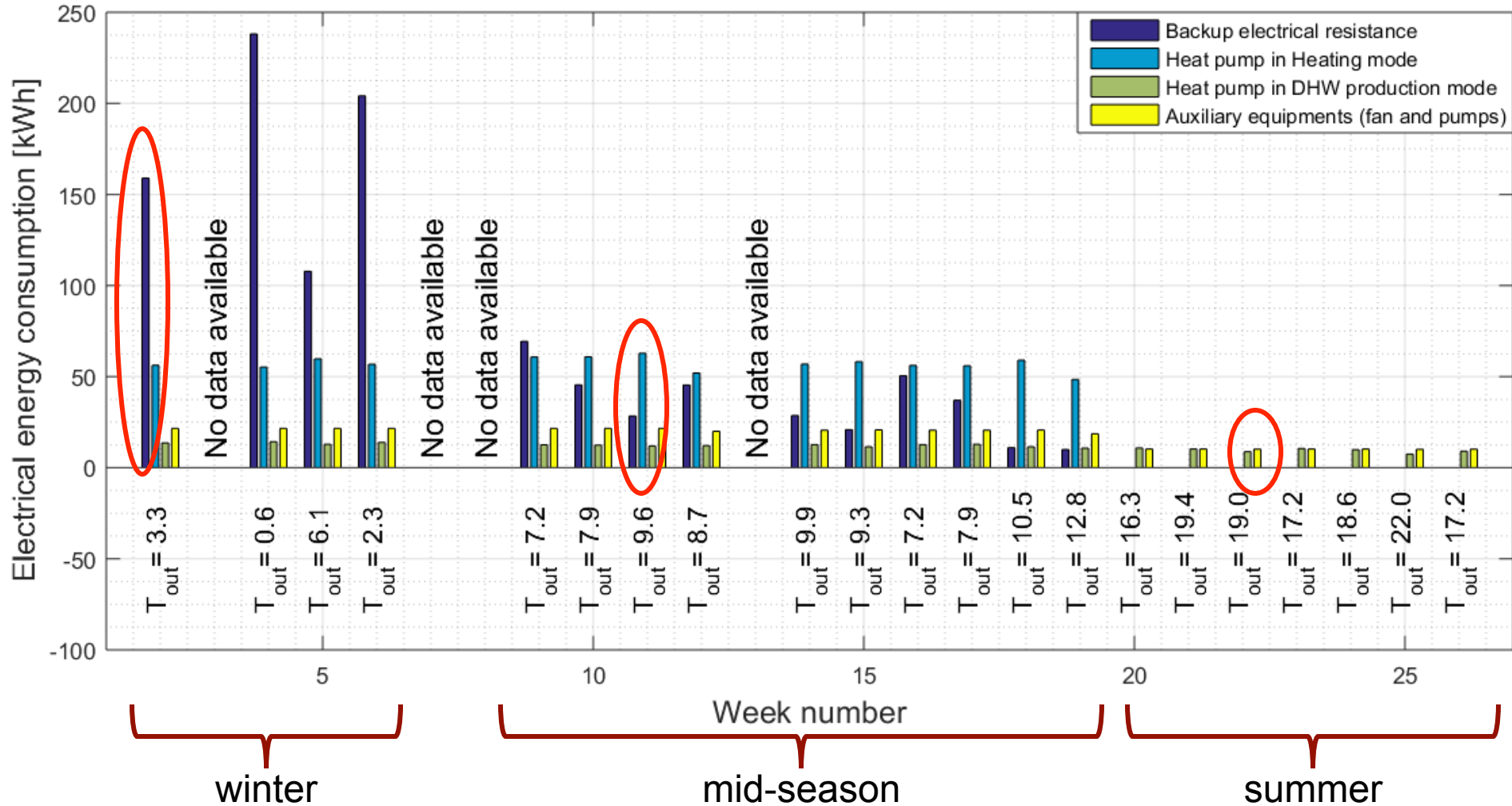
Space heating mode



- ❑ The backup electrical resistance consumption varies from 240 to 0 kWh/week
- ❑ Production of the heat pump in heating mode is almost constant (200 kWh/week)
- ❑ Measured overall heat transfer coefficient is close to the theoretical one



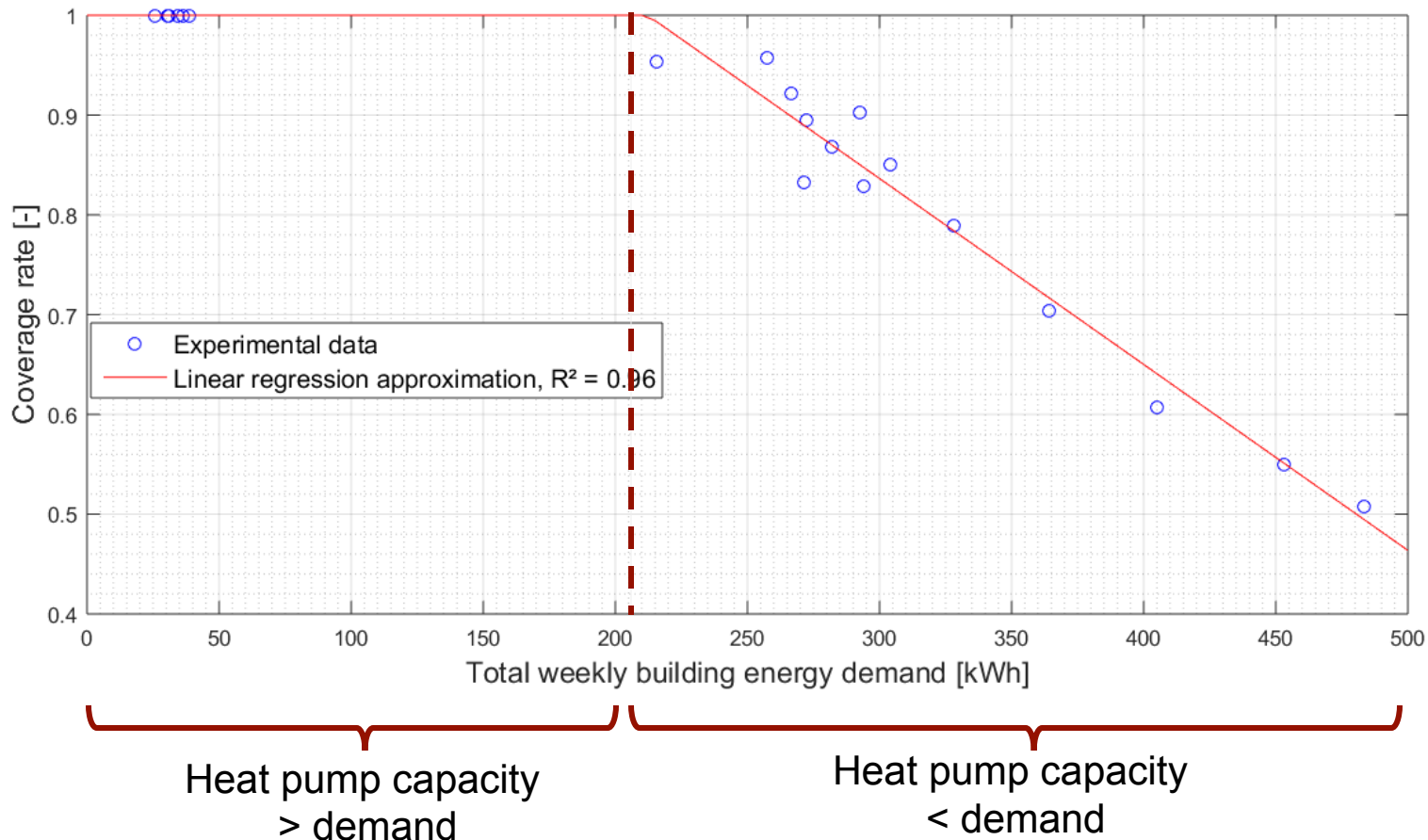
Whole system consumption (DHW + space heating)





Coverage factor

□ HP Coverage factor:
$$\tau = \frac{Q_{HP,DHW,week} + Q_{HP,Heating,week}}{Q_{HP,DHW,week} + Q_{HP,Heating,week} + W_{res,week}}$$





Performance for Standard weather conditions

- ❑ Experimental results cover only part of the year: January 8th to July 2nd 2017
- ❑ During spring 2017, temperatures were above seasonal normal temperatures

=> Heat pump system performance is extrapolated over one year with a standard climate by means of the heating demand computed in the EPB certificate (= approximation).



Performance for Standard weather conditions

| | $T_{out,avg}$ [°C] | $Q_{Heating}$ [kWh] | Q_{DHW} [kWh] | τ [-] | $Q_{HP,Total}$ [kWh] | $Q_{HP,DHW}$ [kWh] | $Q_{HP,Heating}$ [kWh] | W_{Res} [kWh] | $W_{HP,DHW}$ [kWh] | $W_{HP,Heating}$ [kWh] |
|---------|-----------------------|------------------------|--------------------|---------------|-------------------------|-----------------------|---------------------------|--------------------|-----------------------|---------------------------|
| Jan | 3.2 | 1545 | 180 | 0.58 | 994 | 180 | 814 | 731 | 52 | 229 |
| Feb | 3.9 | 1278 | 162 | 0.71 | 1024 | 162 | 862 | 415 | 47 | 243 |
| Mar | 5.9 | 1084 | 180 | 0.80 | 1005 | 180 | 825 | 259 | 52 | 232 |
| Apr | 9.2 | 548 | 174 | 1.00 | 722 | 174 | 548 | 0 | 51 | 154 |
| May | 13.3 | 74 | 180 | 1.00 | 254 | 180 | 74 | 0 | 52 | 21 |
| Jun | 16.2 | 0 | 174 | 1.00 | 174 | 174 | 0 | 0 | 51 | 0 |
| Jul | 17.6 | 0 | 180 | 1.00 | 180 | 180 | 0 | 0 | 52 | 0 |
| Aug | 17.6 | 0 | 180 | 1.00 | 180 | 180 | 0 | 0 | 52 | 0 |
| Sep | 15.2 | 10 | 174 | 1.00 | 184 | 174 | 10 | 0 | 51 | 3 |
| Oct | 11.2 | 376 | 180 | 1.00 | 556 | 180 | 376 | 0 | 52 | 106 |
| Nov | 6.3 | 1061 | 174 | 0.81 | 999 | 174 | 825 | 236 | 51 | 232 |
| Dec | 3.5 | 1511 | 180 | 0.59 | 1001 | 180 | 822 | 689 | 52 | 231 |
| Total | | 7487 | 2118 | | 7273 | 2118 | 5156 | 2330 | 615 | 1451 |
| Average | 10.2 | | | 0.76 | | | | | | |

EPB

Measured correlation

COP of 3.55 in heating mode and 3.44 in DHW

=> Seasonal coverage factor of 76% and COP of 2.18 (heat pump + back-up resistance)



Conclusion

- ❑ Performance of a mini exhaust air heat pump integrated inside a low-energy residential building has been measured for part of 2017:
 - The COP of the machine in DHW production mode varies from 3.36 to 3.59
 - Due to the limited capacity of 1500 W, the backup resistance is frequently switched on in winter
 - Consequently, the efficiency of the whole system decreases

- ❑ An extrapolation of the results over one year gives an estimated coverage factor of 76 % and a seasonal COP of 2.18.



Future work

- ❑ Development of a model of the building and the machine to investigate different strategies to improve the annual coverage factor of the system:
 - ✓ Study the possibility for **increasing the heating capacity** of the machine, and study the impact on the frost formation and on the COP
 - ✓ Study the possibility for a **demand-controlled ventilation** strategy and study the impact on the machine heating capacity and on the COP
 - ✓ Study different control **strategies for the heating system**
 - ✓ Study the **impact of the occupant behavior**



Thank you for your attention!