



A.M. Habraken, L. Duchêne, C. Bouffioux, C. Canalès



# Characterization of Fatigue Behaviour, from Material Science to Civil Engineering Applications

OptiBri-Workshop

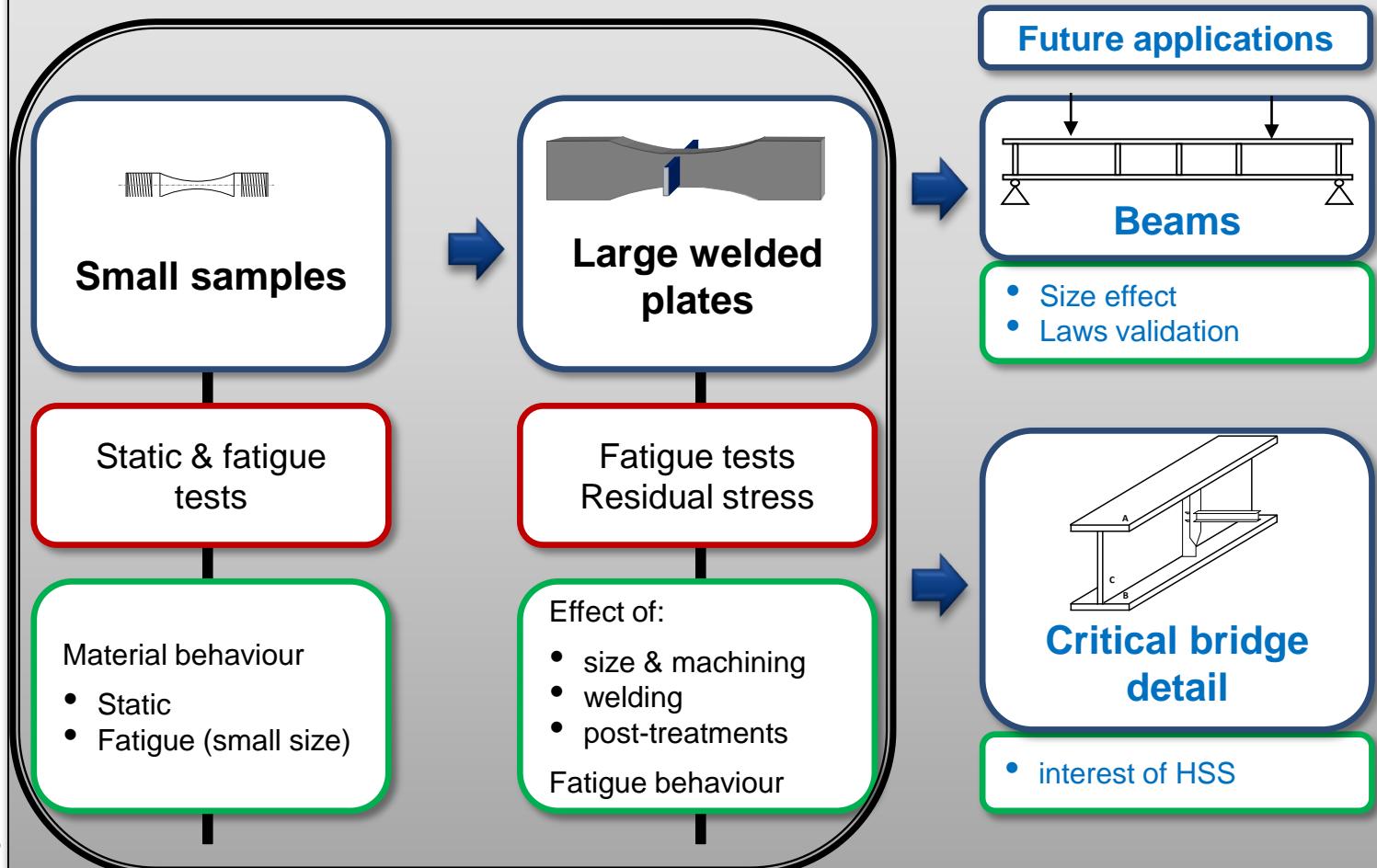
"Design Guidelines for Optimal Use of  
HSS in Bridges"

3<sup>rd</sup> May 2017

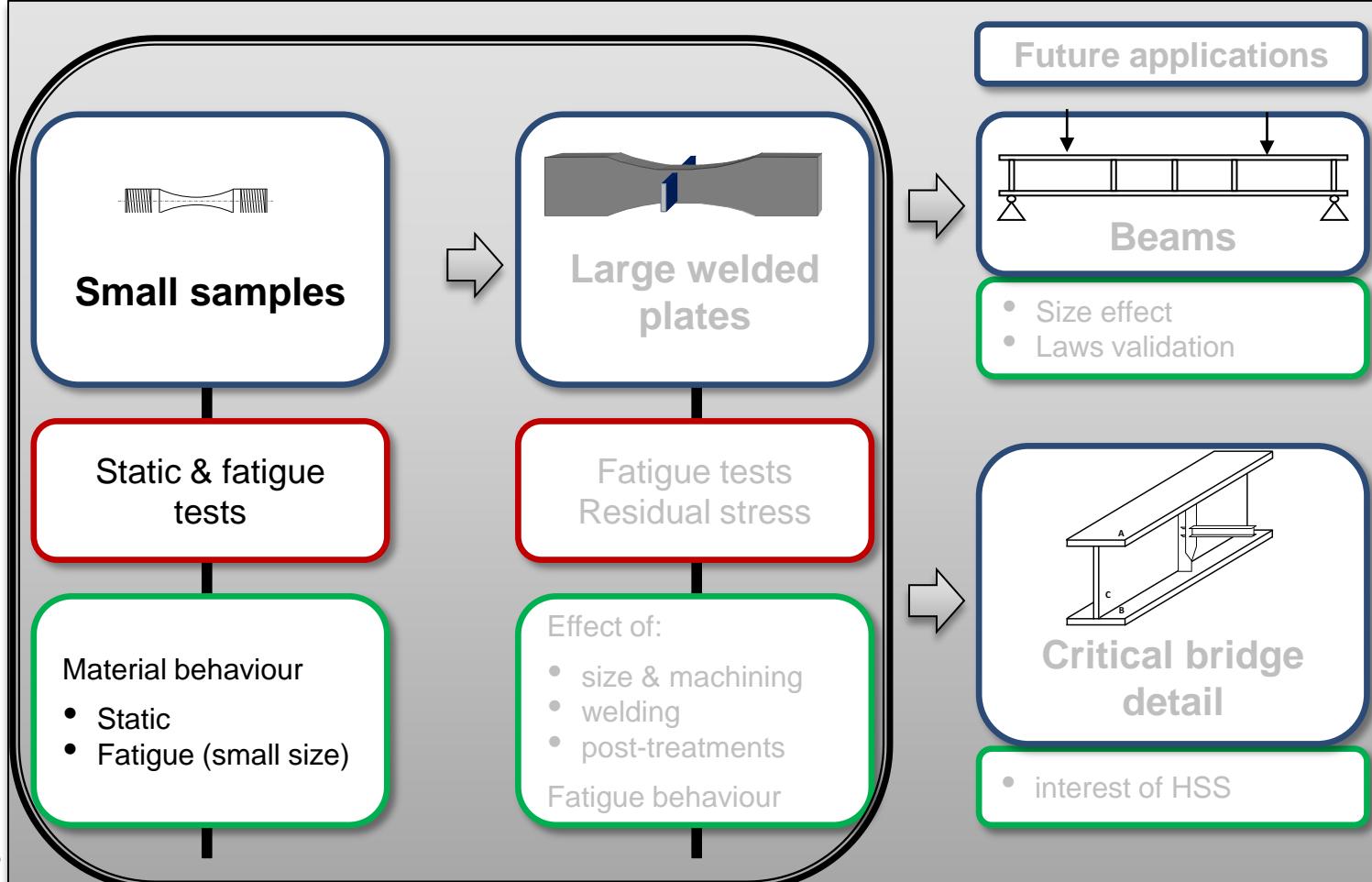
Chantal  
Bouffioux



# Outline



# Outline



## Small samples

- 4 materials:

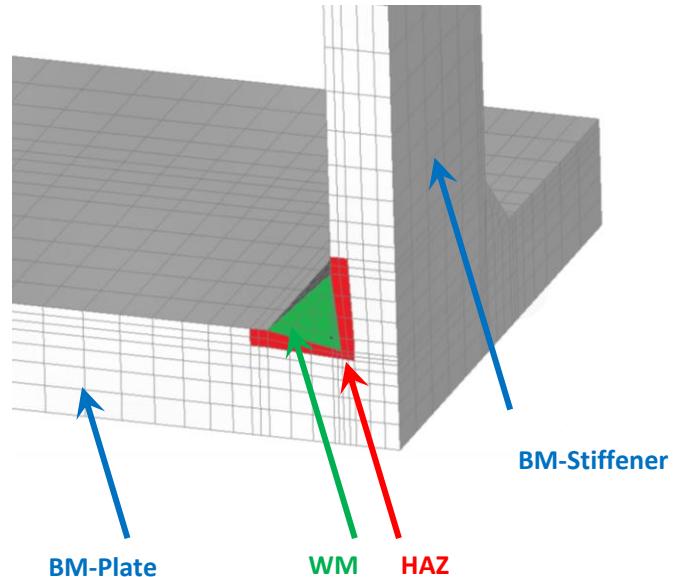
- Base Material: **BM** (HSS - S690QL)

- Heat Affected Zone:

- **HAZ1** (25 mm thick)

- **HAZ2** (40 mm thick)

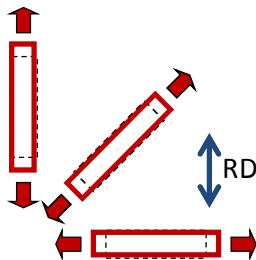
- Welded Metal: **WM**



## Small samples

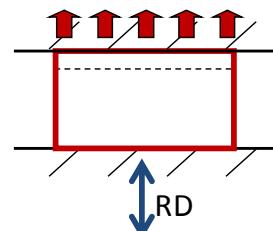
- Static tests:

Tensile test



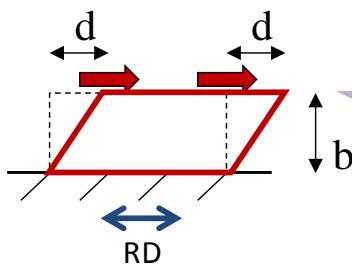
or

Large tensile test



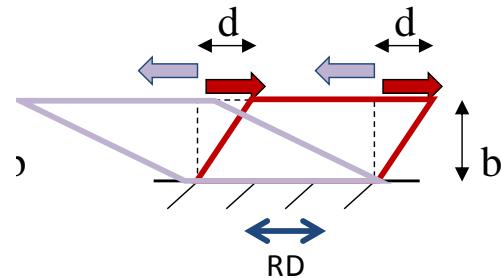
+

Shear test



+

Bauschinger shear test



## Small samples

- Static behavior – material laws & parameters :

Elastic part: Hooke's law:  $E \nu$

Plastic part: Hill's law (Hill48):

$$F_{HILL}(\sigma) = \frac{1}{2} [H(\sigma_{xx} - \sigma_{yy})^2 + G(\sigma_{xx} - \sigma_{zz})^2 + F(\sigma_{yy} - \sigma_{zz})^2 + 2N(\sigma_{xy}^2 + \sigma_{xz}^2 + \sigma_{yz}^2)] - \sigma_F^2 = 0$$

Isotropic hardening: Voce formulation:

$$\sigma_F = \sigma_0 + K(1 - \exp(-n \cdot \varepsilon^{pl}))$$

Back-stress (kinematic hardening): Armstrong-Frederick's equation:

$$\dot{\underline{X}} = C_X (X_{sat} \underline{\dot{\varepsilon}}^{pl} - \bar{\dot{\varepsilon}}^{pl} \cdot \underline{X})$$

**E &  $\nu$ :** defined by tensile tests

**F, G, H:** defined by tensile tests in 3 directions (RD, TD, 45°)

**N,  $\sigma_0$ , K, n,  $C_x$ ,  $X_{sat}$ :** defined by Optim

## Small samples

- Static behavior – material data (inverse method):

- BM:** hardening fully kinematic
- HAZ1 ≈ HAZ2:** same static behaviour
- WM**

For fatigue tests:

$$\sigma_{u,eng} = F_i / A_0$$

For FEM:

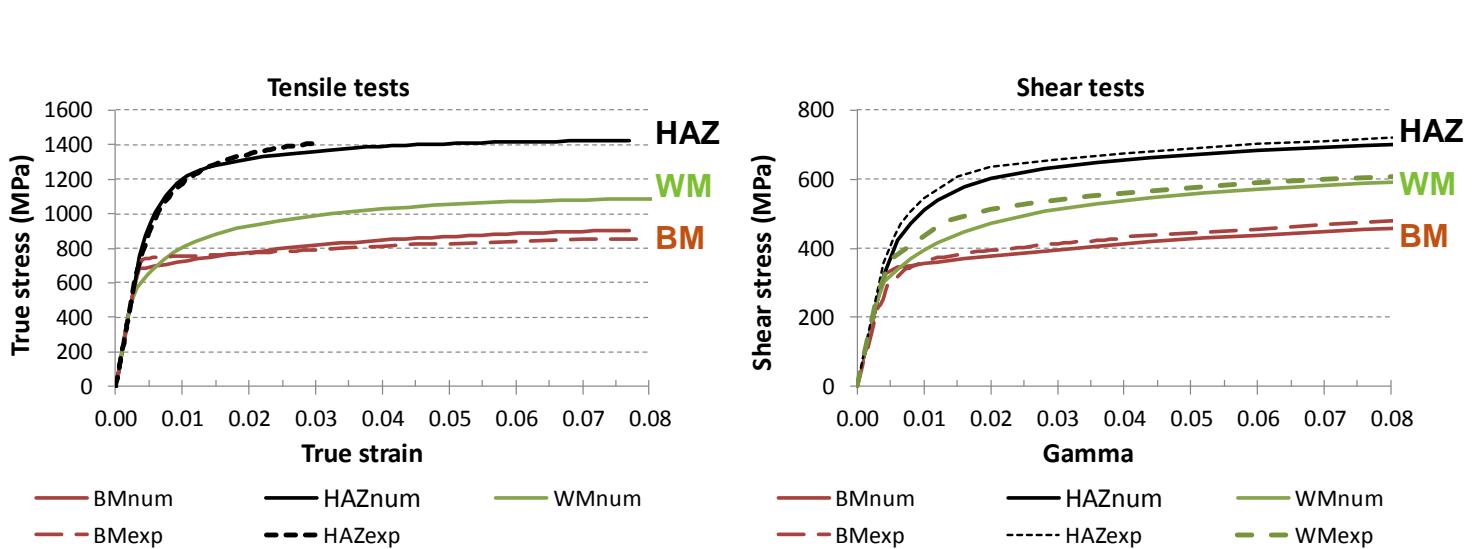
$$\sigma_{u,true} = F_i / A_i$$

		Ultimate tensile strength (Mpa)	
		$\sigma_{u,eng}$	$\sigma_{u,true}$
<b>BM (S690QL)</b>		838	905
<b>HAZ1, HAZ2</b>		1338	1424
<b>WM</b>		1008	1101

Data for Hooke, Hill, Voce and Armstrong-Frederick laws (units: MPa, s)											
Material	Elast. data			Yield locus				Isotropic hardening			Kinematic hardening
	E	v	F	G	H	N=L=M	K	$\sigma_0$	n	$C_x$	$X_{sat}$
<b>BM (S690QL)</b>	210	116	0.3	1	1	1	3.9	0	674	0	31.9
<b>HAZ1, HAZ2</b>	210	000	0.3	1	1	1	4.45	371	827	511	52.5
<b>WM</b>	210	000	0.3	1	1	1	3.2	241	531	285	42.6
											218

## Small samples

- Static behavior – comparison of material behavior:



## Small samples

- **Fatigue tests:**

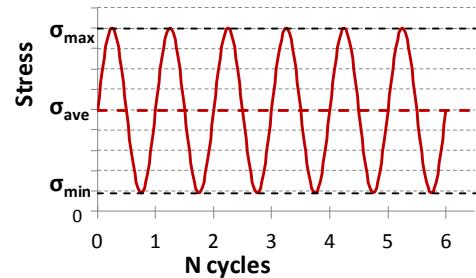
- On vibrophore
- Axial loading
- Frequency: 100 – 150 Hz  
(→ correction factor)

Vibrophore



$$R = \sigma_{\min} / \sigma_{\max} = 0.1 \text{ or } 0.2 \text{ or } \dots$$

Material	Smooth	Notch
BM (S690QL)	4 R	3 geom.
HAZ1	1 R	1 geom.
HAZ2	2 R	1 geom.
WM	2 R	1 geom.



## Small samples

- Fatigue behavior – material laws & parameters :

### Multiaxial Lemaître Chaboche fatigue model

$$\frac{\partial D}{\partial N} = \begin{cases} = 0 & \text{if } f_D < 0 \\ = [1 - (1 - D)^{\beta+1}]^{\alpha} \left(\frac{\bar{A}_{II}}{M}\right)^{\beta} & \text{if } f_D \geq 0 \end{cases}$$

$$f_D = A_{II} - A_{II}^*$$

$$A_{II} = \frac{1}{2} \sqrt{\frac{3}{2} (\hat{\sigma}_{ijmax} - \hat{\sigma}_{ijmin})(\hat{\sigma}_{ijmax} - \hat{\sigma}_{ijmin})} \quad \text{with } \hat{\sigma}_{ij} = \sigma_{ij} - \sum_k \frac{1}{3} \sigma_{kk}$$

$$A_{II}^* = (\sigma_{10} (1 - 3 \cdot b \cdot \sigma_{Hm})) \quad (\text{Sines' criterion})$$

$$\tilde{A}_{II} = \frac{A_{II}}{1 - D}$$

$$M = M_0 (1 - 3 \cdot b \cdot \sigma_{Hm})$$

$$\alpha = 1 - a \left( \frac{A_{II} - A_{II}^*}{\sigma_u - \sigma_{eqmax}} \right)$$

$$\sigma_{Hm} = \frac{1}{3} \left[ \frac{1}{T} \int_T \text{Tr} (\underline{\underline{\sigma}}(t)) dt \right]$$

D: damage val., 0: sound material, 1: rupture

N: number of cycle

$A_{II}$ : 2<sup>nd</sup> invar. of amplit. deviator of  $\sigma$  tensor

$A_{II}^*$ : fatigue limit

$f_D$ : damage yield locus

$\sigma_{Hm}$ : mean hydrostatic stress

$\sigma_{eqmax}$ : maximum Von Mises stress per cycle

$\langle x \rangle$ : = x if  $x > 0$  else = 0

b =  $1/\sigma_u$

$\sigma_u$ : ultimate tensile stress

$\sigma_{10}$ : endurance limit = fatigue limit at null  $\sigma_{mean}$

a, M0, β: other material data to define

## Small samples

- Fatigue behavior – material laws & parameters :  
Volume averaged stress gradient method

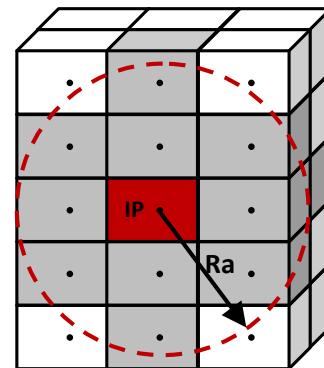
For each element, variables  $\chi_{ip}$ : replaced by an average value of all the elements with their integration point inside the circle with a radius  $R_a$

$$\overline{\chi_{ip}} = \frac{1}{V} \cdot \sum_{i=1}^{N_{elem}} \chi_{ip,i} \cdot V_i$$

$$\chi_{ip} = \{A_{II}, \sigma_{eqmax}, \sigma_{Hm}\}$$

$$V = \sum_{i=1}^{N_{elem}} V_i$$

$A_{II}$ :	2 <sup>nd</sup> invar. of amplit. deviator of $\sigma$ tensor
$\sigma_{eqmax}$ :	maximum Von Mises stress per cycle
$\sigma_{Hm}$ :	mean hydrostatic stress
$R_a$ :	material data to define



## Small samples

- Fatigue behavior – material data (inverse modelling):

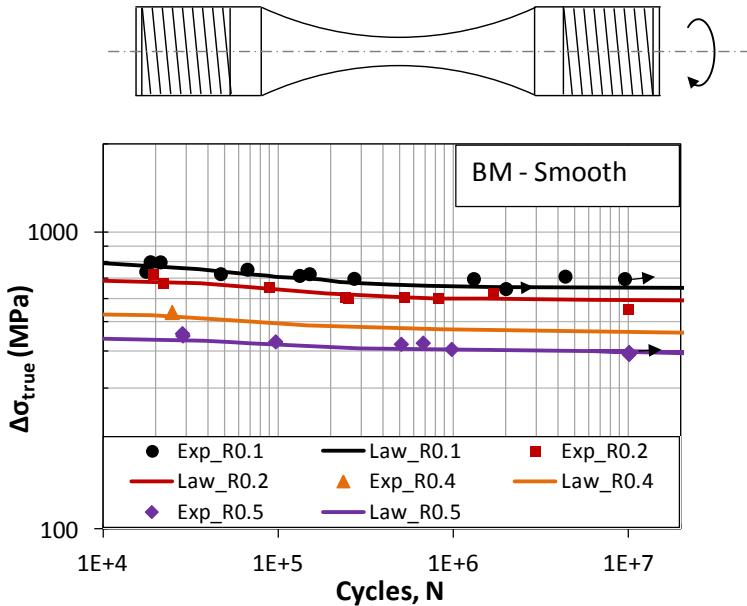
**HAZ1 ≈ HAZ2**: same fatigue behavior

Material	$\sigma_u$ (Mpa)	$\sigma_{l0}$ (Mpa)	b	$\beta$	a	$M_0$	Ra (mm)	$a^*(M_0^{-\beta})$
BM	905.0	580.0	1.10 E-03	0.17	1	5.385 E+30	0.06	5.966E-06
HAZ1, HAZ2	1424.0	428.4	7.02E-04	2.094	1	4.410E+05	0.00	1.516E-12
WM	1101.0	319.4	9.08E-04	0.161	1	7.245E+32	0.00	5.182E-06

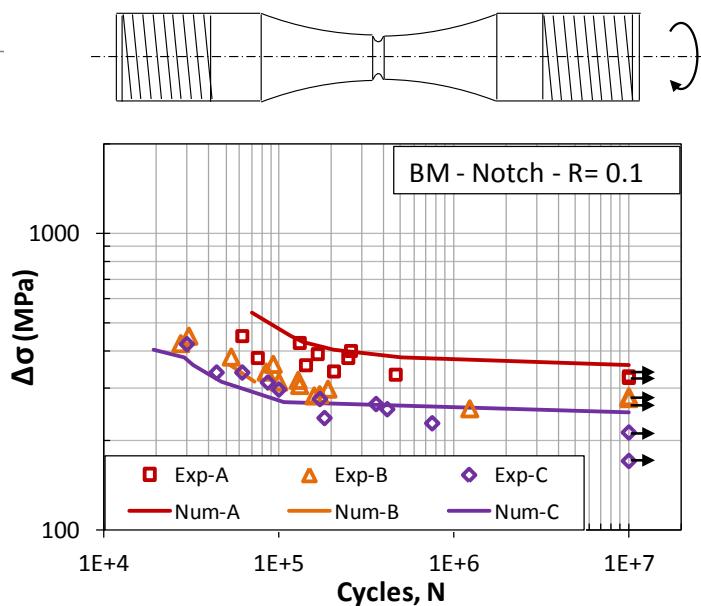
## Small samples

- Fatigue behavior – comparison experiments & fatigue law:  
Base material (BM)

**Smooth samples, 4 R**



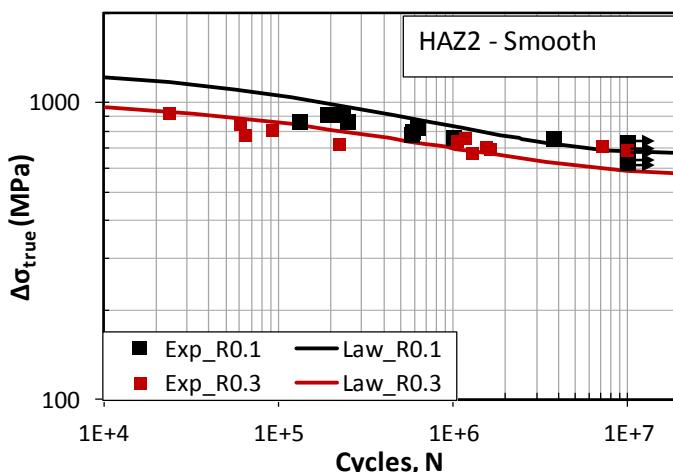
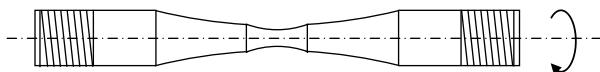
**Notched samples, 3 geom.**



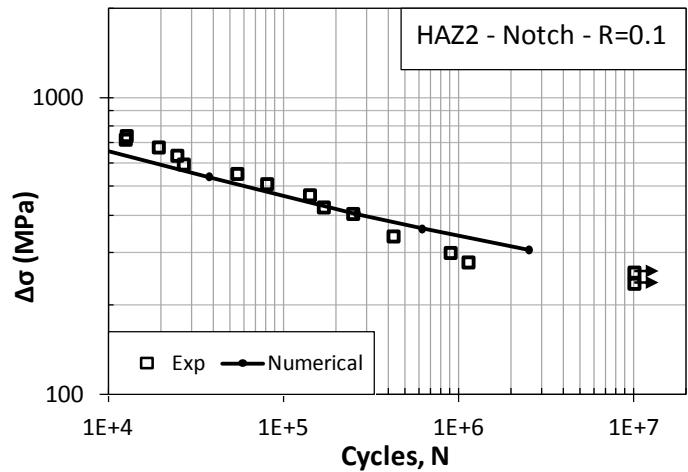
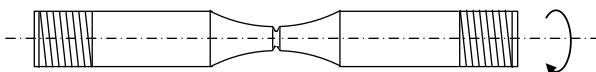
## Small samples

- Fatigue behavior – comparison experiments & fatigue law:  
**Heat affected zone (HAZ) with HAZ1 ≈ HAZ2**

Smooth samples, 2 R



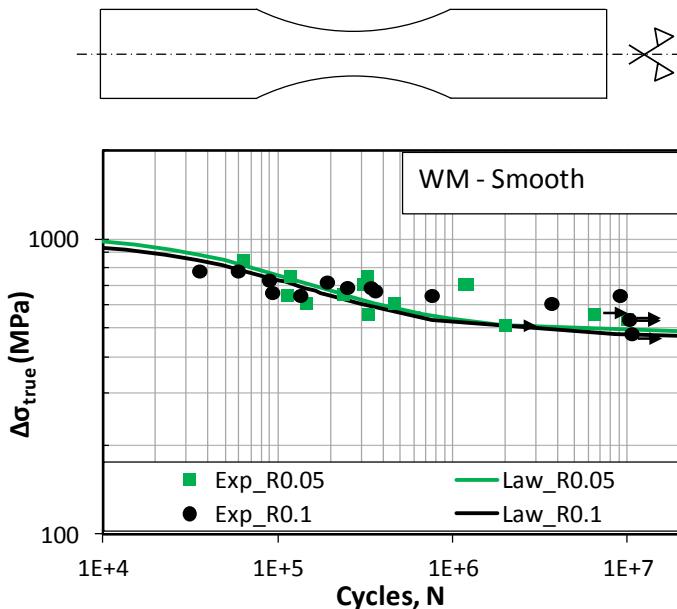
Notched samples, 1 geom.



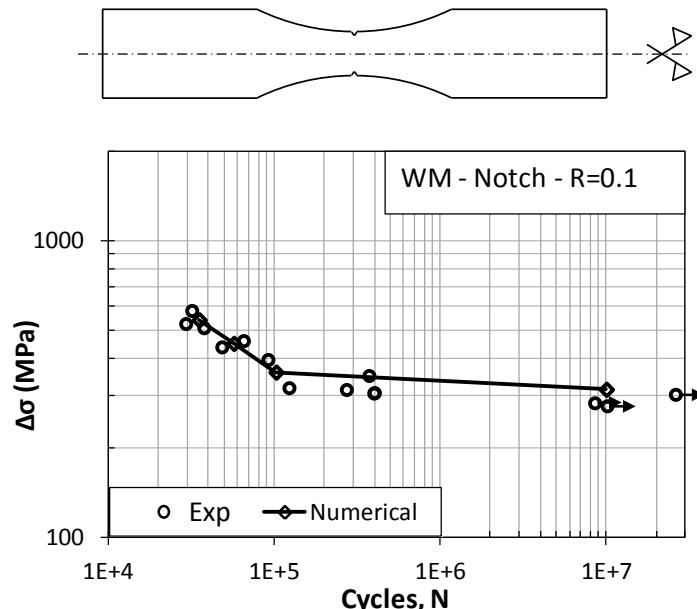
## Small samples

- Fatigue behavior – comparison experiments & fatigue law:  
Weld metal (WM)

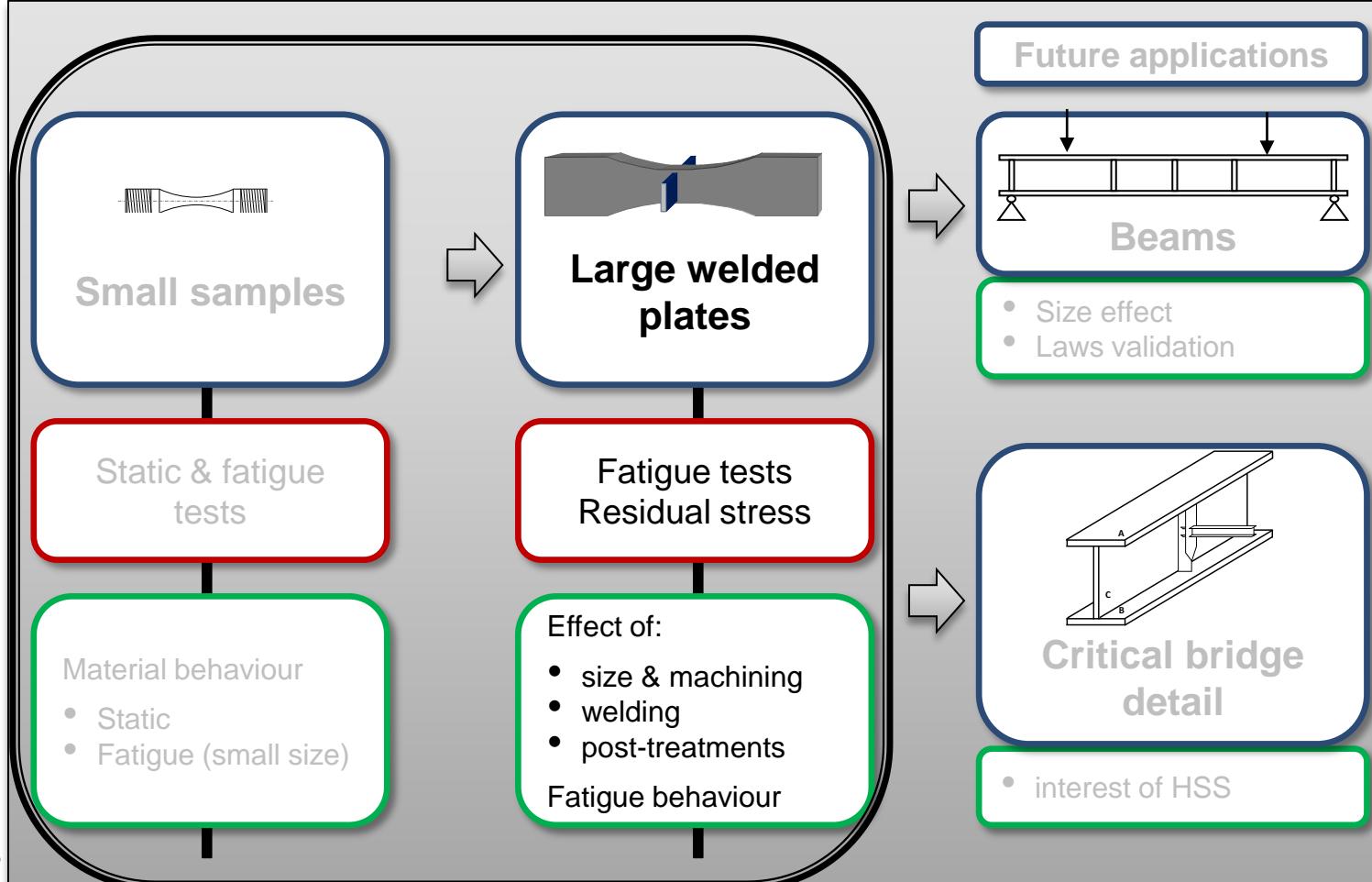
**Smooth samples, 2 R**



**Notched samples, 1 geom.**

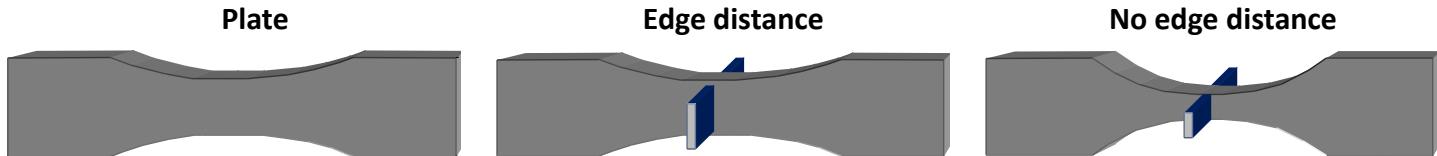


# Outline



## Large welded plates

- Fatigue tests on plates (length= 1070 mm):



Case	Post-treatment	plate thickness (mm)	Stiffener thickness (mm)	Stiffener length (mm)	distance to edge	stress ratio R
Plate	No weld	25	-	-	-	0.1
A (ref case)	PIT	25	15	60	✓	0.1, 0.3, 0.5
B	PIT	15	15	60	✓	0.1
E	PIT	25	15	60	no	0.1
H	PIT	40	15	60	no	0.1
C	TIG remelting	15	15	60	✓	0.1
D	TIG remelting	25	15	60	✓	0.1
F	TIG remelting	25	15	40	✓	0.1
G	TIG remelting	15	6	60	✓	0.1
I	No post-treatment	15	15	60	✓	0.1

## Large welded plates

- Fatigue behavior – material data (inverse modelling):

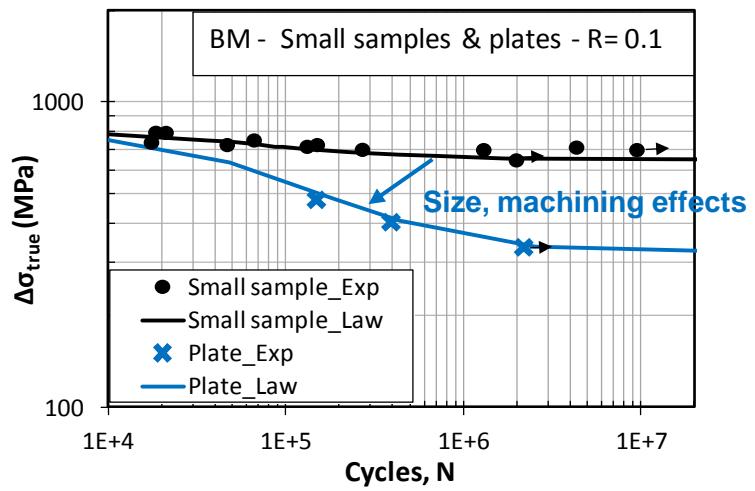
Material	$\sigma_u$ (Mpa)	$\sigma_{I0}$ (Mpa)	b	$\beta$	a	M0	Ra (mm)	$a^*(M0^{-\beta})$
BM-SS	905.0	580.0	1.10 E-03	0.17	1	5.385 E+30	0.06	5.966E-06
BM-plate	905.0	203.0	1.10 E-03	0.17	1	5.385 E+30	0.06	5.966E-06

$\sigma_{I0}$  = endurance limit = fatigue limit at null  $\sigma_{mean}$

Small samples, Ra= 0.8  $\mu$ m,  
length: 96 mm

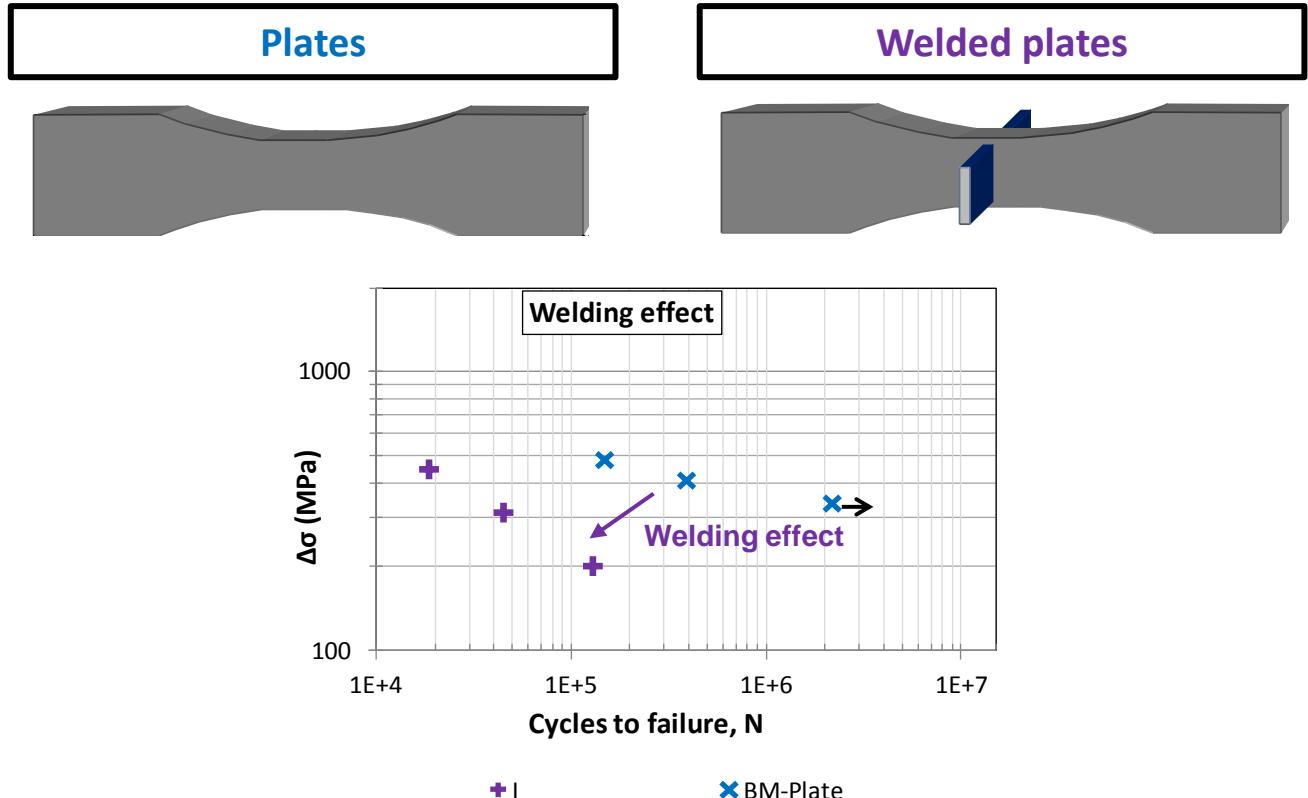


Plates, as produced,  
length: 1070 mm



## Large welded plates

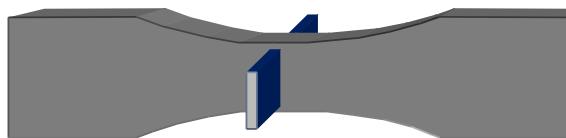
- Fatigue tests: welding effect



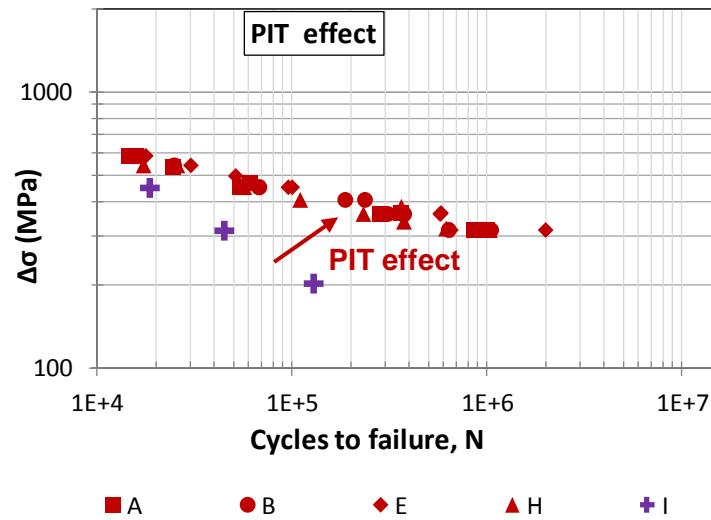
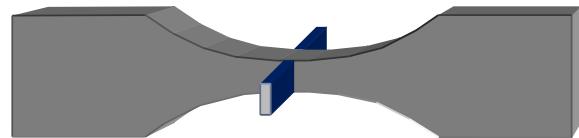
## Large welded plates

- Fatigue tests: PIT post-treatment effect

**Welded plates**



**Welded plates + PIT, ≠ geom.**

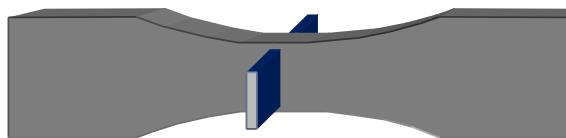


PIT: 4 cases  
low effect of  
geometry

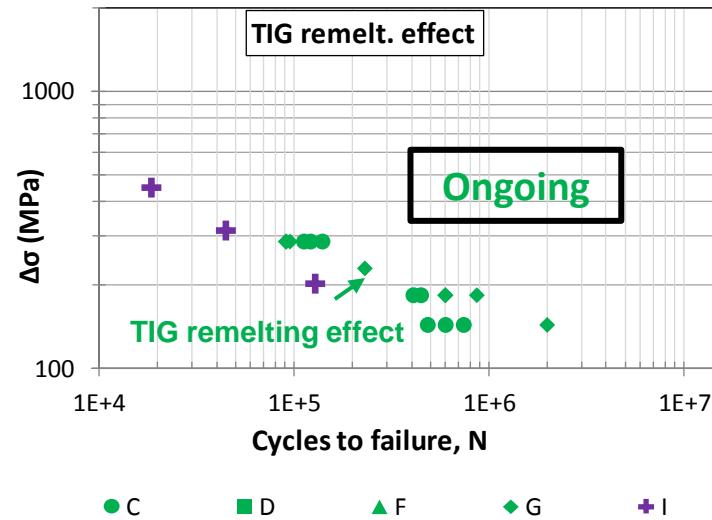
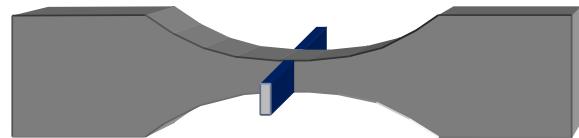
## Large welded plates

- Fatigue tests: TIG remelting post-treatment effect

**Welded plates**



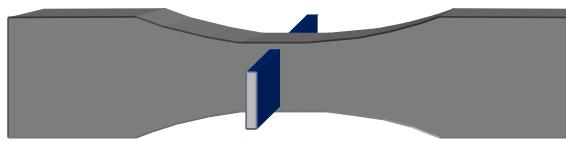
**Weld. plates + TIG rem. , ≠ geom.**



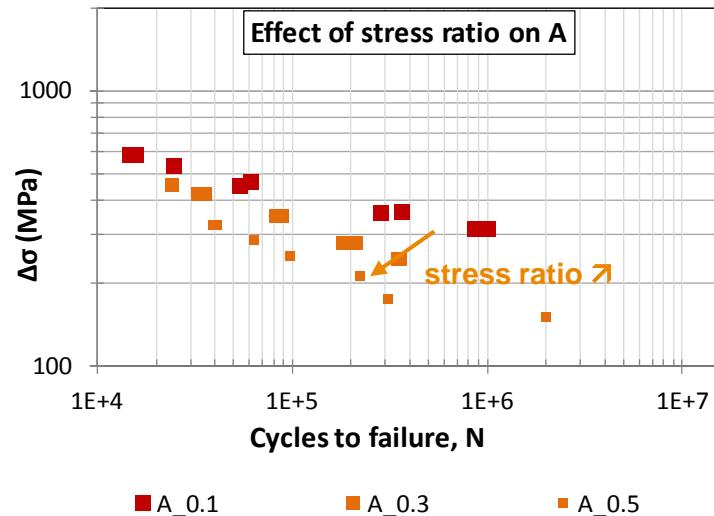
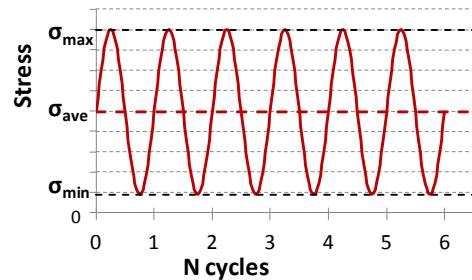
## Large welded plates

- Fatigue tests: Stress ratio effect

**Welded plates + PIT**

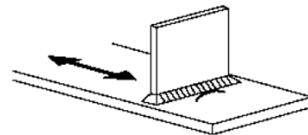


$$R = \sigma_{\min} / \sigma_{\max} = 0.1 \text{ or } 0.3 \text{ or } 0.5$$



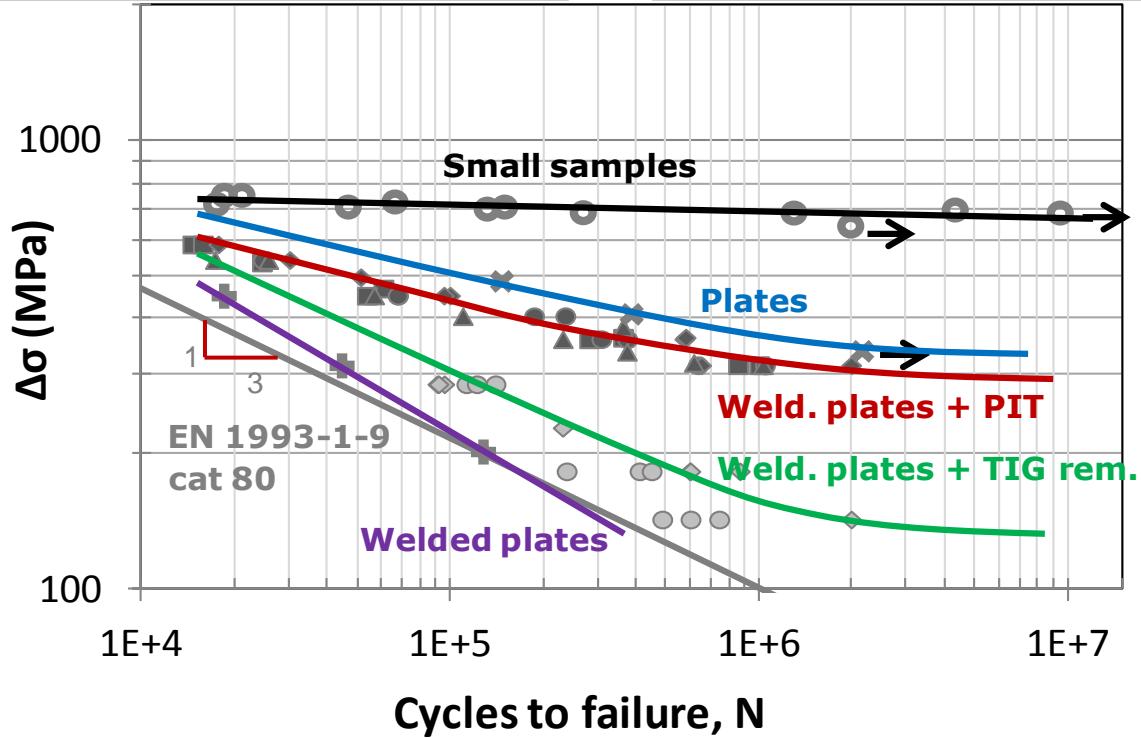
## Large welded plates

- Fatigue tests



Summary, all cases, R= 0.1

EN 1993-1-9,  $\Delta\sigma_c = 80$  MPa



# Large welded plates

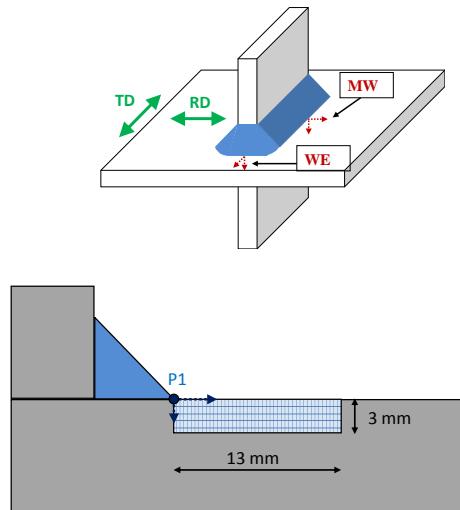
- Residual stress measurement

Several cases:

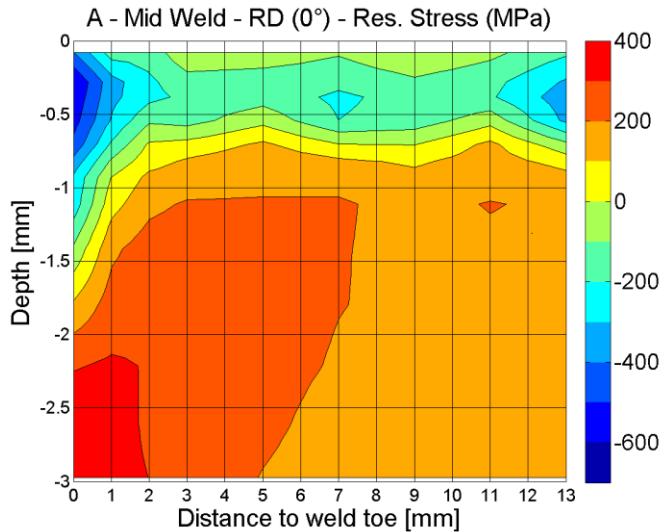
- 3 geometries
- 3 cases: PIT, TIG remelting, no post-treat.
- Mid-weld (MW), weld edge (WE), RD, TD
- X-ray, neutron diffraction

Welding & post-treatment effects:

- up to depth  $\approx$  3-4 mm
- up to weld toe distance  $\approx$  6-7 mm

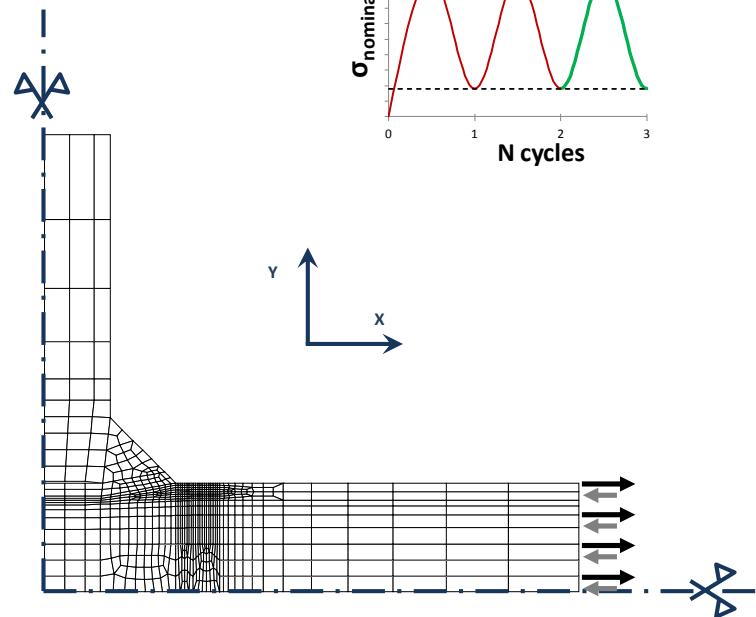
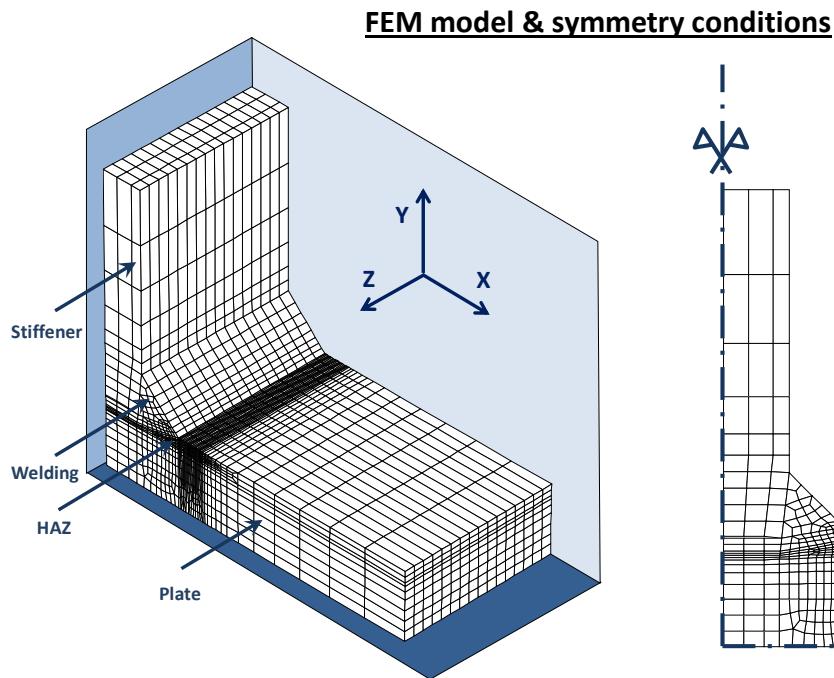


Ex: ref. case: A, with post-treatment (X-ray)



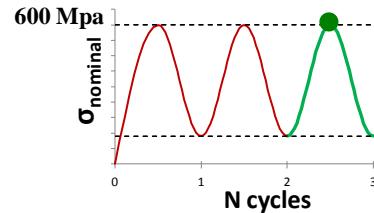
# Large welded plates

- Fatigue - numerical analysis



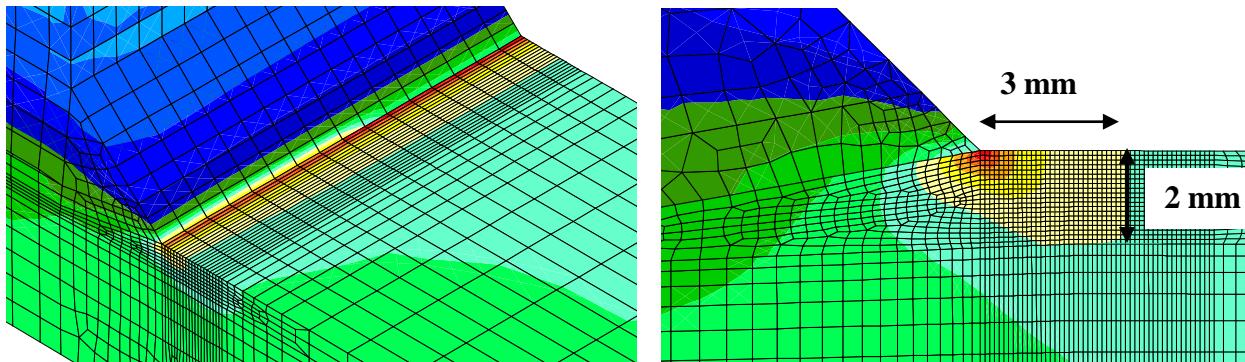
# Large welded plates

- Fatigue - numerical analysis



Example of results without post-treatment , no  $\sigma_{\text{res}}$

$\sigma_{xx}$  (Mpa) for nominal stress = 600 Mpa ( $\sigma_{\text{nom}} = F/A_0$  and  $A_0$ = section of web)



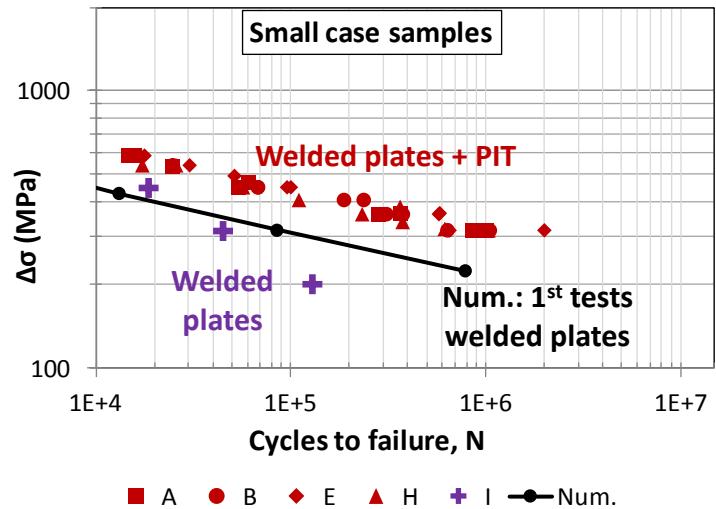
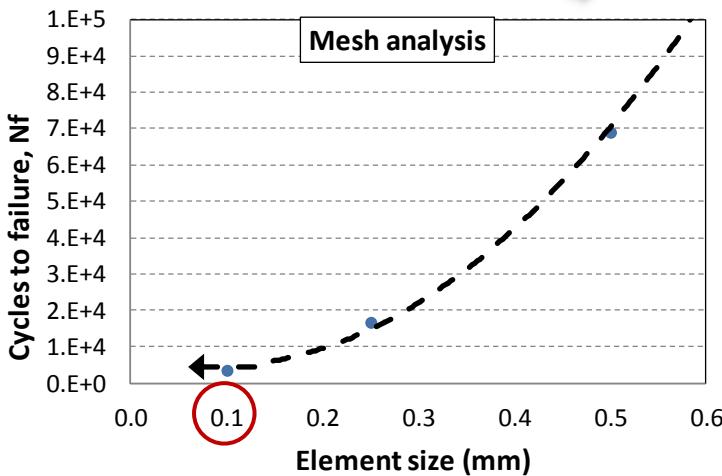
### Stress concentration at weld toe:

- up to depth  $\approx 2$  mm (in HAZ)
- up to weld toe distance  $\approx 3$  mm

## Large welded plates

- Fatigue - numerical analysis

1. Mesh analysis → element size at weld toe: 0.1 mm (results not mesh dependent)
2. For several stress ranges and a specified stress ratio (here: 0.1):
  - Numerical analysis → Stress distribution → number of cycle at rupture

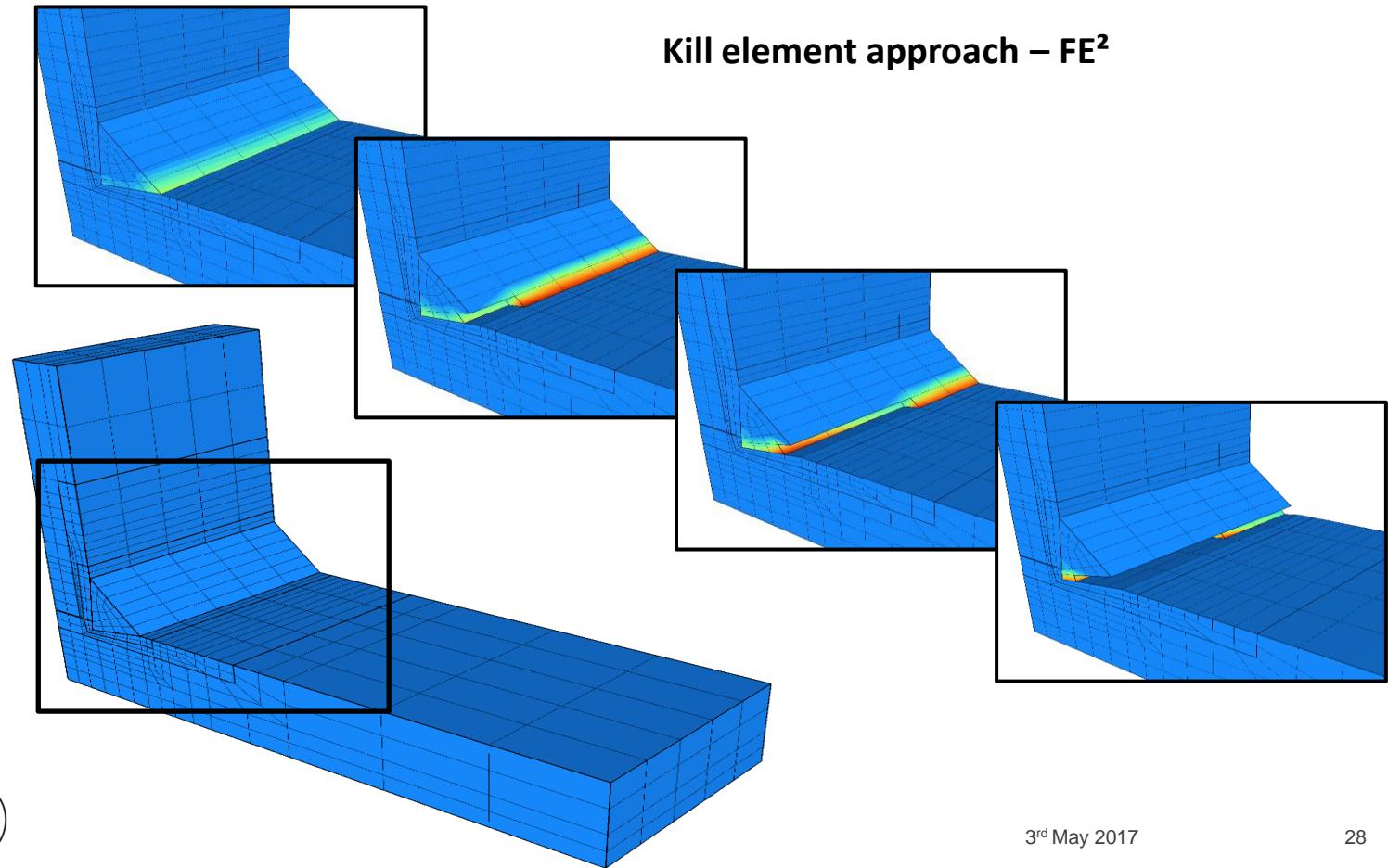


### Next steps:

- to add  $\sigma_{\text{res}}$  to model (welding + post-treatment)
- to improve fatigue mat. data of HAZ ( $\sigma_{10}$ )
- to study beams and critical bridge detail

## Large welded plates

- Fatigue - numerical analysis, crack propagation



## Summary & conclusions

Important test campaign has been done to prepare numerical fatigue study of bridge details:

- Static tests on small samples  
Fatigue tests → Static behaviour  
BM, HAZ, WM
  - Fatigue tests on large welded plates → Effects of size, surface roughness, welding, geometry, post-treatments
  - Residual stresses measurements → Effect of post-treatments for num. analysis
  - Mesh analysis → welded plates: elem. size at weld toe: 0.1 mm
  - Crack propagation → deep analysis of fatigue study
- Positive effect in fatigue life is shown on welded plates  
Fatigue characterisation almost ready for analysis on critical bridge detail

# Thank you for your attention!



E-Mail [chantal.bouffioux@ulg.ac.be](mailto:chantal.bouffioux@ulg.ac.be)

Telefon +32 (0) 4 366 92 19

Fax +32 (0) 4 366 95 34

University of Liège  
ARGENCO department  
Quartier Polytech 1,  
Allée de la découverte, 9 Bât B52/3  
B-4000 Liège BELGIUM