



#### University of Liège Aerospace & Mechanical Engineering Department

# Computational homogenization of cellular materials capturing micro-buckling, macro-localization and size effects

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Cellular materials: "a cellular solid is one made up of an interconnected network of solid struts or plates which form the edges and faces of cells"

(Gibson & Ashby 1999)

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- Classification based on the topology
  - Honeycombs: 2-dimensional arrays of polygons (e.g. Bee hexagons)
  - Foams: 3-dimensional network of cell edges and cell faces
    - Network of cell edges →open-cell foams
    - Network of cell faces → closed-cell foams
    - Partly open- & partly closed-cell foams
- Man-made cellular materials



Aluminum honeycomb



Open-cell nickel foam



Closed-cell polyurethane foam

2

50 µm





• Natural cellular materials

1 mm 200 µm 2mm 1mm

(Gibson & Ashby 1999)

(a) cork

(b) balsa

(c) sponge

(d) trabecular bone

(e) coral

(f) cuttlefish bone

(g) iris leaf

(h) plant stalk





## • Cellular materials are used in many applications

- Light weight structures
- Energy absorption
- Packaging
- Sound absorption
- Thermal insulation
- Etc.





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"From imaging to geometrical modeling of complex micro-structured materials: Bridging computational engineering and material science"









#### Mechanical behavior of cellular materials

 Buckling of thin components (cell struts, cell faces) can occur under compression loads leading to macroscopic localization



Force-displacement crushing response of aluminum honeycomb (Experimental data from Papka & Kyriakides 1999)







## • Mechanical behavior of cellular materials (2)

Size effect



Effect of shear layer thickness in shear stress-strain response of Alporas foam (Experimental data from Chen & Fleck 2002)





## Mechanical behavior of cellular materials (3)

- Multi-scale behavior in nature
- Intrinsic roles of different scale properties have to be accounted for
- Macro-localization, micro-buckling, size effect phenomena have to be evaluated



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- Finite element modeling strategies for cellular materials
  - Direct modeling-based approach
    - Direct models of cell struts, cell faces with beam, shell, bulk elements
- Constitutive modeling-based approach
  - Cellular solids are considered as homogeneous media with suitable constitutive laws:
    - Phenomenological models
      - » Curve fitting, parameters identification from numerical or experimental results
    - Homogenization models
      - » Mean field, FFT, asymptotic, computational, etc.











- Finite element modeling strategies for cellular materials (2)
  - Direct modeling-based approach
    - Advantages:
      - Capture directly localization due to micro-buckling & size effects
    - Drawbacks
      - Enormous number of DOFs
      - Difficult to construct the FE model due to geometry complexities
      - Suitable for small problems with limited dimensions







- Finite element modeling strategies for cellular materials (3)
  - Constitutive modeling-based approach
    - Advantages
      - Suitable for large problems
      - Micro-buckling, macro-localization & size effects can be captured with suitable constitutive models (e.g. computational homogenization)
    - Drawbacks
      - Phenomenological models
        - » Details of the micro-structure during macro-loading cannot be observed
        - » Material models and their parameters are difficult to be identified
      - Homogenization models
        - » Occurrence of micro-buckling phenomena is still limited in the mean field, FFT, asymptotic homogenization frameworks





#### Computational homogenization

 This method is probably the most accurate method to directly account for complex micro-structural behaviors

- This method can model:
  - Micro-buckling of cell walls (e.g. Okumura et al. 2004, Takahashi et al. 2010)
  - Localization problems (e.g. Kouznetsova et al. 2004, Massart et al. 2007, Nguyen et al. 2011, Coenen et al. 2012)
  - Size effects (e.g. Kouznetsova et al. 2004, Ebinger et al. (2005))

- First-order computational homogenization framework (first-order FE<sup>2</sup>)
  - Macro-scale
    - Finite element model
    - At one integration point  $\bar{\mathbf{F}}$  is known,  $\bar{\mathbf{P}}\,$  is sought
  - Micro-scale
    - Representative Volume Element (RVE)
    - Usual finite elements
    - Microscopic boundary conditions
  - Transition
    - Down-scaling:  $\bar{\mathbf{F}}$  is used to define the BCs
    - Up-scaling:  $\bar{\mathbf{P}}$  and  $\partial \bar{\mathbf{P}} / \partial \bar{\mathbf{F}}$  are known from resolutions of micro-scale problems
  - Scale separation

 $l_{discrete} \ll l_{micro} \ll l_{macro}$ 









- Finite element solutions for strain softening problems suffer from:
  - Loss of solution uniqueness and strain localization
  - Mesh dependence



- A generalized continuum is required at the macro-scale
  - Second-order computational homogenization framework (second-order FE<sup>2</sup>) (Kouznetsova et al. 2004)
    - Macroscopic Mindlin strain gradient continuum
    - Microscopic classical continuum
    - Suitable for moderate localization bands & size effects



- Second-order computational homogenization framework (second-order FE<sup>2</sup>)
  - Macro-scale
    - Mindlin strain gradient continuum

$$\bar{\mathbf{P}} \otimes \boldsymbol{\nabla}_0 - \bar{\mathbf{Q}} : (\boldsymbol{\nabla}_0 \otimes \boldsymbol{\nabla}_0) = \mathbf{0} \quad \text{in } B_0 \quad \& \quad \begin{cases} \bar{\mathbf{u}} = \bar{\mathbf{u}}^0 & \text{on } \partial_D B_0 \\ \bar{\mathbf{T}} = \bar{\mathbf{T}}^0 & \text{on } \partial_N B_0 \\ D\bar{\mathbf{u}} = D\bar{\mathbf{u}}^0 & \text{on } \partial_T B_0 \\ \bar{\mathbf{R}} = \bar{\mathbf{R}}^0 & \text{on } \partial_M B_0 \end{cases}$$

- Micro-scale
  - Usual finite elements
  - Second-order microscopic boundary conditions
- Transition
  - Down-scaling:  $\bar{\mathbf{F}},\bar{\mathbf{F}}\otimes\boldsymbol{\nabla}_{0}$  are used to define the BCs
  - Up-scaling: Two stresses P, Q and 4 tangent operators are known from resolutions of micro-scale problems



 $\mathbf{P}$ 

 $\nabla_0$ 

 $\partial ar{\mathbf{Q}}$ 

 $\partial ar{\mathbf{F}}\otimes oldsymbol{
abla}_{0}$ 

 $\overline{\partial \bar{\mathbf{F}}}^{\,,\,}\overline{\partial \bar{\mathbf{F}}}\otimes$ 

 $\partial ar{\mathbf{P}}$ 

 $\partial \mathbf{Q}$ 

 $\overline{\partial ar{\mathbf{F}}}$ 

 $ar{\mathbf{F}},ar{\mathbf{F}}\otimesoldsymbol{
abla}_0$ 





#### • Selected approach & challenges

Microscopic classical continuum with periodic boundary condition (PBC)



new method to enforce PBC on non-conforming meshes

RVE example of random foam

16





• Selected approach & challenges (2)

Macroscopic Mindlin strain gradient continuum

$$\bar{\mathbf{P}} \otimes \boldsymbol{\nabla}_0 - \bar{\mathbf{Q}} : (\boldsymbol{\nabla}_0 \otimes \boldsymbol{\nabla}_0) = \mathbf{0} \text{ in } B_0 \quad \& \quad \begin{cases} \bar{\mathbf{u}} = \bar{\mathbf{u}}^0 & \text{ on } \partial_D B_0 \\ \bar{\mathbf{T}} = \bar{\mathbf{T}}^0 & \text{ on } \partial_N B_0 \\ D\bar{\mathbf{u}} = D\bar{\mathbf{u}}^0 & \text{ on } \partial_T B_0 \\ \bar{\mathbf{R}} = \bar{\mathbf{R}}^0 & \text{ on } \partial_M B_0 \end{cases}$$

## $\rightarrow$ efficient method to solve using usual finite elements



Selected approach & challenges (3)

Instabilities at both scales



Force-displacement crushing response of aluminum honeycomb (Experimental data from Papka & Kyriakides 1999)

 $\rightarrow$  arc-length method to capture instabilities

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PBC enforcement based on the *polynomial interpolation method*

(Nguyen, Geuzaine, Béchet & Noels CMS 2012)

- Second-order multi-scale computational homogenization scheme based on the Discontinuous Galerkin method (called *second-order DG-based FE<sup>2</sup> scheme*)
  - DG method is used to solved the macroscopic Mindlin strain gradient
  - Usual FE
  - Parallel computation

(Nguyen, Becker & Noels CMAME 2013)

- Use of this second-order DG-based FE<sup>2</sup> scheme to capture instabilities in cellular materials
  - Arc-length path following method is adopted at both scales because of the presence of the macroscopic localization and micro-buckling
  - Parallel computation

### (Nguyen & Noels IJSS 2014)





## PBC enforcement based on the *polynomial interpolation method*

(Nguyen, Geuzaine, Béchet & Noels CMS 2012)

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(Nguyen & Noels IJSS 2014)





- Periodic boundary condition
  - Defined from the fluctuation field
    - First-order:  $\mathbf{w} = \mathbf{u} \left(\bar{\mathbf{F}} \mathbf{I}\right) \cdot \mathbf{X}$
    - Second-order:

$$\mathbf{w} = \mathbf{u} - \left(\bar{\mathbf{F}} - \mathbf{I}\right) \cdot \mathbf{X} - \frac{1}{2} \left(\bar{\mathbf{F}} \otimes \boldsymbol{\nabla}_{0}\right) : (\mathbf{X} \otimes \mathbf{X})$$

Stated on the RVE boundary

• First-order PBC:  

$$\mathbf{w}(\mathbf{X}^+) = \mathbf{w}(\mathbf{X}^-) \text{ and }$$

$$\mathbf{w}(\mathbf{X}^I) = \mathbf{0}$$

• Second-order PBC:

$$\mathbf{w}(\mathbf{X}^{+}) = \mathbf{w}(\mathbf{X}^{-}) \text{ and}$$
$$\int_{S_{i}} \mathbf{w}(\mathbf{X}) \, d\partial V = \mathbf{0} \quad \forall S_{i} \subset \partial V$$



Can be achieved by constraining opposite nodes







Polynomial interpolation method for PBC enforcement



- Use of control nodes (new or existing nodes) to interpolate the fluctuation field at the RVE boundary
- Use of interpolant functions (e.g. Lagrange, cubic spline, patch Coons, etc.)
- Fluctuations at boundary nodes are interpolated from control nodes
- PBC is satisfied by using the same interpolation form for opposite parts





- Polynomial interpolation method for PBC enforcement (2)
  - First-order PBC

$$\begin{split} \mathbf{w}^{-}\left(\mathbf{X}\right) &= \sum_{k} \mathbb{N}^{k}\left(\mathbf{X}\right) \mathbf{w}^{k} + \sum_{k} \mathbb{M}^{k}\left(\mathbf{X}\right) \boldsymbol{\theta}^{k},\\ \mathbf{w}^{+}\left(\mathbf{X}\right) &= \sum_{k} \mathbb{N}^{k}\left(\mathbf{X}\right) \mathbf{w}^{k} + \sum_{k} \mathbb{M}^{k}\left(\mathbf{X}\right) \boldsymbol{\theta}^{k} \text{ and}\\ \mathbf{w}\left(\mathbf{X}^{I}\right) &= \mathbf{0} \end{split}$$

Second-order PBC

$$\mathbf{w}^{-}(\mathbf{X}) = \sum_{k} \mathbb{N}^{k} (\mathbf{X}) \mathbf{w}^{k} + \sum_{k} \mathbb{M}^{k} (\mathbf{X}) \boldsymbol{\theta}^{k},$$
$$\mathbf{w}^{+}(\mathbf{X}) = \sum_{k} \mathbb{N}^{k} (\mathbf{X}) \mathbf{w}^{k} + \sum_{k} \mathbb{M}^{k} (\mathbf{X}) \boldsymbol{\theta}^{k} \text{ and}$$
$$\int_{S \subset \partial V^{-}} \left( \sum_{k} \mathbb{N}^{k} (\mathbf{X}) \mathbf{w}^{k} + \sum_{k} \mathbb{M}^{k} (\mathbf{X}) \boldsymbol{\theta}^{k} \right) d\partial V = \mathbf{0}$$



Boundary nodeControl node

- Interpolant functions  $\mathbb{N}^k, \mathbb{M}^k$  and control DOFs  $\mathbf{w}^k, \boldsymbol{\theta}^k$  depend on the interpolation methods





- Polynomial interpolation method for PBC enforcement (3)
  - Results in new constraints in terms of displacements of both boundary and control nodes

$$ilde{\mathbf{C}} ilde{\mathbf{u}}_b - \mathbf{g} = \mathbf{0}$$

- First order:  $\mathbf{g} = \mathbf{g}\left(\bar{\mathbf{F}}\right)$
- Second-order:  $\mathbf{g} = \mathbf{g} \left( \bar{\mathbf{F}}, \bar{\mathbf{F}} \otimes \boldsymbol{\nabla}_0 \right)$
- These linear constraints can be enforced by
  - Constraints elimination
  - Lagrange multipliers
- Suitable for
  - Arbitrary meshes
  - Important void parts on the RVE boundaries
  - Arbitrary interpolation forms





- 2D problems using Lagrange & cubic spline interpolations
  - Periodic hole structures
    - Hole radius = 0.2 mm
    - Elastic material: Young modulus= 70 GPa, Poisson ratio = 0.3



Non-periodic mesh



Convergence of effective properties in terms of new DOFs added to system





- 2D problems using Lagrange & cubic spline interpolations (2)
  - Honeycomb structures
    - Edge length = 1 mm & thickness = 0.1 mm
    - Elastic material: Young modulus= 68.9 GPa, Poisson ratio = 0.33



RVE mesh from honeycomb structure



Convergence of effective properties in terms of new DOFs added to system



 $\bar{\varepsilon} =$ 

0.01

0.005

3D problems using the patch Coons interpolation based on the Lagrange & cubic spline interpolations

$$\bar{\sigma}_{\text{Lagrange}} = \begin{bmatrix} 281.583 & 92.392 & 121.181 \\ 92.392 & 270.111 & -115.835 \\ 121.181 & -115.835 & -247.78 \end{bmatrix} \text{MPa}$$
$$\bar{\sigma}_{\text{spline}} = \begin{bmatrix} 281.399 & 91.9833 & 121.239 \\ 91.983 & 268.85 & -115.614 \\ 121.239 & -115.614 & -248.214 \end{bmatrix} \text{MPa}$$





0.005



Lagrange Coons patch of order 15



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Cubic spline Coons patch with 10 segments





- Conclusions
  - A new method to enforce the PBC is presented
    - By using an interpolation formulation
    - For arbitrary meshes
    - For both 2-dimensional and 3-dimensional problems
  - This method provides a better estimation compared to the linear displacement BC which is usually used for non-conforming meshes
  - The key advantage of this method is the elimination of the need of matching nodes





PBC enforcement based on the *polynomial interpolation method*

(Nguyen, Geuzaine, Béchet & Noels CMS 2012)

- Second-order multi-scale computational homogenization scheme based on the Discontinuous Galerkin method (called *second-order DG-based FE<sup>2</sup> scheme*)
  - DG method is used to solved the macroscopic Mindlin strain gradient
  - Usual FE
  - Parallel computation

#### (Nguyen, Becker & Noels CMAME 2013)

- Use of this second DG-based FE2 scheme to capture instabilities in cellular materials
  - Arc-length path following method is adopted at both scales because of the presence of the macroscopic localization and micro-buckling
  - Parallel computation

(Nguyen & Noels IJSS 2014)





Mindlin strain gradient

$$\bar{\mathbf{P}} \otimes \boldsymbol{\nabla}_0 - \bar{\mathbf{Q}} : (\boldsymbol{\nabla}_0 \otimes \boldsymbol{\nabla}_0) = \mathbf{0} \quad \text{in } B_0 \quad \& \quad \begin{cases} \bar{\mathbf{u}} = \bar{\mathbf{u}}^0 & \text{on } \partial_D B_0 \\ \bar{\mathbf{T}} = \bar{\mathbf{T}}^0 & \text{on } \partial_N B_0 \\ D\bar{\mathbf{u}} = D\bar{\mathbf{u}}^0 & \text{on } \partial_T B_0 \\ \bar{\mathbf{R}} = \bar{\mathbf{R}}^0 & \text{on } \partial_M B_0 \end{cases}$$

- Numerical solution requires the continuity of the displacement field and of its derivatives. Some methods can be considered:
  - Mixed methods (e.g. Shu et al. 1999)
  - Mesh-less method (*e.g.* Askes et al. 2002)
  - **C**<sup>1</sup> finite elements (*e.g.* Papanicolopulos et al. 2012)
  - Discontinuous Galerkin (DG) method (*e.g.* Engel et al. 2002)

- DG method is extended to large deformations and multi-scale analyses to solve the Mindlin strain gradient continuum
  - Using only the displacement field as unknowns
  - Enforcing weakly inter-element continuities



- Finite element discretization
- Same **discontinuous** polynomial approximation for



- Definition of trace operators on the inter-element interfaces
  - Jump operator  $\llbracket \bullet \rrbracket = \bullet^+ \bullet^-$
  - Mean operator  $\langle \bullet \rangle = \frac{1}{2} \left( \bullet^+ + \bullet^- \right)$
- Continuity is weakly enforced, such that the method
  - Is consistent
  - Is stable
  - Has an optimal convergence rate

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- As the strain gradient solution requires the C<sup>0</sup> & C<sup>1</sup> continuities, two formulations can be used:
  - Full Discontinuous Galerkin (FDG) formulation

$$\begin{cases} \mathbf{U}^{k} = \left\{ \bar{\mathbf{u}} \in \mathbf{L}^{2} \left( B_{0} \right) & | \quad \bar{\mathbf{u}} |_{\Omega_{0}^{e}} \in \mathbb{P}^{k} \forall \Omega_{e}^{0} \in B_{0} \right\} \\ \mathbf{U}_{c}^{k} = \left\{ \delta \bar{\mathbf{u}} \in \mathbf{U}^{k} & | \quad \delta \bar{\mathbf{u}} |_{\partial_{D} B_{0}} = \mathbf{0} \right\} \end{cases}$$

- Weak enforcement of *C*<sup>0</sup> & *C*<sup>1</sup> continuities
- Different DOFs at inter-element interfaces
- Usual shape functions



- Enriched Discontinuous Galerkin (EDG) formulation

$$\begin{cases} \mathbf{U}^{k} = \left\{ \bar{\mathbf{u}} \in \mathbf{H}^{1}\left(B_{0}\right) & | \quad \bar{\mathbf{u}}|_{\Omega_{0}^{e}} \in \mathbb{P}^{k} \quad \forall \Omega_{e}^{0} \in B_{0} \right\} \\ \mathbf{U}_{c}^{k} = \left\{ \delta \bar{\mathbf{u}} \in \mathbf{U}^{k} & | \quad \delta \bar{\mathbf{u}}|_{\partial_{D}B_{0}} = \mathbf{0} \right\} \end{cases}$$

- Weak enforcement of *C*<sup>1</sup> continuity
- Same DOFs as conventional FEM
- Usual shape functions







Weak formulation obtained by repeating integrations by parts on each element :







First-order interface term

$$\int_{\partial_I B_0} \left[ \! \left[ \delta \bar{\mathbf{u}} \cdot \bar{\hat{\mathbf{P}}} \right] \! \right] \cdot \bar{\mathbf{N}} \, d\partial B$$

## is rewritten as the sum of three terms:

Consistency

$$\int_{\partial_I B_0} \left[\!\!\left[ \delta \bar{\mathbf{u}} \right]\!\!\right] \cdot \left\langle \bar{\hat{\mathbf{P}}} \right\rangle \cdot \bar{\mathbf{N}}^- \, d\partial B$$

Compatibility

$$\int_{\partial_I B_0} \left[\!\!\left[ \bar{\mathbf{u}} \right]\!\!\right] \cdot \left\langle \bar{\hat{\mathbf{P}}} \left( \delta \bar{\mathbf{u}} \right) \right\rangle \cdot \bar{\mathbf{N}}^- \, d\partial B$$

Stability controlled by  $\beta_P$ 

$$\int_{\partial_I B_0} \left( \llbracket \bar{\mathbf{u}} \rrbracket \otimes \bar{\mathbf{N}}^- \right) : \left\langle \frac{\beta_P}{h_s} \mathbb{C}^0 \right\rangle : \left( \delta \bar{\mathbf{u}} \otimes \bar{\mathbf{N}}^- \right) \, d\partial B$$

These terms vanish in the case of EDG formulation

• Second-order interface term

$$\int_{\partial_I B_0} \left[\!\!\left[ (\delta \bar{\mathbf{u}} \otimes \boldsymbol{\nabla}_0) : \bar{\mathbf{Q}} \right]\!\!\right] \cdot \bar{\mathbf{N}} \, d\partial B$$

## is rewritten as the sum of three terms:

Consistency

$$\int_{\partial_I B_0} \left[\!\left[\delta \bar{\mathbf{u}} \otimes \boldsymbol{\nabla}_0\right]\!\right] : \left\langle \bar{\mathbf{Q}} \right\rangle \cdot \bar{\mathbf{N}}^- \, d\partial B$$

Compatibility

$$\int_{\partial_{I}B_{0}} \left[\!\left[\bar{\mathbf{u}} \otimes \boldsymbol{\nabla}_{0}\right]\!\right] : \left\langle \bar{\mathbf{Q}}\left(\delta\bar{\mathbf{u}}\right)\right\rangle \cdot \bar{\mathbf{N}}^{-} \, d\partial B$$

– Stability controlled by 
$$\beta_Q$$

$$\int_{\partial_I B_0} \left( \llbracket \bar{\mathbf{u}} \otimes \boldsymbol{\nabla}_0 \rrbracket \otimes \bar{\mathbf{N}}^- \right) \stackrel{!}{:} \left\langle \frac{\beta_Q}{h_s} \mathbb{J}^0 \right\rangle \stackrel{!}{:} \\ \left( \llbracket \delta \bar{\mathbf{u}} \otimes \boldsymbol{\nabla}_0 \rrbracket \otimes \bar{\mathbf{N}}^- \right) \, d\partial B$$



- Weak formulation obtained by DG method:  $a(\bar{u}, \delta \bar{u}) = b(\delta \bar{u})$
- Bi-nonlinear term:  $a(\bar{\mathbf{u}}, \delta \bar{\mathbf{u}}) = a^{\text{bulk}}(\bar{\mathbf{u}}, \delta \bar{\mathbf{u}}) + a^{\text{PI}}(\bar{\mathbf{u}}, \delta \bar{\mathbf{u}}) + a^{\text{QI}}(\bar{\mathbf{u}}, \delta \bar{\mathbf{u}})$ 
  - Bulk term

$$\mathbf{a}^{\mathrm{bulk}}\left(\bar{\mathbf{u}},\delta\bar{\mathbf{u}}\right) = \int_{B_0} \left[\bar{\mathbf{P}}:\left(\delta\bar{\mathbf{u}}\otimes\boldsymbol{\nabla}_0\right) + \bar{\mathbf{Q}}\overset{\cdot}{:}\left(\delta\bar{\mathbf{u}}\otimes\boldsymbol{\nabla}_0\otimes\boldsymbol{\nabla}_0\right)\right] \, dB$$

- First-order interface term (vanishes if using the EDG formulation)

$$\begin{split} \mathbf{a}^{\mathrm{PI}}\left(\bar{\mathbf{u}},\delta\bar{\mathbf{u}}\right) &= \int_{\partial_{I}B_{0}} \left[ \left[\!\left[\delta\bar{\mathbf{u}}\right]\!\right] \cdot \left\langle \bar{\hat{\mathbf{P}}}\left(\bar{\mathbf{u}}\right)\right\rangle \cdot \bar{\mathbf{N}}^{-} + \left[\!\left[\bar{\mathbf{u}}\right]\!\right] \cdot \left\langle \bar{\hat{\mathbf{P}}}\left(\delta\bar{\mathbf{u}}\right)\right\rangle \cdot \bar{\mathbf{N}}^{-} \\ &+ \left[\!\left[\bar{\mathbf{u}}\right]\!\right] \otimes \bar{\mathbf{N}}^{-} : \left\langle \frac{\beta_{P}}{h_{s}} \mathbf{C}^{0} \right\rangle : \left[\!\left[\delta\bar{\mathbf{u}}\right]\!\right] \otimes \bar{\mathbf{N}}^{-} \right] d\partial B \,, \end{split}$$

Second-order interface term

$$\begin{aligned} \mathbf{a}^{\mathrm{QI}}\left(\bar{\mathbf{u}},\delta\bar{\mathbf{u}}\right) &= \int_{\partial_{I}B_{0}} \left[ \left[ \left[ \delta\bar{\mathbf{u}}\otimes\nabla_{0} \right] \right] : \left\langle \bar{\mathbf{Q}}\left(\bar{\mathbf{u}}\right) \right\rangle \cdot \bar{\mathbf{N}}^{-} + \left[ \left[ \bar{\mathbf{u}}\otimes\nabla_{0} \right] \right] : \left\langle \bar{\mathbf{Q}}\left(\delta\bar{\mathbf{u}}\right) \right\rangle \cdot \bar{\mathbf{N}}^{-} \\ &+ \left[ \left[ \bar{\mathbf{u}}\otimes\nabla_{0} \right] \right] \otimes \bar{\mathbf{N}}^{-} \vdots \left\langle \frac{\beta_{Q}}{h_{s}} \mathbf{J}^{0} \right\rangle \vdots \left[ \left[ \delta\bar{\mathbf{u}}\otimes\nabla_{0} \right] \right] \otimes \bar{\mathbf{N}}^{-} \right] d\partial B \,. \end{aligned}$$

• Load term  $\mathbf{b} \left( \delta \bar{\mathbf{u}} \right) = \left( \int_{\partial_N B_0} \bar{\mathbf{T}}^0 \cdot \delta \bar{\mathbf{u}} \, d\partial B + \int_{\partial_M B_0} \bar{\mathbf{R}}^0 \cdot \mathbf{D} \delta \bar{\mathbf{u}} \, d\partial B \right)$  Univers

 $\forall \delta \bar{\mathbf{u}} \in \mathbf{U}_c^k$ 

## DG method for strain gradient problems



- Second-order DG-based FE<sup>2</sup> scheme
  - DG solution of macroscopic Mindlin strain gradient problems
    - Interface & bulk integration points
  - Microscopic problems
    - Associated with both interface & bulk integration points
  - Scale transitions
    - Down-scaling: two kinematic strains are used to define the microscopic BCs
    - Up-scaling: two stresses and 4 tangent operators are extracted from the resolutions of the microscopic problems



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- Parallel second-order DG-based FE<sup>2</sup> scheme
  - Computation strategy







- At the macro-scale
  - Macro-scale parallelization
  - Computation using FDG formulation with "ghost elements"
    - Communications required at each time step to exchange the nodal displacements at inter-partition interfaces only
- At the micro-scale
  - All microscopic problems are separate in nature











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(d) Communications

(Becker et al. 2012, Wu et al. 2013)





- Multi-scale study of the shear layer test
  - H = 1cm , 2cm, 4cm and 8cm
     in order to consider size effects
  - RVE size d = 0.2cm
  - Material law
    - Bulk modulus = 175 GPa
    - Shear modulus = 81 GPa
    - Yield stress = 507 MPa
    - Hardening modulus = 200 MPa



(a) Macroscopic mesh











- Multi-scale study of the shear layer test (2)
  - Size effect







Deformation profile

Deformed shape with H/d = 10

Deformed shape with H/d = 40





#### Conclusions

- The Mindlin strain gradient problems are solved using the discontinuous
   Galerkin formulations with conventional finite elements
- The resulting one-field formulation can easily be implemented in existing software
- This formulation is used for second-order multi-scale computational homogenizations
- The micro and macro-scale problems consider finite strains
- Size effects in heterogeneous elasto-plastic materials can be studied





PBC enforcement based on the polynomial interpolation method

(Nguyen, Geuzaine, Béchet & Noels CMS 2012)

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  - DG method is used to solved the macroscopic Mindlin strain gradient
  - Usual FE
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#### (Nguyen, Becker & Noels CMAME 2013)

- Use of this second-order DG-based FE<sup>2</sup> scheme to capture instabilities in cellular materials
  - Arc-length path following method is adopted at both scales because of the presence of the macroscopic localization and micro-buckling
  - Parallel computation

#### (Nguyen & Noels IJSS 2014)





#### Microscopic classical continuum

- Enforcement of PBC using the polynomial interpolation method
- Arc-length path following method
- Macroscopic Mindlin strain gradient continuum
  - Resolution with Discontinuous Galerkin formulation
  - Arc-length path following method
- Full parallel computations
  - Macroscopic parallel distribution using ghost elements
  - Microscopic parallel distribution
- Why the arc-length path following method?
  - Load-based increments (pure load, arc-length increments, etc.) are preferred to improve the Newton-Raphson convergence
  - Presence of critical points (e.g. limit points) & unstable equilibrium paths for which the conventional Newton-Raphson method fails -> arc-length increments



45

## • Path following method

- Macro-scale problem
  - Path following with applied loading

 $\mathbf{a}\left(\bar{\mathbf{u}},\delta\bar{\mathbf{u}}\right)=\bar{\mu}\mathbf{b}\left(\delta\bar{\mathbf{u}}\right)$ 

• Arc-length constraint

$$\bar{h}(\Delta \bar{\mathbf{u}}, \Delta \bar{\mu}) = \frac{\Delta \bar{\mathbf{u}}^T \Delta \bar{\mathbf{u}}}{\Psi^2} + \Delta \bar{\mu}^2 - \Delta L^2 = 0,$$

- Micro-scale problems
  - Path following method on the applied boundary conditions

$$ilde{\mathbf{C}} ilde{\mathbf{u}}_b - \mathbf{g}\left(ar{\mathbf{F}},ar{\mathbf{F}}\otimesoldsymbol{
abla}_0
ight) = oldsymbol{0}$$

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Arc-length constraint

$$h(\Delta \mathbf{u}, \Delta \mu) = \frac{\Delta \mathbf{u}^T \Delta \mathbf{u}}{\psi^2} + \Delta \mu^2 - \Delta l^2 = 0$$

- Compute by increments until  $\,\mu=1$ 









- Compression of an hexagonal honeycomb plate
  - Plate: H = 102 mm, L = 65.8mm
  - Honeycomb: I = 1mm, t = 0.1 mm
  - Elasto-plastic material
    - Bulk modulus = 67.55 GPa
    - Shear modulus = 25.9 GPa
    - Initial yield stress = 276 MPa









## • Compression of an hexagonal honeycomb plate (2)

- Captures the softening onset
- Captures the softening response
- No macro-mesh size effect





- Compression of an hexagonal honeycomb plate (3)
  - Influence of cell size
    - Same honeycomb structure: I = 1mm, t = 0.1 mm
    - Different plate dimensions







- Compression of a hexagonal honeycomb plate with a centered hole
  - Central radius: r = 15mm
  - Honeycomb: I = 1mm, t= 0.1mm
  - Elasto-plastic material
    - Bulk modulus = 67.55 GPa
    - Shear modulus = 25.9 GPa
    - Initial yield stress = 276 MPa



Full model with 300.000 quadratic triangles





Multi-scale model

Geometry & BCs





- Compression of a hexagonal honeycomb plate with a centered hole (2)
  - Results given by full and multi-scale models are comparable











- A second-order DG-based FE<sup>2</sup> scheme was developed with the following novelties:
  - The periodic boundary condition is enforced using the polynomial interpolation method without the need of conforming meshes
  - The macroscopic Mindlin strain gradient problem is solved using the Discontinuous Galerkin method with conventional finite elements
  - The arc-length path following method is applied at both scales to capture their instabilities

## Current works





#### Polypropylene foam

- Experimental tests (F. Wan)



SEM image of a PP foam with 4% CNTs

## Imperfections → reduce the stiffness

- Random cell sizes and shapes
- Non-uniform distribution of solid materials of cell walls
- Curvature of cell walls
- Loss of cell walls
- Corrugation of cell walls
- Fracture of cell walls



#### Current works





- Homogenized properties based on the tetrakaidecahedron unit cell
  - Ideal unit cell models



Open unit cell

## →imperfections must be considered





- Homogenized properties based on the tetrakaidecahedron unit cell (2)
  - Unit cell with mass concentration at cell edges
    - Mass concentration parameter: **φ**= mass at cell edge/ total mass







 Experimental validation in the context of the ARC project with the RVE meshes coming from tomographical images of the microstructure

• Electromagnetic-mechanical coupling problems since electromagnetic properties are modified during the mechanical loading (as the shape is deformed)

• Discontinuous-continuous schemes for sharper localization problems following the works of Massart et al. 2007, Nguyen et al. 2011 or Coenen et al. 2012

• Material tailoring with required properties by computational homogenization schemes







## Thank you for your attention!





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Method	CPU time per iteration	Used memory
Full model	92 seconds	5.6 gigabytes
Multi-scale model, mesh 0	36 seconds	1.3 gigabytes
Multi-scale model, mesh 1	84 seconds	2.6 gigabytes
Multi-scale model, mesh 2	146 seconds	4.0 gigabytes

Computation time and used memory of the full model and multi–scale models. These computations were performed in the same machine with one processor