

Investigations on a Level Set based approach for the optimization of flexible components in multibody systems with a fixed mesh grid

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Introduction

Evolution of virtual prototyping

Evolution of virtual prototyping

Motivations

■ Classical FEM approach

(Quasi-) Static load cases (Empirical, experience,…)

■ Weak coupling between FEM and MBS

- Coupling with pre / post processing
- Define equivalent static load cases (Kang, Park and Arora, 2005)
- Optimization of isolated components
- **Extrange coupling between FEM and MBS**
	- Define realistic dynamic loadings
	- Take care of the coupling between large overall rigid-body motions and deformations
	- **Dependengelem** with function depending on time
- « Fully Integrated Method »
	- **MBS approach based on nonlinear FEM (Samcef Mecano)**
	- Coupled with an optimization shell (Boss Quattro)

Finite Element Approach Of Multibody System Dynamics

Equation of FEM-MBS dynamics

- Motion of the flexible body (FEM) is represented by absolute nodal coordinates **q** (Geradin & Cardona, 2001)
- Dynamic equations of multibody system

$$
M\ddot{q} = g(\dot{q}, q, t) = g^{ext} - g^{int}
$$

- Subject to kinematic constraints of the motion $\Phi(q,t) = 0$
- **Solution based on an augmented Lagrangian approach** of the total energy

$$
\begin{bmatrix} \mathbf{M}\ddot{\mathbf{q}} + \mathbf{B}^T(k\lambda + p\Phi) = \mathbf{g}(\dot{\mathbf{q}}, \mathbf{q}, t) & \mathbf{B} = \frac{\partial \Phi}{\partial \mathbf{q}} \\ k\Phi(\mathbf{q}, t) = 0 & \\ \mathbf{q}'(0) = \mathbf{q}'_0 \text{ and } \dot{\mathbf{q}}'(0) = \dot{\mathbf{q}}_0 \end{bmatrix}
$$

- The set of nonlinear DAE is solved using the generalized- α method by Chung and Hulbert (1993)
- Definition of a pseudo acceleration vector **a**:

$$
(1 - \alpha_m)\mathbf{a}_{n+1} + \alpha_m \mathbf{a}_n = (1 - \alpha_f)\ddot{\mathbf{q}}_{n+1} + \alpha_f \ddot{\mathbf{q}}_n
$$

Newmark integration formulae

$$
\dot{\mathbf{q}}_{n+1} = \dot{\mathbf{q}}_n + h(1 - \gamma)\mathbf{a}_n + h\gamma \mathbf{a}_{n+1}
$$

$$
\mathbf{q}_{n+1} = \mathbf{q}_n + h\dot{\mathbf{q}}_{n+1} + h^2(1/2 - \beta)\mathbf{a}_n + h\beta \mathbf{a}_{n+1}
$$

■ Solve iteratively the dynamic equation system (Newton-Raphson)

$$
\begin{bmatrix} \mathbf{M}\Delta \ddot{\mathbf{q}} + \mathbf{C}_t \Delta \dot{\mathbf{q}} + \mathbf{K}_t \Delta \mathbf{q} + \mathbf{B}^T \Delta \lambda = \Delta \mathbf{r} & \mathbf{r} = \mathbf{M} \ddot{\mathbf{q}} - \mathbf{g} + \mathbf{B}^T \lambda \\ \mathbf{B} = \mathbf{0} & \end{bmatrix}
$$

The Level Set Description

Principle (Sethian & Osher, 1988)

- Numerical technique for tracking interfaces
	- **Introduce a higher dimension function**
	- **I**mplicit boundary representation $\psi(x,t) = 0$
	- Interface = the zero level of the function

Advantages - Drawbacks

Advantages Drawbacks

- Easiness of combining entities (min, max,…)
	- **Remove entities**
	- **Separate entities**
	- **Nerge entities**
- \rightarrow Topology modifications
- Extension 2D/3D
- Nicely coupled to XFEM

- Construction:
	- Specific tools
	- **Analytical functions**
	- Point set NURBS
- Mesh "adaptation" necessary but not in the method proposed here

- Necessary to have an initial design of the component
- Parametric model
- Design variables = Geometrical parameters of flexible body shape

■ The finite element mesh is modified to follow the shape modifications.

- → It leads to mesh distortion. Major Problem!
- \rightarrow The quality of the mesh decreases and the solution accuracy of the FEA decreases after the first iteration. Re-meshing techniques exist to avoid this problem.
- \rightarrow Velocity field for the sensitivity analysis
- The adopted parameterization influences the performance of the optimization process.

Topology optimization

- Can be seen as an optimal material distribution within a design domain
- No initial knowledge on the component
	- Only have to define:
		- The design domain
		- **The loading**
		- The boundary conditions
		- A volume constraint
- The optimization process gives the best design for these pieces of information.

 The design variables are the density of each finite element. Large number of variables – local optima

Feasibility of manufacturing:

Difficulties to determine structural boundary shape from the topology optimization results.

But…

Fixed mesh grid

- **The** *Level Set Description* of the geometry leads to an intermediate type of optimization between shape optimization and topology optimization.
	- **Fixed mesh grid: No mesh distortion**
	- The geometry is based on CAD entities: can easily be manufactured.
	- Remove, separate, merge entities: Modification of the topology
	- Design variables: parameters of the level sets (quite small number)
- The proposed approach is different from the topology optimization based on the level set (in static optimization) proposed by G. Allaire because we use the level set for the description of the component geometry.

(G. Allaire *et al.*, A level-set method for shape optimization. C.R. Acad. Sci. Paris, 334:1125–1130, 2002)

The proposed approach

The method: Square plate with a hole

- Mesh definition (fixed during all the process) + Level Set definition: Fixed mesh grid: 6*6 elements Level Set: a cone
- No element is removed to create the hole but the properties of elements are modified: the density and the Young modulus.

- For each node: Computation of the level set value.
- Different possibilities can happen for each element:

- For the boundary elements \rightarrow SIMP law
	- Introduction of a pseudo-density

Volume of material $\mu = \frac{1}{\text{Volume of the element}}$

SIMP law

$$
\rho = \mu \rho_0 \text{ and } E = \mu^3 E_0
$$

- Consequences:
	- Smooth transition
	- **Sensitivity analysis:** Fixed number of elements

 \sim

Formulations Of Flexible Multibody System Optimization Problem

General form of the optimization problem

- Provides a general and robust framework to the solution procedure
- Efficient solver :
	- Sequential Convex Programming (Gradient based algorithm)
		- → GCM (Bruyneel et al. 2002)

- Gradient-based optimization methods require the first order derivatives of the responses
- **Finite differences** $\int_{0}^{x} \frac{f(x + \delta x) - f(x)}{\delta x}$
 A variables

MBS code
 P coach (Not yet developed)
 $\frac{\partial \Phi}{\partial x} \approx \frac{\Phi(x + \delta x) - \Phi(x)}{\delta x}$ $\frac{1}{x} \approx \frac{1(x + \delta x)}{\delta x}$ δ δ ∂f $\int f(x+\delta x) - f(x)$ \approx ∂

Perturbation of design variables

→ Additional calls to MBS code

■ Semi-analytical approach (Not yet developed)

$$
\frac{\partial \mathbf{r}}{\partial x} \approx \frac{\mathbf{r}(x + \delta x) - \mathbf{r}(x)}{\delta x} \qquad \frac{\partial \mathbf{\Phi}}{\partial x} \approx \frac{\mathbf{\Phi}(x + \delta x) - \mathbf{\Phi}(x)}{\delta x}
$$

The formulation

■ The formulation is a key point for this type of problems: Very complex nonlinear behavior Impact on the design space Mean Deviation [mm]
ರ ಶಿ ಶಿ ಶಿ ಶಿ 45 Deviation (Time Step 20)
ခုံးခုံးနှံ့ မှ မှ နှံ့ န 20 25 $30⁷$ 35 35 $T5$ [mm] $40²$ $T6$ [mm] $T5$ [mm] 45 50^{\degree} $T6$ [mm] 50 $50 50$

- Extremely important for gradient based algorithm
- Genetic algorithms
	- Do not necessary give better results
	- Computation time much more important

Numerical Applications

- **Minimization of the connecting rod mass in a real** combustion engine (Diesel).
- **During the exhaust phase, the connecting rod** elongates which can destroy the engine. \rightarrow Collision between the piston and the valves.
- Constraints imposed on the elongation

Modeling of the connecting rod

- Consideration of one single complete cycle as the behavior is cyclic (720°) for the optimization
- Rotation speed 4000 Rpm
- Gas pressure taken into account.

Local formulation

 $\min m(\mathbf{x})$ \mathbf{x} s.t. $\Delta l(\mathbf{x}, t_i) \leq \Delta l_{max}$

with $i = 1, \ldots$, nbr time step

- **The constraint on the elongation** Δl (\mathbf{x}, t_i) is considered at each time step.
	- \rightarrow As many constraints as the number of time steps (134)

- The level set is defined in order to have an ellipse as interface.
- **5** different design variables: a, b, c_x , c_y and d. Here only

- Convergence obtained after 12 iterations
- **Monotonous behavior of the optimization process**

■ As the boundary is defined by a CAD entity, the connecting rod can be directly manufactured without any post processing.

Second application – 3 level sets

Results

- Convergence obtained after 15 iterations
- Monotonous behavior of the optimization process
- Even better than the simpler case

Modification of the topology during the evolution of the optimization process

 \blacksquare It should be interesting to add the position of the level sets in the design variable set.

Conclusions and Perspectives

- **•** Optimization of flexible components carried out in the framework of flexible dynamic multibody system simulation
- **Type of optimization between shape optimization and** topology optimization
- Validation of the proposed approach
- Combine the advantages of both methods:
	- **No mesh problem**
	- Possibility of changing the topology but must be introduced before the optimization.
	- **The geometry is expressed by CAD entities**
		- \rightarrow Can be directly manufactured.

Semi-analytical approach for the derivatives

Nam H. Kim, Youngmin Chang, 2005

Need to establish the relation between the velocity field of the level set boundaries and the design variables.

Progress on the optimization problem formulation

Thank You Very Much For Your Attention

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