

# Contributions to handle maximum size constraints in density-based topology optimization

*ECCOMAS Young Investigators Conference 2017*

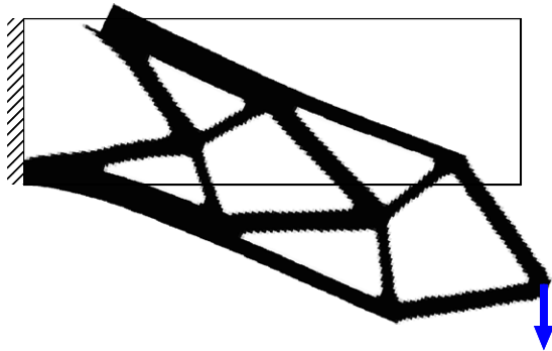
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Lemaire<sup>\$</sup>, Pierre Duysinx<sup>\*</sup>

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# Maximum size in the design process

## Indirect desired properties

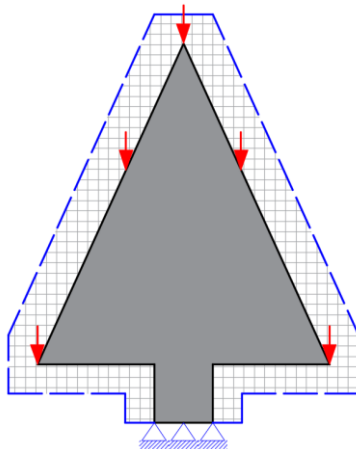


Optimized Cantilever beam  
under local damage scenarios ,  
**Jansen et al (2014).**

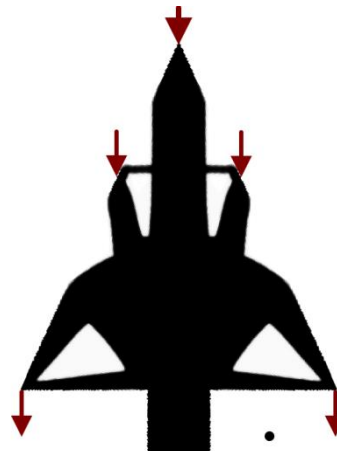


Cantilever beam **with  
maximum size and  
minimum gap** control.

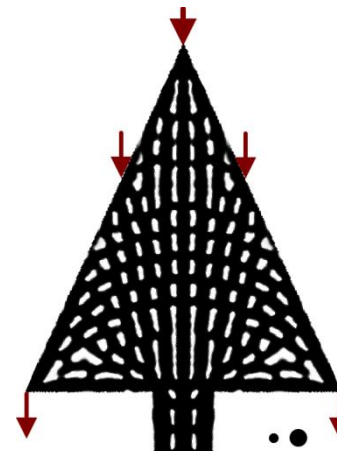
## Aesthetic reasons



Design domain

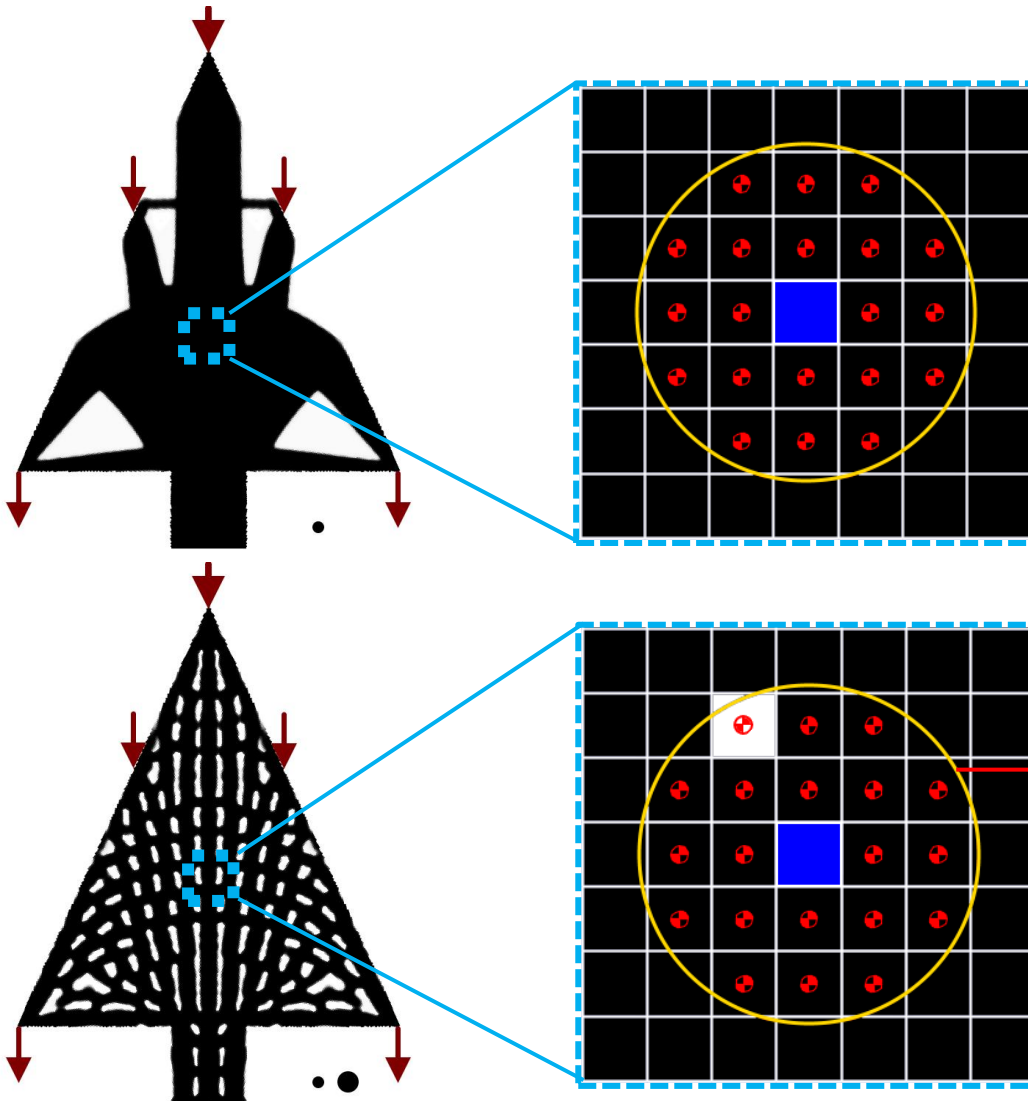


Minimum size



Maximum size

# Guest's approach (2009)



Constraint:

$$\text{Amount of voids} \geq \varepsilon$$

**Guest (2009)**

**Guest** JK, *Imposing maximum length scale in topology optimization*, *Struct Multidisc Optim*, 37, 463-473, **2009**.

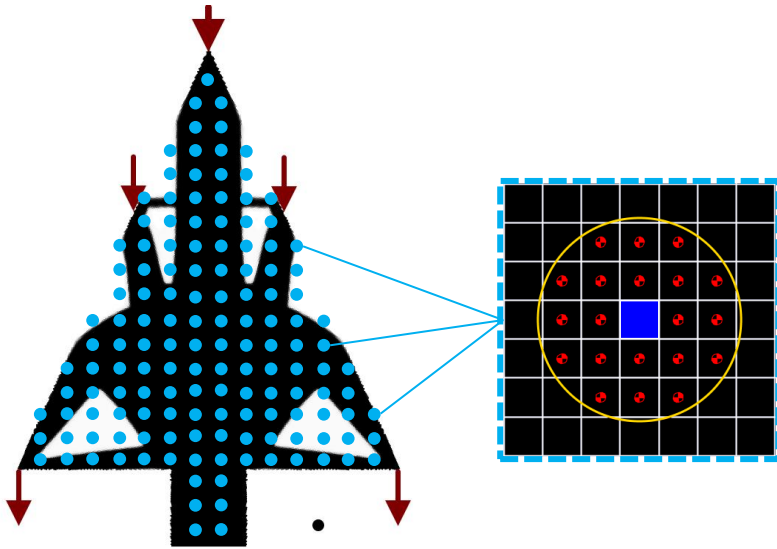
→  $\text{Amount of voids} = 4.8\%$

Condition satisfied if  $\varepsilon = 3\%$

# Maximum size formulation

## Local Constraint :

$$g_e = \varepsilon - \frac{\sum_{k \in \Omega_e} v_k (1 - \tilde{\rho}_k)^q}{V_{\Omega_e}} \leq 0$$



## Compliance minimization with maximum size constraints

$$\begin{aligned} \min \quad & f^\top u \\ \text{s.t. :} \quad & \mathbf{K}(\rho) u = f \\ & \sum_e v_e \tilde{\rho}_e \leq V^* \end{aligned}$$

$$\left. \begin{aligned} & g_1 \leq 0 \\ & g_2 \leq 0 \\ & g_3 \leq 0 \\ & \vdots \\ & g_N \leq 0 \end{aligned} \right\} \text{Max. size}$$
$$0 \leq \rho \leq 1$$

# Compliance minimization with maximum size constraints

Do not change during the optimization

$$\varepsilon - \frac{\sum_{k \in \Omega_e} v_k (1 - \tilde{\rho}_k)^q}{V_{\Omega_e}} \leq 0$$

**Matrix** with neighbors information

$$D_{II(e,j)} = \begin{cases} \frac{v_j}{V_{\Omega^e}} & \text{if } x_j \in \Omega^e \\ 0 & \text{if } x_j \notin \Omega^e \end{cases}$$

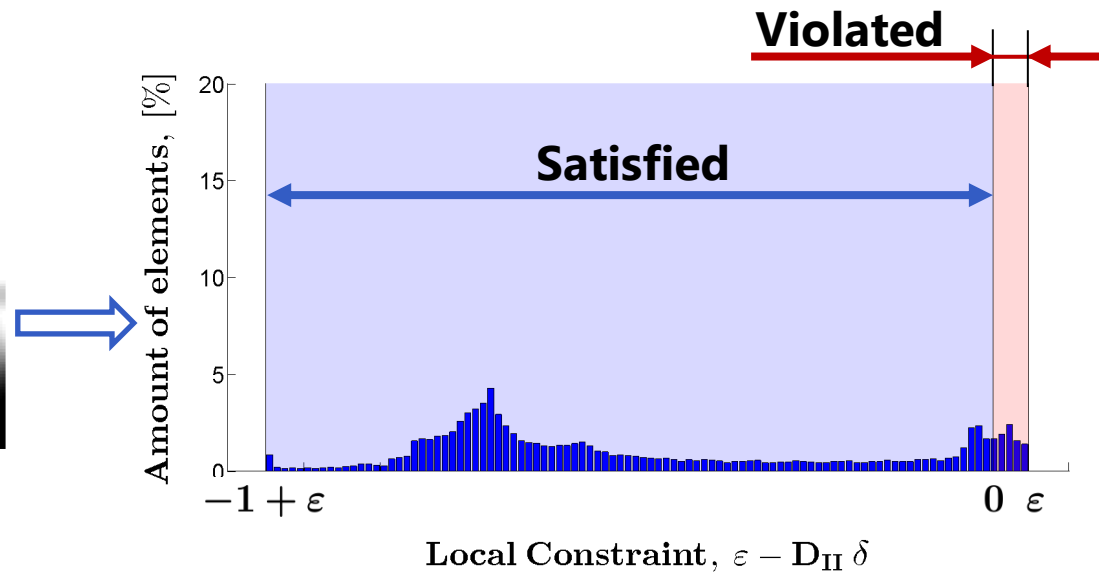
**Voids** vector with penalization

$$\delta_e = (1 - \tilde{\rho}_e)^q$$

$$\begin{aligned} \min \quad & f^\top u \\ \text{s.t. :} \quad & \mathbf{K}(\rho) u = f \\ & \sum_e v_e \tilde{\rho}_e \leq V^* \\ & \boxed{\varepsilon - \mathbf{D}_{II} \boldsymbol{\delta} \leq 0} \\ & 0 \leq \rho \leq 1 \end{aligned}$$

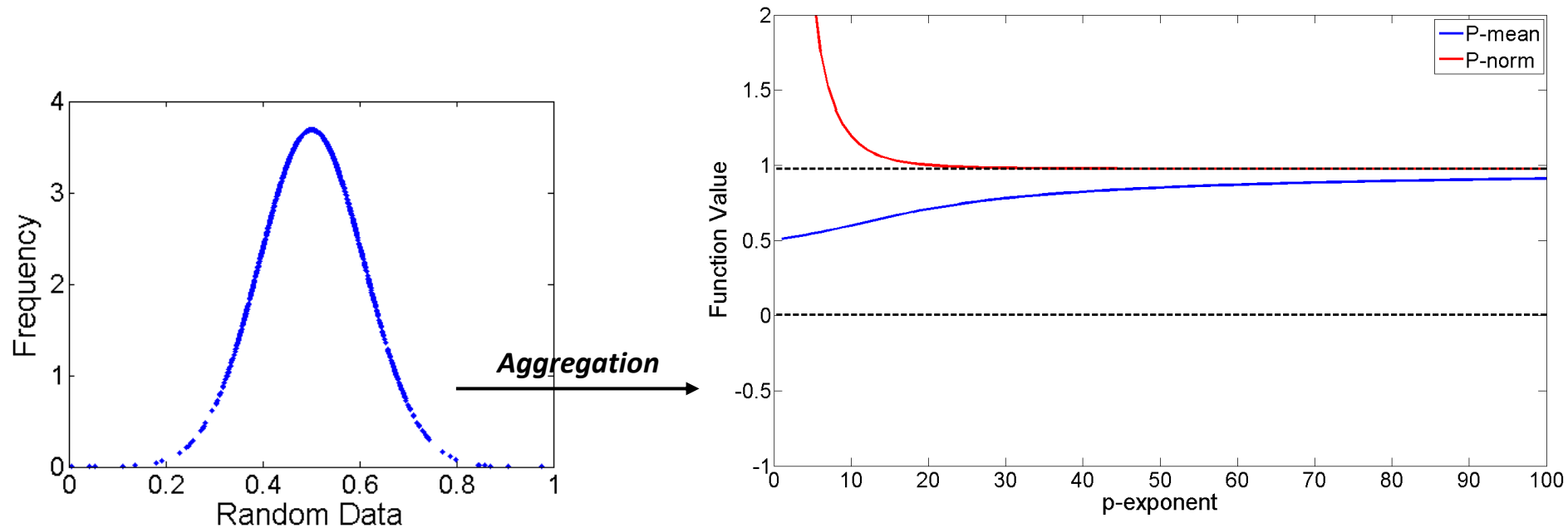
# Constraints distribution and aggregation

Iteration of the MBB beam with  
**maximum size** constraints



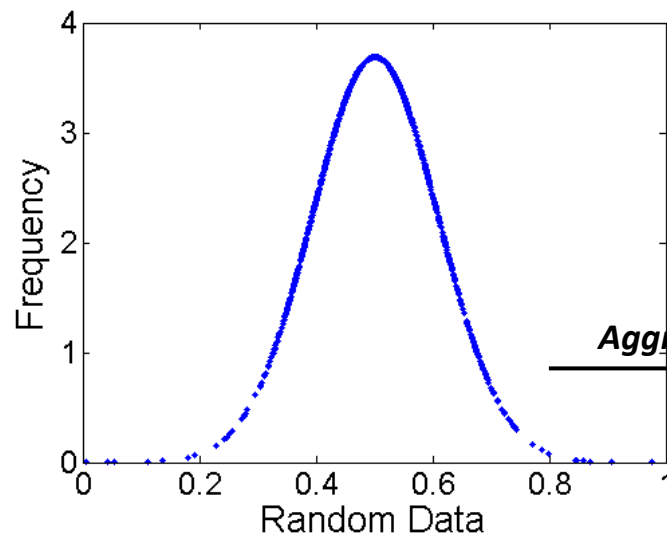
# Aggregation functions: p-mean & p-norm

$$P_m = \left( \frac{1}{N} \sum_{e=1}^N (s^e)^p \right)^{\frac{1}{p}} \quad P_n = \left( \sum_{e=1}^N (s^e)^p \right)^{\frac{1}{p}}$$

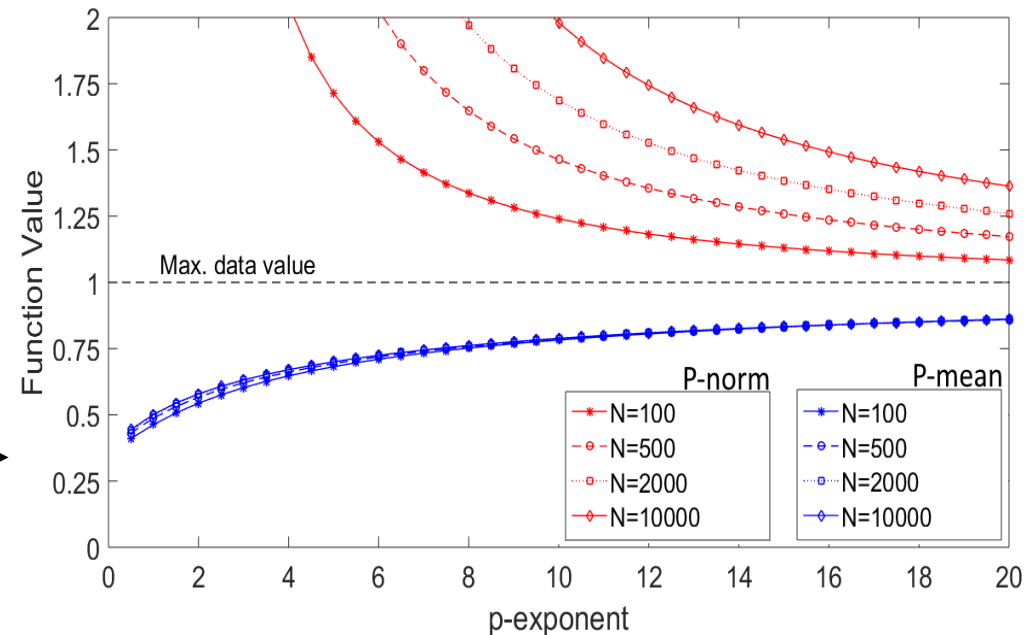


# Aggregation functions: p-mean & p-norm

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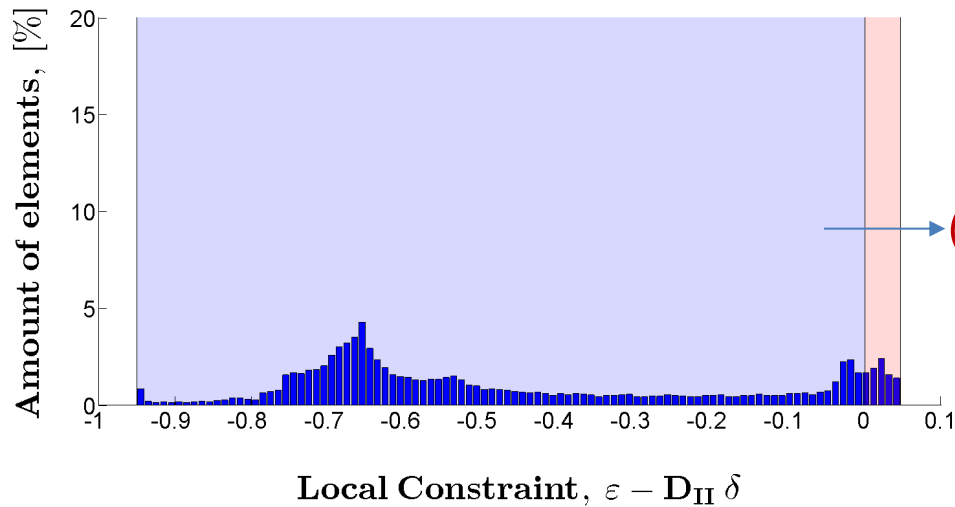


**Aggregation**



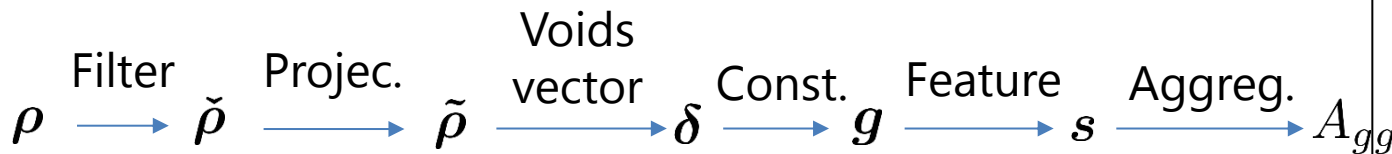


# Aggregation of maximum size constraints



$$s_i = \max(g_i, 0) \rightarrow P_n = \left( \sum_{e=1}^N (s^e)^p \right)^{\frac{1}{p}}$$

$$\begin{aligned} \min \quad & f^\top u \\ \text{s.t. :} \quad & \mathbf{K}(\rho) u = f \\ & \sum_e v_e \tilde{\rho}_e \leq V^* \\ & G_{ms} \leq 0 \\ & 0 \leq \rho \leq 1 \end{aligned}$$

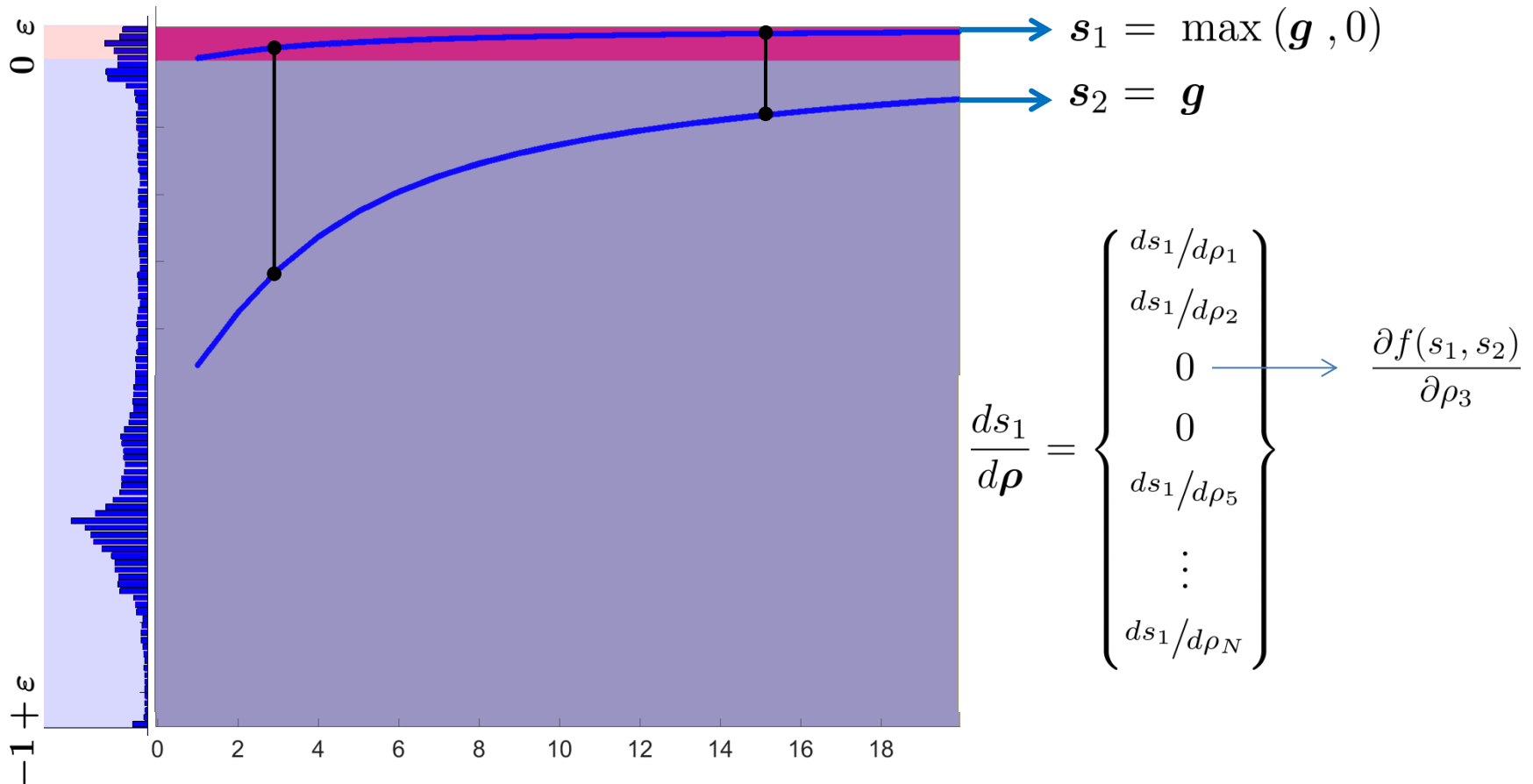


$$\frac{dG_{ms}}{d\rho} = \frac{d\tilde{\rho}}{d\rho} \frac{d\tilde{\rho}}{d\tilde{\rho}} \frac{d\delta}{d\tilde{\rho}} \frac{dg}{d\delta} \left( \frac{ds}{dg} \right) \frac{dA_{gg}}{ds} \frac{dG_{ms}}{dA_{gg}}$$

# Aggregation of maximum size constraints

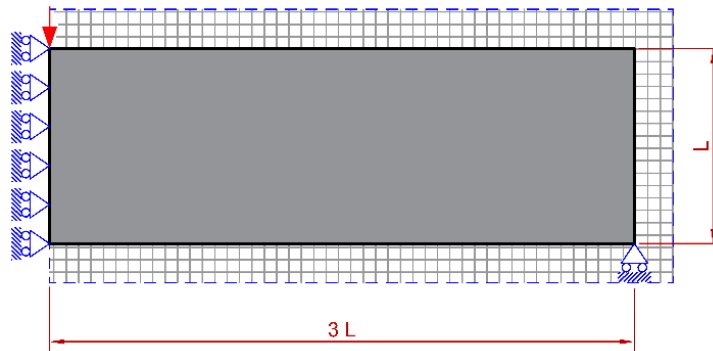
## Modification of the sensitivities to improve convergence

P-mean curves

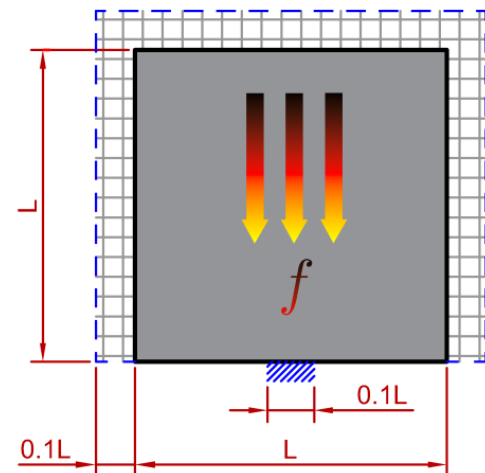


# Addressed problems in linear elasticity

## Compliance Minimization

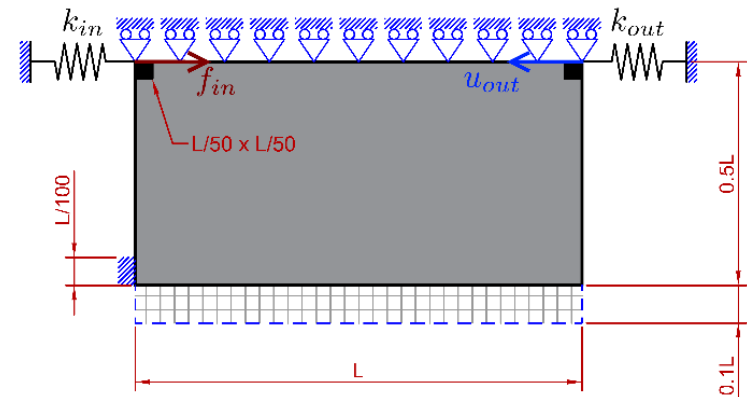


## Heat conduction problem

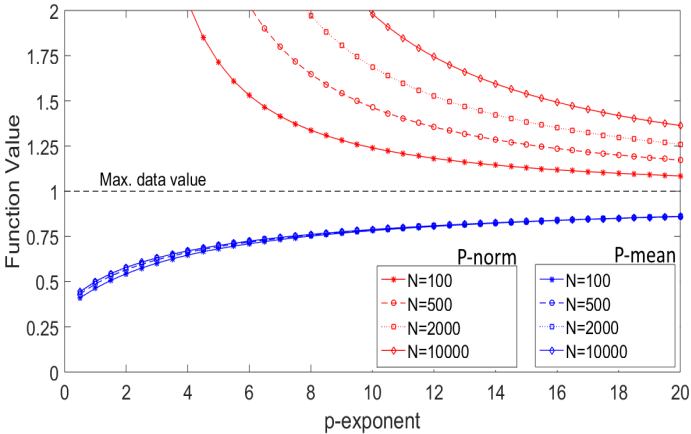


$$\begin{aligned} \min_{\rho} \quad & t^T u \\ \text{s.t. :} \quad & \mathbf{K}(\rho) u = f \\ & \sum_i v_i \bar{\rho}_i \leq V^* \\ & G_{ms} \leq 0 \\ & 0 \leq \rho \leq 1 \end{aligned}$$

## Compliant Mechanism



# MBB-beam with different mesh sizes



P-norm

P-mean

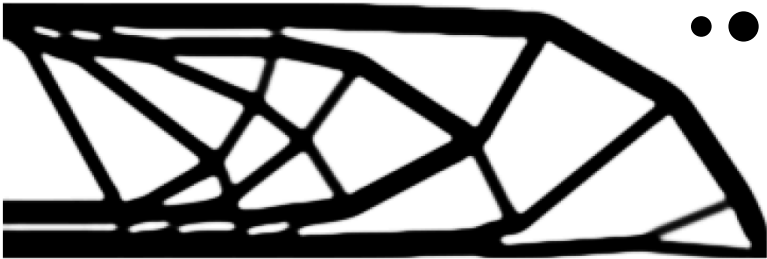


$C = 57.7$

6144  
Elements

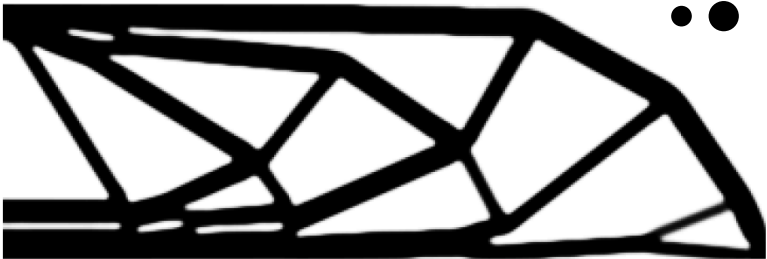


$C = 56.4$

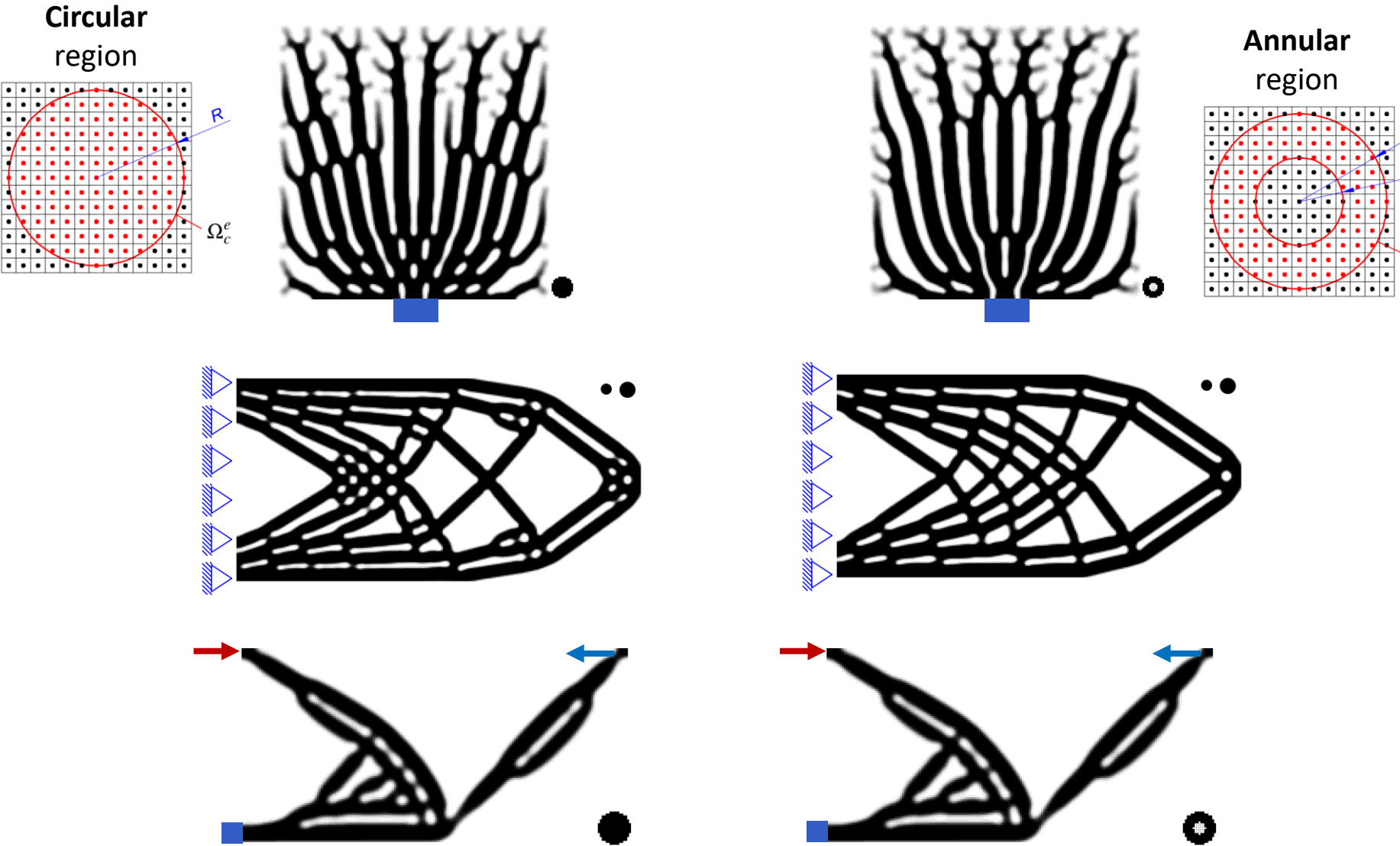


$C = 57.6$

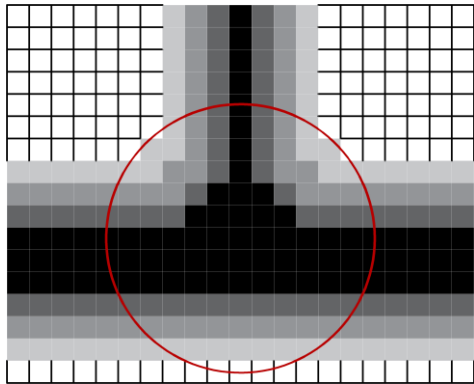
55296  
Elements



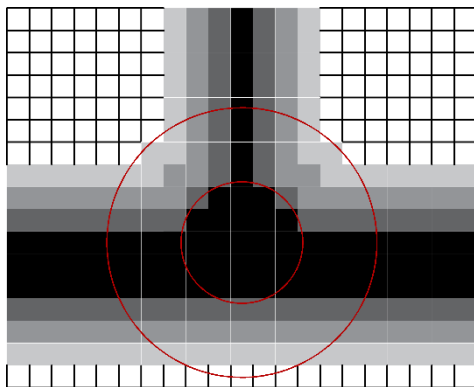
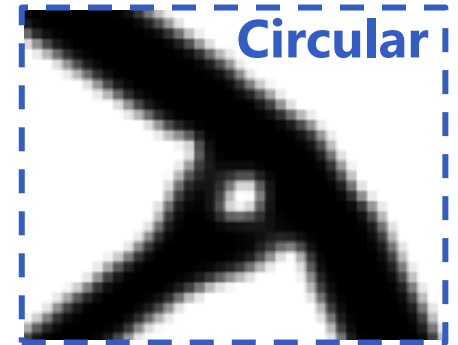
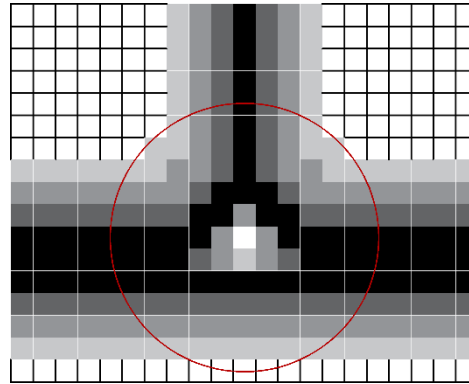
# New test region



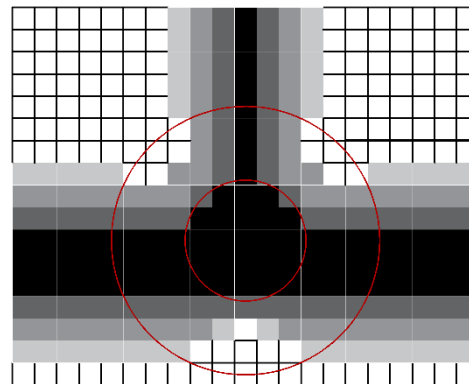
# New test region



**Circular**



**Annular**



# Compliance minimization with maximum size

$$\begin{aligned} \min \quad & \mathbf{f}^\top \mathbf{u} \\ \text{s.t. : } \quad & \mathbf{K}(\rho) \mathbf{u} = \mathbf{f} \\ & \sum_e v_e \tilde{\rho}_e \leq V^* \\ & G_{ms} \leq 0 \\ & 0 \leq \rho \leq 1 \end{aligned}$$

SIMP

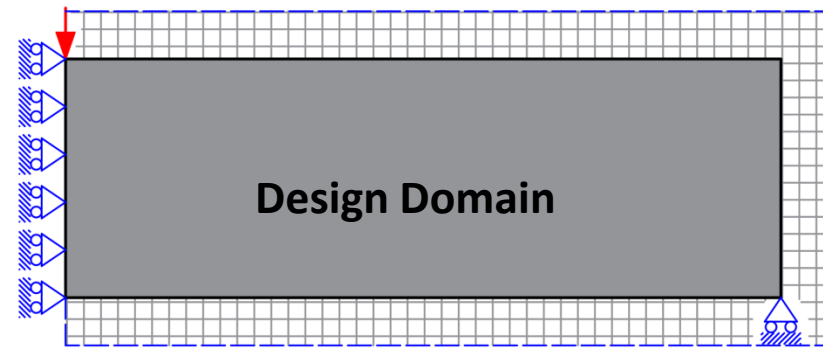
$$\mathbf{K}_e = (E_{min} + \tilde{\rho}_e^\eta (E_0 - E_{min})) \mathbf{K}_0$$

Continuation procedure :

$$\eta = 1 : 0.25 : 3.5$$

$$\beta = 2^\eta - 1$$

$$iter = 20_{iter} \times 11_{steps} = 220$$

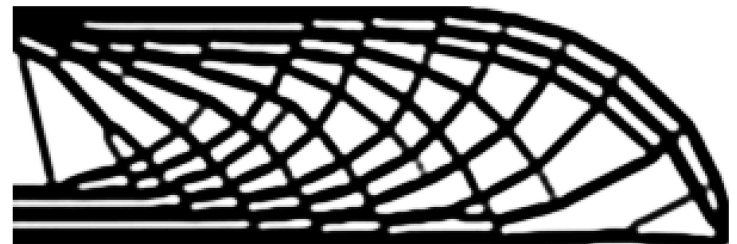


Minimum Size



Time/Iter = Ref.

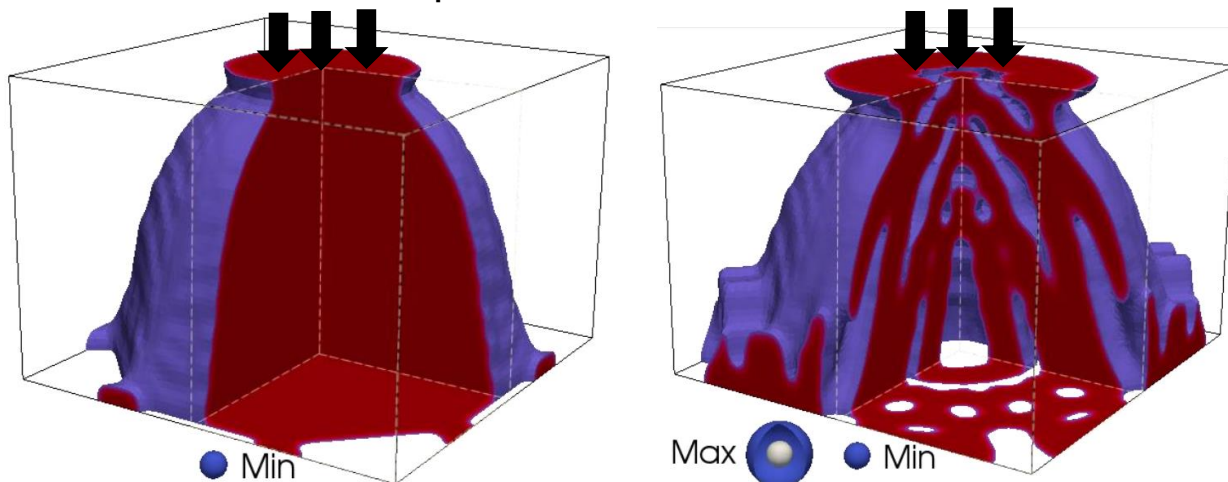
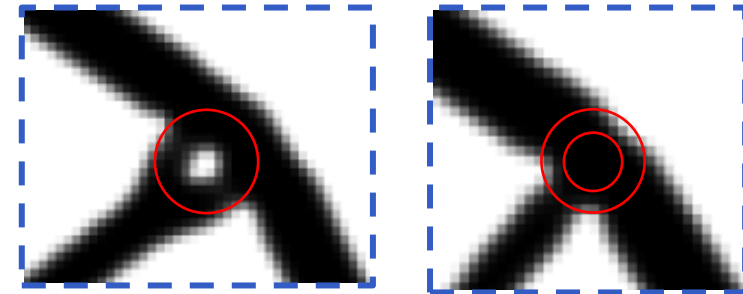
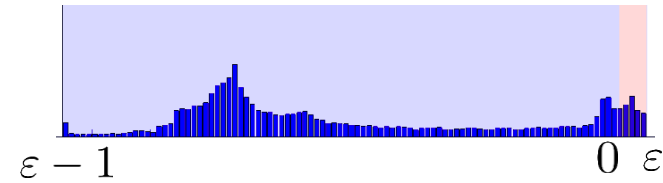
Maximum size



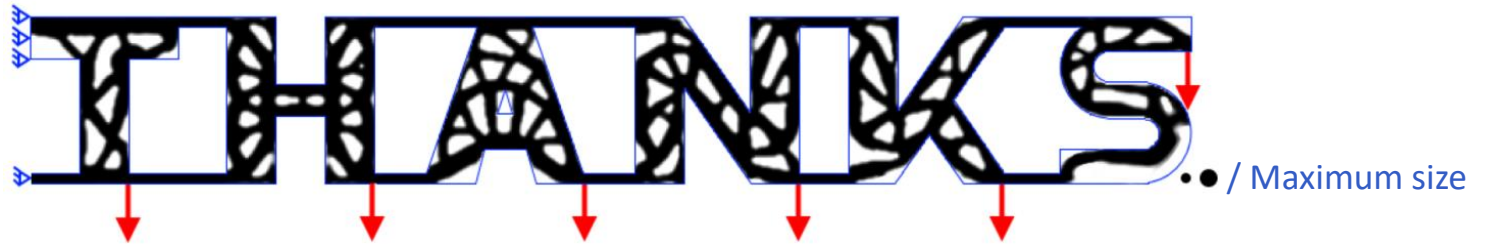
Time/Iter = **+12%**

# Concluding remarks

- Maximum size constraints where aggregated using **p-mean** and **p-norm** functions.
- With p-norm is not possible to get mesh-independent solutions.
- The **ring-shaped** region reduces the amount of holes in the design
- The aggregation reduces the computation time compared to a highly constrained problem and allows to solve 3D problems in a reasonable time.







### Acknowledgement

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