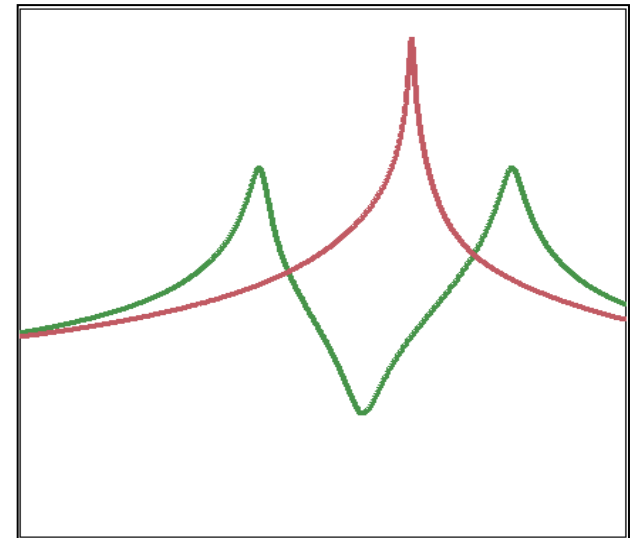


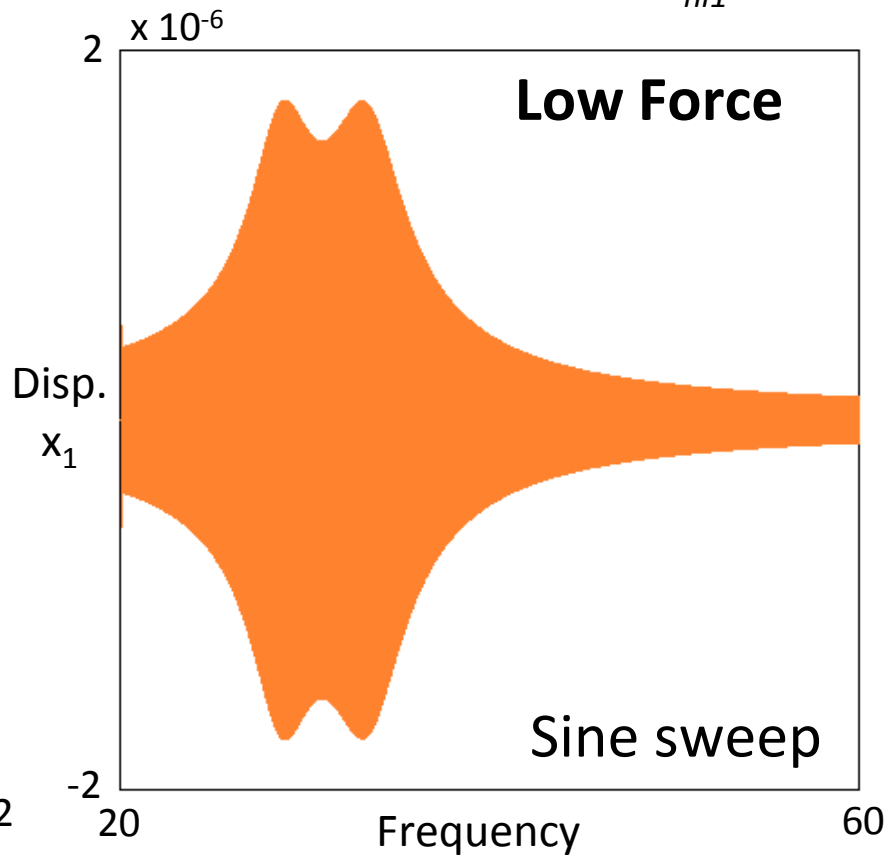
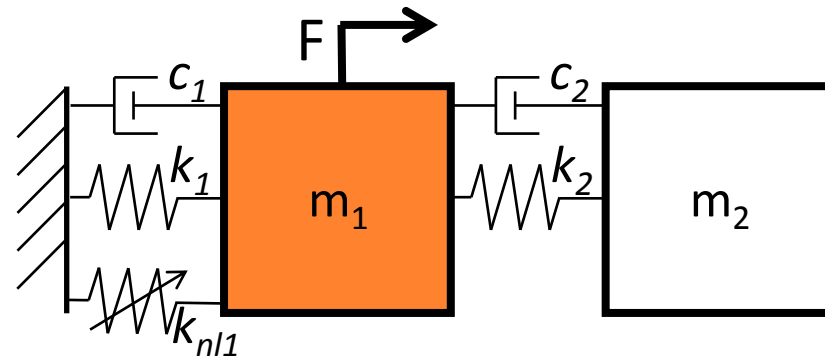
The nonlinear tuned vibration absorber in practice

C. Grappasonni, G. Habib,
T. Detroux, G. Kerschen

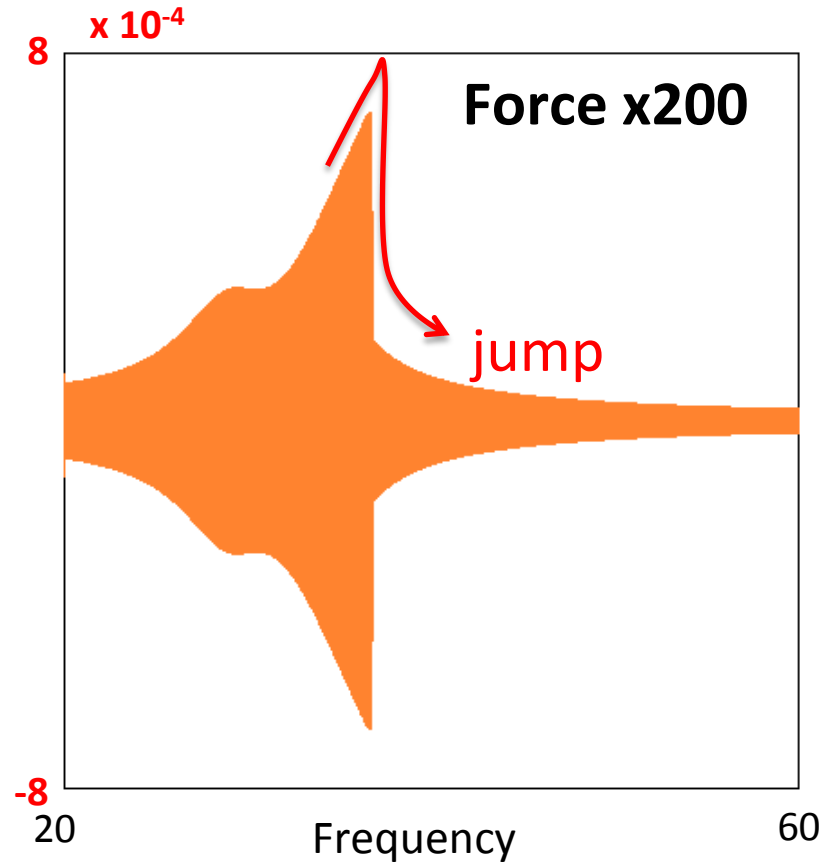
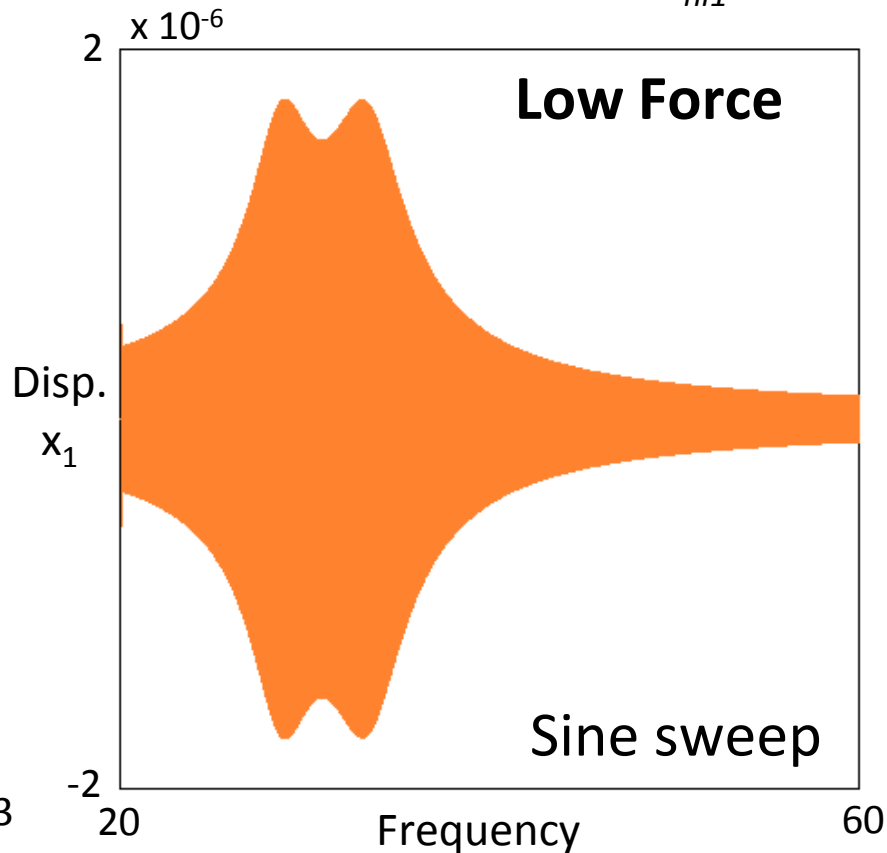
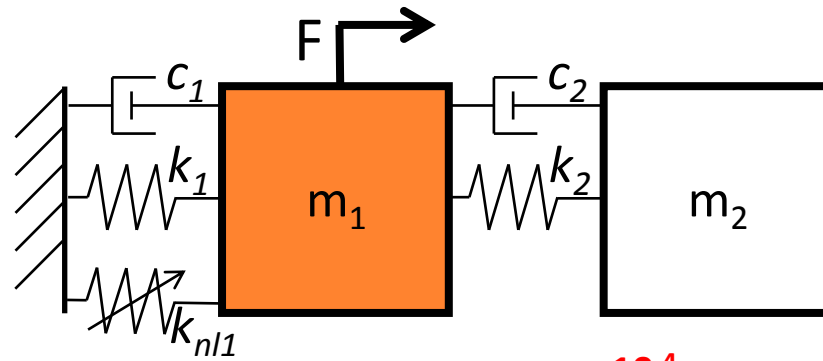
Space Structures and Systems Laboratory
Dept. of Aerospace and Mechanical Eng.
University of Liège, Belgium



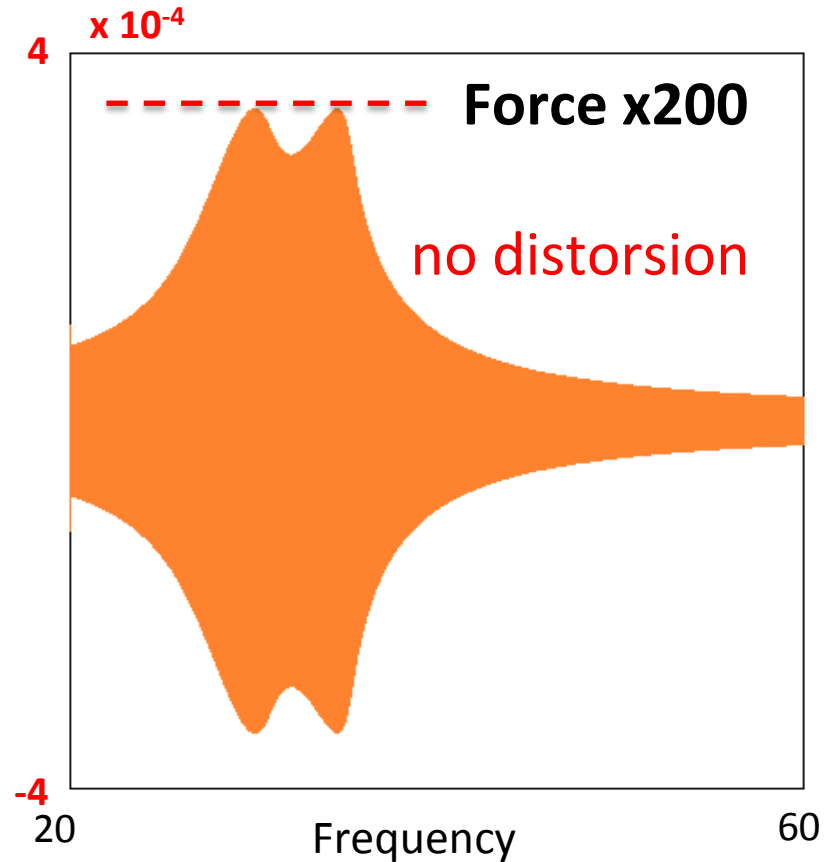
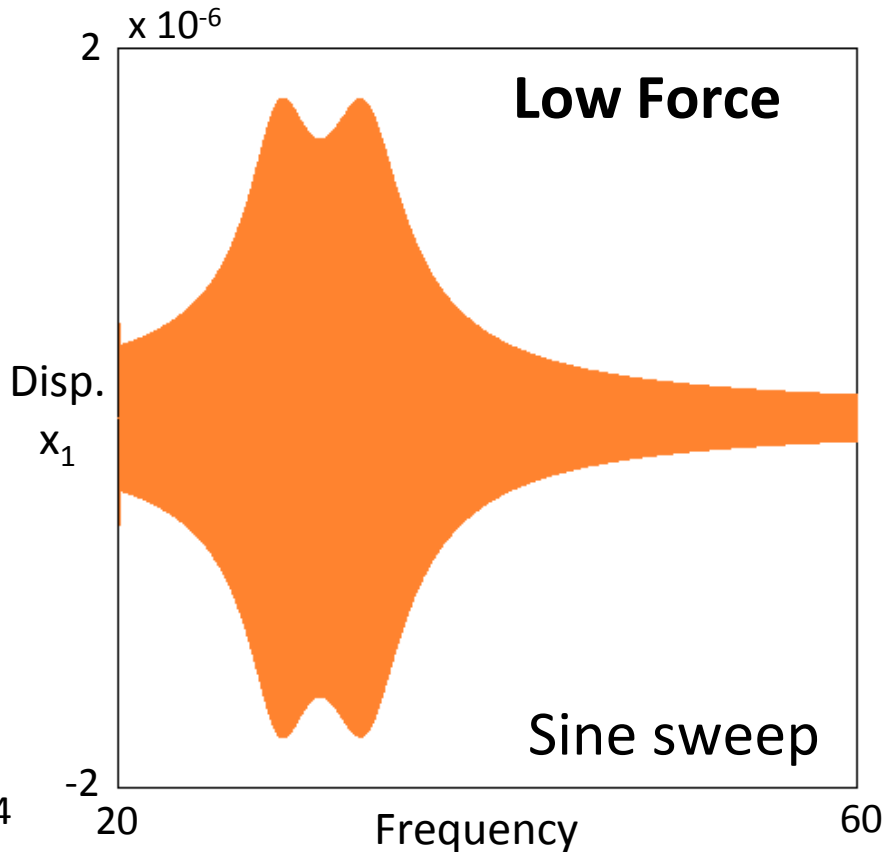
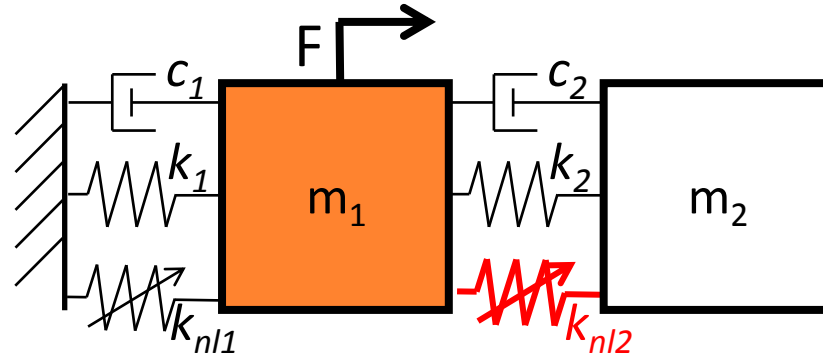
Detrimental effect of nonlinearity



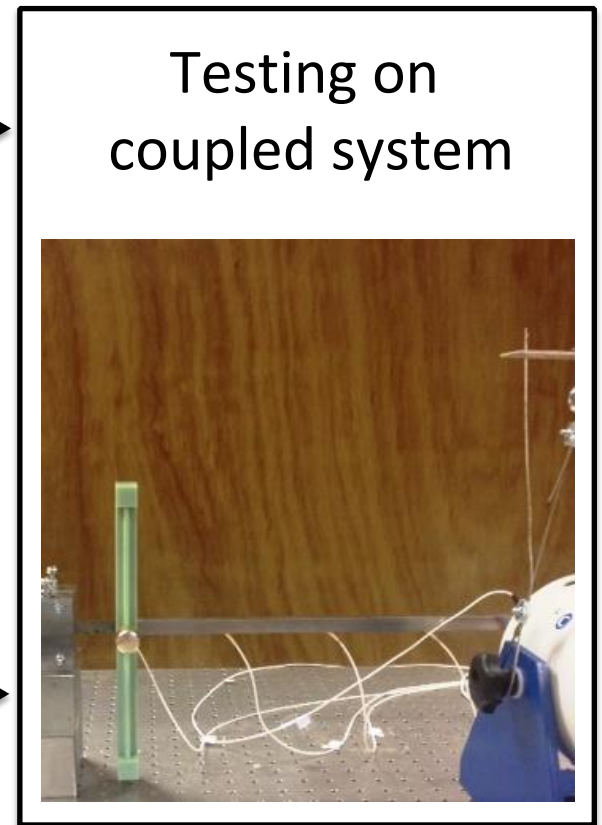
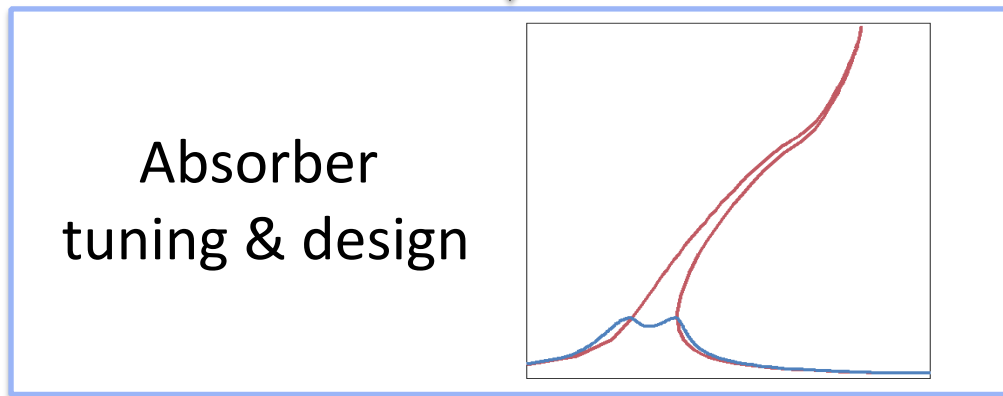
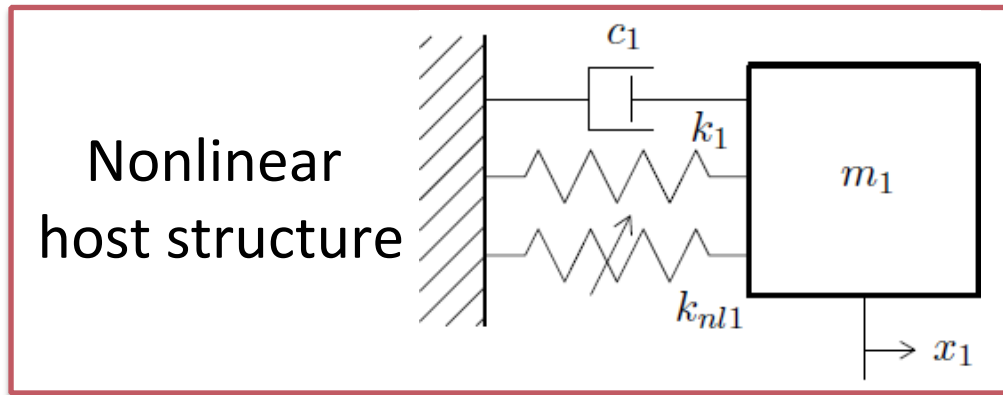
Detrimental effect of nonlinearity



Beneficial effect of nonlinearity: NL + NL = LIN

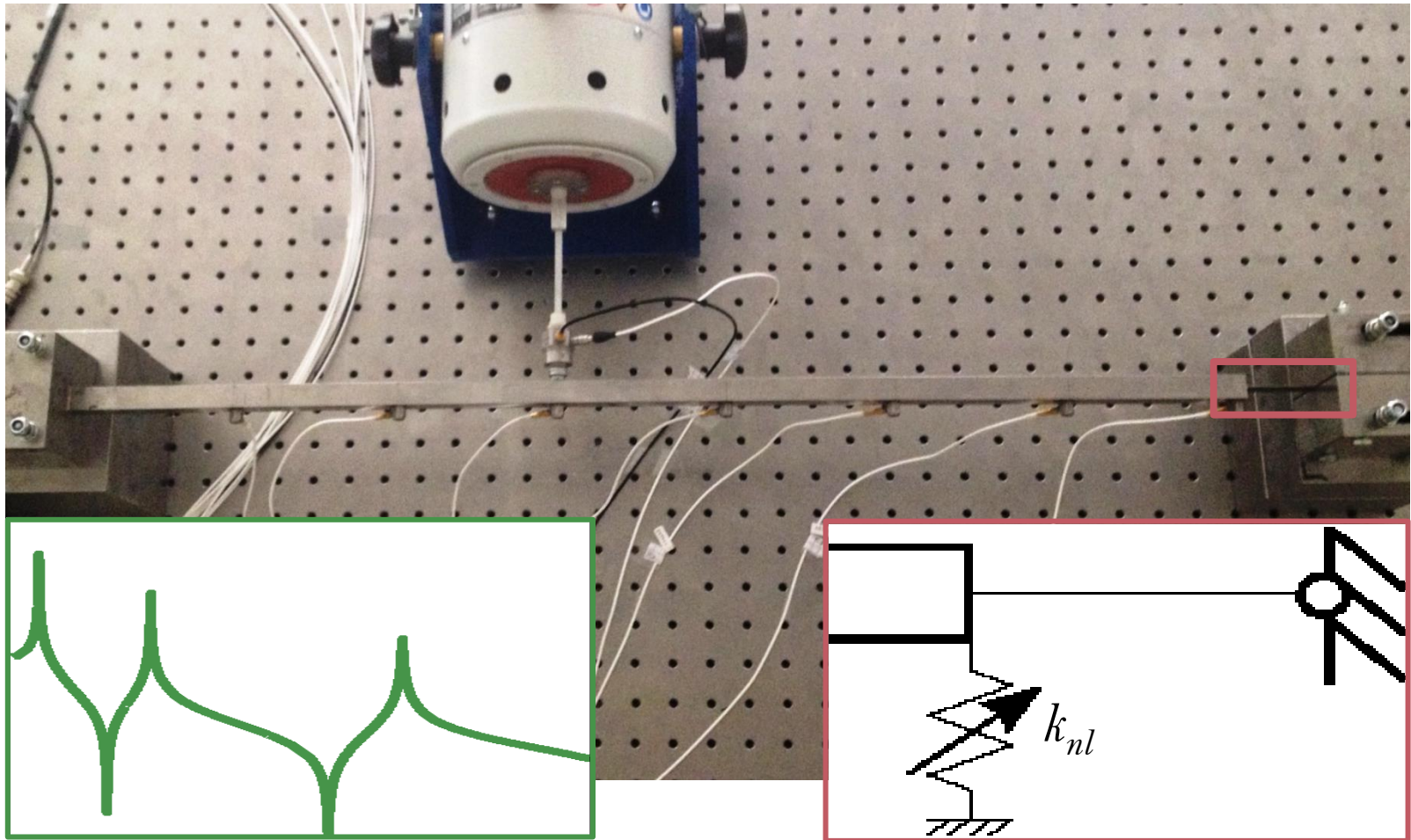


Outline



Our nonlinear experimental host structure

Thouverez (2003) Presentation of the ECL benchmark. Mech Syst Signal Process 17:195–202

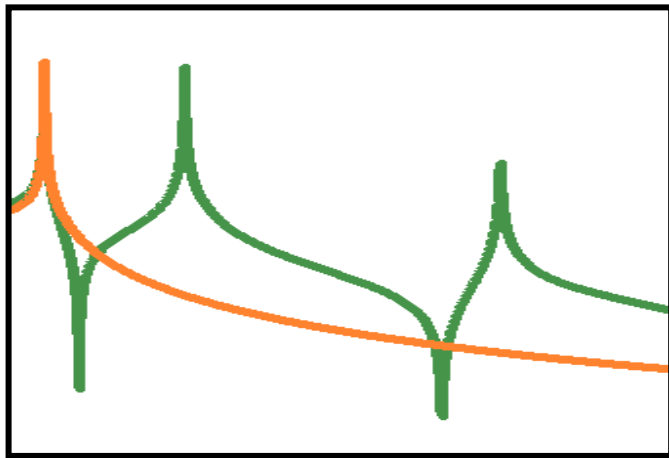


MDOF system

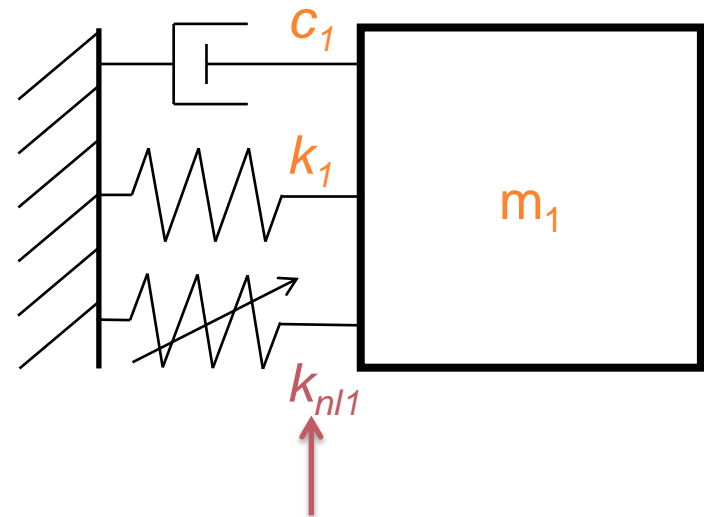
Concentrated nonlinearity

Our target is to mitigate the 1st nonlinear resonance

Nonlinear experimental host structure



SDOF system

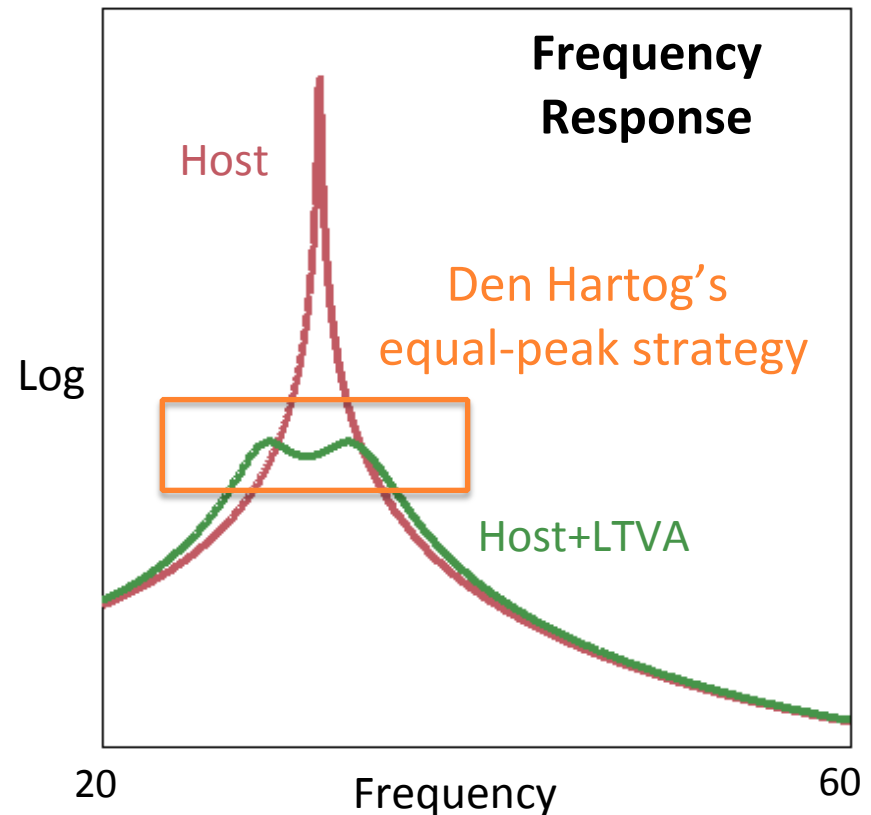
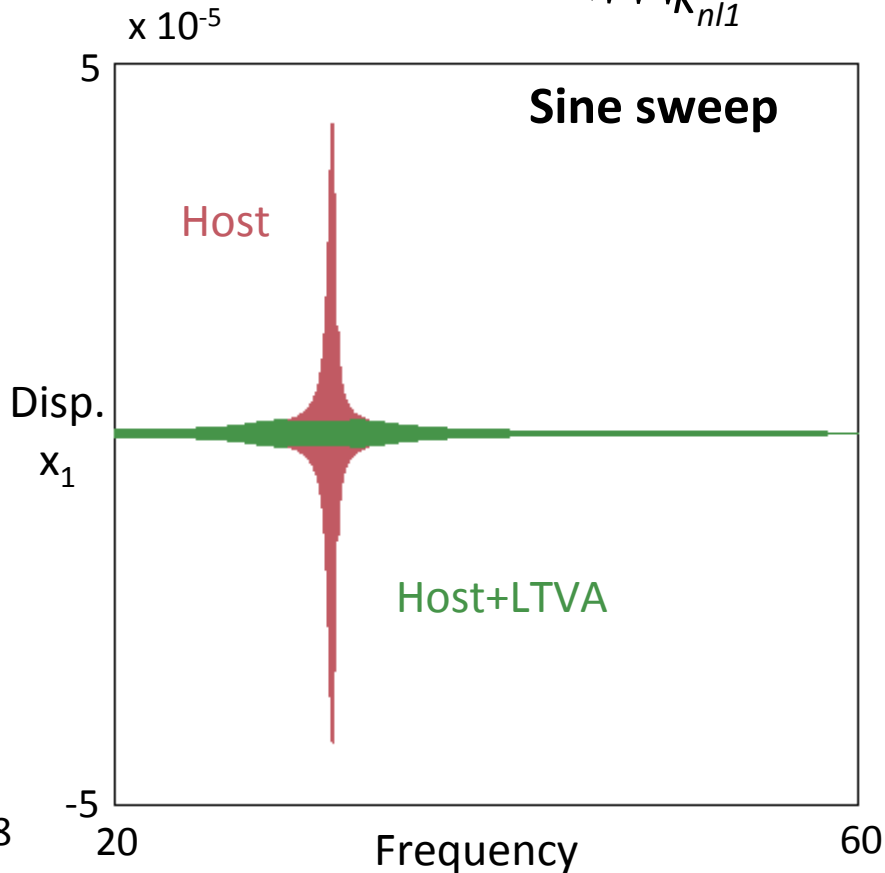
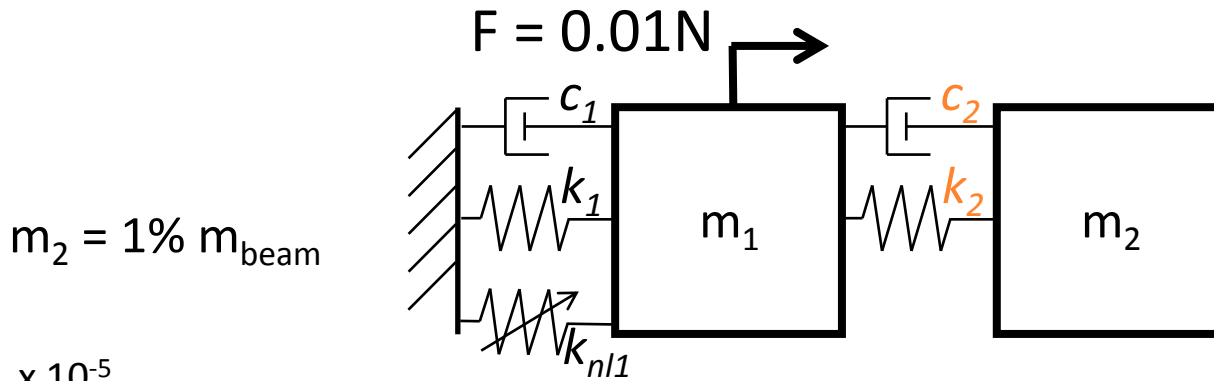


Concentrated nonlinearity

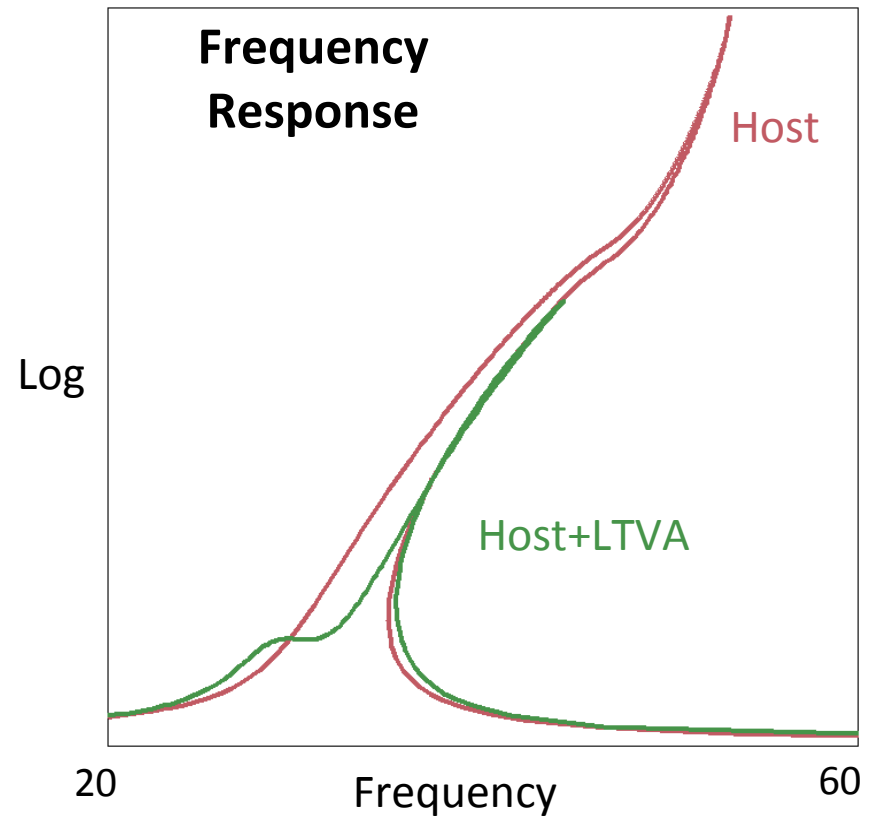
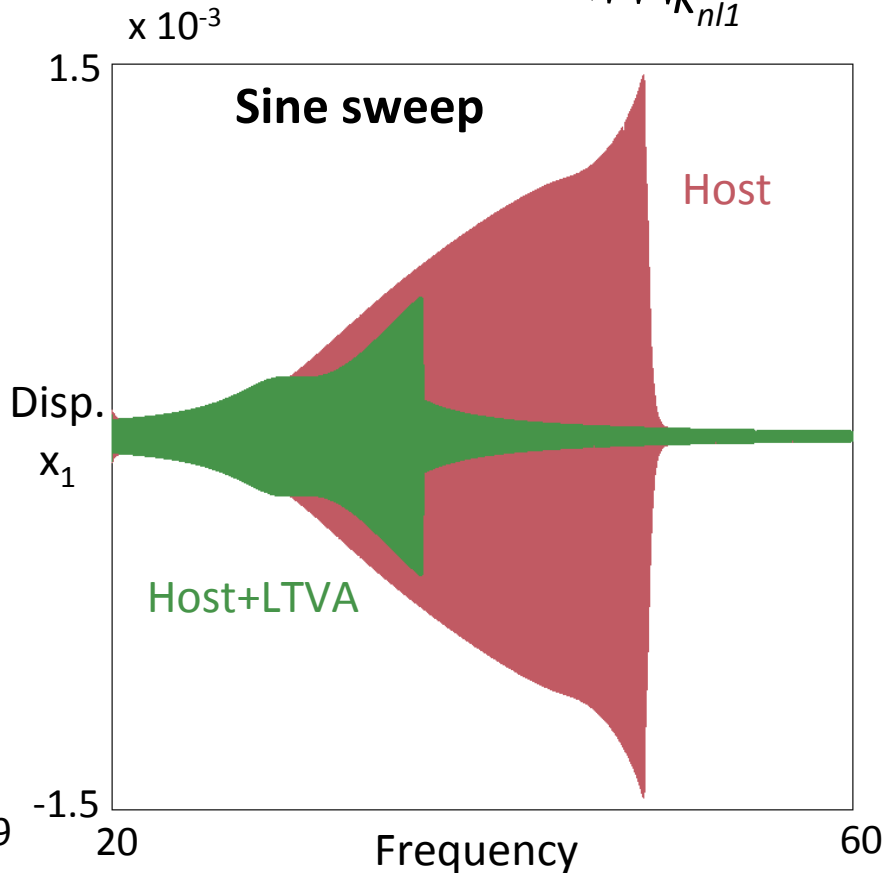
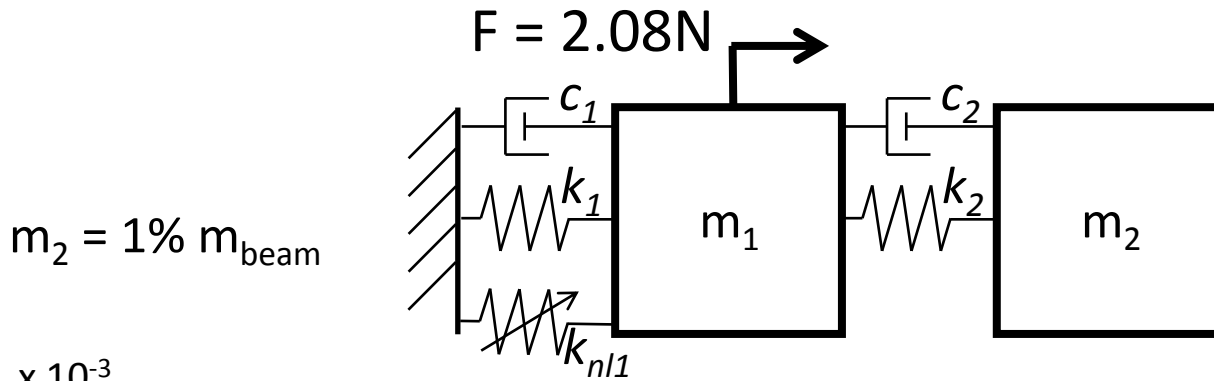
Nonlinear modal model

Mitigating resonant vibrations with a LTVA

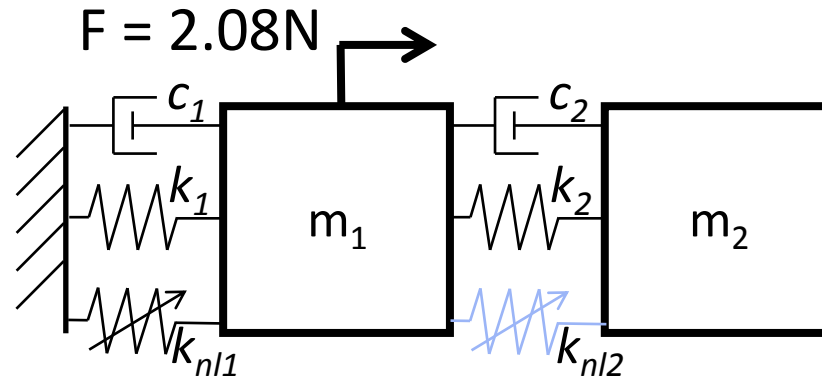
[Den Hartog, 1928]
[Brock, 1946]
[Asami, 2003]



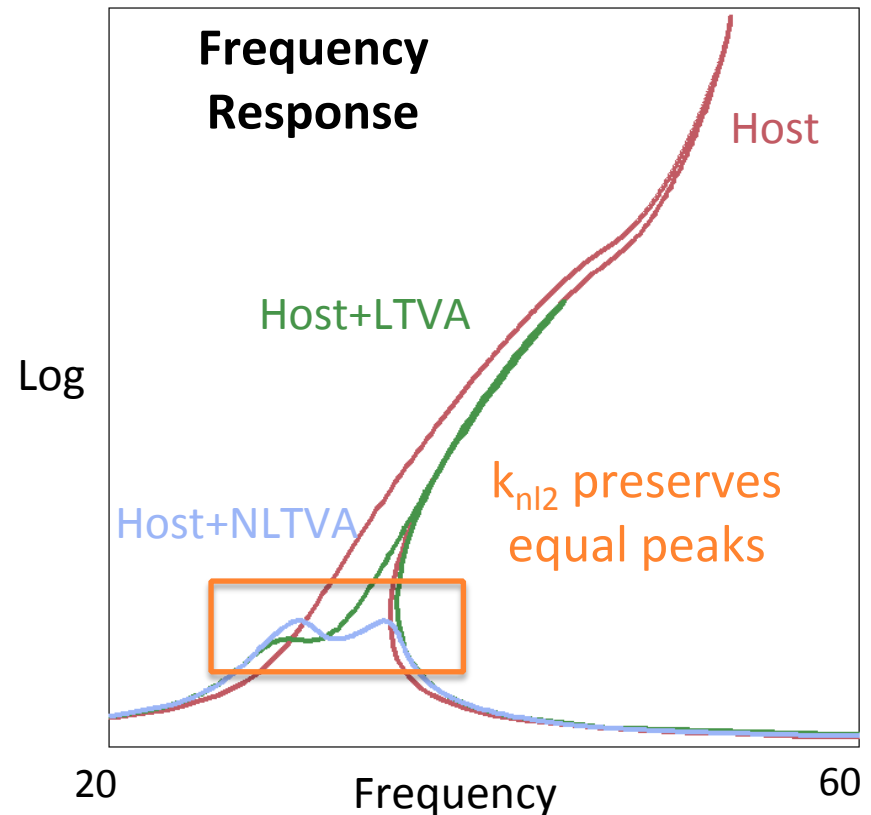
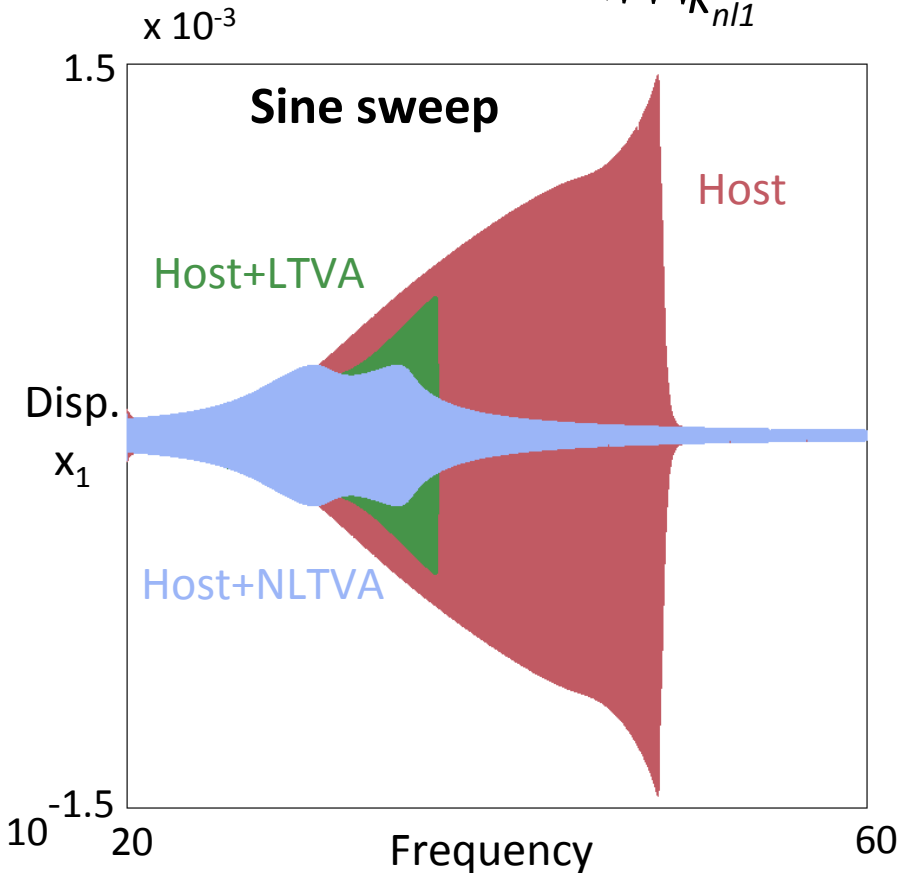
The LTVA detunes at high excitation



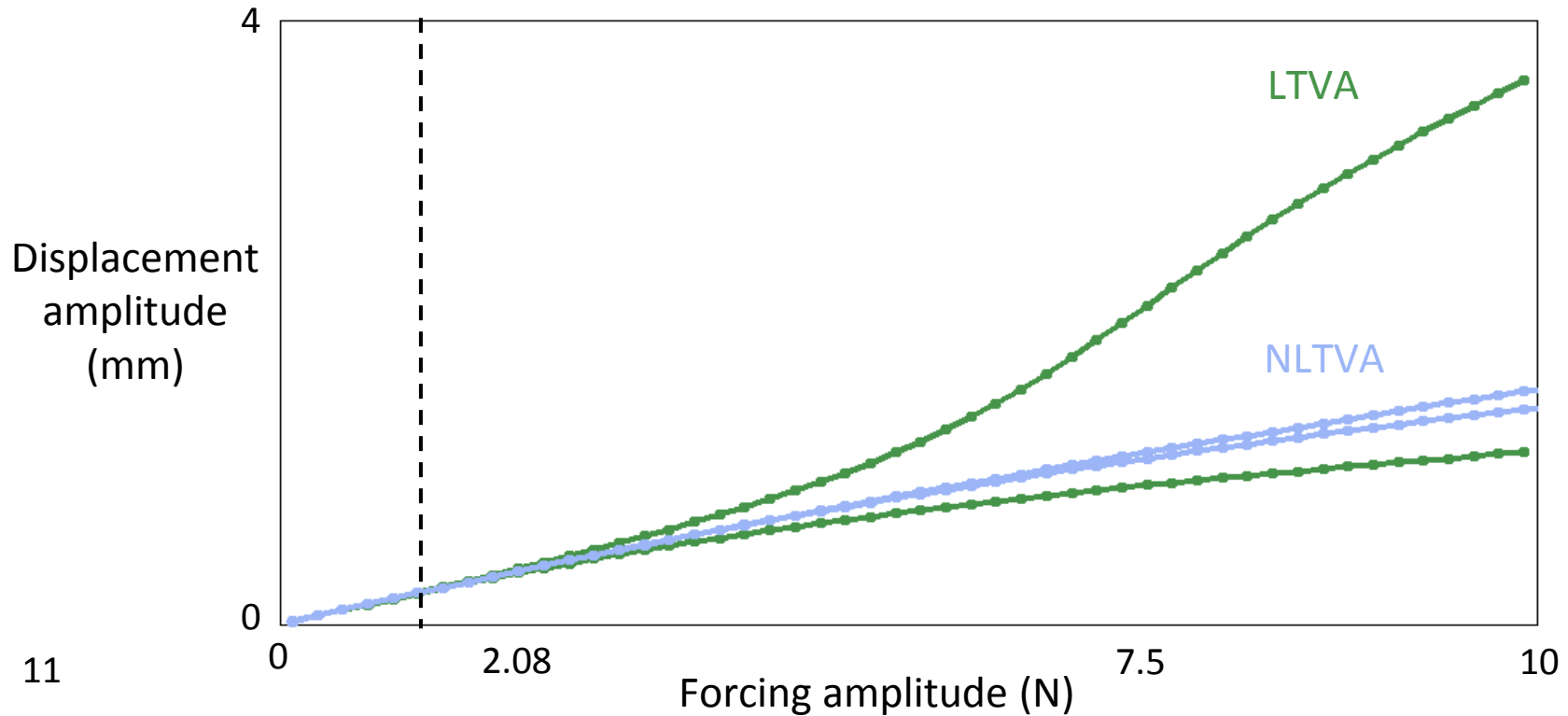
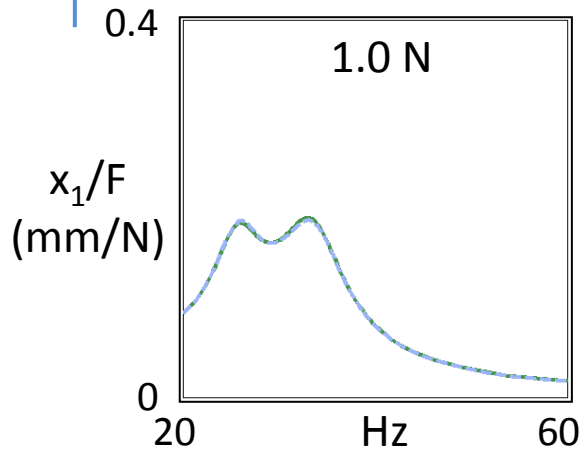
Mitigating resonant vibrations with a NLTV



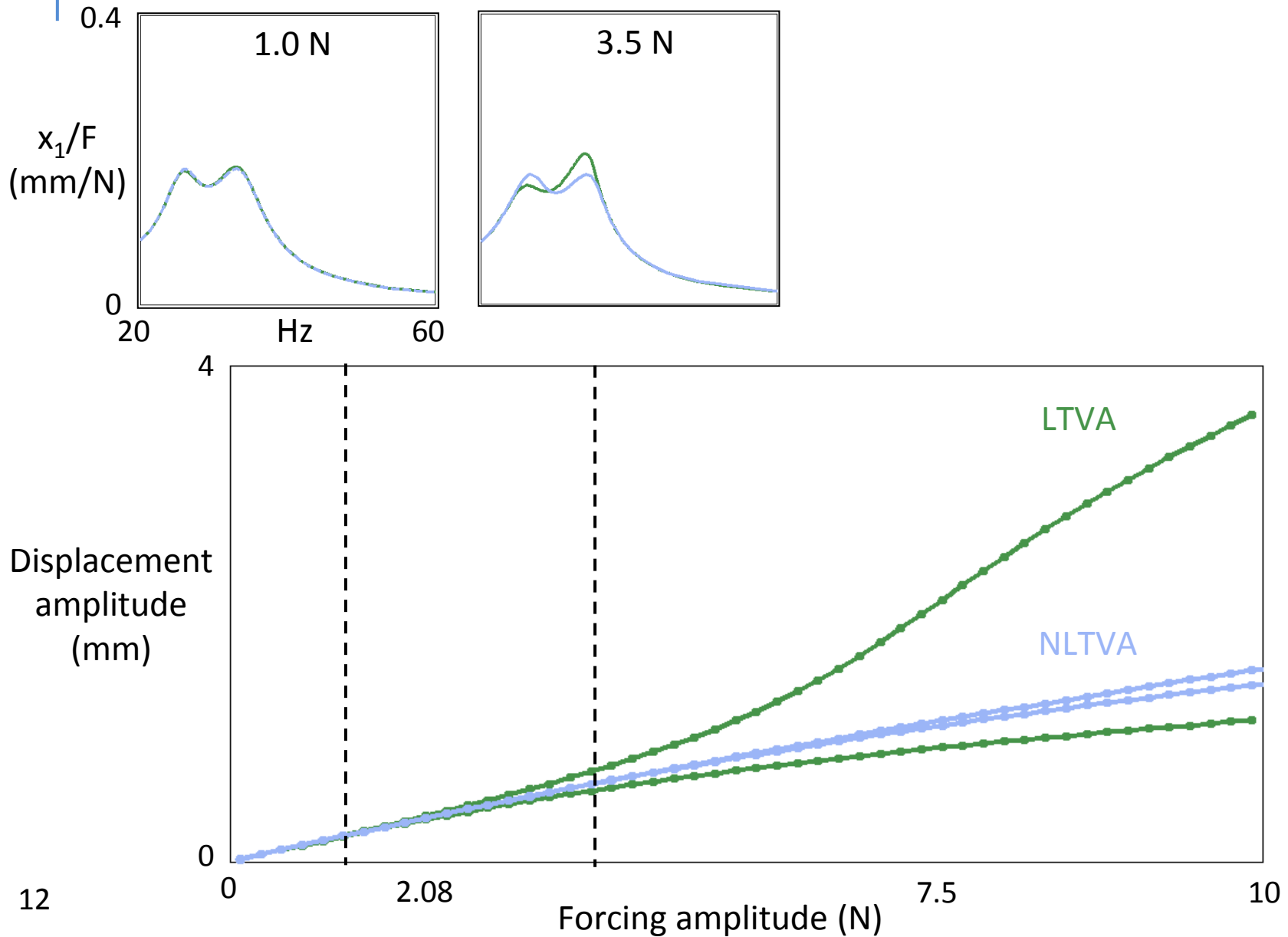
$$k_{nl2} = \frac{2\epsilon^2 k_{nl1}}{1 + 4\epsilon}$$



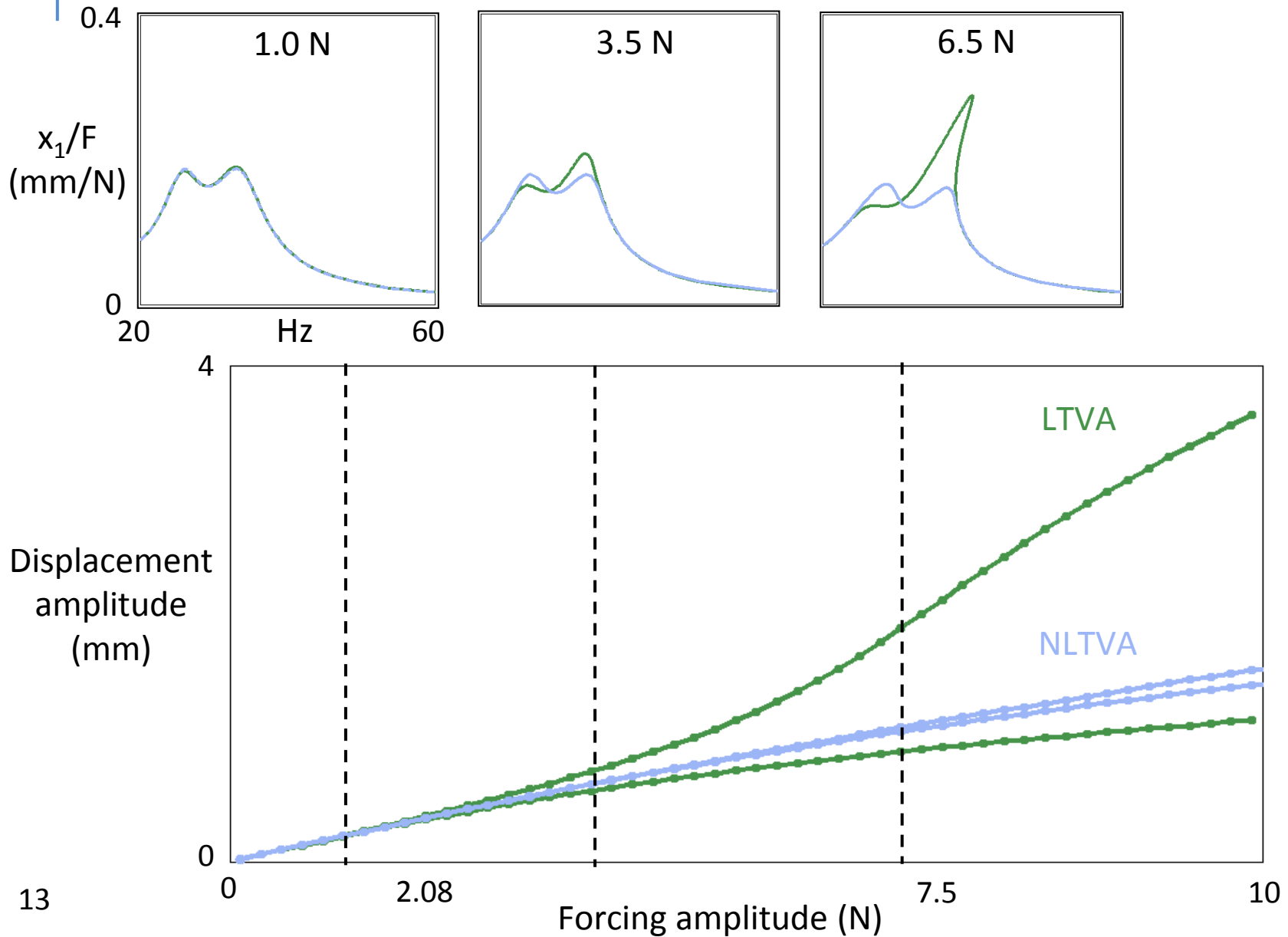
The NLTVA outperforms the LTVA by design



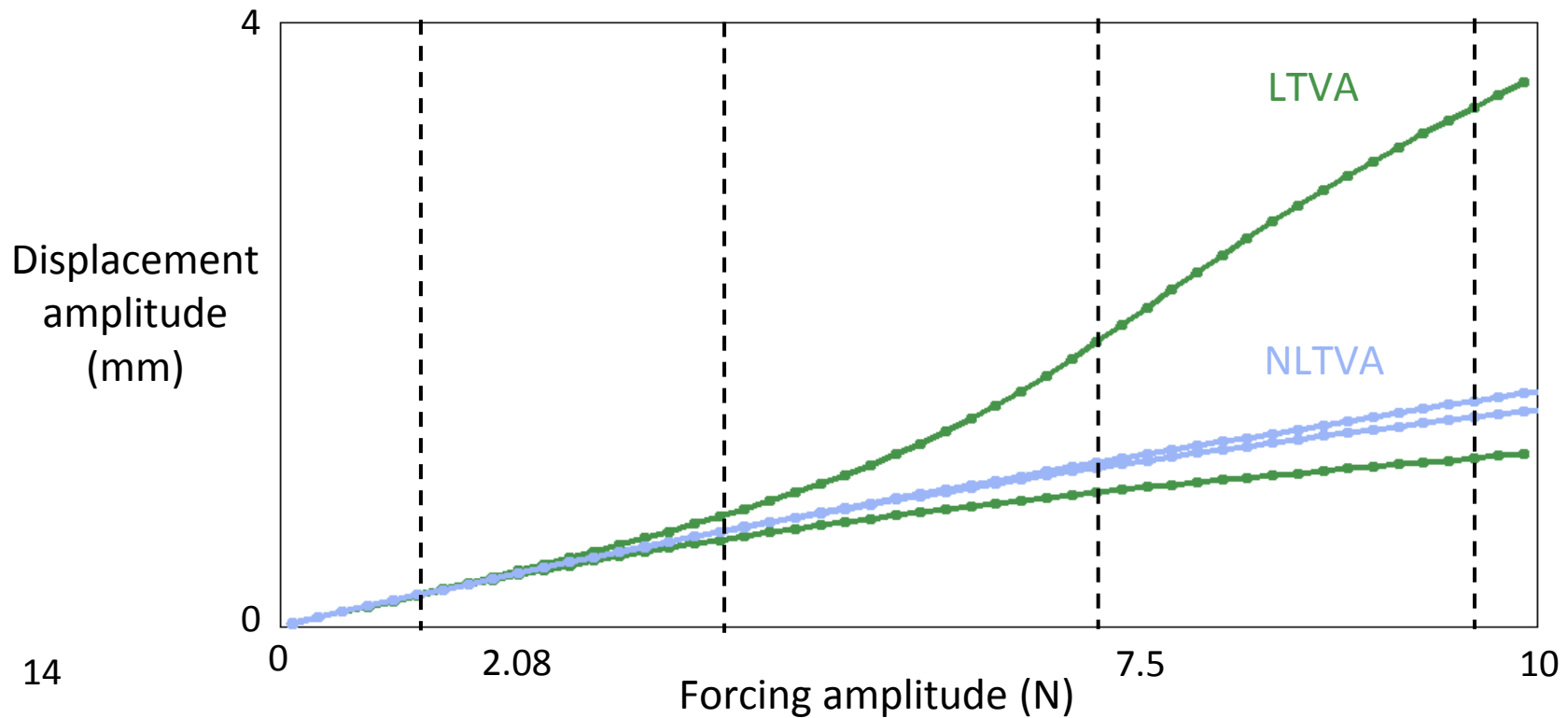
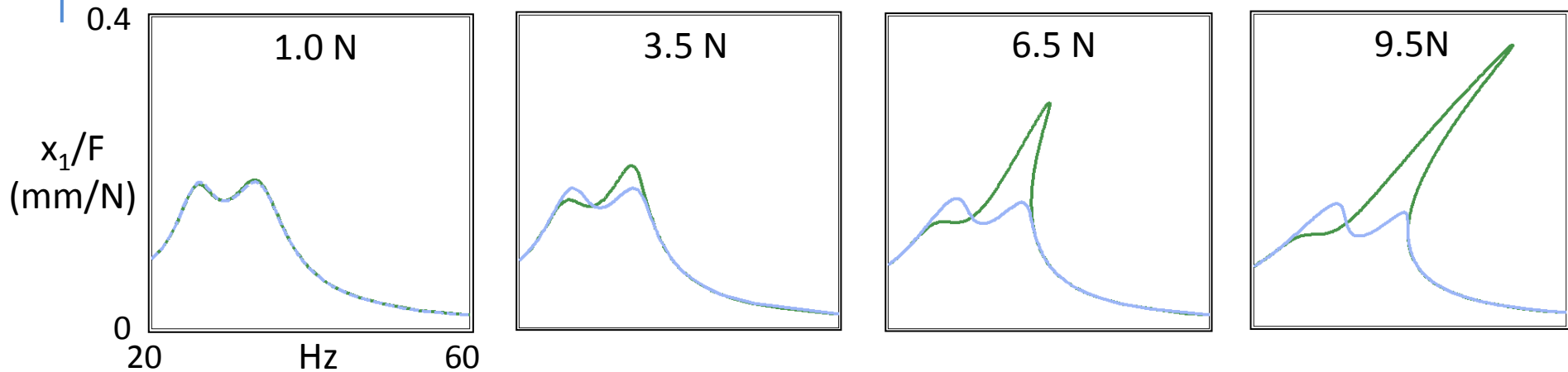
The NLTVA outperforms the LTVA by design



The NLTVA outperforms the LTVA by design



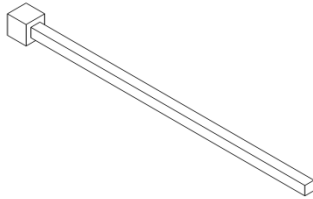
The NLTVA outperforms the LTVA by design



Practical realization based on “simple” beams

LTVA

linear cantilever beam

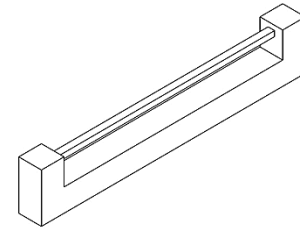


$$m_2 = \rho AL$$

$$\sqrt{\frac{k_2}{m_2}} = \sqrt{\frac{1.875^4 EI}{12L^5 \rho^2}}$$

NLTVA

NL doubly-clamped beam



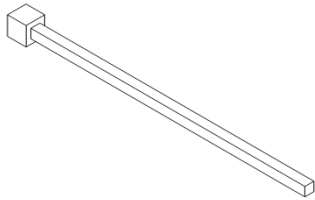
$$k_2 = \frac{2\pi^4 EI}{L^3}$$

$$k_{nl2} = \frac{\pi^4 EA}{8L^3}$$

Practical realization using 3D printing

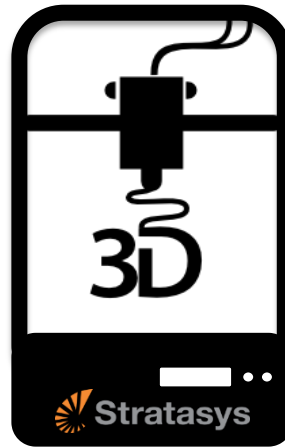
LTVA

linear cantilever beam



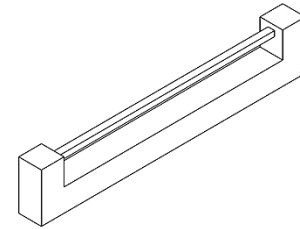
$$m_2 = \rho AL$$

$$\sqrt{\frac{k_2}{m_2}} = \sqrt{\frac{1.875^4 EI}{12L^5 \rho^2}}$$



NLTVA

NL doubly-clamped beam



$$k_2 = \frac{2\pi^4 EI}{L^3}$$

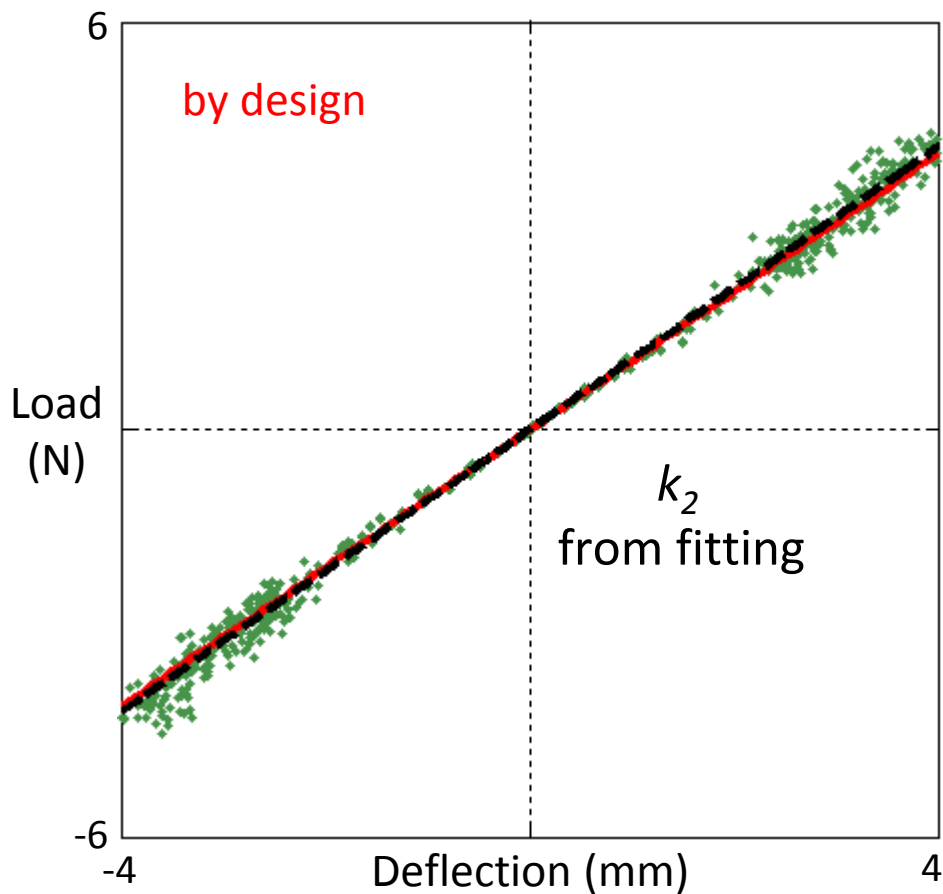
$$k_{nl2} = \frac{\pi^4 EA}{8L^3}$$



The printed TVAs follow the prescribed stiffness curves

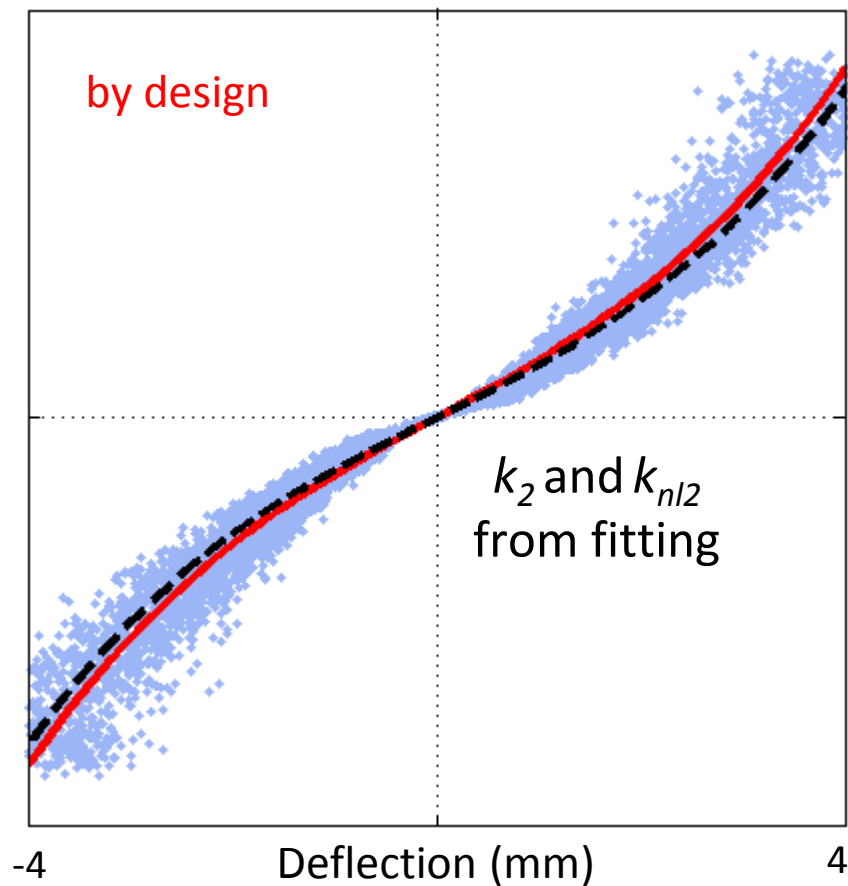
LTVA

linear cantilever beam

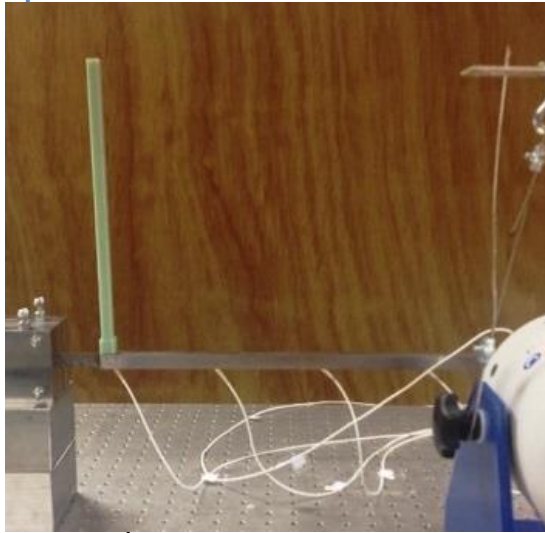


NLTVA

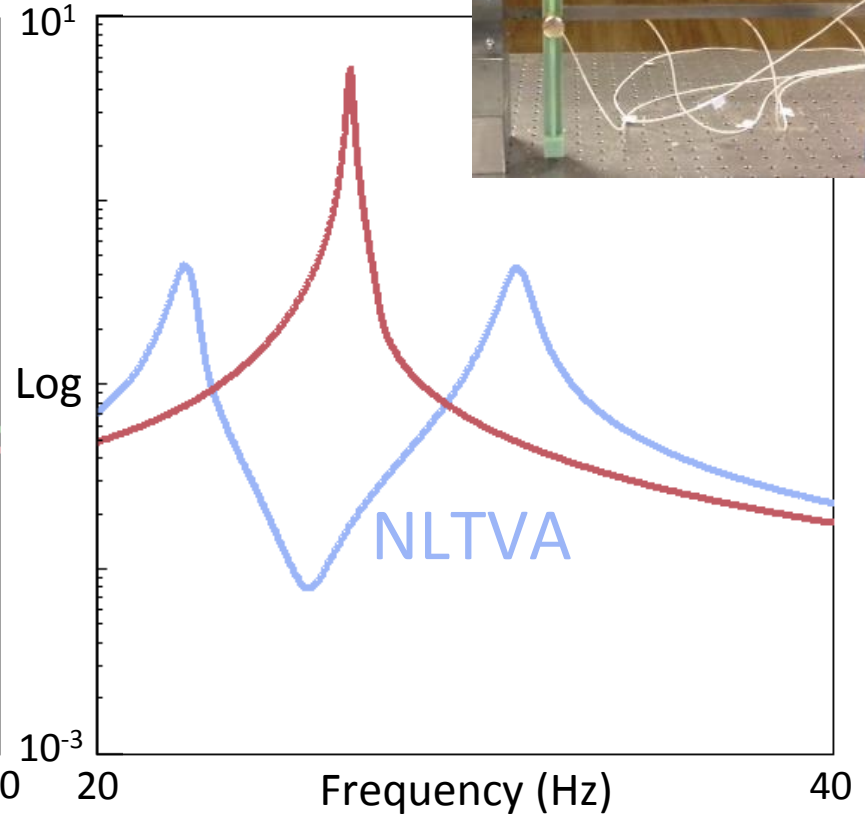
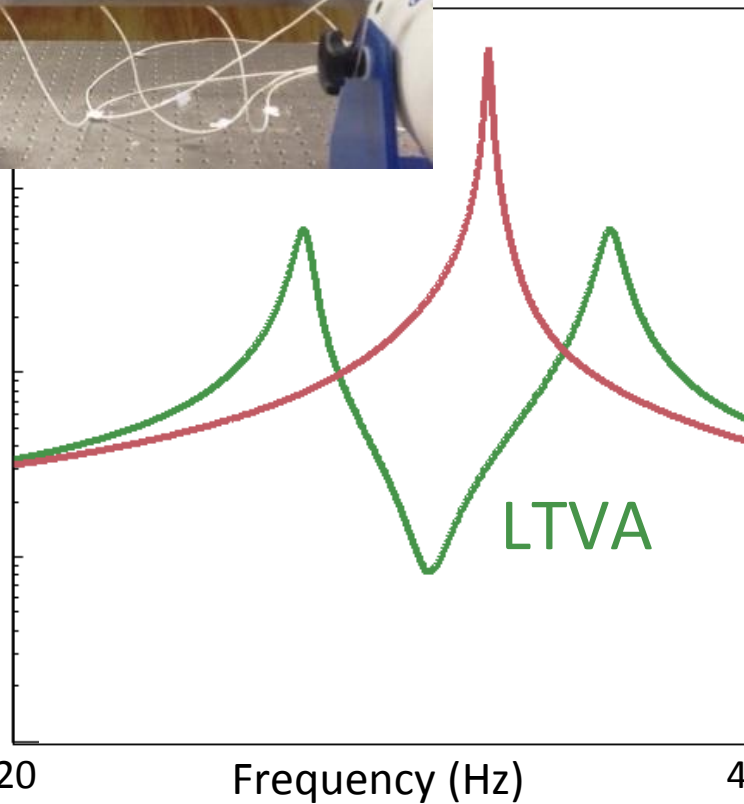
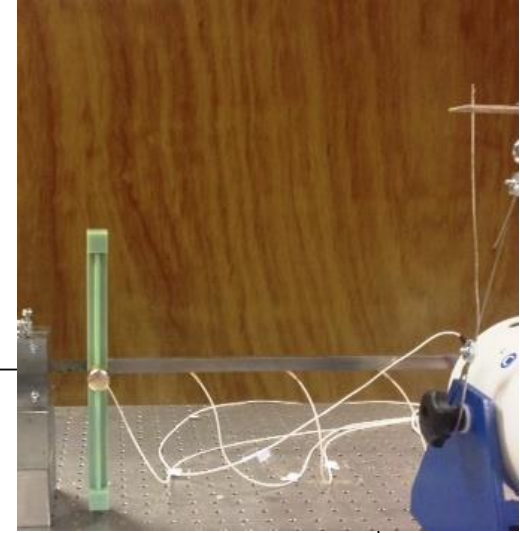
NL doubly-clamped beam



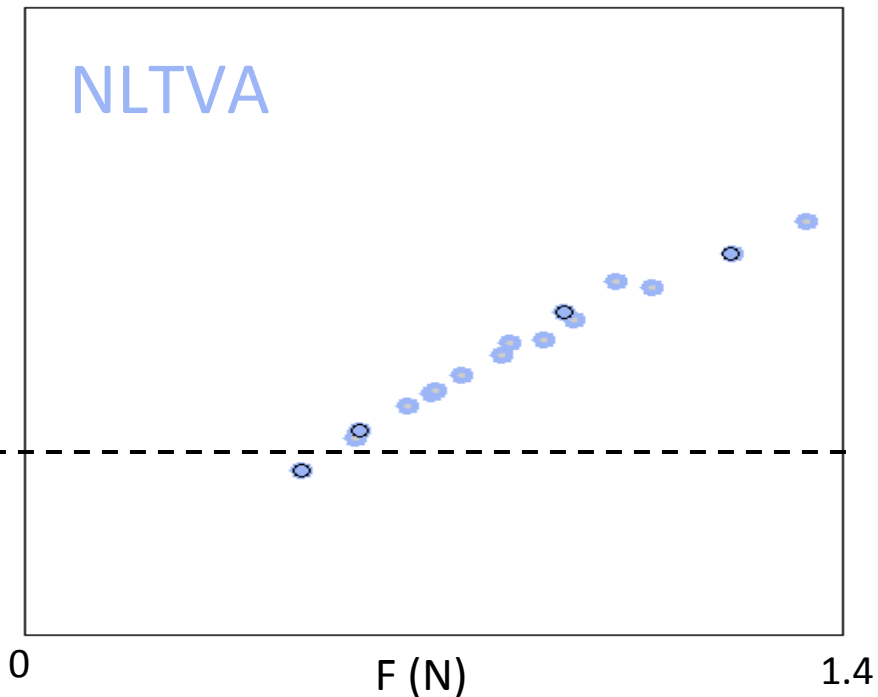
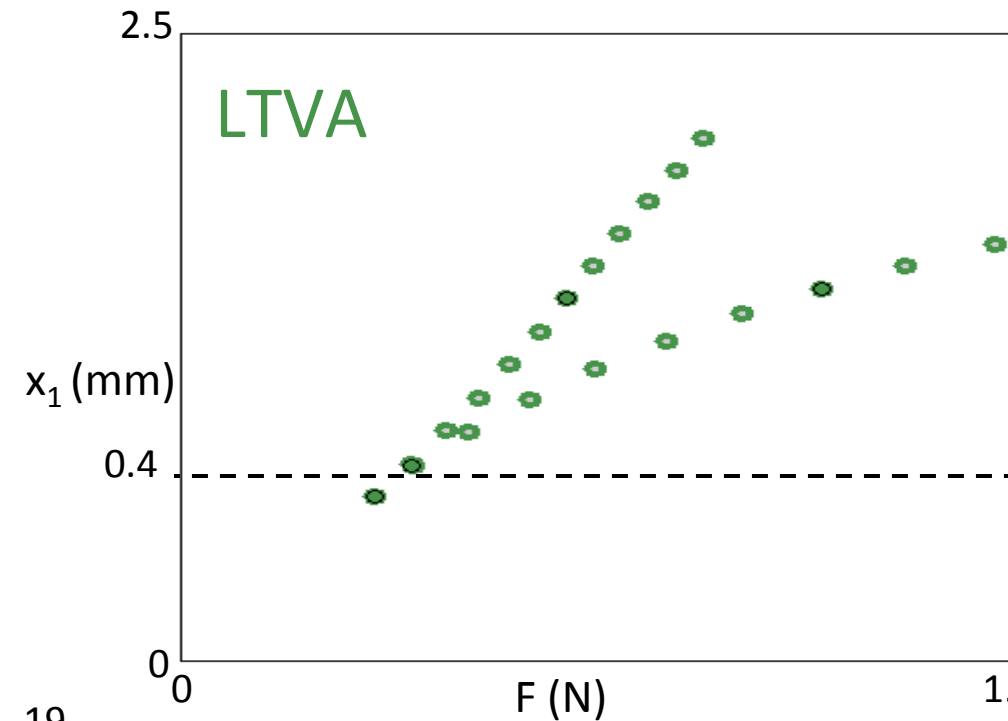
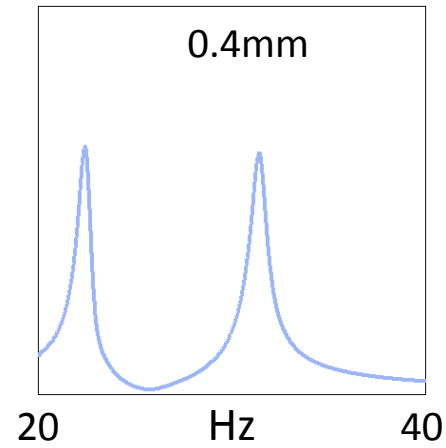
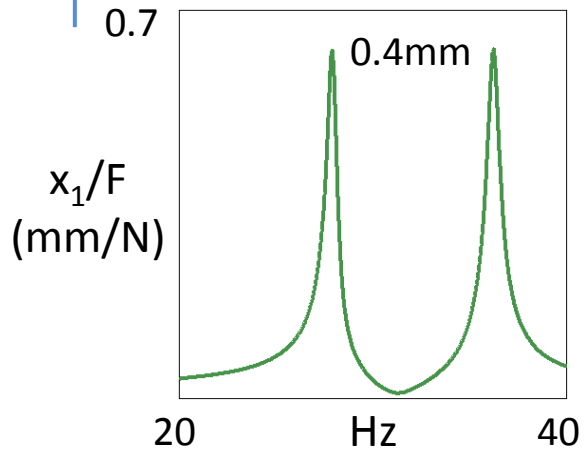
The effects of the **real** absorbers on the host structure



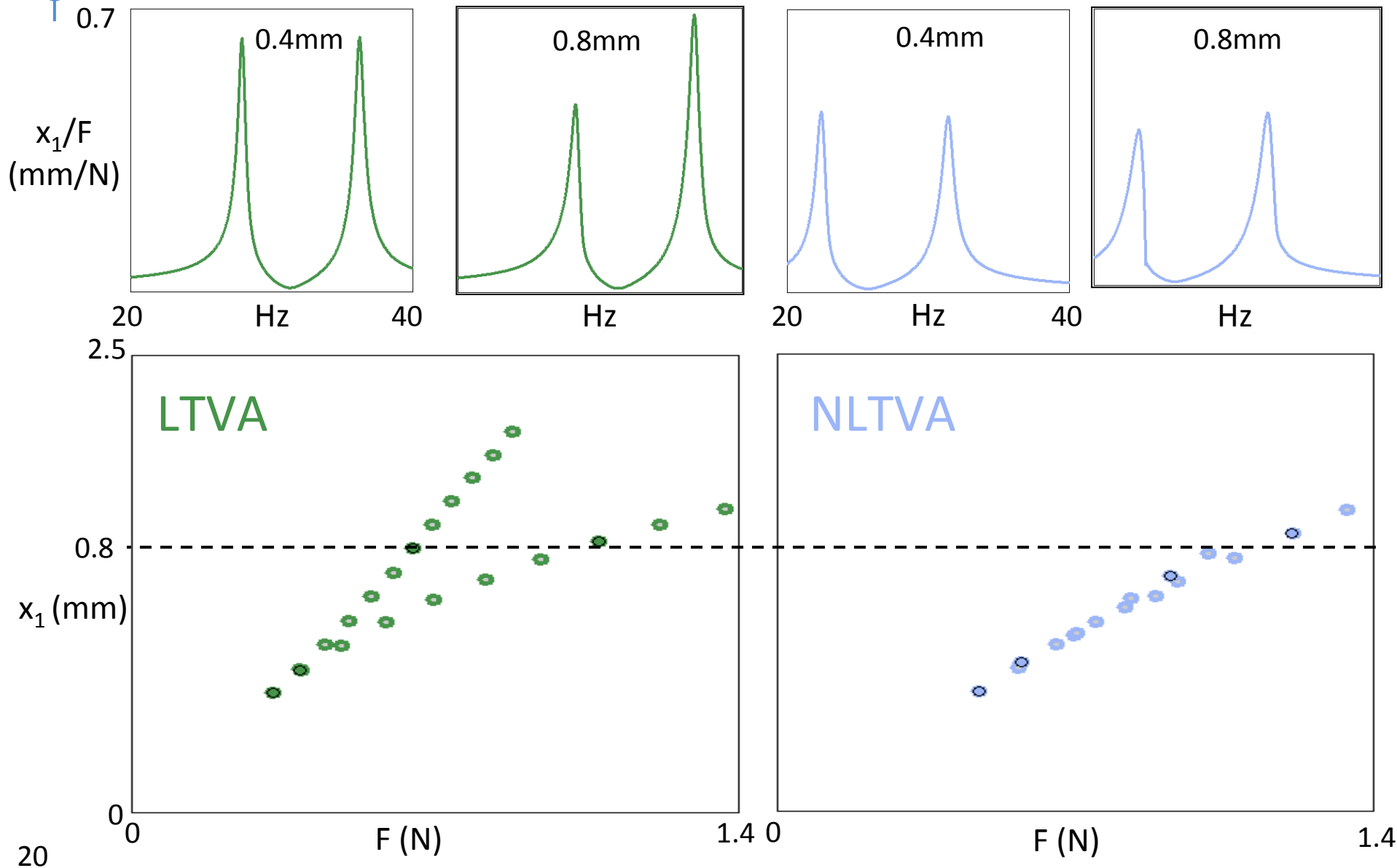
FRF amplitude
(mm/N)



The NLTVA outperforms the LTVA, for real!

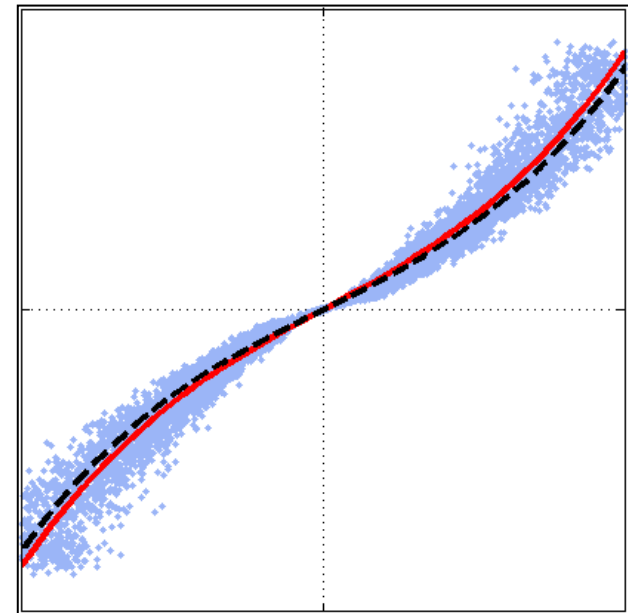
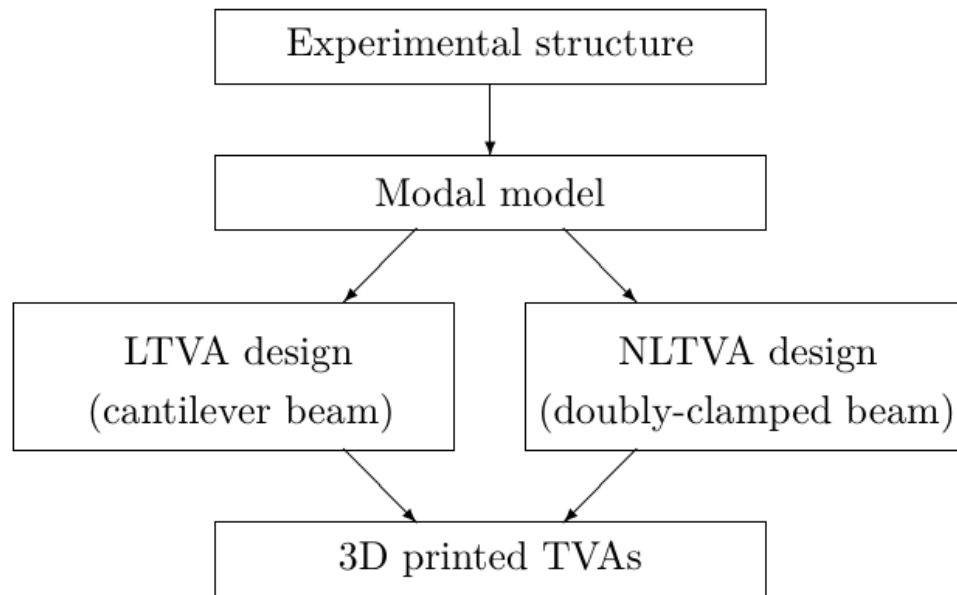


The NLTVA outperforms the LTVA, for real!



Conclusions

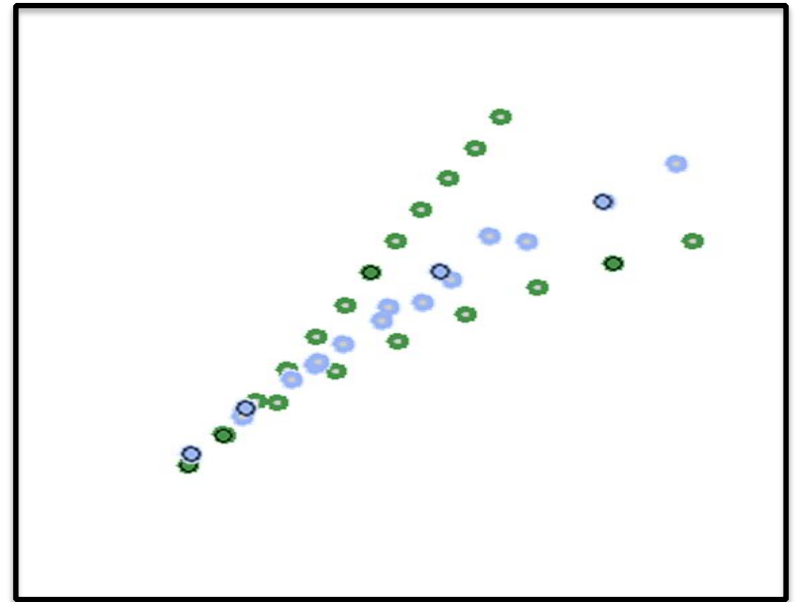
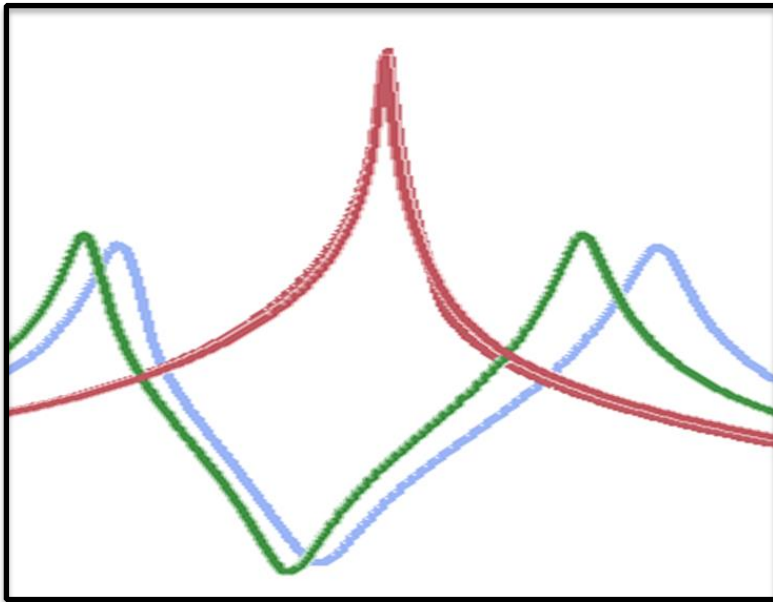
The TVA can be designed analytically:



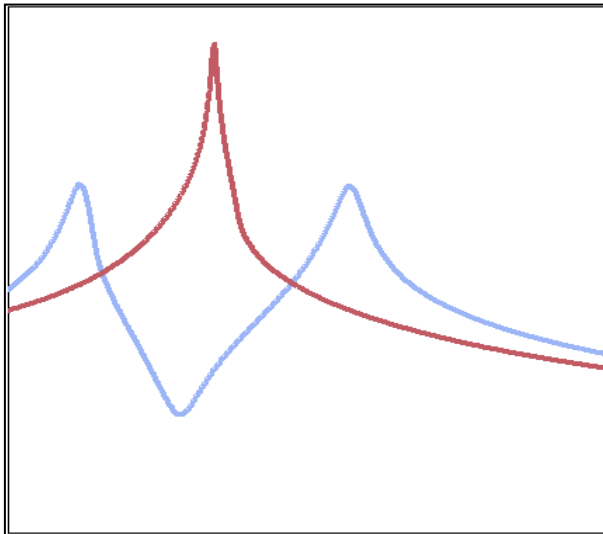
and the real TVAs follow the objective NL load-deflection function

Conclusions

NLTVA allows to maintain equal peaks in nonlinear regimes



and the experiments validated the concept: $N\text{Lin} + N\text{Lin} = \text{Lin}$



Thanks for your attention

ACKNOWLEDGEMENTS:

- financial support of the European Union (ERC Starting Grant NoVib 307265)
- LMS A Siemens Business provided access to the LMS Test.Lab software